Goals

- Develop means to improve heat rejection from power electronics > 250 W/cm²
- Reducing system cost, increasing reliability, specific power, power density, and efficiency

Objectives for FY04

Develop and demonstrate the viability and advantages of twophase cooling techniques such as spray cooling, and Jet impingement

Deliverable for FY04

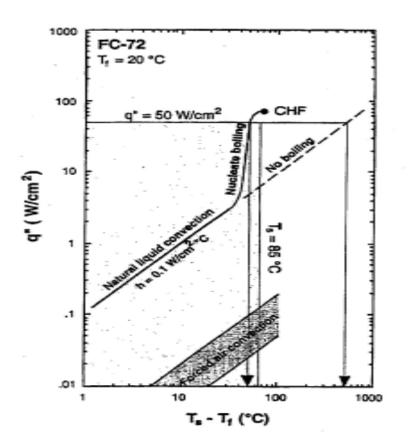
Technical report on viability of spray-cooling and jet impingement for high heat flux heat removal

Task Description: 2-phase heat management

- Investigation of spray cooling fluid dynamics: droplet size and pattern, orientation, surface treatment, spray behavior in critical system pressure, heat load, and vibration ranges.
- Investigation of jet impingement fluid dynamics: Jet nozzle design, orientation, surface treatment, jet behavior in critical system pressure, heat load, and vibration ranges.
- Surface preparation studies

High Heat Flux Thermal Management Techniques

- Pool Boiling/Thermosyphons (30-70 W/cm²)
 - 2-phase Microchannel/Minichannel Cooling (100-250 W/cm²)
- Jet Impingement Cooling (70-110 W/cm²)
 - Spray Cooling (80-120 W/cm²)
- Surface Enhancement
- Choice of Coolants



Pool Boiling

- Coolant that boils 10 to 40 C below the operating temp.
- **30-70** W/cm²
- Governing Parameters
 - Surface Enhancement
 - Surface Orientation
 - Nucleation Sites
 - Fluid Properties

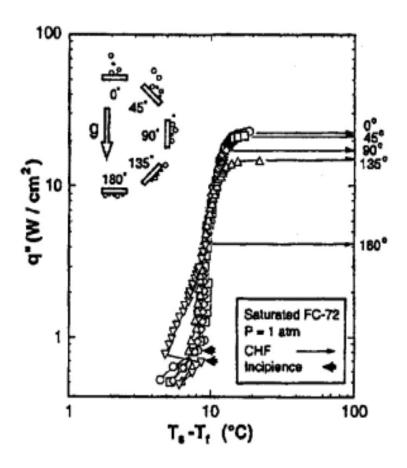


Fig. 8. Pool boiling curves for a 12.7-mm heated disc in saturated FC-72 at different surface orientations (adapted from [18]).

MUDAWAR: ASSESSMENT OF HIGH-HEAT-FLUX THERMAL MANAGEMENT SCHEME!

2-Phase Microchannel/Minichannel Cooling

- Require minimal coolant flow rates
- Flow channels with dimensionsranging from hundreds of microns tcfew millimeters
- 100-250 W/cm²
- Governing Parameters
 - Microfabrication methods
 - Pressure drop
 - Choice of hydraulic diameter
 - Fluid Properties

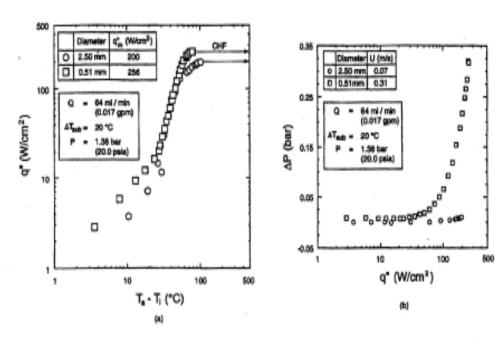
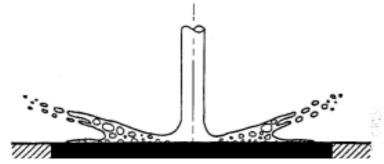


Fig. 18. Comparison of microchannel and minichannel heat sink characteristics relative to (a) cooling performance and (b) pressure drop (adapted from)

Jet Impingement Cooling

- Free jet (vapor or gaseous environment)
- Submerged jet (liquid jet in liquid environment)
- Confined jet (liquid jet confined between the nozzle and target)
- Aggressive form of cooling (large impact momentum)
 - 70-110 W/cm²
- Governing Parameters
 - Jet velocity
 - Jet diameter
 - Subcooling of working fluid
 - Fluid Properties



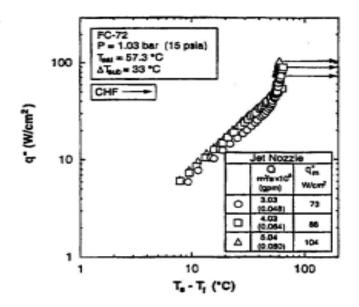


Fig. 21. Free circular jet boiling curves for different flow rates (adapted from [67]).

Spray Cooling

- Water or FCs
- 80-120 W/cm²
- Governing Parameters
 - Droplet Diameter
 - Surface Orientation
 - Surface Texture
 - Fluid Properties

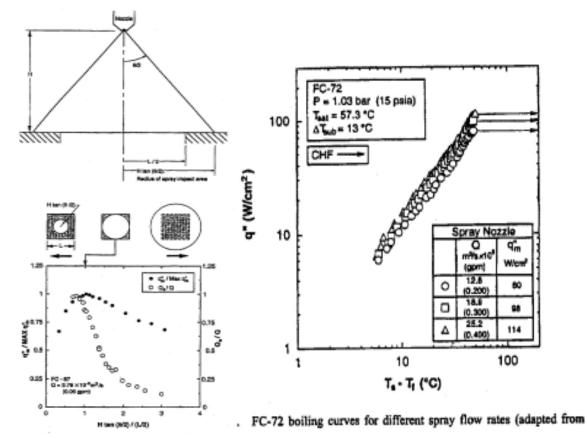
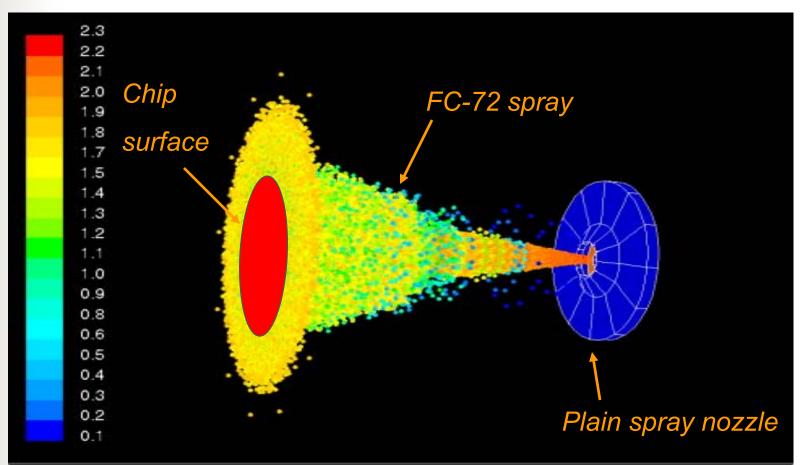


Fig. 24. Optimization of spray norzh-to-wall distance to maximize CHF (adapted from [16]).

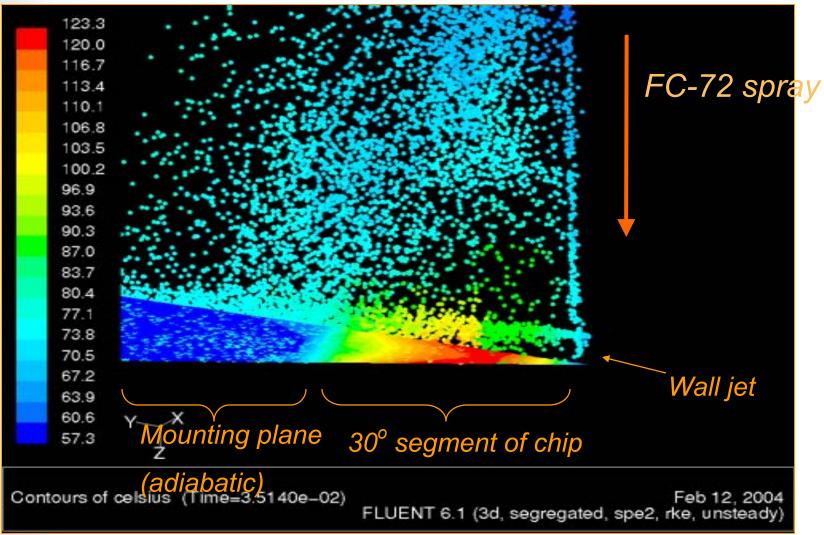
- Surface Modification Investigations
 - Grooved Surface Configurations
 - Dimpled Surface Configurations
 - Patterned Structure Configurations
- Spray Orientation Effects on Modified Surfaces
- Generally Focus on Thermal Effects On Underside of Die/Substrate

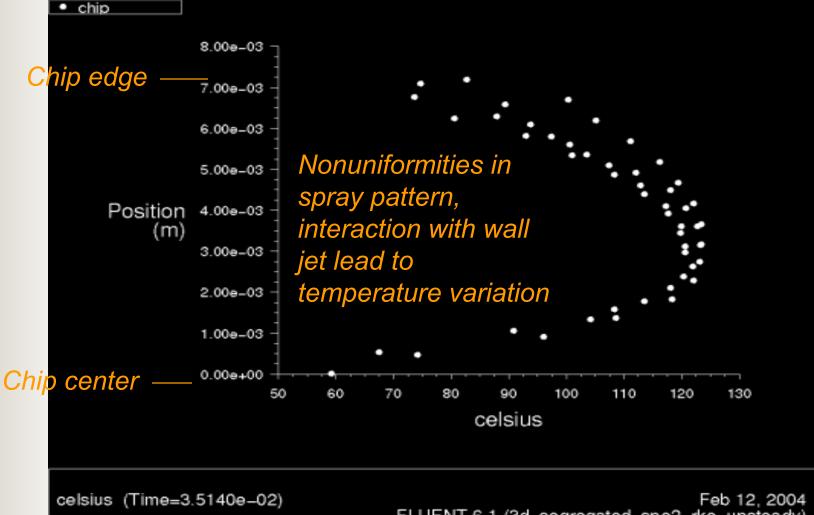
- FC-72 used in initial modeling
- Mass flow rate 0.0084 kg/s
- Chip surface area 161 mm²
- Conditions match those in the literature
- Heat transfer now through droplet impingement, evaporation to be incorporated



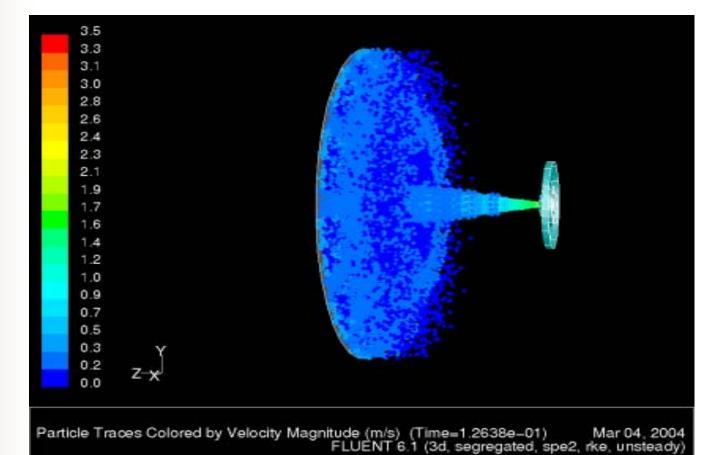
Velocity magnitude (m/s) Grid (Time=3.1140e–02)

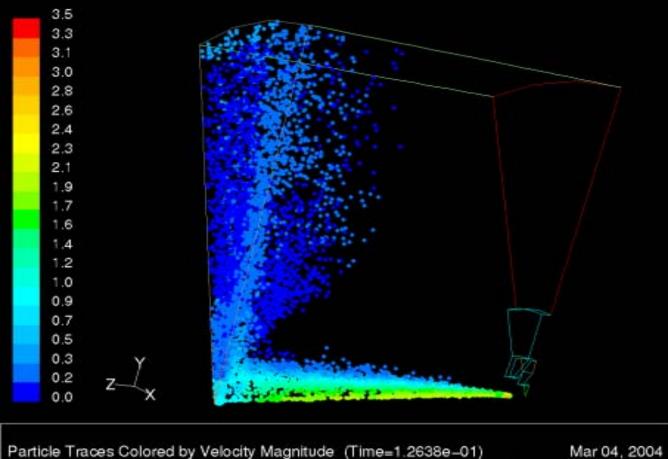
Feb 09, 2004 FLUENT 6.1 (3d, segregated, spe2, rke, unsteady)





FLUENT 6.1 (3d, segregated, spe2, rke, unsteady)





Particle Traces Colored by Velocity Magnitude (Time=1.2638e-01) Mar 04, 2004 FLUENT 6.1 (3d, segregated, spe2, rke, unsteady)

NREL's Experimental Capability

- Spray and jet impingement cooling testing capability, localized heat source and total inverter box
- High speed camera for flow visualization
- Infrared camera for temperature distribution

