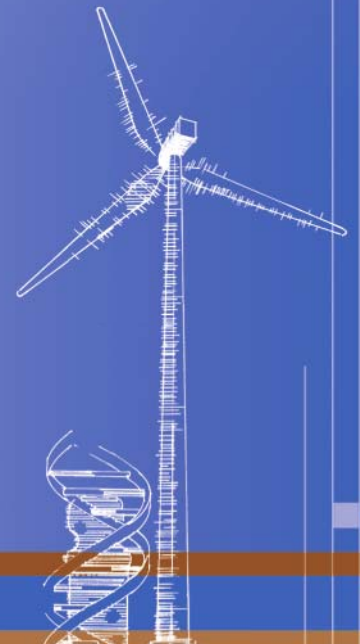


Determination of Elastic Twist in Horizontal Axis Wind Turbines (HAWTs)

Final Report: August 1989

F. Stoddard, V. Nelson, K. Starcher, B. Andrews
*Alternative Energy Institute – West Texas State University
Canyon, Texas*

Subcontract Report
NREL/SR-500-12087
June 2006



NREL is operated by Midwest Research Institute • Battelle Contract No. DE-AC36-99-GO10337



Determination of Elastic Twist in Horizontal Axis Wind Turbines (HAWTs)

Final Report: August 1989

F. Stoddard, V. Nelson, K. Starcher, B. Andrews
*Alternative Energy Institute – West Texas State University
Canyon, Texas*

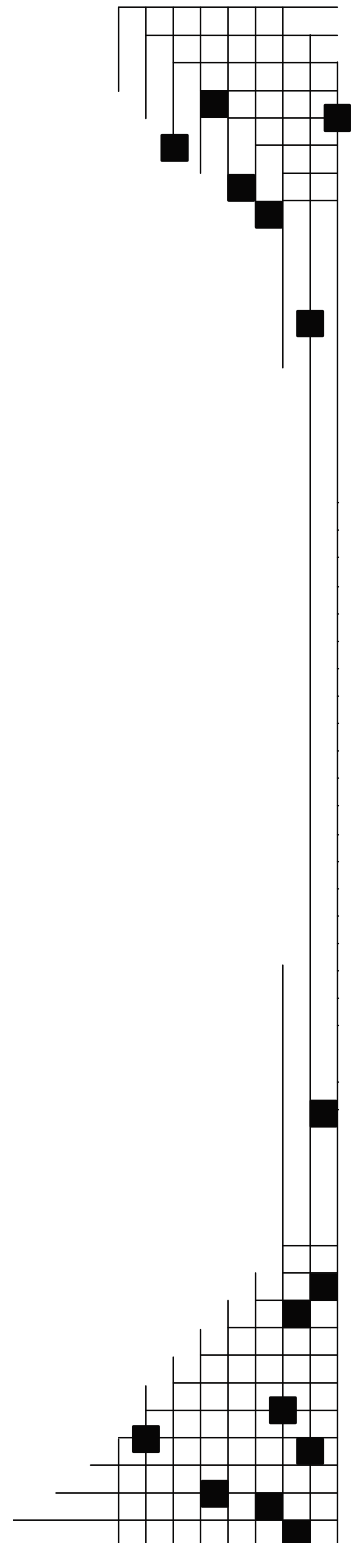
NREL Technical Monitor:

Prepared under Subcontract No RL-6-06013

Subcontract Report

NREL/SR-500-12087

June 2006



National Renewable Energy Laboratory

1617 Cole Boulevard, Golden, Colorado 80401-3393
303-275-3000 • www.nrel.gov

Operated for the U.S. Department of Energy
Office of Energy Efficiency and Renewable Energy
by Midwest Research Institute • Battelle

Contract No. DE-AC36-99-GO10337

**This publication was reproduced from the best available copy
Submitted by the subcontractor and received no editorial review at NREL**

NOTICE

This report was prepared as an account of work sponsored by an agency of the United States government. Neither the United States government nor any agency thereof, nor any of their employees, makes any warranty, express or implied, or assumes any legal liability or responsibility for the accuracy, completeness, or usefulness of any information, apparatus, product, or process disclosed, or represents that its use would not infringe privately owned rights. Reference herein to any specific commercial product, process, or service by trade name, trademark, manufacturer, or otherwise does not necessarily constitute or imply its endorsement, recommendation, or favoring by the United States government or any agency thereof. The views and opinions of authors expressed herein do not necessarily state or reflect those of the United States government or any agency thereof.

Available electronically at <http://www.osti.gov/bridge>

Available for a processing fee to U.S. Department of Energy
and its contractors, in paper, from:

U.S. Department of Energy
Office of Scientific and Technical Information
P.O. Box 62
Oak Ridge, TN 37831-0062
phone: 865.576.8401
fax: 865.576.5728
email: <mailto:reports@adonis.osti.gov>

Available for sale to the public, in paper, from:

U.S. Department of Commerce
National Technical Information Service
5285 Port Royal Road
Springfield, VA 22161
phone: 800.553.6847
fax: 703.605.6900
email: orders@ntis.fedworld.gov
online ordering: <http://www.ntis.gov/ordering.htm>



SUMMARY

This report presents the results of a project at the Alternative Energy Institute (AEI) which measured and calculated the elastic twist of three representative composite horizontal-axis blades: Carter 300, Gougeon ESI 54, and UTRC 8 kW. The report covers the experimental method used to inspect and test the blades, the theory and computer programs used to develop semi-empirical analytical blade models, calculation of the steady elastic twist of the three blades in simulated operation, and comparison with actual field performance of the wind turbines.

The experimental method employed is suitable for normal engineering blade testing, including quality control, verification of theoretical results, and iteration of design parameters. The computer codes developed under this project are suitable for engineering design of composite wind turbine blades. Suitable instructions for use of the programs are given.

Theory developed under this project can calculate a practical aeroelastic model of geometrically and structurally complex composite wind turbine blades. Such blades show great promise for reducing the weight and cost of future wind turbines; however, they are usually highly twisted and tapered, have variation in mass and stiffness properties in both spanwise and chordwise directions, and employ various forms and orientations of reinforcement, all of which present significant uncertainties in determining the engineering stiffness constants. The method developed here allows straightforward derivation of the blade section properties (without resorting to difficult-to-implement and error-sensitive finite element computer codes) by combining classical beam analysis with laboratory test results in a semi-empirical process here termed "mass-elastic triangulation." The aeroelastic blade model thus achieved has the same static deflection and the same fundamental frequencies of vibration, in flapping, lead-lag, and torsion, as the actual blade, and can thus be used in aeroelastic calculations.

Linear beam theory with a simple unidirectional stress state is assumed. The analysis calculates area-weighted flexural and torsional moduli for the elastic beam, determines the beam properties relative to the principal axes, and then performs coordinate transformations to develop blade axis sectional properties. Deflections are calculated by referring the applied blade section loading and moments to the aerodynamic center (quarterchord), the mass center, the bending centroid (weighted area centroid), or the shear center (elastic axis) and transforming the deflections by superposition. Section geometry and moduli are verified by a series of laboratory bench tests including straightforward static load measurements on the actual blade, close blade inspection, and optional blade sectioning and coupon testing, which are also covered in this report. The section moduli and mass moments were verified by measuring experimental fundamental frequencies and modes of vibration.

Elastic twist results are presented for the three test blades: Carter 300 glass-epoxy, Gougeon ESI 54 wood-epoxy, and UTRC 8 kW pultrusion, illustrating the wide variation in the section properties predicted by (1) composite analysis alone, (2) static deflection empirical tests alone, (3) natural frequency tests alone, and (4) the present method, which combines all three.

Elastic twist of the UTRC blade varied from 3.3 degrees nose down at 4.5 m/s (10 mph, 0 kW) to 0.25 degrees nose up at 13.4 m/s (30 mph, 13 kW); the section torsion was dominated by the lift offset from the shear center and most of the twist occurred in the flexbeam. This was seen to greatly affect the performance and helped explain the actual vs. calculated UTRC 8 kW power curve.

The ESI blade, for both the 77 and 90 rpm production versions, would have had less than 0.1 degree of elastic twist for all conditions if no tip flaps were present, due to its very high torsional stiffness. However, the elastic twist increased to about 0.33 degrees nose up for the standard flat plate tip, and 0.41 degrees nose down for the larger "whisper tip." Still, the effects of this live twist on the ESI performance were negligible.

The Carter blade has substantial blade leading edge ballast weight, which was seen to dominate the section torsion, resulting in over two degrees nose down live twist at high wind speeds. The ballast produced a very beneficial reduction of blade tip deflection, and also counteracted the large nose up elastic moment due to aerodynamic lift offset. Neither the ESI nor the Carter power curve test vs. theory discrepancies could be explained by elastic twist effects. The most significant factors affecting elastic twist were found to be shear center location, flap bending deflection, and added ballast weight.

TABLE OF CONTENTS

1.0 Introduction	
.1	Background..... 1
.2	Composite Blade Sources of Uncertainty..... 2
.3	Sources of Elastic Twisting Moments..... 4
2.0 Project Scope and Methodology	
.1	Goal of the Project..... 5
.2	Semi-Empirical Approach..... 5
.3	Blades Chosen for this Project..... 6
3.0 Experimental Method	
.1	General..... 7
.2	Initial Blade Inspection..... 7
.3	Blade Control Axis Location and Twist..... 7
.4	Blade Template Construction..... 7
.5	Blade Sectioning..... 11
.6	Blade Mass Distribution..... 15
.7	Section Aerodynamic Corrections..... 16
.8	Blade Laminate Coupons..... 17
.9	Static Deflection Testing..... 18
.9.1	Setup and Method..... 18
.9.2	Load Cases..... 21
.9.3	Deflection Measurement Method..... 22
.9.4	Deflection Corrections to Quarterchord..... 23
.9.5	Deflection Results..... 24
.10	Blade Vibration Modal Survey..... 24
4.0 Theoretical Method	
.1	General..... 32
.2	Composite Blade Section Properties Calculation..... 32
.2.1	Section Description..... 32
.2.2	Section Area and Stiffness Properties..... 33
.2.3	Section Torsional Rigidity..... 36
.2.4	Section Masses and Inertias..... 37
.3	Blade Static Deflection Calculation..... 38
.4	Blade Natural Bending Frequencies and Modes..... 41
.5	Blade Torsional Frequencies and Modes..... 45
5.0 Development of Blade Mass/Elastic Models	
.1	Triangulation of Mass/Elastic Properties..... 47
.2	Uniform FRP Blade Example: UTRC Blade..... 47
.2.1	Method..... 47
.2.2	Prediction of Engineering Moduli and Densities..... 47
.2.3	Prediction of Spar Properties..... 50
.2.4	Initial Predicted Results..... 50
.2.5	Specification Verification by Blade Sectioning..... 50
.2.6	Verification by Deflection Tests..... 51
.2.7	Verification by Frequency Tests..... 53
.2.8	Final Aeroelastic Model..... 55

.3	Wood-Epoxy Blade Example: Gougeon ESI 54 Blade.....	56
.3.1	ESI Blade Description.....	56
.3.2	Derive Initial Section Properties.....	57
.3.3	Verification by Flexural Tests.....	61
.3.4	Verification of Static Twist.....	61
.3.5	Verification by Frequency Test.....	64
.3.6	Final Aeroelastic Model.....	66
.3.7	ESI Tip Flaps.....	66
.4	Complex Composite FRP Example: Carter 300 Blade.....	68
.4.1	Carter Blade Description.....	69
.4.2	Ascertain Blade Geometry.....	70
.4.3	Ascertain Ballast and Weight Distribution.....	73
.4.4	Derive Carter Design Composite Values.....	75
.4.5	Derive Contribution of I-Beam.....	79
.4.6	AEI Measurements for Composite section Properties.....	80
.4.7	Verification by Flexural Test.....	81
.4.8	Verification of Static Twist.....	83
.4.9	Comparisons of Running Mass Distributions.....	84
.4.10	Verification by Frequency of Vibration Tests.....	86
.4.11	Final Aeroelastic Model.....	87
	6.0 Elastic Twist Calculation	
.1	General.....	88
.2	Calculation of Blade Centrifugal Tension and Moments.....	88
.3	Derivation of Section Torsion.....	90
.4	Aerodynamic Loads.....	92
.5	General Discussion: Operating Blade Elastic Deflections.....	93
.6	UTRC Blade Deflections.....	95
.7	ESI Blade Deflections.....	97
.8	Carter Blade Deflections.....	98
	7.0 Performance Calculations and Results	
.1	Aerodynamic Calculations.....	106
.2	Operational Cases Considered.....	110
.3	Turbine Power Curve Performance Results.....	110
.3.1	UTRC 8 kW Blade.....	112
.3.2	ESI 54S Blade.....	112
.3.3	Carter 300 Blade.....	112
.4	Discussion.....	112
	8.0 Conclusions	
.1	Comparison of Predicted and Experimental Results.....	114
.2	use of Method for Analysis of Composite Blades.....	114
.3	use of Method for Blade Quality Control for Manufacturers.....	114
.4	Design Implications of Results.....	114
.5	Recommendations for Further Work and HAWT Improvement.....	115
	9.0 References	
	116

10.0 Appendices

.1	Computer Code: Determination of Composite Blade Section Properties.....	118
.1.1	Brief Discussion.....	118
.1.2	Program Listing, STRESSM.....	118
.1.3	Sample Run, Carter Blade.....	126
.1.4	Sample Run, ESI Blade. No Tips or Ballast.....	130
.2	Computer Code: Blade Deflections and Stresses Under Loading.....	134
.2.1	Brief Discussion.....	134
.2.2	Program Listing, STRESSDATA.....	134
.2.3	Sample Run, Carter Blade.....	136
.2.4	Sample Run, ESI Blade, With Tip Weight.....	138
.3	Computer Code: Composite Blade Bending Frequencies of Vibration.....	139
.3.1	Brief Discussion.....	139
.3.2	Program Listing, VIBRATE.....	140
.3.3	Sample Run, UTRC Blade, Static and 108 rpm.....	141
.3.4	Program Listing, VIBETIP, (Tip Weight Version).....	143
.4	Computer Code: Composite Blade Torsional Frequencies of Vibration.....	143
.4.1	Summary.....	143
.4.2	Program Listing, TORSVIB.....	143
.4.3	Sample Run, ESI Blade.....	145
.4.4	Program Listing, TVIBTIP, (ESI Blade With Tip Inertia).....	145
.4.5	Sample Run, ESI Blade with Tip Inertia.....	146
.5	Computer Code: Steady Elastic Twist Calculation.....	146
.5.1	Program Listing, STRESSELASTIC.....	146
.5.2	Sample Run, UTRC Blade.....	156
.6	Calculation of Drag Polars for Eight Wind Turbine Blade Sections.....	158
.6.1	Procedure and Analysis.....	158
.6.2	Planform Data Files and Drag Polars.....	158
.7	Checklist Sent to Blade Manufacturers.....	159

LIST OF FIGURES

1.1	Carter 300 Blade.....	1
1.2	ESI 54 Blade.....	2
1.3	UTRC 8 kW Blade.....	2
1.4	Composite Blade Sources of Uncertainty.....	3
2.1	Triangulation of Mass/Elastic Properties.....	5
3.1	Test Stand Fixture.....	8
3.2	Blade axis / station lines.....	9
3.3	Template construction.....	10
3.4	Twist Measurement by Mechanical Clinometer.....	11
3.5	Carter Blade interior Arrangement.....	14
3.6	Lift-Drag Polars Compared (Specified vs. Measured / EPPLER Code).....	18
3.7	Laser Base Plane Alignment.....	19
3.8	Carter Blade: Deflection and Creep.....	20
3.9	Clinometer Error Idiosyncrasy.....	21
3.10	Load Cases.....	22
3.11	Test Table.....	23
3.12	Deflection Results, Carter (uncorrected).....	25
3.13	Deflection Results, ESI (uncorrected).....	26
3.14	Deflection Results, UTRC (uncorrected).....	27
3.15	Frequency of Vibration: Carter blade.....	29
3.16	Frequency of Vibration: ESI blade.....	30
3.17	Frequency of Vibration: UTRC blade.....	31
4.1	General Spar-Shell Crosssection.....	32
4.2	Actual vs. Equivalent Laminate.....	33
4.3	Mohr's Circle for Principal Axes.....	35
4.4	Section Torsional Moments.....	39
4.5	General Deflected Blade: Holzer/Myklestad.....	42
4.6	Free Body Diagram: Holzer Table.....	42
4.7	Unit Load and Moment Diagrams.....	43
4.8	Free Body Diagram for Torsional Vibration.....	46
5.1	UTRC Cross section.....	48
5.2	Rule of Thumb, FRP Directional Properties.....	49
5.3	UTRC Flap Deflection, Test vs. Calculation vs. Specifications.....	51
5.4	UTRC Vibration Modes, Test (modal survey).....	53
5.5	UTRC Flapwise Vibration Modes, Predicted vs. Test.....	54
5.6	ESI Section EA: AEI Calculation vs. GBI Specification.....	57
5.7	ESI Section EI _{flap} : AEI Calculation vs. GBI Specification.....	58
5.8	ESI Section EI _{lag} : AEI Calculation vs. GBI Specification.....	58
5.9	ESI Blade Running Mass: AEI Calculation vs. GBI Specification.....	59
5.10	ESI Section Shear Center: AEI Calculation vs. GBI Specification.....	59
5.11	ESI Section GJ: AEI Calculation vs. GBI Specification.....	60
5.12	ESI Blade Deflection: Predictions vs. Test.....	61
5.13	ESI Blade Static Twist.....	62
5.14	ESI Blade Static Twist: (50 lb torsion load case).....	62

5.15	ESI Blade Static Twist: Predictions vs. Test	63
5.16	ESI Blade Twist: Predictions vs. Test (flexure load case)	63
5.17	ESI Blade Frequency Modes	64
5.18	ESI Blade Final GJ Values	66
5.19	ESI Tip Flaps	67
5.20	Carter Blade Internal Structure	69
5.21	Carter Blade Free Body Diagram	69
5.22	Carter Dead Load Twist	71
5.23	Carter Blade Twist from Measurement	71
5.24	Carter Blade Error Scatter Plot, Twist Data	72
5.25	Carter Blade Ballast Summary	74
5.26	Carter Blade Laminate Schedule	77
5.27	Carter Spanwise Weight Density (Spar Portion)	78
5.28	Carter Static Deflection: Three Load Cases (Corrected)	82
5.29	Carter Static Deflection: AEI Calculation vs. Measured (Corrected)	82
5.30	Carter Measured Twist: (225 lb test) Predicted vs. AEI Calculation vs. Specification	83
5.31	Carter Measured Twist: 325 lb test	84
5.32	Carter Measured Twist: Torsion Load Case	84
5.33	Carter Blade Running Mass Comparisons	85
6.1	Discrete Lumped Mass Case	88
6.2	Blade Section Free Body Diagram	90
6.3	Code Output: UTRC Blade, 30 mph Case	93
6.4	UTRC Flap Deflection (10-70 mph)	96
6.5	UTRC Elastic Twist (10-70 mph)	96
6.6	ESI Elastic Twist Distribution	99
6.7	Carter Elastic Twist (10 and 30 mph Cases)	101
6.8	Carter Pitching Moment Distribution (10 and 30 mph Cases)	102
6.9	Carter Section Free Body Diagrams	103
6.10	Carter Elastic Twist and Section Moment, No Ballast (30 mph)	104
7.1	2-D Section Data Comparison: Standard vs. Viterna vs. Tangler	107
7.2	UTRC: Comparison of Post-stall Routines	108
7.3	ESI: Comparison of Post-stall Routines	108
7.4	Carter: Comparison of Post-stall Routines	109
7.5	UTRC Power Output: Comparison of Test, Predictions, and Elastic Twist	110
7.6	ESI 54 Power Output: Comparison of Data, Predictions, and Elastic Twist	110
7.7	Carter 300 Power Output: Comparison of Data, Predictions, and Elastic Twist	110

LIST OF TABLES

3.1	Internal Structural Measurements: Carter blade.....	12
3.2	Internal Structural Measurements: ESI blade.....	13
3.3	Internal Structural Measurements: UTRC blade.....	13
3.4	Sectioned cg and Mass Data: Carter blade.....	15
3.5	Sectioned cg and Mass Data: ESI blade.....	16
3.6	Sectioned cg and Mass Data: UTRC blade.....	16
3.7	Frequency of Vibration Results from test (HZ).....	28
4.1	FRP Unidirectional Weights and Moduli.....	37
5.1	UTRC blade Section Properties Compared to Manufacturers' Specifications and Sectioned Values.....	50
5.2	UTRC Frequency of Vibration Results.....	54
5.3	UTRC Final Blade Model Analysis.....	56
5.4	ESI Blade Frequency of Vibration: Predictions vs. Test.....	65
5.5	ESI Blade Final Aeroelastic Model.....	66
5.6	ESI Vibration Frequencies with Tips.....	68
5.7	Carter Blade Blade Geometry, Measured vs. Carter Specifications.....	73
5.8	Carter Blade Weights Summary.....	74
5.9	Carter Reinforcements, As-Built Summary.....	76
5.10	Carter Blade Laminate Design.....	78
5.11	Carter I-Beam Section Data.....	79
5.12	Carter I-Beam Moments of Inertia.....	80
5.13	Carter Blade Section Measurements vs. Carter Specifications.....	80
5.14	Carter Blade Running Mass Comparison.....	85
5.15	Carter Blade Flapwise Frequency Comparison.....	86
5.16	Carter Blade Final Masses and Section cg's.....	87
5.17	Carter Blade Final Aeroelastic Blade Model.....	87
6.1	UTRC Flap Deflection.....	95
6.2	UTRC Elastic Twist.....	95
6.3	ESI Tip Flap Deflection.....	98
6.4	ESI Tip Elastic Twist.....	98
6.5	Carter Flap Deflection.....	100
6.6	Carter Lead-Lag Deflection.....	100
6.7	Carter Elastic Twist.....	100
6.8	Carter Blade Section Torsion Components.....	101
7.1	Airfoil Lift and Drag Data Input to PROPPC.....	106

1.0 INTRODUCTION

1.1: BACKGROUND

High performance wind turbine rotors in the future may have light, flexible blades to increase cost competitiveness. Blades which are elastic in torsion will experience small pitch changes during operation which will cause large changes in aerodynamic loading leading to errors in wind turbine performance, especially near rated power conditions where blade loading and thrust are high. New airfoils for wind turbines have been and are being developed in order to attenuate the real-world detrimental effects of surface degradation, manufacturing tolerance, and unsteady aerodynamics in turbulent winds (Ref. 1).

Certain wind turbine rotors now in use have demonstrated aeroelastic instabilities and blade flexibility problems in operation (Ref. 2). Invariably the primary cause of flutter and divergence is blade torsional flexibility. In stall-regulated (fixed pitch) rotor systems an added potential flexibility problem is stall flutter. In pitch-regulated (controlled pitch) rotor systems, blade elastic twist in operation can lead to major increases in pitching moment and the attendant pitching system power required. Elastic blade twist near rated conditions, if nose-down or towards reducing angle of attack, causes delay of stall and negates the desired load alleviation, thereby causing potential rotor overloading. Elastic blade twist if nose-up, or toward increasing angle of attack, causes premature blade stall, the potential for stall flutter, and loss of power. The same arguments, though less significant, apply at operating conditions below rated or stalled conditions.

Up to now, the problem of calculating the blade elastic twist in operation has not been addressed except as qualitative or grossly estimated minor effects. Certainly that approach is defensible for heavy, massive blades which have little bending deflection and virtually no torsional deflection. For wind turbines to be cost-competitive in the future, however, the blade and rotor weights must be low, leading to unavoidably increased blade flexibility which must be accurately estimated. This project analyzed in a quantitative, practical, and reproducible way the amount of elastic twist for three representative wind turbines (Carter 300, ESI 54, and UTRC, Figs. 1.1 - 1.3), the origins of the elastic twist, and the likely changes in the wind turbine field performance.

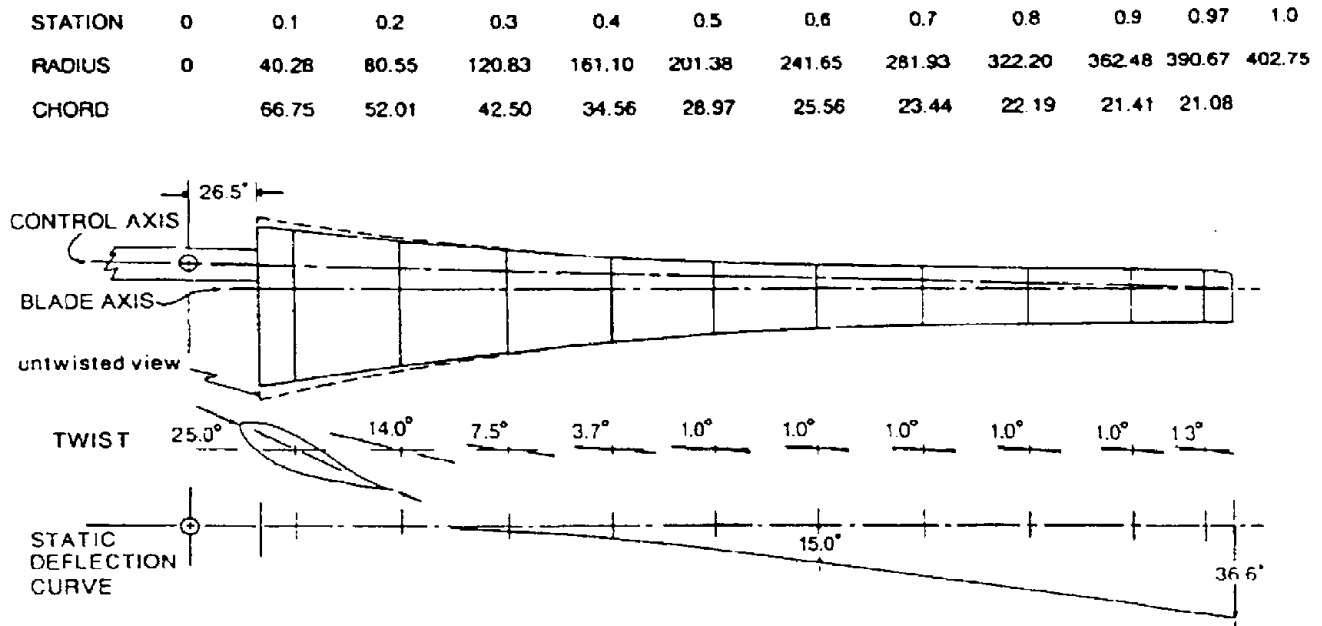


FIG 1.1 CARTER 300 BLADE

STATION	0	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
RADIUS	0	32.41	64.81	97.22	129.62	162.03	194.44	226.84	259.25	291.65	324.06
CHORD		16.80	20.85	24.20	22.72	19.13	18.09	17.28	15.43	13.66	11.65

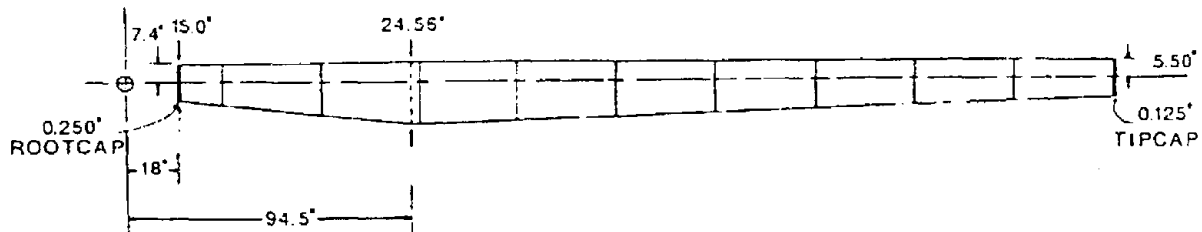


FIG. 1.2 ESI 54 BLADE

STATION	0	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
RADIUS	0	19.2	38.4	57.6	76.8	96.0	115.2	134.4	153.6	172.8	192.0
CHORD				15.0	15.0	15.0	15.0	15.0	15.0	15.0	15.0

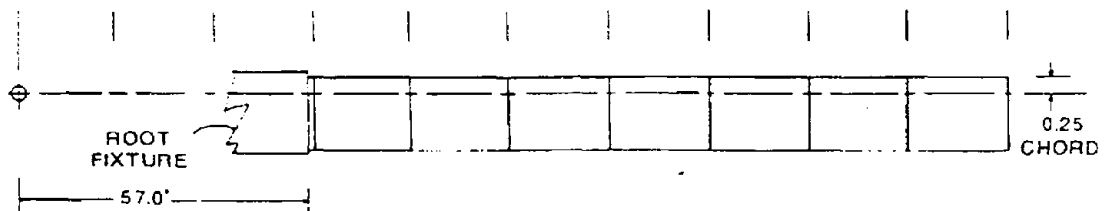


FIG. 1.3 UTRC 8 kW BLADE

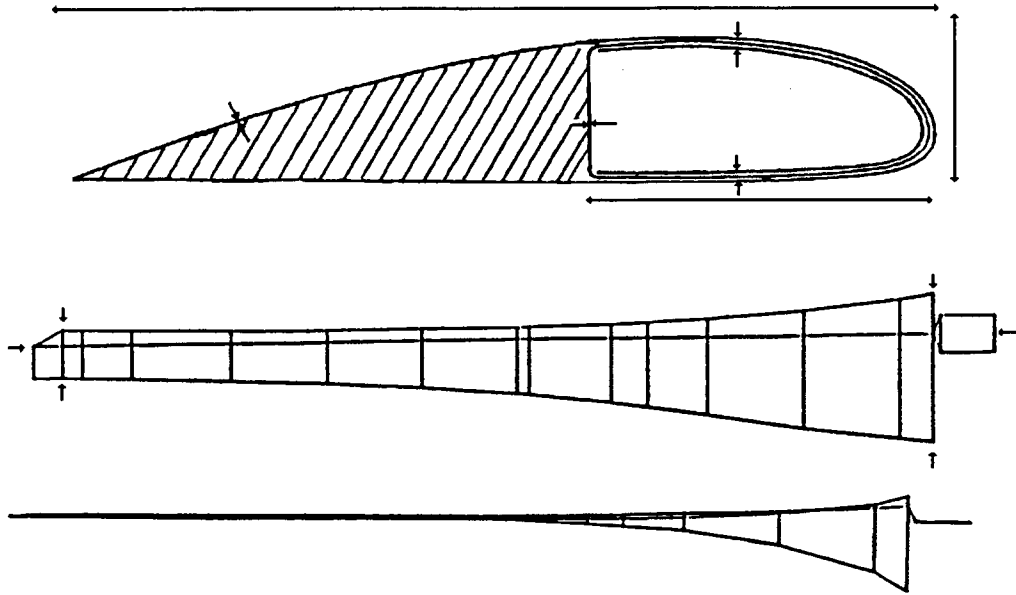
1.2: COMPOSITE BLADE SOURCES OF UNCERTAINTY

The combined bending and torsion of composite structures, especially high aspect ratio (span-to-width) HAWT blades, is difficult to analyze, characterize, and predict (Ref. 3). The degree of structural anisotropy of a composite laminate in general extends to a stiffness matrix of 64 possible engineering constants. The methodology of isotropic strength of materials does not work since the molecular and physical structures of composites are radically different from metallic crystal structures (Ref. 4, 5, 6). Therefore, simplifying assumptions were made. For the unidirectional bending plus torsion indicative of wind turbine (and helicopter) blades a simple stress state was assumed, and the laminates were considered to be bidirectional, needing only four elasticity constants for engineering analysis: primary and transverse tensile and shear moduli. It was originally thought that these four could be verified in coupon tests, but that was found to be too difficult for the scope of this project.

In addition to the modulus uncertainties referred to above, manufactured composite blades, particularly those which are molded by hand (Carter, ESI) have significant geometric and internal structural sources of uncertainty (Fig. 1.4). These generally boil down to geometric errors (twist, taper, location of control axis), mass location errors (structural weight distribution, ballast weights), and the unavoidable nonuniformities in the individual laminates or plies (section errors).

Section Errors:

- Airfoil Shape
- Chord
- Thickness
- Web Placement
- Skin Thickness
- Spar Thickness



Geometry Errors:

- Twist Angle
- Control Axis Placement
- Alignment
- Tip Geometry

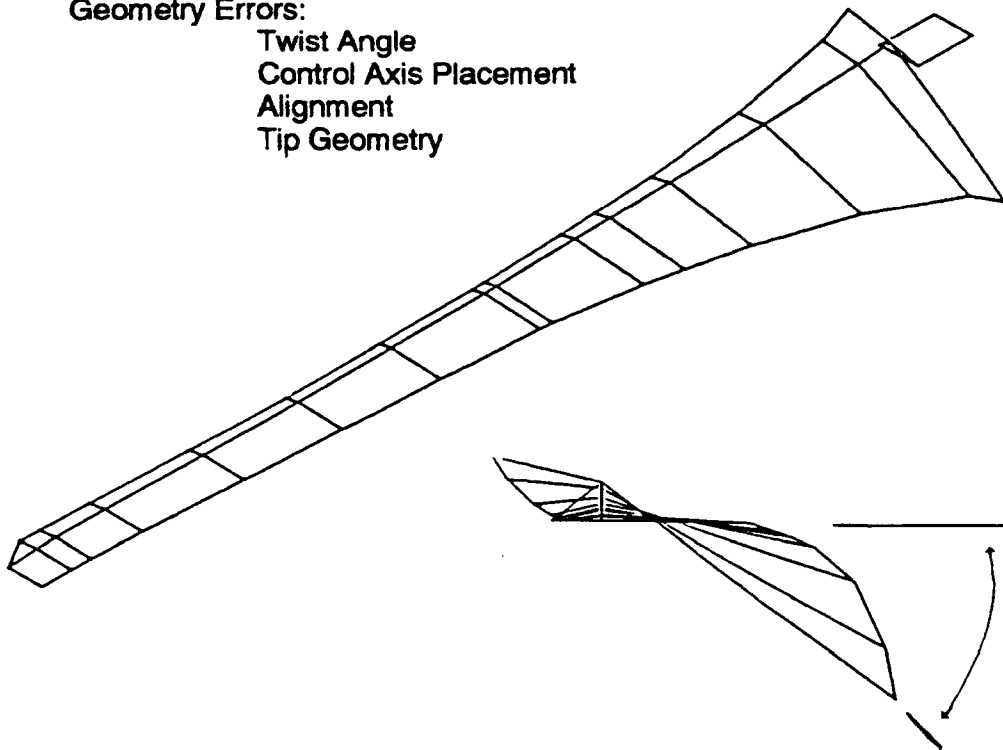


FIG. 1.4 COMPOSITE BLADE SOURCES OF UNCERTAINTY

1.3: SOURCES OF ELASTIC TWISTING MOMENTS

The elastic twist of interest is the steady state twist due to steady forces and pitching moments on the blade. These produce the torsion on the blade which is resisted by the elastic blade structural sections. Additional elastic twist is due to structural coupling between bending and torsion. There are two major sources of the torsional moments: aerodynamic and inertial; that is, the moments due to aerodynamic pressure distributions and the moments due to centrifugal forces. The analysis of these moments is complicated by the difficult blade aerodynamics, which depends on historical 2-D data which does not model the true post stall and unsteady aerodynamic behavior in wind turbulence, and is further aggravated by manufacturing errors in blade airfoil post stall and section geometry. The analysis is also complicated by the typical composite wind turbine blade structure, which has complex geometry and is composed of fiber-reinforced resin, both of which introduce nonuniformities and structural coupling. A further complication is the blade-hub attachment. Soft attachments, such as the Carter "snubber," can cause large elastic pitch errors at the hub, which then add to the elastic twist, as verified during this testing program. Elastic twist introduced by the flexbeam and the pendulum weights for the UTRC was treated as an assumed root twisting moment.

2.0 PROJECT SCOPE AND METHODOLOGY

2.1: GOAL OF THE PROJECT

The major goal of this study was to ascertain the elastic blade twist due to wind turbine operation which leads to errors in the prediction of loads and performance for three promising and representative wind turbine rotors. A secondary goal was to develop, verify, and present a practical engineering methodology for the estimation of this elastic twist on other wind turbine rotors.

2.2: SEMI-EMPIRICAL APPROACH

This program was divided into two parts of equal importance, experimental and theoretical. There were not enough resources in the program to permit a sophisticated engineering analysis in either area, so a practical engineering approach was taken (Fig. 2.1), and has proved efficient and effective. This allowed the derivation of semi-empirical blade aeroelastic models with which to calculate the steady elastic twist.

Computer Codes:

- BLADE SECTION PROPERTIES
- STATIC DEFLECTION & STRESSES
- BLADE VIBRATION MODES

Laboratory Tests:

- EXTERNAL INSPECTION
 - SIMPLE FREQUENCY OF VIBRATION
 - STATIC DEFLECTION UNDER LOAD
 - BLADE SECTIONING & CLOSE INSPECTION
 - MODAL SURVEY
-

FIG. 2.1 TRIANGULATION OF MASS / ELASTIC PROPERTIES

The structural analysis programs employ practical and verified engineering techniques for the analysis of complex composite structure beams (Ref. 3). The wind turbine performance programs were developed by Solar Energy Research Institute (SERI), and are the standard aerodynamic codes in use in the industry (Ref. 7). The blade tests were performed at the Agricultural Research Service, U.S. Department of Agriculture (USDA) facility in Bushland, TX, under the cooperative agreement with AEI. The test equipment and testing techniques employed are standard laboratory items and methods generally available, with the exception of the Fast Fourier Transform (FFT) modal survey equipment provided by SERI, which was more sophisticated (Ref. 8).

The experimental testing program consisted of inspection, static loading, and frequency of vibration tests. Practical engineering laboratory techniques were used, which required only modest testing equipment (e.g., strip chart recorder, laboratory optical laser, laboratory strain gages), and permitted full-scale bending and torsion tests of the blades cantilevered from floor weldments. Simple mechanical measurement techniques were used at the cost of great accuracy (e.g., section twist measurements were to ± 10 minutes of arc), but provided a simpler physical feel for the experiments.

The theoretical program used a mainframe computer (PRIME) for the calculation of blade structural parameters, blade aero- and inertial elastic torsion and loading, blade bending and twist, and frequencies of vibration. The mainframe computer was also used for the calculation of aerodynamic loading and wind turbine rotor performance. However, all the programs could be compiled and run on personal computers. Again as in the testing program, a practical engineering approach was taken for the analysis of the blade structure, insofar that a beam analysis program was used that does not calculate the complex shear stress and shear flow, but instead assumes a principal stress state, unidirectional component

laminates which are representable by only four (empirical) engineering constants, and linear beam theory (plane sections remain plane). Again, the practical approach allowed many more calculations and iterations to be made, and permitted a better and more transferable physical feel for the structural effects.

2.3: BLADES CHOSEN FOR THIS PROJECT

The blades selected for this project (Carter 300, Gougeon ESI 54, and UTRC 8 kW) represent a wide variety of promising blade characteristics: hub retention, airfoils, rotor diameters, aerodynamic geometries, and structural arrangements. The choice was limited somewhat by these major constraints (1) available field test data, (2) verified manufacturing quality control of mechanical properties, (3) enough flexibility for potential aero- and inertial elastic effects to be seen, and (4) cost and time constraints. These blades were also chosen to represent a range of composite construction and manufacturing techniques which are all potential weight- and cost-savers for future wind turbines.

3.0 EXPERIMENTAL METHOD

3.1: GENERAL

The Carter and Gougeon blades (two blades of each) was procured directly from the manufacturer. One was returned for credit after static testing and the second was kept for sectioning. The UTRC blades were already the property of AEI. Additionally, a Carter hub was acquired for mounting its blade and ascertaining root compliance. ESI tip flaps were also acquired to determine their weights and predict their moments of inertia. The UTRC blade overspeed mechanism, the flexbeam/pendulum, was not studied in detail, only the outboard pultrusion.

In addition to the blades and rotor hardware, manufacturers were asked to provide specific data for the rotors (Appendix 10.7). The Gougeon blade was the only well-documented blade in manufacturer's correspondence and in the open literature. The Carter blade required considerable effort to ascertain internal structure, arrangement, and geometry.

3.2: INITIAL BLADE INSPECTION

The first requirement was to determine the actual blade external geometry and airfoil shape (measured at .75 and .95 span). These were used as the basis for the aerodynamic loading on the blade. The internal structure including laminate thickness and geometry, and their spanwise distributions also had to be found at each tenth station (STA) for input to the structural model. Blade manufacturers had been asked to provide the specification for blade external geometry and structural geometry, and the mechanical properties of the individual laminates used, but this job ultimately rested on the test/inspection program. The hub attachment had to be measured in the same way for its input to the structural model.

The blade geometric parameters, structural details, and airfoil were determined first. A test stand fixture was constructed for each blade (Fig. 3.1) which allowed the individual blades to be cantilevered above the laboratory floor. Each blade was either bolt-on or clamped mounted, or in the case of the Carter, mounted in its own rotor hub.

3.3: BLADE CONTROL AXIS LOCATION AND TWIST

A blade "control axis" is defined as the pitch axis for a pitching blade, hence "feathering axis" (e.g. as in the case of the Carter blade), or as the centroid of the root fixation for a rigid blade (ESI). Linear spanwise and chordwise measurements were taken to determine the blade taper and radius. Blade control, or feathering axis, was determined and then inscribed on the blades along with 0.1 radius section lines (Fig. 3.2). Control axis was assumed to be the root centroid axis for the ESI blade; it was the quarterchord axis for the other two. The blade control axis was used as the primary blade reference axis. Twist was first measured on the cantilevered blades by using a laboratory laser, optical bench, and mirrors mounted on the templates. This laser/mirror method was discarded since on the cantilevered blades a precise clinometer gave the same accuracy and was much easier to use. Blade static gravity deflection was zeroed by taking twist measurements for 0° and 180° pitch orientation. Root pitch alignment was also measured for the blade/hub attachment. There were minor discrepancies between the measured values and the manufacturers' specifications (see Section 5.0).

3.4: BLADE TEMPLATE CONSTRUCTION

Template construction was done by two methods: the first was nondestructive, the second was from the sectioned blade. The former method consisted of pine blanks (Fig. 3.3), which were rough cut on the inside, leaving a small gap between template and blade, to fit both the upper and lower surfaces of the test airfoils at each 0.1 radii (tenth station). The surface of the blade was coated with a thin layer of floor wax and then auto body filler was used to bridge the gap between template and blade surface, with care taken to

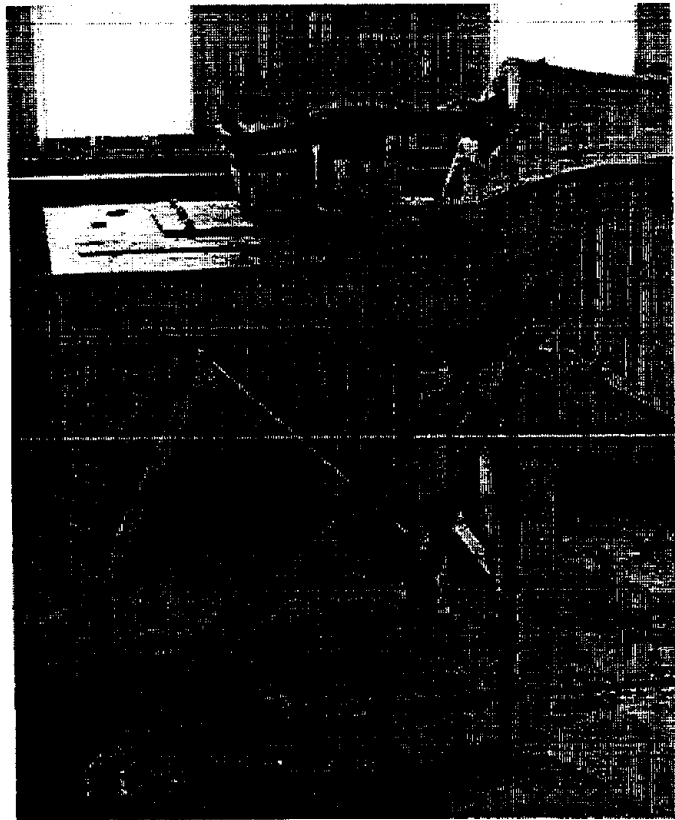
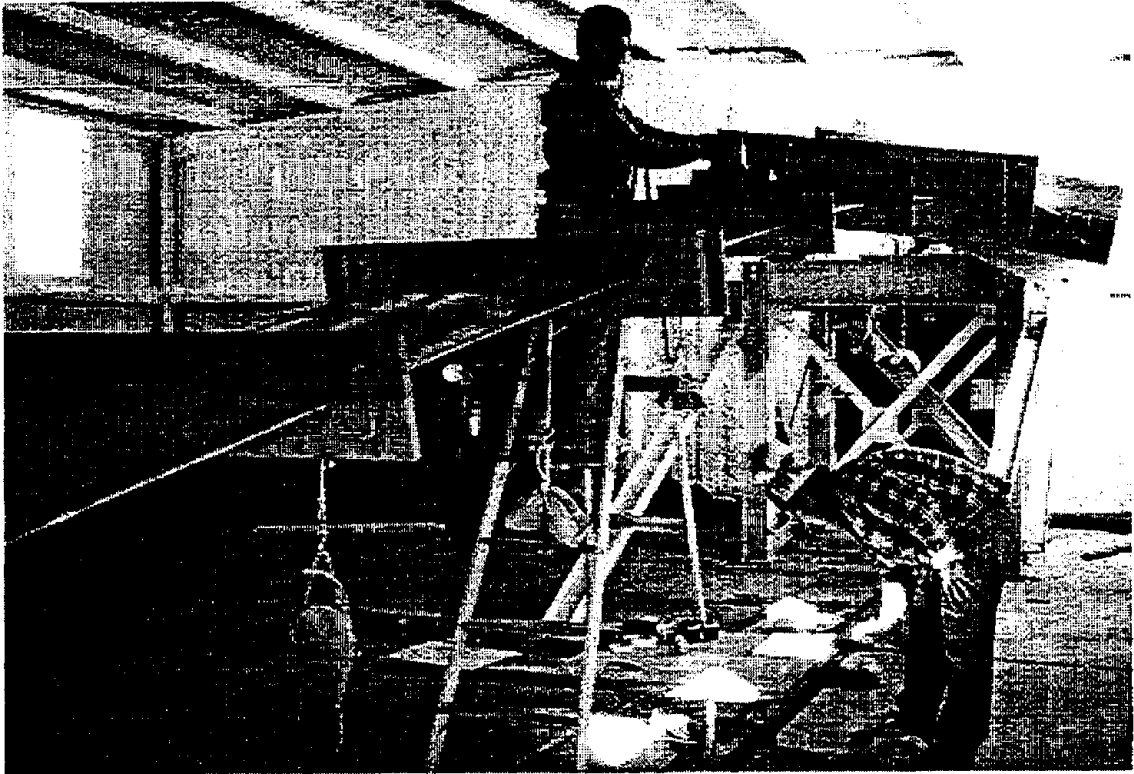


FIG. 3.1 TEST STAND FIXTURE

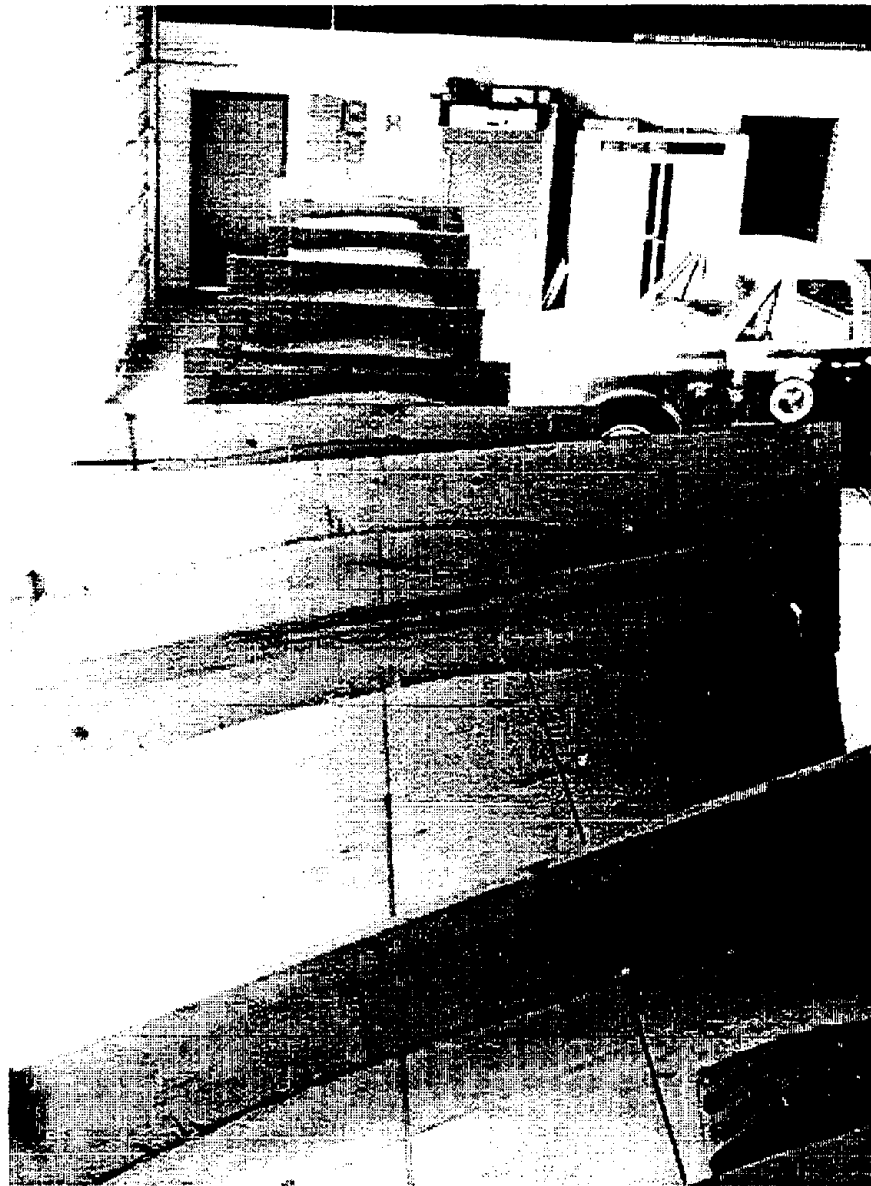


FIG. 3.2 BLADE AXIS / STATION LINES

preserve the chordline at the separation line between upper and lower template halves. Holes were drilled in the leading and trailing edges of the upper and lower parts of each station's template and then dowel rods were inserted. Rubber bands and strapping tape were used to secure the templates together for inspection and installation on the blades. The clinometer was placed on top of the templates above the control line for measuring twist angles (Fig. 3.4). The template surfaces that butted together were located at the relative chordline for the section at that radius. The floor wax allowed easy removal of the hardened templates after the tests were completed. The squared outer edges of the templates were used as the reference lines for determining the relative twist at each radii using the clinometer. The problems with this method were establishing an accurate chordline and achieving accurate airfoil shape.

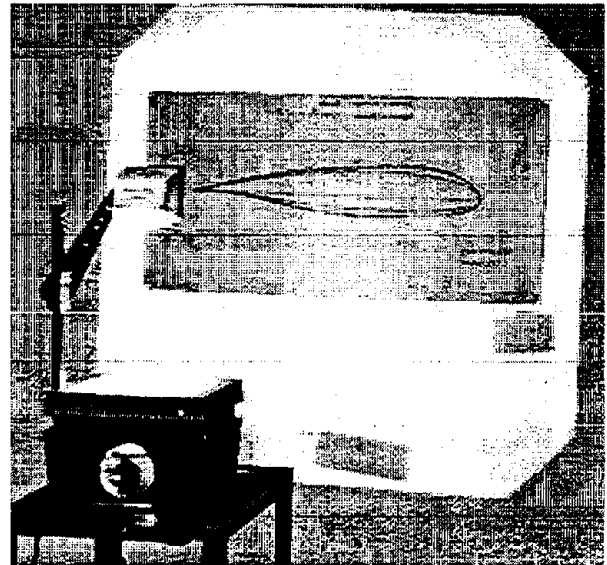
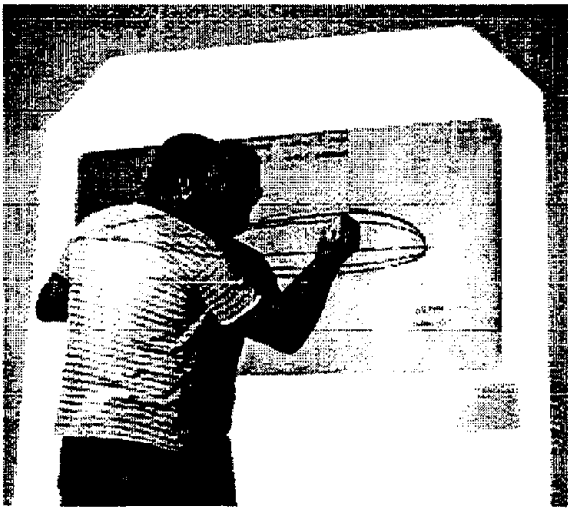


FIG. 3.3 TEMPLATE CONSTRUCTION

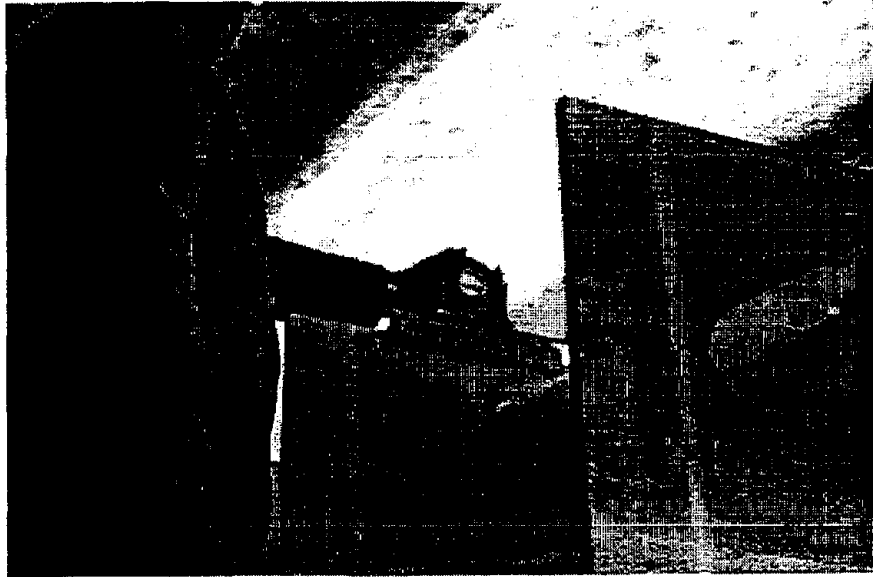


FIG. 3.4 TWIST MEASUREMENT BY MECHANICAL CLINOMETER

The latter method was much easier but depended on the sectioned (cut up) blade portions. The sections were traced onto craft paper and the chordlines marked on the tracing using a projection technique (Fig 3.3). The paper tracings were hung on a wall and transparencies of the members of the airfoil family were projected onto the tracing and judgment used in the location of the chordline. The chordline was then marked on the tracing. The various airfoils were also marked on the tracing for qualitative analysis of the cut sections, which included sending the tracings for EPPLER Code analysis (Sec. 3.7 below).

To construct the templates, the sections were placed on the above tracings with the chordline marked. The section then was placed on the two pine blanks with the chord line split between upper and lower surface. The section was then traced onto the wood for the construction of female templates for each station. The wood was finally cut and filed to closely approximate airfoil shape. Additional female templates were made at stations .75 and .95 for the aerodynamic evaluation.

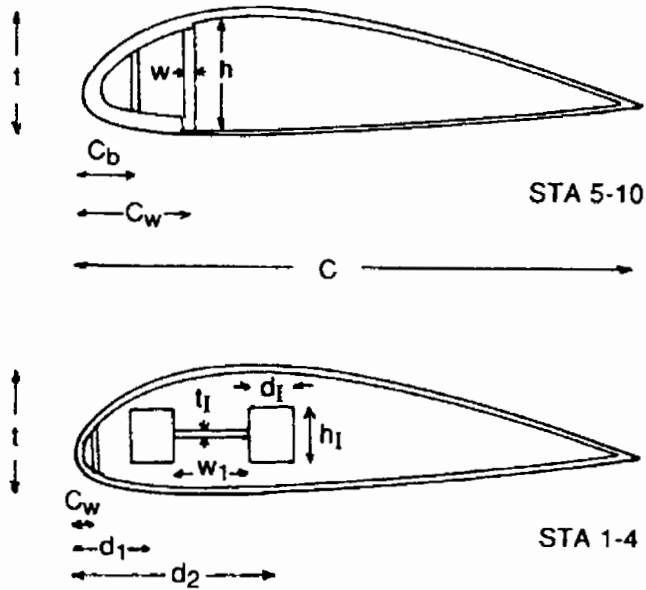
3.5: BLADE SECTIONING

The internal structural geometry could only be accurately determined by cutting a blade into sections. A blade from each set was cut into sections corresponding to its 10 spanwise stations. For each station internal structural measurements were taken to verify the manufacturer's specification (Tables 3.1, 3.2, and 3.3). This allowed internal arrangement verification of ballast and inserts; this was especially important for the Carter blade (Fig. 3.5). Another reason for sectioning the blade was to determine the chordwise center of gravity (cg) position (Sec. 3.6).

An accurate mass axis was drawn from the spanwise cg positions; this turned out to be a very sensitive indicator of actual laminate thickness, reinforcement/resin ratio, and weighted area centroid. The lumped mass model of the blade, which was used in the frequency of vibration program, was readily determined from sectioned data.

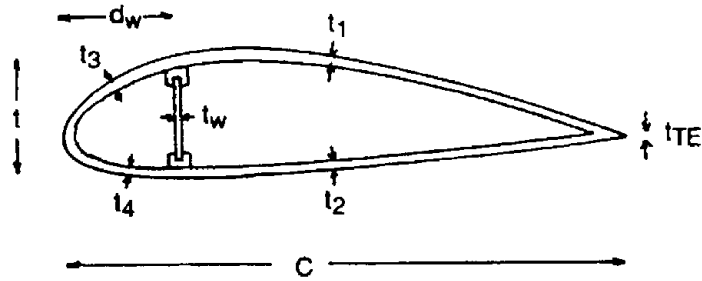
The actual airfoil section geometry was determined accurately from the sectioned blade. In the Carter blade, it was thought that shear relief could modify the airfoil shape when the section was cut, so the .75 and .95 airfoils were determined both before and after sectioning (Sec. 3.7).

TABLE 3.1: INTERNAL STRUCTURAL MEASUREMENTS: CARTER BLADE (in cm)



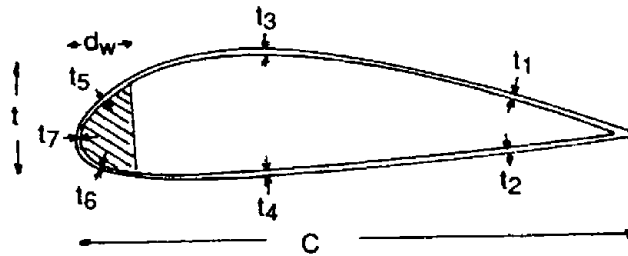
STATION	1	2	3	4	5	6	7	8	9	9.5
C	169.55	134.14	107.95	87.78	73.58	64.92	59.54	56.36	53.98	53.19
t	35.71	28.09	20.80	13.34	9.68	8.33	7.54	7.06	6.83	6.58
C _b	-	-	-	-	6.35	6.50	6.35	6.99	6.35	7.32
C _w	13.34	8.74	4.78	-	17.30	16.99	14.30	12.22	-	-
h	22.23	15.88	-	-	6.35	5.41	4.29	4.45	-	-
w	1.12	1.12	-	-	.97	.97	.97	.97	-	-
d ₁	-	-	9.22	6.20	-	-	-	-	-	-
d ₂	-	-	35.89	27.15	-	-	-	-	-	-
w ₁	25.10	27.00	21.92	16.84	-	-	-	-	-	-
t ₁	.64	.64	.64	.64	-	-	-	-	-	-
h ₁	6.35	6.05	6.05	5.00	-	-	-	-	-	-
d ₁	6.99	3.89	3.96	3.96	-	-	-	-	-	-

TABLE 3.2: INTERNAL STRUCTURAL MEASUREMENTS: ESI BLADE (in cm)



STATION	1	2	3	4	5	6	7	8	9
C	42.67	52.96	62.00	57.70	48.60	48.50	43.80	39.20	34.70
t	23.10	21.00	19.10	16.70	14.55	12.49	12.56	8.43	6.44
dw	-	20.70	20.25	18.60	17.30	16.00	14.30	12.45	10.96
tw	-	0.67	0.67	0.66	0.68	0.68	0.66	0.65	0.68
tTE	6.28	2.70	2.56	1.98	1.69	1.78	1.43	1.17	1.19
t1	6.37	2.80	2.45	1.99	1.73	1.79	1.42	1.24	1.20
t2	6.20	2.72	2.49	1.98	1.71	1.73	1.43	1.19	1.20
t3	6.43	2.72	2.49	1.99	1.79	1.68	1.38	1.23	1.16
t4	6.23	2.75	2.54	1.98	1.73	1.70	1.44	1.17	1.20
plies	24	10	9	7	6	6	5	4	4

TABLE 3.3: INTERNAL STRUCTURAL MEASUREMENTS: UTRC BLADE (in cm)



STATION	(Flexbeam)	4	5	6	7	8	9	10
C		37.96	37.82	37.42	37.90	37.95	37.90	37.90
t		-	5.00	4.96	4.93	4.88	4.96	4.96
dw		6.50	6.50	6.50	6.50	6.50	6.50	6.50
t1		.33	.27	.29	.31	.28	.38	.30
t2		.33	.27	.29	.31	.28	.38	.30
t3		.29	.27	.23	.26	.23	.27	.28
t4		-	.23	.22	.24	.32	.27	.32
t5		.27	.24	.25	.26	.29	.29	.29
t6		.23	.25	.23	.24	.24	.23	.22
t7		.36	.39	.41	.43	.41	.40	.40

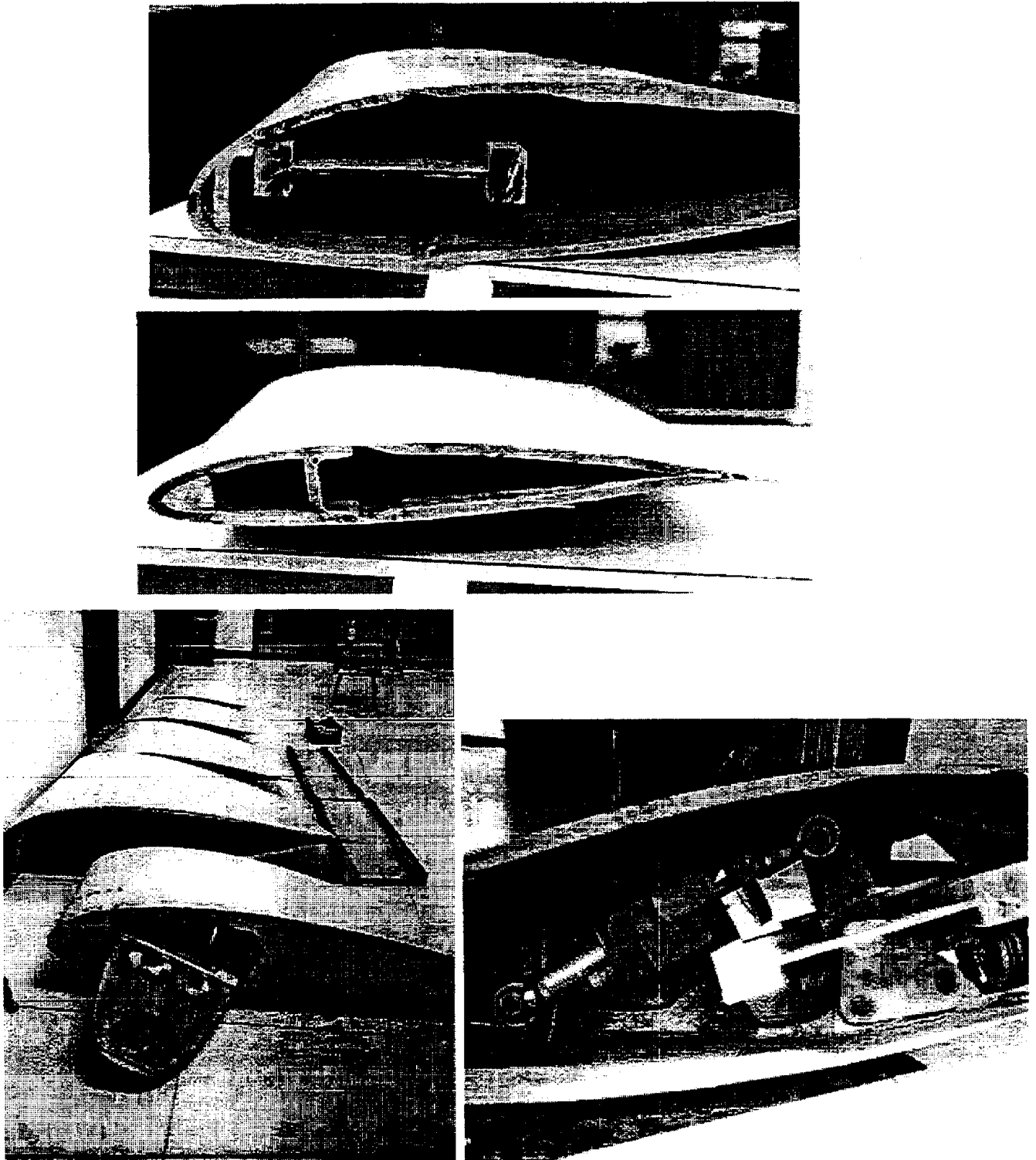


FIG 3.5 CARTER BLADE INTERIOR ARRANGEMENT

3.6: BLADE MASS DISTRIBUTION

The actual mass distribution of the individual blades was determined by measuring the mass of the individual sections on a balance scale and locating the local cg of each section using an edge balancing beam. The individual sections were placed on a sharp edge (angle iron) and the balance point was determined and the line marked on the section; this was done spanwise, chordwise and at a diagonal. The intersection of these lines was taken as the point mass location and was converted to X,Y,Z coordinates using the local quarterchord point as a 0,0 point (X,Y origin), and the center of rotation as the Z origin. These points and masses are summarized in Tables 3.4, 3.5, and 3.6, and were included as input data points for the computer models.

TABLE 3.4: SECTIONED cg AND MASS DATA: CARTER BLADE

STA	PANEL	cg POSITION (cm)				MASS (kg)	
		I-BEAM		BLADE SHELL		I-BEAM	SHELL
		SPAN W	CHORD W	SPAN W	CHORD W		
0 - 1	1	-	-	-	-	-	-
1 - 2	2	45.2	19.0	47.8	55.8	15.3	26.3
2 - 3	3	50.8	16.5	48.8	43.0	12.7	21.7
3 - 4	4			47.0	27.6	(4)	34.1
4 - 5	5			49.2	21.4		30.3
5 - 6	6	(see note 4)		48.3	16.6		25.3
6 - 7	7			52.2	13.7		23.1
7 - 8	8			45.7	12.2		20.6
8 - 9	9			55.4	15.2		12.5
9 - 10	10			44.3	11.6		19.4

NOTES:

- (1) BLADE WAS SECTIONED ON THE TENTH STATION LINES.
- (2) PANEL NUMBER REFERS TO SPANWISE SEQUENCE.
- (3) PANELS 4-6 CONTAIN SOME EMBEDDED I-BEAM AND POTTING RESIN.
- (4) ROOT RIB AND SNUBBER MECHANISMS ARE IN PANEL 1 PORTION.
- (5) PANEL 10 (BLADE TIP) CONTAINS BALANCE WEIGHT.
- (6) PANELS 9 & 10 FROM CARTER BLADE #29B, OTHER PANELS FROM CARTER #47A.

TABLE 3.5: SECTIONED cg AND MASS DATA: ESI BLADE

STATION	PANEL	cg POSITION (cm)		MASS (kg)
		SPANWISE	CHORDWISE	
0 - 1	1	18.4	18.6	12.36
1 - 2	2	44.8	21.0	19.85
2 - 3	3	41.7	25.4	18.03
3 - 4	4	43.4	27.5	16.10
4 - 5	5	42.8	24.6	12.81
5 - 6	6	41.5	22.5	11.23
6 - 7	7	42.0	21.0	9.07
7 - 8	8	43.5	18.5	7.60
8 - 9	9	41.8	15.8	6.01
9 - 10	10	40.7	14.1	6.33

NOTES:

- (1) PANEL 1 INCLUDED THE BLADE ROOT BUT NO STEEL STUDS.
- (2) NO TIP WEIGHT OR MECHANISMS INSTALLED.

TABLE 3.6: SECTIONED cg AND MASS DATA: UTRC BLADE

STATION	PANEL	cg POSITION (cm)		MASS (kg)
		SPANWISE	CHORDWISE	
0 - 1	1	-	-	2.50
1 - 2	2	-	-	2.50
2 - 3	3	-	-	18.24
3 - 4	4	23.0	12.1	3.75
4 - 5	5	25.2	12.0	3.64
5 - 6	6	24.4	11.8	3.64
6 - 7	7	24.5	11.6	3.64
7 - 8	8	25.0	11.9	3.64
8 - 9	9	24.2	11.6	3.70
9 - 10	10	24.6	12.2	3.70

NOTES:

- (1) STATIONS 0 - 3 COMPRISED THE FLEXBEAM/PENDULUM.
- (2) STATIONS 4 -10 COMPRISED THE PULTRUSION.
- (3) PANEL 3 CONTAINED THE ATTACHMENT HARDWARE TO THE FLEXBEAM.

3.7: SECTION AERODYNAMIC CORRECTIONS

The airfoil shape manufacturing errors (for STA 0.75 and 0.95) were characterized in two methods (Appendix 10.6). The first method was to trace the airfoil shape from the cut edge of the sectioned blade. The second method used the undisturbed airfoil and auto body putty; a fine layer of wax was applied and the putty was placed around the entire airfoil at the desired station. The putty was allowed to harden and then was removed with breaks at the leading and trailing edges. The putty was reassembled and then finished so that one side was flat, then a Xerox copy was made to show the airfoil shape. The shapes were then digitized and smoothed by Prof. Stan Miley, Texas A&M University, who was the aerodynamics consultant for the project. EPPLER code calculation determined a C_L at various angles of attack up to stall

(Appendix 10.6). The zero lift angle and the lift at 90 degrees were also computed. The differences between the blade specifications and the computer lift/drag polar were compared (Fig. 3.6).

3.8: BLADE LAMINATE COUPONS

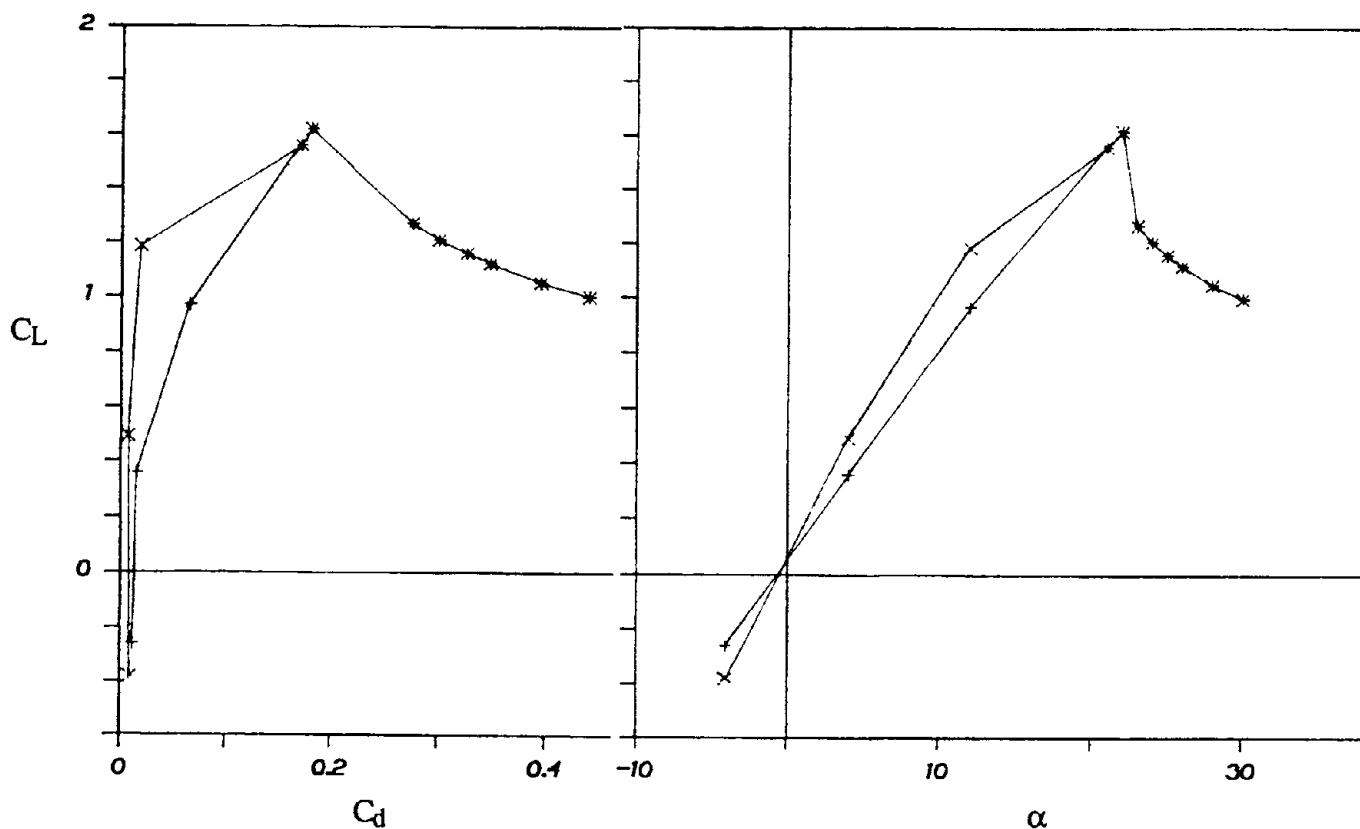
It was originally thought that the blade material engineering constants could be determined from laboratory bench tests on coupons obtained from samples cut from the discarded blade sections, for each laminate (Ref. 9, 10). The four modulus values needed were tensile (longitudinal and transverse) and shear (longitudinal and transverse). Samples measuring roughly 1 in. by 14 in. and .25 in. thick can be readily used with standard tensile testing equipment to give bulk modulus results. These results are accurate to about 5% with fiberglass, provided that the sample is truly planar and without residual strain. This turned out to be impossible with coupons from these blades, although the methodology could still work under properly controlled conditions. If time and resources permit, at least three representative samples should be tested for each laminate. Four graph points with test weights or torsion springs would allow adequate accuracy. It is helpful to verify the actual composite engineering constants since the static deflections and frequencies of vibration are very sensitive to them; however, a design range of values can be adopted and used, as was done in this case.

3.9: STATIC DEFLECTION TESTING

3.9.1: SETUP AND METHOD

The building at USDA, Bushland, TX, was large enough to allow all three blades to be separately tested from the same platform. The test stand was constructed from the leftover base of a 100 kW vertical axis wind turbine. This stand was bolted to the concrete floor and braced with two 8-inch I-beams. The mounting connection for the Carter 300 blade was the spare hub borrowed from Carter Wind Systems. Mounts for the ESI and UTRC blades were manufactured in house. The most important part of the test rig was the establishing of a baseline using a laser. The line was defined by mounting the laser under the test stand and shining the light through a small peep hole and aligning the laser dot on the far wall (Fig. 3.7). The light line was leveled using a transit and placing a mark on the wall.

When there was no load, the Carter blade had a large deflection just due to its weight. Therefore it was supported at the tip between tests. The template halves were secured together on the blade with strapping tape and rubber bands. When the blade deflected, the cross sectional area (blade shape) changed at stations 2-4. At station 3 the template came apart with a gap of 1 cm the first time the blade was loaded and then template 3 was loose and had to be repositioned several times during the different load cases.



NACAA 23112 +, Eppler Code (Miley) x
 (Eppler separate until stall, then copies NACA 23112)

FIG. 3.6 LIFT-DRAG POLARS COMPARED (SPECIFIED VS. MEASURED / EPPLER CODE)

The static deflection tests were performed by using the setup shown in Figure 3.7. Lead-weighted bags were suspended from taped hooks located at the control axis of the blade sections. Lead shot was used for accuracy and tape was used to simulate suction (Sec. 3.9.2 below and Fig. 3.11). Experience with this type of test has shown that temporary creep is a problem with composites (Fig. 3.8 illustrates the Carter blade); therefore, the linear measurements were made relative to a relaxed datum, and averages of several trials were made. The elastic twist was measured in two ways: first, by optical laser and template mirrors providing accuracy to ± 10 minutes of arc and second by the clinometer. The Fowler clinometer was acquired to take ideal local inclination measurements to within 1 minute accuracy. The base of the unit is beveled so that it can sit on a rounded edge and still give an accurate reading of inclination. The unit was able to read degrees of deviation from 0° (horizontal) to $\pm 180^\circ$. The major problem with this unit was in reading minor angles in the (-) direction. The unit has a manual dial caliper gauge and the negative readings must be read as subtraction's from the displayed degrees. That is, a reading of 45 minutes was 45 minutes when the angle was positive (nose up) but that it was meant to be read as 15 minutes subtracted from the next higher degree reading if the angle was negative. This stet idiosyncrasy of the unit resulted in operator misreading data when very close to 0° and negative angle. The graph presented Figure 3.9 shows this in the Carter 300 data twist under load vs. station in stations 4 - 7.

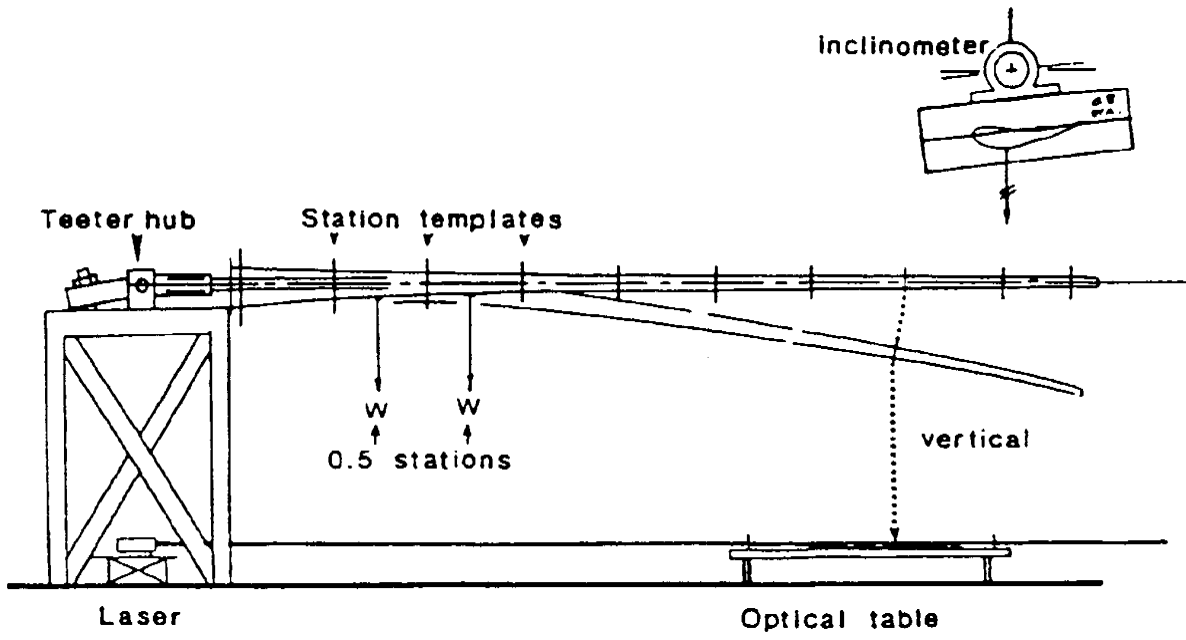
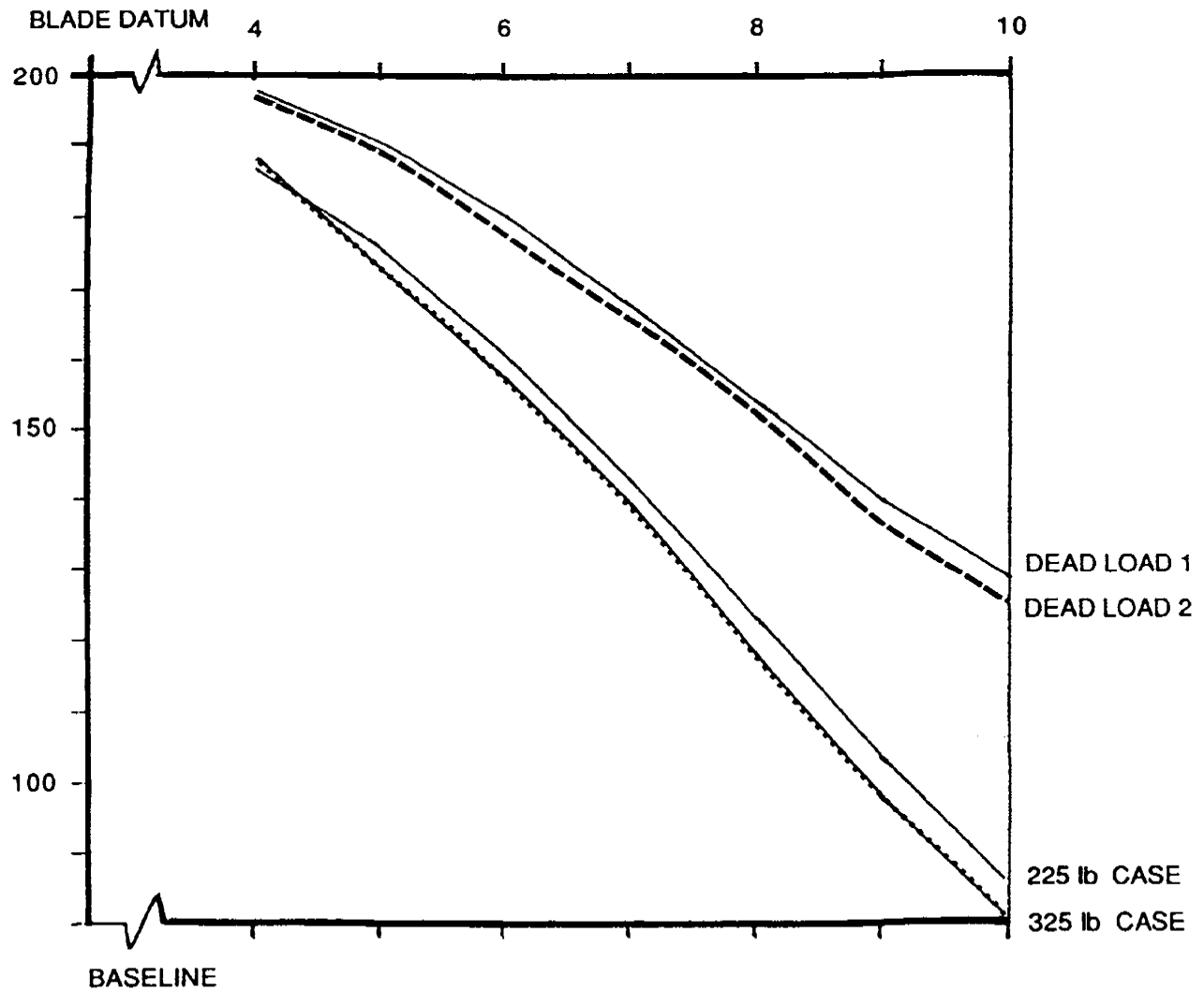


FIG. 3.7 LASER BASE PLANE ALIGNMENT



NOTE: Difference between Dead Loads 1 (initial) and 2 (final) represents temporary creep (2 hour test run, 11/9/86 , 50° F)

FIG. 3.8 CARTER BLADE: DEFLECTION AND CREEP

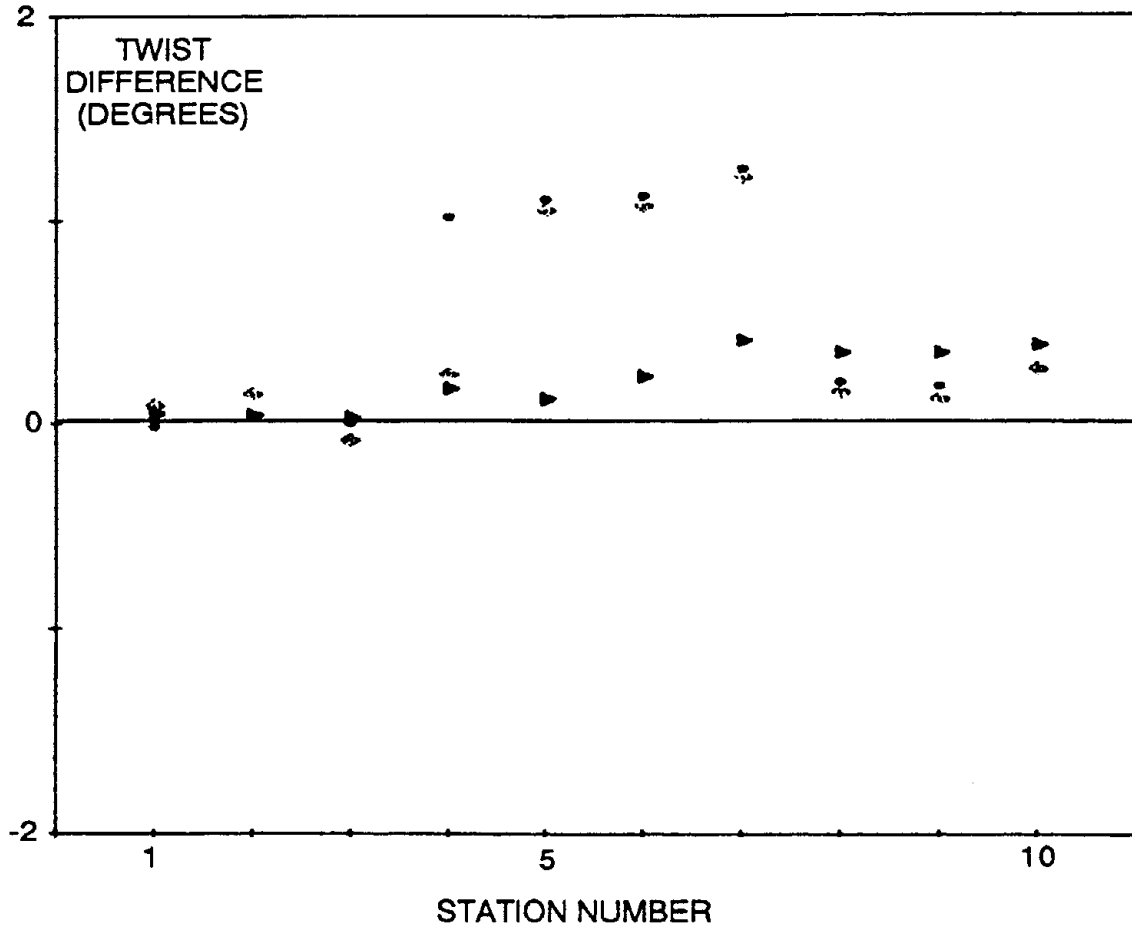
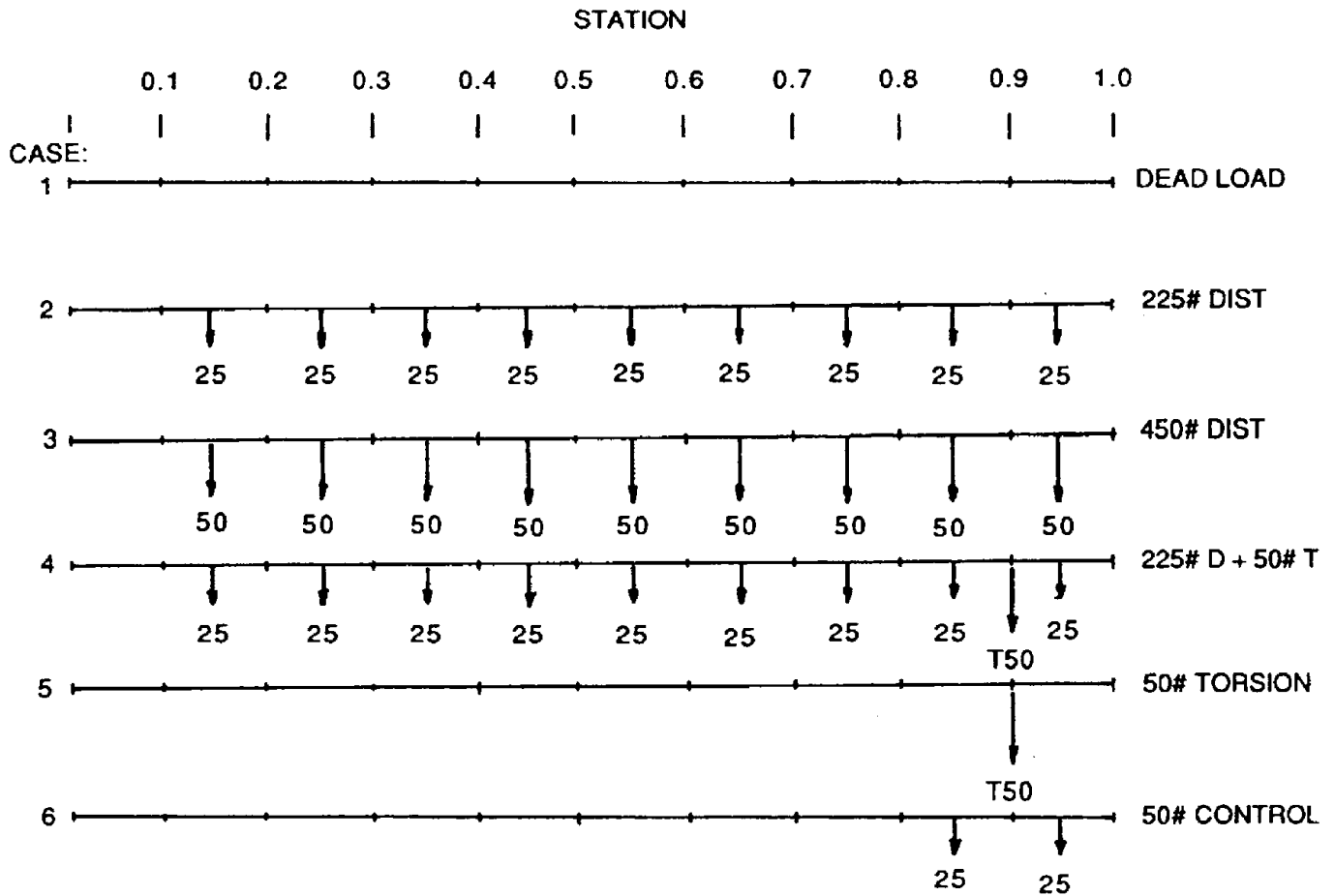


FIG. 3.9 CLINOMETER ERROR IDIOSYNCRASY

3.9.2: LOAD CASES

Load cases used for all blades were (1) no load (right side up and up side down), (2) 25 lb hung from the quarterchord line at each mid station [225 lb total], (3) 50 lb producing torsion at 0.70 span on the Carter blade and 0.90 span on the other two blades (with a zero torsion control case), and (4) 50 lb at each station as possible [up to 375 lb total]. Other load cases were tried as well with the Carter blade, totaling 19 different case loads (Fig. 3.10). These cases were chosen not to proof test the actual loads experienced in operation (which would have been greater and much more difficult to reproduce in the lab, e.g., including tension) but to establish the elastic properties of the blades. Hence, it was decided to opt for small linear deflections and the greater measurement accuracy thus required in testing.

The Carter blade was the first blade tested and the initial trial and error learning was done with this blade. The placement of the test weights was attempted in three methods: (1) placement on the blade surface at the quarterchord line, (2) hung from eyebolts attached to a wooden bar attached to the blade, and (3) using wire rods formed into eyelets taped to the blade with an S-hook to support the bags of lead shot. The latter was determined to be the best method and was used on the rest of the blades. Concentrated loads were adequately supported by using the rope/eyelet method, and furthermore allowed placement of the bags directly on the quarterchord lines. The UTRC blade was not able to support a high load since the root began to delaminate at the 25 lb per station load.



- NOTES:
1. All weights were lead shot in bags hung from s-hooks taped to blade at control axis
 2. Measurements were made on templates at each station
 3. Torsion weights (cases 4 and 5) were hung on templates
 4. Carter 325 lb. case same as case 3 with 25 lb. from STA .55 to tip

FIG. 3.10 LOAD CASES

3.9.3: DEFLECTION MEASUREMENT METHOD

The blades were loaded and the data collected for each station in turn for a known set of loads. The test table (base plane register) was moved from station to station and the loads repeated. The first station was routinely remeasured at the end of the test run to check for creep in the blade or a shifting of the blade mount. The portable base plane was constructed from an oak door and three pipe legs, one threaded (Fig. 3.11). The pipe mounts were placed so that two were in line with the established laser line, and the third centered between these and placed at the opposite end of the table. The laser line ran over a massive 20-foot I-beam trough that was relatively flat. This trough held two legs of the table. Level adjustment was made by moving a nut up and down the all-thread rod on the third leg. Two pin holes were mounted on an optical bench on the table and placed outside of and in line with the two legs in the trough. The base plane could be raised and lowered by adjusting the nuts until the laser light could pass through both holes of the leveled optical bench. A large bubble level at right angles to the laser line edge and the third leg of the table was used to level the plane of the table relative to the laser line. Droop data then could be recorded from the tip side of each template relative to a registration point on the template (nail). A monofilament line with a construction plumb bob was used to find the relative X,Y position of the nail projected onto the laser base plane. A tape measure was then hung from the nail and the total droop distance from the nail to the

laser plane was taken. In this way (X,Y,Z) measurements could all be taken to an accuracy of ± 0.1 in. (2.5 mm).

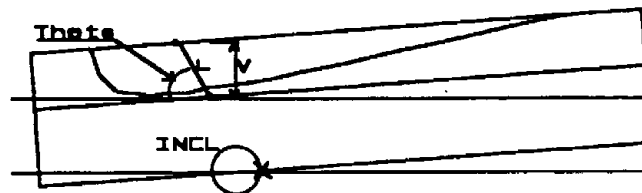


FIG. 3.11 TEST TABLE

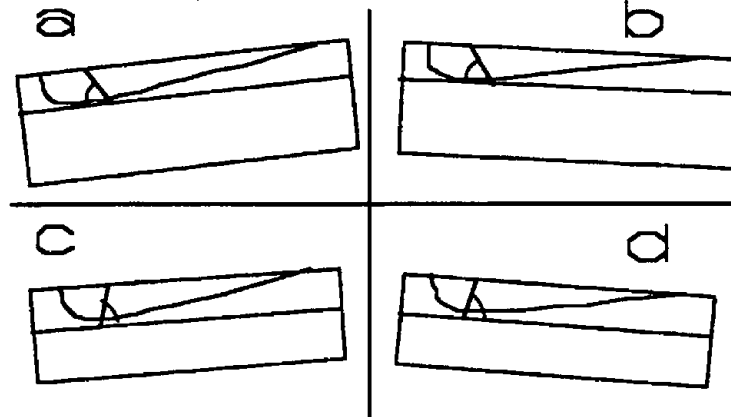
3.9.4: DEFLECTION CORRECTIONS TO QUARTERCHORD

The procedure for the correction of the droop deflection measurements to the quarterchord point is described here. First, the original data (i.e., template nail to base plane) were summarized in a computer data file for each load case. These data consisted of the station position (station number), load case number, X direction deflection (relative to laser baseline), Y deflection (relative to a moving origin based on the laser table), Z droop measured relative to the laser plane, angular displacement of the chord in degrees and minutes, and the flapwise root fixture bending in degrees and minutes. During the collection of these data a clinometer convention was adopted: positive was defined as airfoil nose down (i.e., tendency for the nose of the airfoil to go into the wind).

The method to calculate the coordinate shifts is described here. In the opening lines of each program were the measured data of the plumb bob support arm (nail) positions by station relative to the quarterchord point. These values were determined from the templates. The nails were already as close as possible to the quarterchord position. Both upper and lower (low and high pressure) sides of the templates were equipped with registration nails. These nail positions were related to the actual quarterchord by their angular deflection and offset (Θ and L). V, the vertical offset, was measured as depicted in the sketch below:



The following sketches show the four cases of nail position that were accommodated: the nail ahead and behind the quarterchord point, and with positive and negative local pitch.



The input data and the constants (Θ and L) were used to compute three new values: T , which is the radian equivalent of the degree measurements; P , the radians of the angle Θ ; and the calculated Z droop deflection based on measured local station twist and relative nail position:

$$V = L \{ \sin [(P \text{ SGN}(P)) + (T C)] \}$$

V = vertical correction to droop

L = distance from nail to quarterchord

P = radian angle measure from nail to quarterchord

SGN = sign function [returns the sign of the variable where -1 is returned for negative numbers and 1 is returned for positive numbers (0 is considered positive)]

T = radian measure of local station twist

C = correction sign based on if the nail position is ahead (noseward) or behind the quarterchord point.

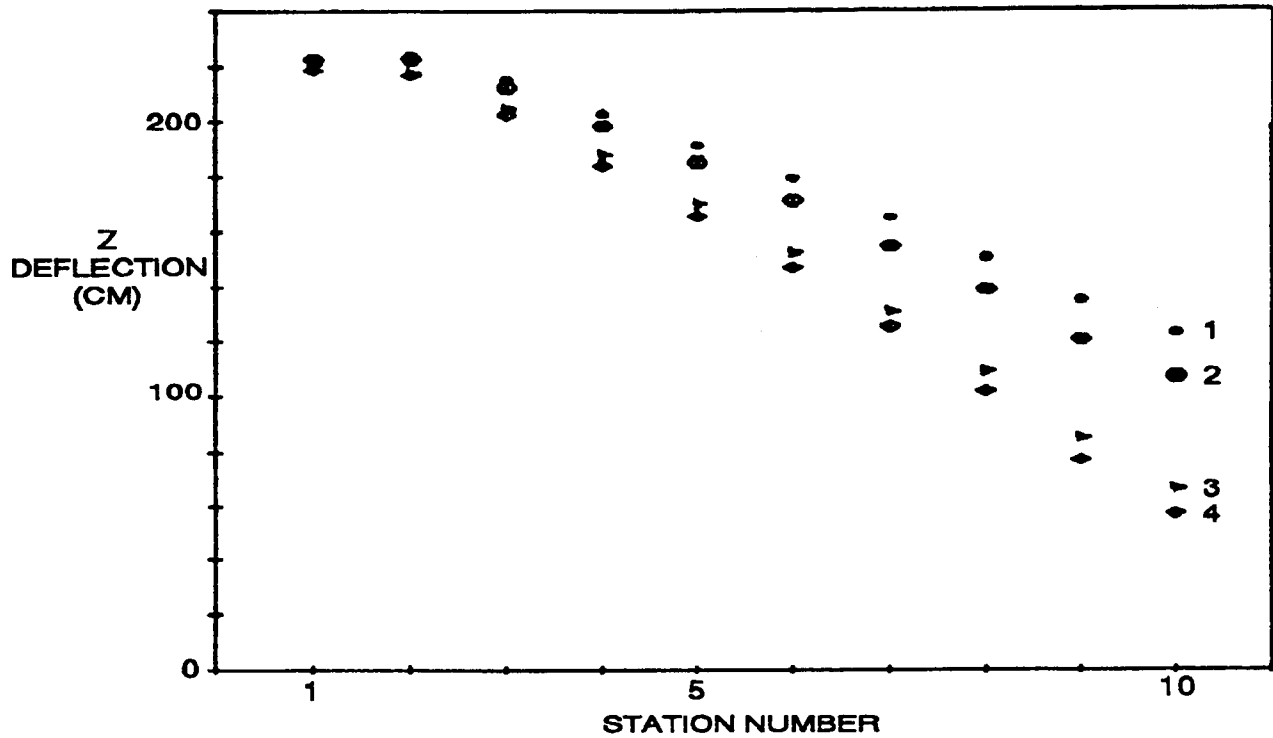
$$Z_{\text{new}} = V + Z_{\text{old}}$$

3.9.5: DEFLECTION RESULTS

The deflection results under load and no-load conditions are best seen in the graphs of deflection versus station number (Figs. 3.12 to 3.14) for various load cases (Fig 3.10). These curves are very smooth even with varying loading methods. Of course, this is to be expected since the blades were purposely kept inside their linear limits of elasticity.

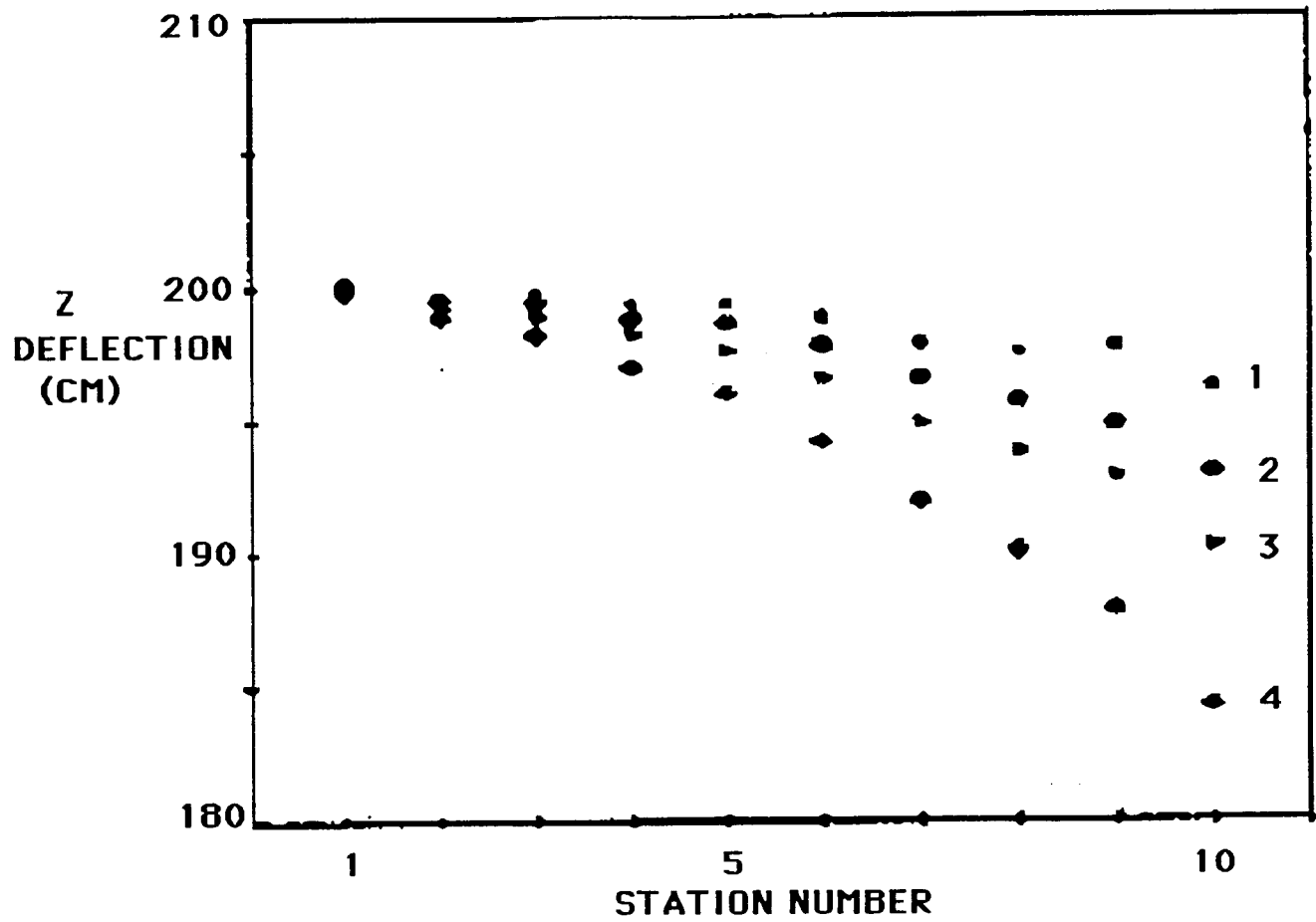
3.10: BLADE VIBRATION MODAL SURVEY

The laboratory frequency-of-vibration tests were performed in the same test fixtures. It was originally thought that the fundamental frequencies of vibration could be obtained simply with an accelerometer and strip chart recorder with a frequency response higher than the expected frequency. It was expected that the required frequency response would be about 100 Hz. However, it became clear that particularly for the Carter blade, no simple fundamental bending modes could be easily identified. Also, the torsional frequencies turned out to be higher than expected for the ESI (40.5 Hz) and UTRC (29 Hz) blades. Therefore, SERI was asked to perform a more rigorous modal survey on the blades. The method and results are fully presented in Reference 8, and summarized here.



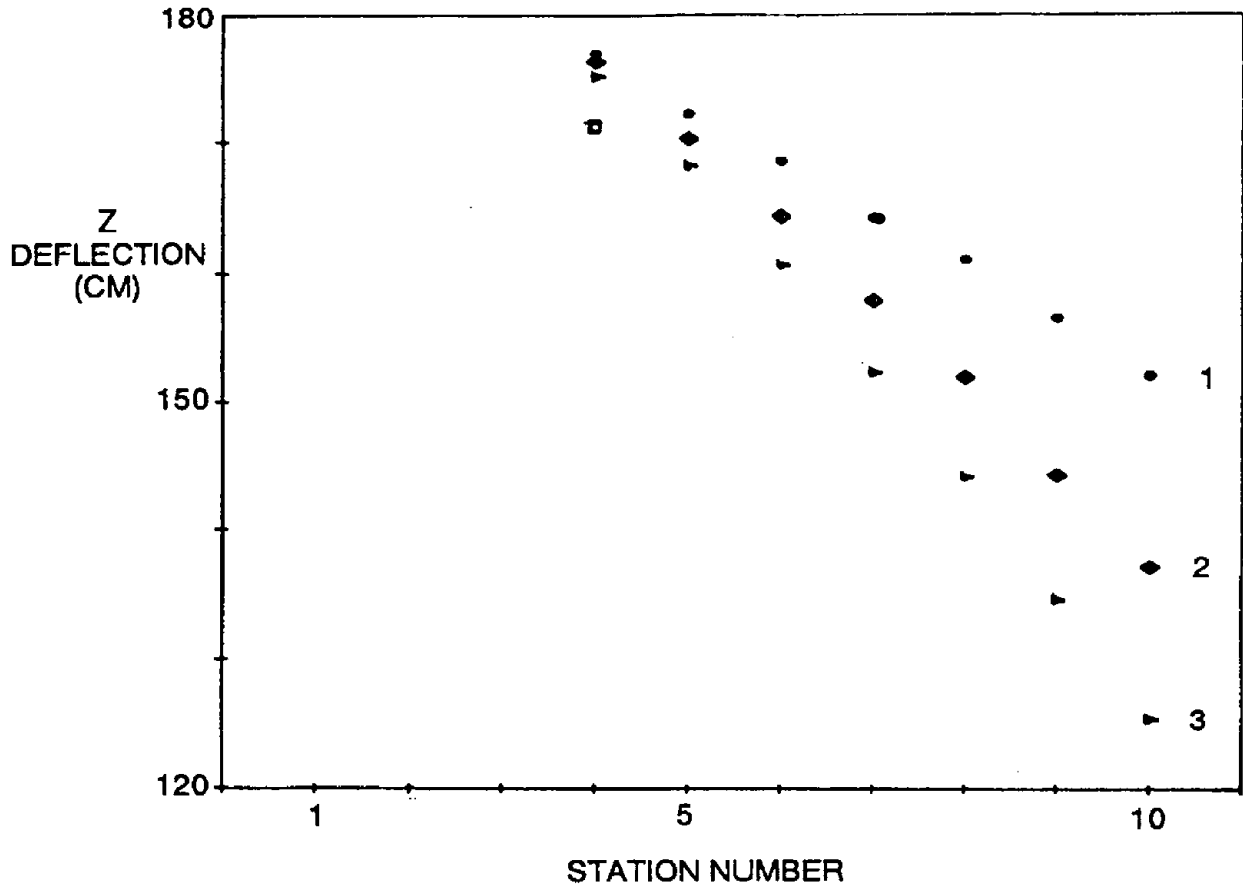
LOAD KEY: (1) Dead Load, (2) Torsion, (3) 225 lb, (4) 325 lb

FIG. 3.12 DEFLECTION RESULTS, CARTER (UNCORRECTED)



LOAD KEY: (1) Dead Load, (2) Torsion, (3) 225 lb, (4) 450 lb

FIG. 3.13 DEFLECTION RESULTS, ESI (UNCORRECTED)



LOAD KEY: (1) Dead Load, (2) Torsion, (3) 175 lb

FIG. 3.14 DEFLECTION RESULTS, UTRC (UNCORRECTED)

Modal data collection primarily consists of the judicious application of a known input force and then measurement of the structure's response. In general, the dynamic response to an external excitation is a superposition of all the normal vibration modes of the structure. This collection of large amounts of data and the extensive harmonic analysis can be done with available electronic equipment and desktop computers, as in this project. The modal survey equipment sent by SERI included a Hewlett Packard HP 5420A two-channel spectrum analyzer and an HP 9825 desktop calculator, calibrated impact hammer with charge amplifier, and piezoresistive accelerometers and bridge amplifiers. The outputs from the hammer (forcing function) and the amplified accelerometer signal (response function) were the inputs to the spectrum analyzer (S.A.). The S.A. computed frequency response functions from the impact testing which were then stored for later analysis on the desktop calculator. The comprehensive software for modal survey (Fast Fourier Transform) and the particular hardware system had been used by SERI on previous projects. The impulse method was used since the shakers available from SERI did not have sufficient mass/amplitude for large blades. The shake test method would have given better mode resolution and would have been easier to install in addition to enabling the discovery of hidden modes with less effort. The zero-crossing (peak-picking) method was used to determine mode frequencies.

Frequencies and mode shapes (Table 3.7 and Figs. 3.15 - 3.17) were determined. The Carter 300 was the most complicated blade; it had a dashpot snubber in the root that had to be replaced with a rigid member to eliminate the interference of the snubber (pitching) action. The first flap bending mode turned out to be 0.59 Hz and the first torsion 13.25 Hz. Above the second flap bending the modes were more complex due to the complicated internal structure and ballast weights. Near 40 Hz, the modal frequencies

were very highly coupled. The ESI was much stiffer and had a simpler internal structure. The vibration modes were well separated and classic in shape. First flap bending was at 3.5 Hz and first torsion at 40.5 Hz. The UTRC blade was the simplest: constant chord and no twist. The first flap bending was 2.1 Hz and the first torsion 29 Hz. The modal survey results compare favorably with the results from the theory of elasticity (see Ref. 11, for example).

TABLE 3.7: FREQUENCY OF VIBRATION RESULTS FROM TEST (HZ)

MODE	UTRC	ESI	CARTER
1ST FLAP	2.1	3.5	0.59
2ND FLAP	13.0	11.0	2.73
3RD FLAP	35.0	25.0	7.38
1ST LAG	13.0	8.0	3.0
2ND LAG	78.0	33.0	16.0
3RD LAG	200.0	80.0	40.4
1ST TORSION	29.0	40.5	*13.25
2ND TORSION	84.0	*68.0	*19.73

NOTES

- (1) ALL BLADES TESTED IN FLYING CONDITION EXCEPT: CARTER SNUBBER LOCKED, NO ESI TIPS.
- (2) ROOT FIXITY IDENTICAL TO STATIC TESTS.
- (3) ESI 2ND TORSION IS COUPLED WITH 5TH FLAP.
- (4) CARTER 1ST TORSION IS I-BEAM ROOT FLEXURE ONLY.
- (5) CARTER 2ND TORSION IS OUTBOARD TORSION ONLY.

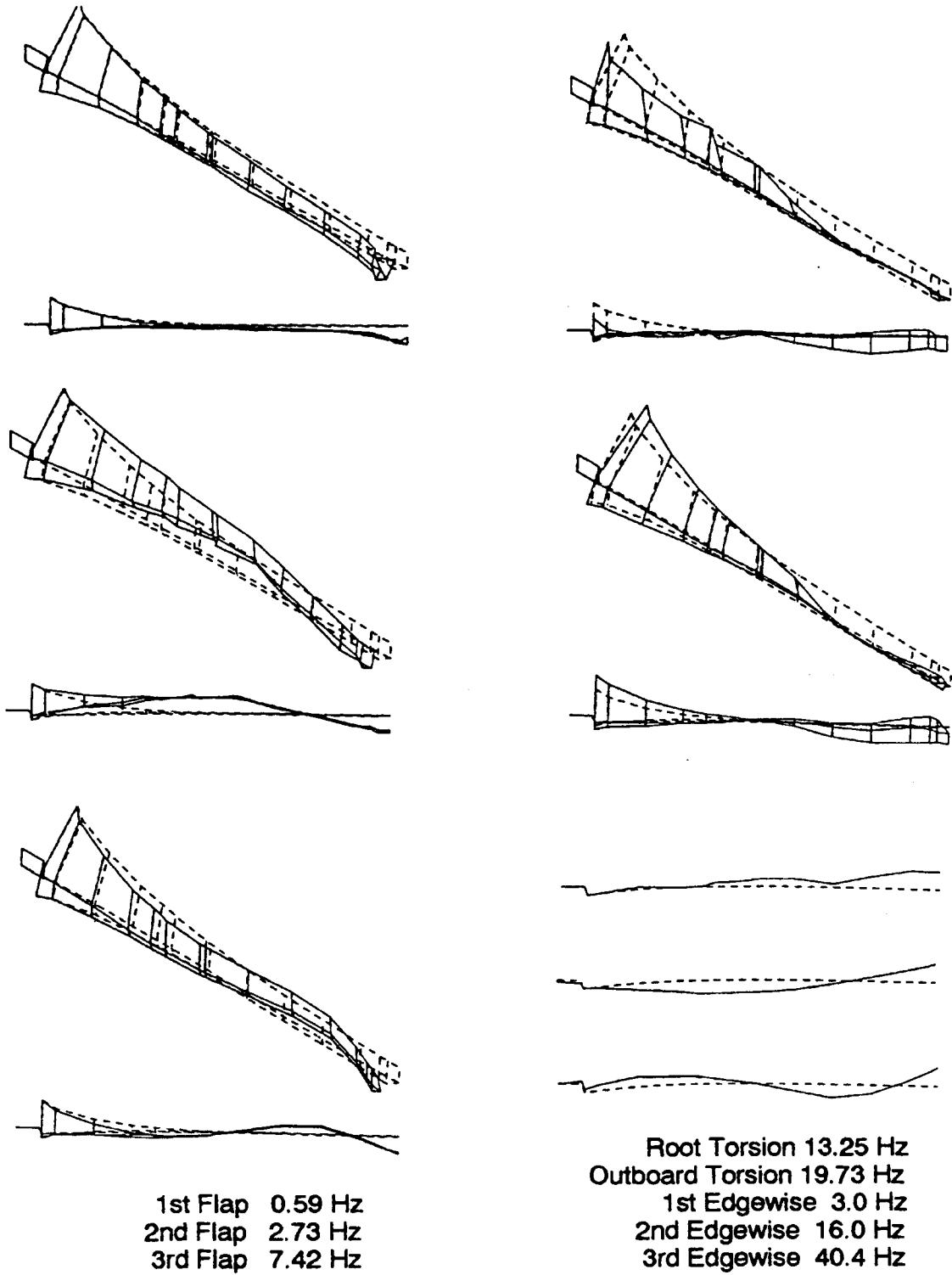
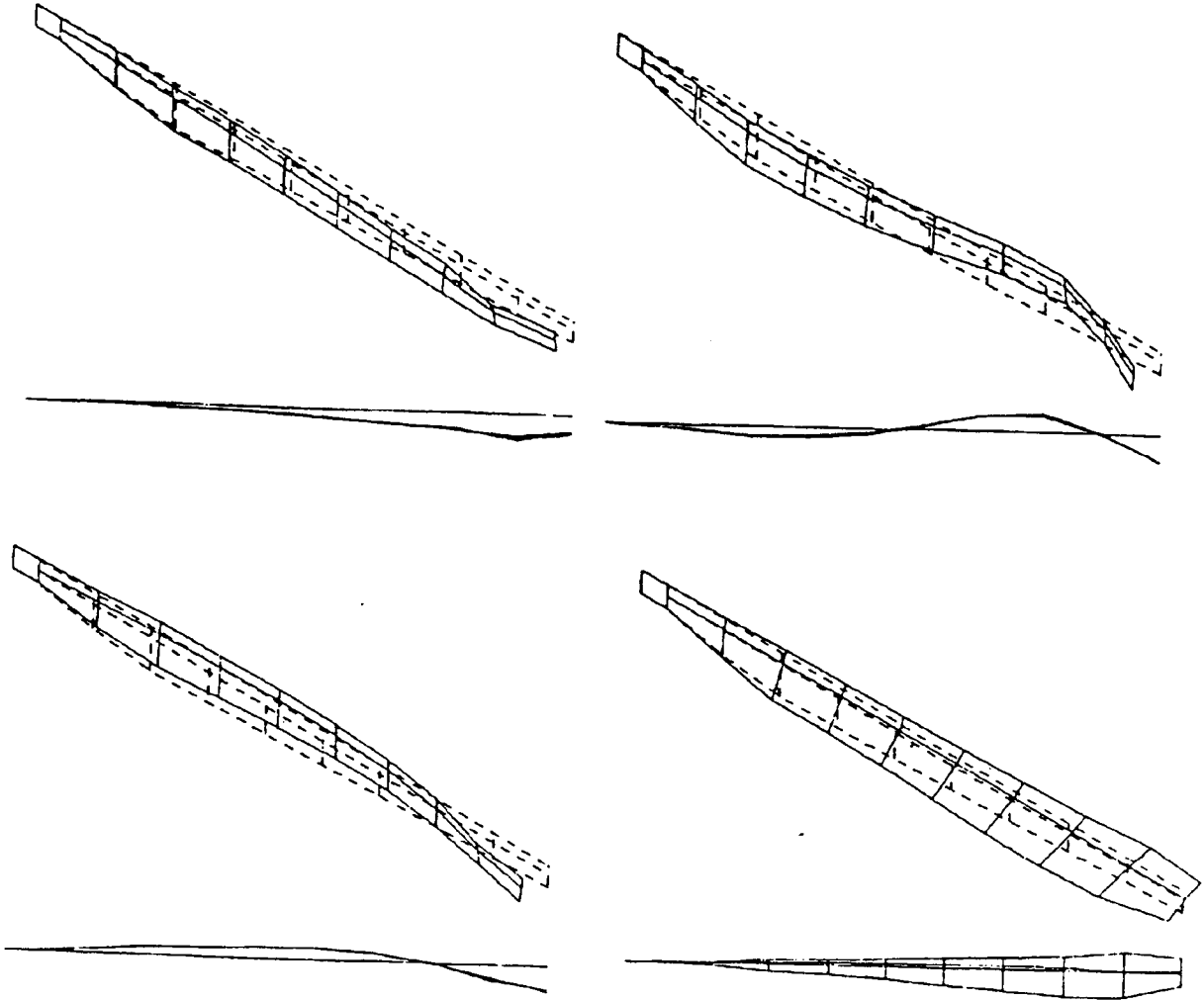


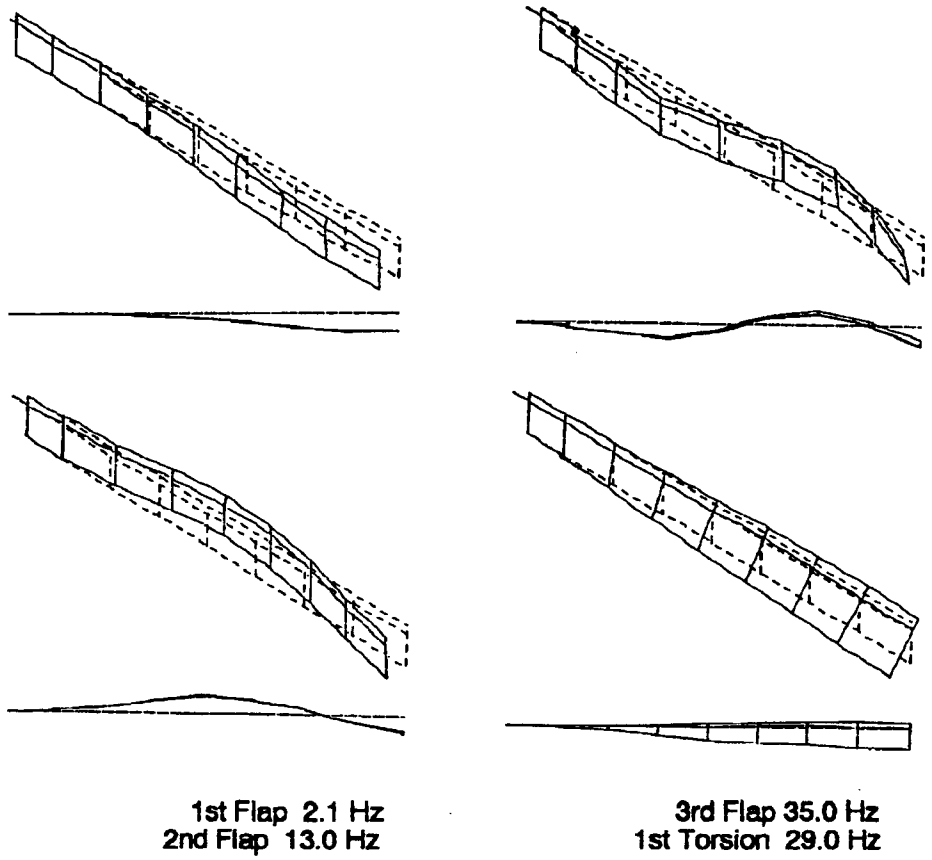
FIG. 3.15 FREQUENCY OF VIBRATION: CARTER BLADE



1st Flap 3.5 Hz
2nd Flap 11.0 Hz

3rd Flap 25.0 Hz
1st Torsion 40.5 Hz

FIG. 3.16 FREQUENCY OF VIBRATION: ESI BLADE



— CALCULATED AND TEST VALUES —

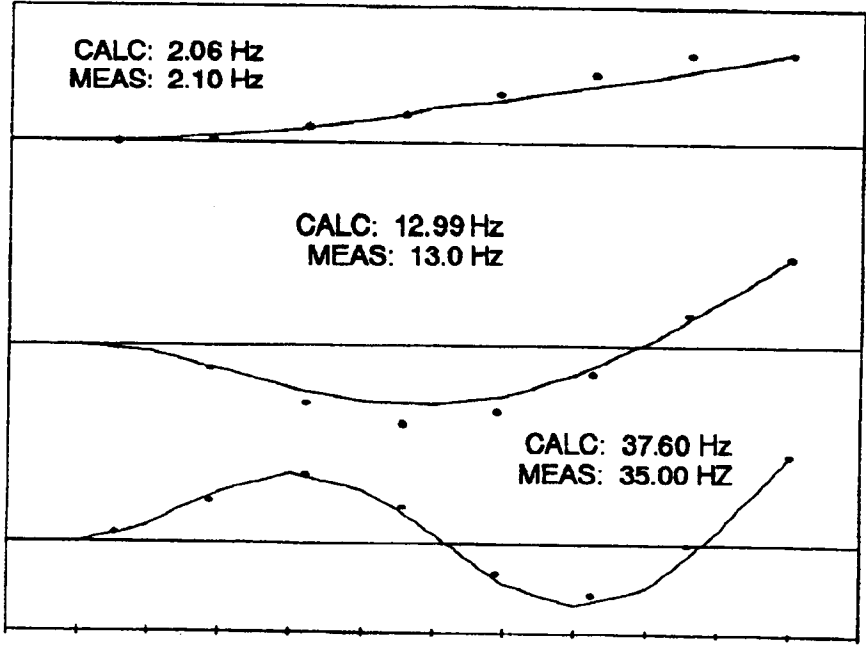


FIG. 3.17 FREQUENCY OF VIBRATION: UTRC BLADE

4.0 THEORETICAL METHOD

4.1: GENERAL

An arbitrary cross-section, 3-dimensional beam subject to arbitrary distributed loading in bending and torsion is a formidable mathematical problem. It is usually accomplished in the aerospace industry by employing expensive finite-element structural programs (e.g. NASTRAN). This study did not have sufficient resources to employ such a state-of-the-art model, nor was it warranted, since the static bending and frequency of vibration results are very sensitive to the material mechanical properties, which were composite laminates for this effort. In using such computer programs a lot of effort can be wasted needlessly if these engineering constants are not known to a high degree of accuracy.

In view of this, this project used a simpler elastic beam approach which modeled the composite blade sections by calculating area-weighted flexural and torsional moduli, calculated the beam properties relative to the principal axes, and then performed coordinate transformation to develop blade axis sectional properties. Arbitrary cross section including blade skin, spar, shear webs, and afterbodies are all handled in the above approach. The deflections were found, in a likewise manner, by referring the applied loading and moments to the aerodynamic center (quarterchord), the mass center (blade section cg), the bending centroid (weighted area centroid), and the shear center (elastic axis), and transforming the deflections by superposition. The deflections, when compared to the test results, then provide a reiteration error of the original assumed laminate constants, hence the term "mass/elastic triangulation."

4.2: COMPOSITE BLADE SECTION PROPERTIES CALCULATION

4.2.1: SECTION DESCRIPTION

This section describes the beam section properties calculations (Ref. 3). The Bernoulli-Euler theory of bending ("engineers' theory of bending") calculation can be followed in any thorough strength of materials text (e.g., Ref. 12, 13). The key assumptions are 1) deflections are small and linear and 2) plane sections remain plane and normal to the beam axis. These assumptions make it possible to determine the deflections of any point on the beam in terms of points on the axis of the beam. Figure 4.1 shows a general blade cross section for a spar-shell composite blade.

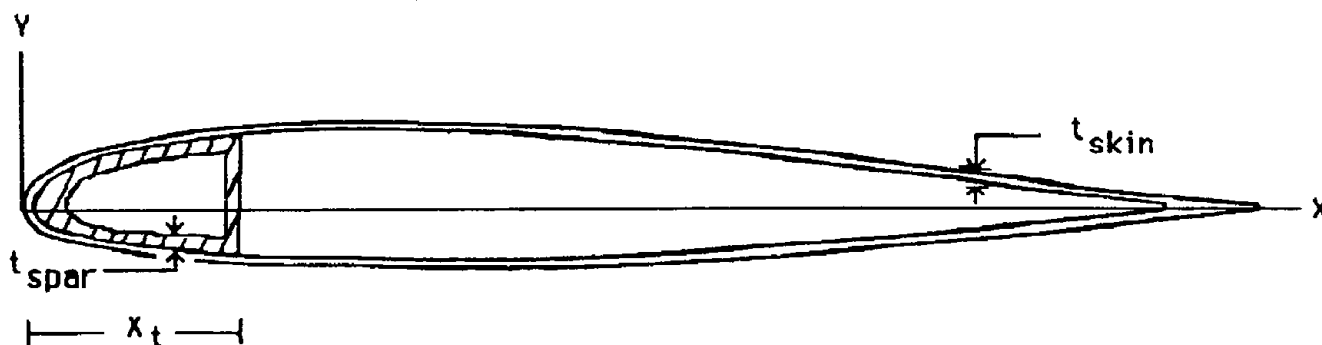


FIG. 4.1 GENERAL SPAR-SHELL CROSS SECTION

The spar is shown as a constant thickness D-spar and a web at X_t in the nose, and the skin is shown also as constant thickness. Section axes have origin at the nose and abscissa along the airfoil chordline. In general the spar and skin will have entirely different constituents (laminate schedules). The spar is usually the principal bending structure, and the skin the torsion structure, also providing a shear load path for aerodynamic pressure forces into the spar. More complicated versions will have more webs and variation in the laminate within the skin or the spar. For now we assume the skin and spar to be uniform.

In general, the spar and skin constituents have directional properties; that is, the elastic constants vary depending on the orientation (0 degrees reference is spanwise and 90 degrees is chordwise). For

simple bending only the longitudinal (0°) values are used. This gives four engineering constants which describe the longitudinal and shear (torsion) behavior of the section: E_{skin} , G_{skin} , E_{spar} , and G_{spar} , or elastic and shear moduli of elasticity. These engineering moduli can be calculated for a given laminate by the area-weighting method. For instance, suppose a laminate is composed of various layers as shown in Figure 4.2.

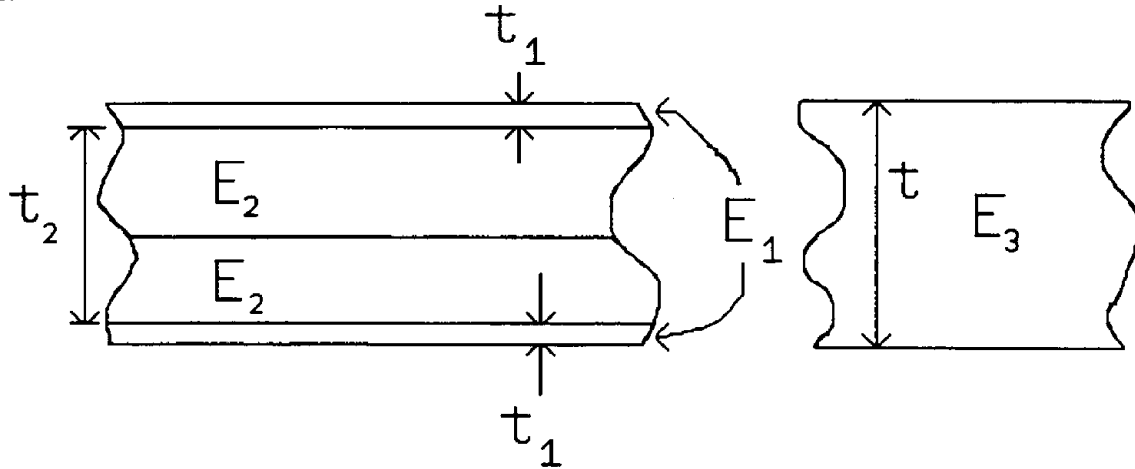


FIG. 4.2 ACTUAL VS. EQUIVALENT LAMINATE

Then the equivalent laminate has the same actual thickness as the aggregate ($t = 2t_1 + t_2$), and an elastic modulus based on area ratios is

$$E_3 = \frac{\sum E_n t_n}{\sum t_n} = \frac{(2E_1 t_1 + E_2 t_2)}{(2t_1 + t_2)}$$

Clearly the complicated laminate on the left can only be approximately represented by the bulk laminate on the right; internal shear stresses (interlaminar shear), for instance, could be calculated also for each layer interface. This detail can be left for the end of the engineering process, after the overall beam deflection and bending stresses have been calculated. Then these residual internal stresses can be checked. In this application the actual thicknesses of the layers are maintained since these will also be used to calculate weights and moments of inertia, and will also be checked with measurements. Alternative approaches may use modulus-weighted thicknesses to simplify calculations.

4.2.2: SECTION AREA AND STIFFNESS PROPERTIES

Section area properties are found by simple integration. The airfoil shape (X,Y) is first calculated based on a specific table of offsets and a chordline, and using a simple second-order curve-fit subroutine through each series of three points. The contribution of each differential skin and spar segment is calculated and summed around the airfoil, starting at the nose and also ending there. These integrals are simply

- A_t = Total area = $\int dA$
- M_x = 1st moment of the area about the X axis = $\int y dA$
- M_y = 1st moment of the area about the Y axis = $\int x dA$
- I_x = Moment of inertia about the X axis = $\int y^2 dA$
- I_y = Moment of inertia about the Y axis = $\int x^2 dA$
- I_{xy} = Product of inertia = $\int xy dA$

$$\bar{X}_c = \text{Bending centroid, chordwise position} = \frac{M_y}{A_t}$$

$$\bar{Y}_c = \text{Bending centroid, flapwise position} = \frac{M_x}{A_t}$$

The program corrects for shell thickness by using the skin (or spar) segment centers to define the moment arm.

Section stiffness properties are found by the area-weighting method, combining the contributions from the skin and spar, and adding the spar web as a separate item for simplicity. This approach allows the addition of other shear webs or structural elements in the total composite section values. The spar web and further elements are simply described by their area properties:

- A_w = Spar web area
- I_{xw} = Spar web moments of inertia
- I_{yw} = Spar web moments of inertia
- I_{xyw} = Spar web product of inertia
- $(X,Y)_w$ = Centroid

Bending values relative to the (X,Y) axis system origin are

$$EA = E_{skin} A_{skin} + E_{spar} A_{spar} + E_{web} A_{web}$$

Bending centroids for total section are

$$\bar{X}_T = \frac{1}{EA} [E_{skin} M_y \text{ skin} + E_{spar} M_y \text{ spar} + E_{web} M_y \text{ web}]$$

$$\bar{Y}_T = \frac{1}{EA} [E_{skin} M_x \text{ skin} + E_{spar} M_x \text{ spar} + E_{web} M_x \text{ web}]$$

Now the bending stiffness for the section is found, but is referenced to the centroid of the composite section. This makes the calculation simpler, and correctly assesses the stiffness contribution of each element by superposition. The parallel axis theorem is used to move each (X,Y) moment of inertia value to the section centroid (X_T, Y_T); an intermediate step moves the point to its own centroid. For example, for the skin only, the X-moment of inertia about the total composite section centroid is

$$EI_{x \text{ skin}} = E_{skin} \{ [I_{x \text{ skin}} - A_{skin} \bar{Y}_{skin}^2] + A_{skin} [\bar{Y}_{skin} - \bar{Y}_T]^2 \}$$

Therefore, the section stiffnesses are simply the sums of the contributions :

$$EI_{x \text{ total structure}} = \sum EI_x \quad (\text{where } I_x \text{ is referred to the bending centroid in each case.)}$$

$$\begin{aligned} &= E_{skin} \{ [I_{x \text{ skin}} - A_{skin} \bar{Y}_{skin}^2] + A_{skin} [\bar{Y}_{skin} - \bar{Y}_T]^2 \} \\ &+ E_{spar} \{ [I_{x \text{ spar}} - A_{spar} \bar{Y}_{spar}^2] + A_{spar} [\bar{Y}_{spar} - \bar{Y}_T]^2 \} \\ &+ E_{web} \{ I_{x \text{ web}} + A_{web} [\bar{Y}_{web} - \bar{Y}_T]^2 \} \end{aligned}$$

$$EI_y \text{ total structure} = E_{\text{skin}} \{ [I_y \text{ skin} - A_{\text{skin}} \bar{X}^2_{\text{skin}}] + A_{\text{skin}} [\bar{X}_{\text{skin}} - \bar{X}_T]^2 \}$$

$$+ E_{\text{spar}} \{ [I_y \text{ spar} - A_{\text{spar}} \bar{X}^2_{\text{spar}}] + A_{\text{spar}} [\bar{X}_{\text{spar}} - \bar{X}_T]^2 \}$$

$$+ E_{\text{web}} \{ I_y \text{ web} + A_{\text{web}} [\bar{X}_{\text{web}} - \bar{X}_T]^2 \}$$

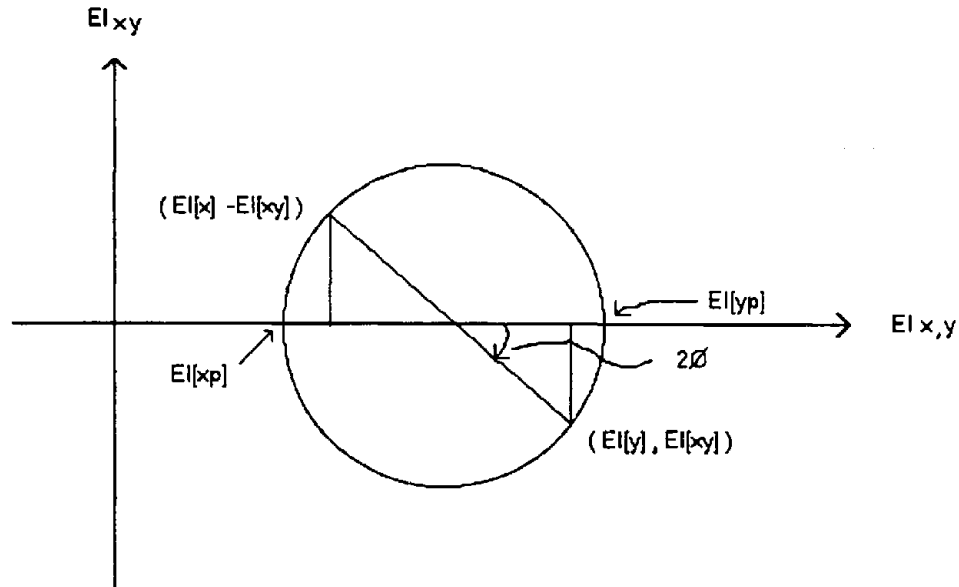
$$EI_{xy} \text{ total structure} = \sum EI_{xy}$$

$$= E_{\text{skin}} \{ [I_{xy} \text{ skin} - A_{\text{skin}} \bar{X} \bar{Y}_{\text{skin}}] + A_{\text{skin}} [\bar{Y} - \bar{Y}_T]_{\text{skin}} [\bar{X} - \bar{X}_T]_{\text{skin}} \}$$

$$+ E_{\text{spar}} \{ [I_{xy} \text{ spar} - A_{\text{spar}} \bar{X} \bar{Y}_{\text{spar}}] + A_{\text{spar}} [\bar{Y} - \bar{Y}_T]_{\text{spar}} [\bar{X} - \bar{X}_T]_{\text{spar}} \}$$

$$+ E_{\text{web}} \{ I_{xy} \text{ web} + A_{\text{web}} [\bar{Y} - \bar{Y}_T]_{\text{web}} [\bar{X} - \bar{X}_T]_{\text{web}} \}$$

The last step is the calculation of the section principal axis values, which are used later to calculate blade deflections. The principal axes of inertia are found in the program from a Mohr's circle representation depicted here in Figure 4.3:



where θ = principal axis angle
 $EI[xp], EI[yp]$ = sectional stiffnesses in the principal axes

FIG. 4.3 MOHR'S CIRCLE FOR PRINCIPAL AXES

These principal moments of inertia (stiffnesses) are the maximum and minimum values for this section. For the usual airfoil shapes, these are close to the original (X,Y) directions: chordwise and flapwise. This step simplifies the bending deflection calculation by eliminating the products of inertia.

4.2.3: SECTION TORSIONAL RIGIDITY

To calculate section torsional rigidity also requires care in referencing the contributions to the total torsional centroid (shear center), which in general is different from the bending centroid since the (E/G) ratio of the components are not the same.

$$GA = G_{skin} A_{skin} + G_{spar} A_{spar} + G_{web} A_{web}$$

The torsional centroid (shear center) is then

$$\bar{Y}_{TT} = [1 / GA] \{ G_{skin} M_{x \text{ skin}} + G_{spar} M_{x \text{ spar}} + G_{web} M_{x \text{ web}} \}$$

$$\bar{X}_{TT} = [1 / GA] \{ G_{skin} M_{y \text{ skin}} + G_{spar} M_{y \text{ spar}} + G_{web} M_{y \text{ web}} \}$$

And the torsional rigidity about the shear center is found by using the parallel axis theorem in sequence as before for the bending stiffness:

$$\begin{aligned} GJ = & G_{skin} \{ [I_{x \text{ skin}} - A_{skin} \bar{Y}_{skin}^2] + A_{skin} [\bar{Y}_{skin} - \bar{Y}_{TT}]^2 \} \\ & + G_{spar} \{ [I_{x \text{ spar}} - A_{spar} \bar{Y}_{spar}^2] + A_{spar} [\bar{Y}_{spar} - \bar{Y}_{TT}]^2 \} \\ & + G_{web} \{ I_{x \text{ web}} + A_{web} [\bar{Y}_{web} - \bar{Y}_{TT}]^2 \} \\ & + G_{skin} \{ [I_{y \text{ skin}} - A_{skin} \bar{X}_{skin}^2] + A_{skin} [\bar{X}_{skin} - \bar{X}_{TT}]^2 \} \\ & + G_{spar} \{ [I_{y \text{ spar}} - A_{spar} \bar{X}_{spar}^2] + A_{spar} [\bar{X}_{spar} - \bar{X}_{TT}]^2 \} \\ & + G_{web} \{ I_{y \text{ web}} + A_{web} [\bar{X}_{web} - \bar{X}_{TT}]^2 \} \end{aligned}$$

Having derived the above expression we now reluctantly relate that this value of GJ is not usually practical for thin shells, since skin warping greatly reduces the stiffness. For example, in a usual thin-skin airfoil with a D-spar, the above calculation is approximately four times the actual values of GJ from experiment. Therefore, the AEI program uses a shear flow approach in calculating GJ:

$$GJ = \frac{4 A^2 G}{\int ds / t}$$

where: A = cross-sectional area
t = skin thickness
ds = differential skin length

In the above calculation the thicknesses (hence areas) are modulus-weighted to the skin value of shear modulus and the contribution of the spar web is ignored. The GJ calculated from the above expression has proven to be accurate enough for practical purposes. Further refinement of this calculation could be done for more complex section geometries. [Note: From later calculations, the major error introduced by dropping the web contribution is to incorrectly locate the shear center. For the ESI blade, which has a thick skin and substantial web, this error amounted to 40% chord (calculated) vs. 25% chord (test)].

4.2.4: SECTION MASSES AND INERTIAS

At this point the required stiffness properties in bending and torsion have been calculated for each section of the composite beam. It remains to calculate the masses and inertias. As for the elasticity constants, the blade skin and spar will have different mass densities based on the percentage of reinforcement to resin volume. For the simplest unidirectional laminate with glass fiber reinforcement, the relationship between density and modulus is straightforward (Table 4.1).

TABLE 4.1: FRP UNIDIRECTIONAL WEIGHTS AND MODULI

% Resin Volume	% Fiber Volume	% Resin Weight	Density (lb/cu in.)	E(ksi)
100	0	100	.04340	0
90	10	80	.04890	1
80	20	64	.05440	2
70	30	51	.05990	3
60	40	40	.06593	4
50	50	31	.07089	5
40	60	23	.07639	6
30	70	16	.08189	7
20	80	10	.08738	8
10	90	5	.09288	9
0	100	0	.09838	10

where resin = .0434 lb/cu in. and glass = .09838 lb/cu in.
 Practical laminates lie between these extremes in modulus and weight.

The section mass integrals are formed in the same manner as the stiffness integrals above, by summing the individual components using the parallel axis theorem:

$$W_A = W_{skin} A_{skin} + W_{spar} A_{spar} + W_{web} A_{web} = \text{total running weight}$$

$$X_g = \text{Section cg} = \sum W_n X_n / W_A$$

$$Y_g = \text{Section cg} = \sum W_n Y_n / W_A$$

$$I_{x0} = X - \text{component mass moment of inertia about origin } X, Y = 0, 0$$

$$I_{y0} = W_{skin} I_{x0 \text{ skin}} + W_{spar} I_{x0 \text{ spar}} + W_{web} [I_{xweb} + A_w (\bar{Y})^2]$$

= Y - component mass moment of inertia about origin X, Y = 0, 0

$$I_{xg} = W_{skin} I_{y0 \text{ skin}} + W_{spar} I_{y0 \text{ spar}} + W_{web} [I_{yweb} + A_w (\bar{X})^2]$$

= Mass Moments of inertia referenced to the section cg = $I_{x0} - W_A y_g^2$

$$I_{yg} = I_{y0} - W_A x_g^2$$

$$I_{xT} = \text{Now referenced to the elastic axis (shear center)}$$

$$= I_{xg} + W_A [\bar{Y}_T - \bar{Y}_g]^2$$

$$I_{yT} = I_{yg} + W_A [\bar{X}_T - \bar{X}_g]^2$$

Finally the section mass moment of inertia is the sum of both the X- and Y- components, for both the section cg and the shear center:

$$I_T = I_{xT} + I_{yT} \quad \text{and} \quad I_g = I_{xg} + I_{yg}$$

The mass moment of inertia about the cg, I_g , is the minimum value for the section.

Lastly, the program adds any ballast weight contribution to the mass integrals. In some blades, lead ballast is added to move the basic section cg position; in others the "ballast" contribution might refer to nonstandard elements embedded in the blade which change the masses but not the stiffness (e.g. tip control mechanism). These ballast contributions are again added by using the parallel axis theorem, as above.

4.3: BLADE STATIC DEFLECTION CALCULATIONS

The Bernoulli-Euler theory of bending of slender beams ("engineer's theory of bending") is used. The key assumptions are flexural deflections are small and linear, and plane sections remain plane and normal to the beam (centroidal) axis as the blade deforms. Stresses in the transverse direction are also neglected. This gives a uniaxial stress-strain relationship and satisfies the equilibrium condition since the stress resultants are in equilibrium with the applied forces. These assumptions also make it possible to determine the deflection of any point on the blade in terms of the deflections of the points on the centroidal axis of the blade since strain distribution is linear across the cross section.

The cross products of inertia are dropped out by the adoption of the principal axes:

$$\epsilon_i = \frac{M_j Y}{E I_i} \text{ (simple flexure formula)}$$

and the total strain (or stress) can be found by superposition :

$$\sigma = \sigma_y + \sigma_x = \frac{M_{zp} y_p}{I_{zp}} - \frac{M_{yp} z_p}{I_{yp}}$$

where y_p and z_p are now the principal axes of the beam sections, and x is the axis down the spar of the blade. The 3-D coordinate system is now

- X-axis: lies along the (undeflected) axis of the blade, with the origin at the blade root.
- Y-axis: lies in the lagging direction (i.e., toward the trailing edge of the airfoil).
- Z-axis: lies in the flapwise direction (i.e., in the direction of the free stream wind).

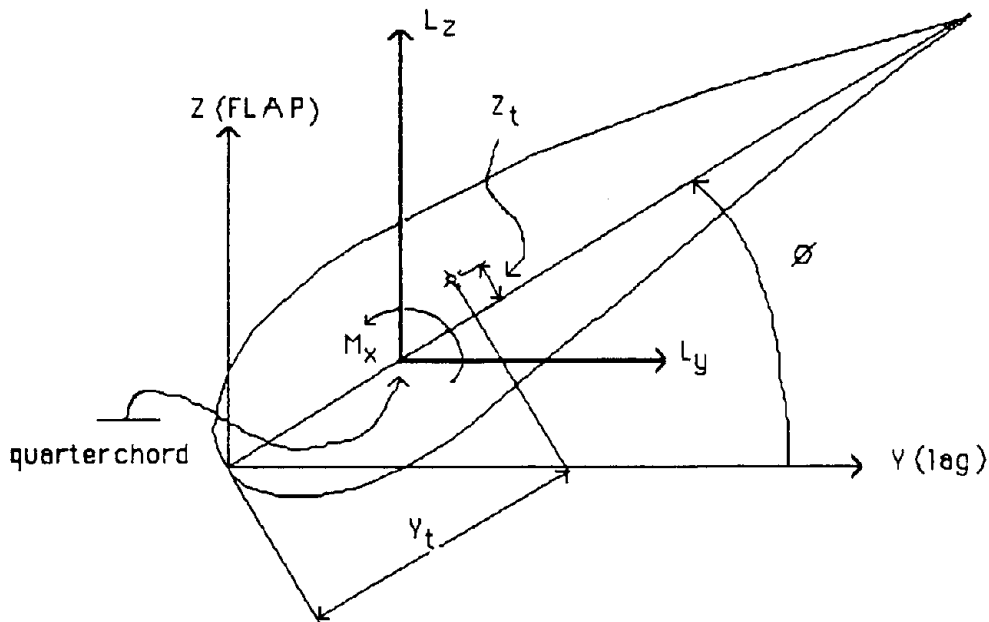
The construction of the 3-D blade model now includes the manufactured blade twist angle at each station. Clearly this affects the principal axes for highly twisted sections, and has the effect of "softening" the blade in the lag direction. As seen below in Figure 4.4, the elastic twist must also be added to the twist before the final equilibrium principal axes and the final equilibrium blade deflections can be found.

The applied loads, L_y and L_z , are assumed to be applied at the airfoil quarterchord (aerodynamic center) as would be true for airloading. For a bench test the applied loading would have to be carefully placed at the quarterchord to satisfy this requirement. Alternatively, the (arbitrary) applied load can be established as a load at the quarterchord plus a moment (couple) about the quarterchord. This way a twisting moment can be applied in the bench test and accurately reflected in the beam calculations. For example, a concentrated weight at .75 span, located on a jig forward of the blade leading edge, produces a load and a substantial moment about the .75 station quarterchord. This was the method used to produce primary blade torsion in the tests.

The total quarterchord moment (M_x) and loading (L_y, L_z) are calculated by the program. Then the twisting moment is calculated; this is the section moment about the shear center, which is the center of twisting. Then, using the blade torsional rigidity (GJ) values calculated above via the shear flow method, the blade twist is found. This must be first integrated from the blade tip inward, and then summed from the root to the tip, to accumulate all the differential twist of the blade. Finally the program finds the "elastic twist," which is simply the differential section twist at each station. This will usually be a maximum at the tip, for all practical cases. The blade twist calculation is simpler than the flexural calculation since

principal axes are not needed. However, the twist must be solved first since it affects the actual resulting principal axes location for each station. The moment about the shear center is simply

$$Q_c = [L_y \cos \theta + L_z \sin \theta] Z_T + [L_y \sin \theta - L_z \cos \theta] [Y_T - .25 C] + M_x$$



where:

- C = blade chord
- θ = principal axis angle + twist angle + elastic twist
- L_z = applied flapping load (quarterchord)
- L_y = applied lag load (quarterchord)
- M_x = applied pitching moment (nose down)
- $(Y, Z)_t$ = shear center

FIG. 4.4 SECTION TORSIONAL MOMENTS

With the applied loading in the Y and Z directions (L_y, L_z) given, as for a static bench test, the blade shears and moments, in the lag and flap directions, and then in the principal axes directions, can be calculated. When the moments in the principal axes have been calculated, the blade strains can be found directly from the simple flexure formula above for each section of the blade.

$$V_y = \text{inplane shear} = \int_{\text{root}}^{\text{tip}} L_y dx$$

$$V_z = \text{flapping shear} = \int_{\text{root}}^{\text{tip}} L_z dx$$

$$M_z = \text{inplane moment (moment about the Z axis)} = \int_{\text{root}}^{\text{tip}} V_y dx$$

$$M_y = \text{flapping moment (moment about the Y axis)} = \int_{\text{root}}^{\text{tip}} V_z dx$$

The root shears are important because they combine with other blade root shears to yield hub shears. Likewise, the inplane moment combines with the other blade moments also to form the rotor torque.

Applied moments in the principal axes are

$$M_{yp} = M_y \cos \varnothing - M_z \sin \varnothing$$

$$M_{zp} = M_z \cos \varnothing + M_y \sin \varnothing$$

where \varnothing is the principal axis angle.

Therefore, the bending strains at any given point on the airfoil are given simply by the $(y,z)_p$ contributions

$$\epsilon (x,y)_i = \frac{M_{zp} y_{pi}}{E I_{zp}} - \frac{M_{yp} z_{pi}}{E I_{yp}}$$

In the program the selected points on the airfoil ("stress offsets") are calculated first, and appear above as $(y,z)_{pi}$. Also, provision for blade tension (spanwise axial) loading is included, as would occur with centrifugal tension for a rotating blade.

The linear flexural deflections of the blade are found simply by integrating the moments, now expressed in the true flap and lag (y,z) directions:

$$\frac{M}{E I_y} = \frac{M}{E I_{yp}} \cos \varnothing + \frac{M}{E I_{zp}} \sin \varnothing$$

$$\frac{M}{E I_z} = \frac{M}{E I_{zp}} \cos \varnothing - \frac{M}{E I_{yp}} \sin \varnothing$$

$$\frac{d^2z}{dx^2} = \frac{M}{E I_y} \quad ; \quad \frac{d^2y}{dx^2} = \frac{M}{E I_z}$$

The z and y deflection curves are simply the flap and lag deflections of the blade axis.

In this calculation, a cantilevered blade has been assumed, as would be true for a rigid hub, or for a cantilevered blade bench test. Computationally, this is expressed as boundary conditions of the slope and deflection at the root ($x = 0$) being zero:

$$z_o = y_o = 0 \quad \text{and} \quad \frac{dz_o}{dx} = \frac{dy_o}{dx} = 0$$

Analytically from St. Venant torsion theory, statically (Ref. 11):

$$\tau(x) = GJ(x) \frac{d\theta(x)}{dx}$$

where:

$\tau(x)$ = applied torque (at the elastic axis)
 $GJ(x)$ = distribution of torsional rigidity

$$\frac{d\theta(x)}{dx} = \text{angle of twist}$$

The general torsional equation of motion is

$$\frac{\partial}{\partial x} \left[GJ(x) \frac{\partial \theta}{\partial x} \right] - I_o \theta - I_o \Omega^2 \theta = \frac{\partial \tau(x)}{\partial x}$$

where I_o = torsional mass moment of inertia (polar moment of inertia) per unit length.

4.4: BLADE NATURAL BENDING FREQUENCIES AND MODES

The static analysis so far has produced a beam model that has distributed elasticity (section properties) and masses (section masses) from a complex geometry composite blade. Bench tests can be expected to verify (and refine) the elastic constants, and blade sectioning will verify the masses. It remains to verify the dynamics of the blade model, which can only be done by calculating the natural frequencies of vibration (and mode shapes) in flexure and torsion, and checking these also with tests. The static deflection is most sensitive to the stiffness properties, but the frequencies of vibration are most sensitive to the masses.

In the spirit of practical engineering calculation, the flexural frequencies and modes are considered to be uncoupled, flapwise from chordwise. This assumption is less valid for very highly twisted blades, but still accurate enough for present purposes.

The method for solution was suggested by Holzer (Ref. 14) for the purpose of finding natural torsional frequencies of crankshafts, and was then modified by Myklestad (Ref. 15) for the bending vibrations of nonlinear beams. Later work for propellers included the centrifugal effect of rotation (Refs. 16, 17). The continuous blade (beam) is represented by a number of discrete segments; analytically speaking, the partial differential equation for bending is thus replaced by a set of ordinary differential equations, one for each segment. The free body diagrams for each segment are easy to analyze, and lead to a direct physical knowledge of the vibrating beam; 10 segments are used here. The deflected blade is shown in Figure 4.5. The blade is segmented into ten equal lengths and the mass of each length is equally divided into lumped masses at either end. The elastic properties (EI and GJ) are assumed to be constant over the beam element.

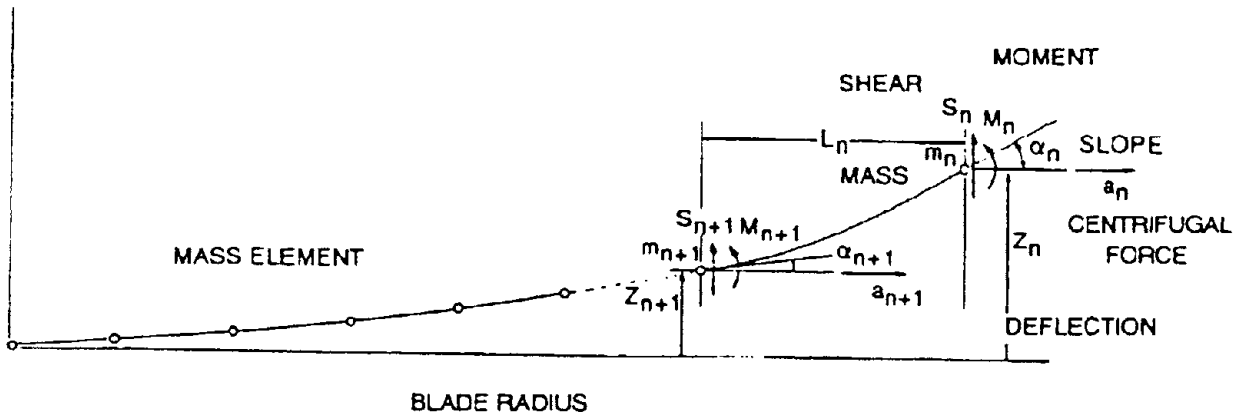


FIG. 4.5 GENERAL DEFLECTED BLADE: HOLZER / MYKLESTAD

The free body diagram for the n th element is shown in Figure 4.6.

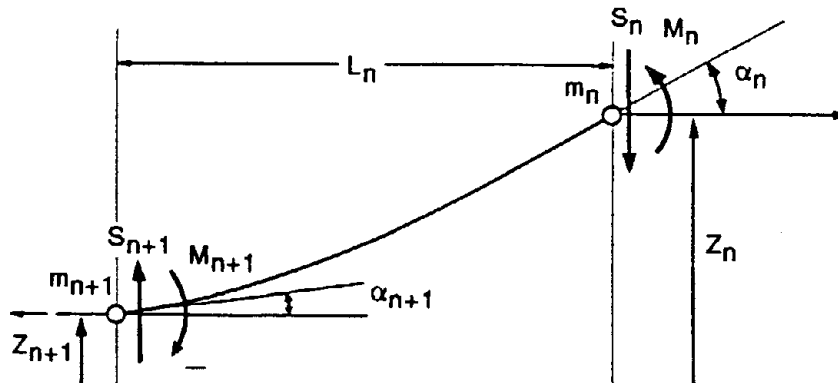


FIG. 4.6 FREE BODY DIAGRAM: HOLZER TABLE

where:

- m_n = n th lumped mass
- α_n = n th slope
- a_n = n th tension (centrifugal force)
- M_n = n th bending moment
- L_n = n th segment length
- S_n = n th shear
- Z_n = n th bending deflection

The partial differential equation of bending of the beam is fourth order; therefore, four coefficients are needed to express unit load effects due to an applied moment and an applied force. The unit load and moment coefficients are derived from a simple cantilever with a unit end load, F , or a unit moment, M (Figure 4.7).

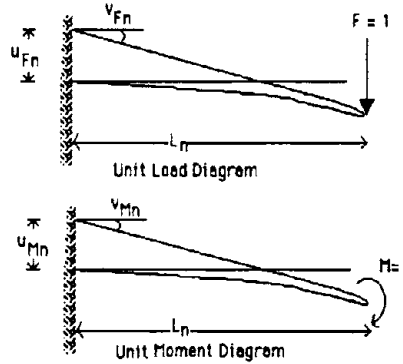


FIG. 4.7 UNIT LOAD AND MOMENT DIAGRAMS

The resulting coefficients are

$$\begin{aligned}
 u_{Fn} &= \text{deflection due to unit force} = L_n^3 / 3(EI)_n \\
 v_{Fn} &= \text{slope due to unit force} = L_n^2 / 2(EI)_n \\
 u_{Mn} &= \text{deflection due to unit moment} = L_n^2 / 2(EI)_n \\
 v_{Mn} &= \text{slope due to unit moment} = L_n / (EI)_n
 \end{aligned}$$

Note: the above coefficients are sometimes written in non-unit form as

$$\begin{aligned}
 u_{Fn} &= u_F = F L^3 / (6 EI) \\
 v_{Fn} &= \theta_F = F L^2 / (2 EI) \\
 u_{Mn} &= u_M = M L^2 / (2 EI) \\
 v_{Mn} &= \theta_M = M L / (EI)
 \end{aligned}$$

Also, the Maxwell Reciprocal Relationship "the u due to the $M = \theta$ due to the F " is easily seen above. Now the segmented blade, rotating at frequency Ω , is assumed to vibrate at frequency ω . An additional inertial force ($M_{n+1} \omega^2 z_{n+1}$) is added to the free body diagram above. Force and moment equilibrium then yield

$$\begin{aligned}
 \text{tension} &= a_{n+1} = a_n + m_{n+1} \Omega^2 r_{n+1} \\
 \text{shear} &= S_{n+1} = S_n + m_{n+1} \omega^2 z_{n+1} \\
 \text{moment} &= M_{n+1} = M_n - S_n L_n + a_n (z_n - z_{n+1}) \\
 \text{slope} &= \alpha_{n+1} = \alpha_n (1 + a_n v_{Fn}) - S_n v_{Fn} + M_n v_{Mn} \\
 \text{displacement} &= z_{n+1} = z_n - (L_n + u_{Fn} a_n) \alpha_n + u_{Fn} S_n - u_{Mn} M_n
 \end{aligned}$$

The tension at the n th segment can be rewritten as a sum of tensions:

$$a_n = \sum_{i=1}^n m_i \Omega^2 r_i$$

And the deflection can be substituted into the moment expression to give

$$M_{n+1} = M_n (1 + u_{Mn} a_n) - S_n (L_n + u_{Fn} a_n) + \alpha_n a_n (L_n + u_{Fn} a_n)$$

These recurrence formulas could be solved by successive substitution, working from the tip of the blade to the root. Shear, moment, slope, and deflection would be calculated for each segment, and resultant distributions would be known. The slope is assumed to be an arbitrary angle, θ , and the deflection a unit quantity. Then, trial values for frequency (or ω^2) are introduced. For the trial ω to be the correct frequency, the slope at the root and the deflection must correspond to the appropriate root boundary condition; viz:

Cantilevered:

$$\text{Slope at root} = \text{deflection at root} = 0$$

Hinged:

$$\text{Moment at root} = \text{deflection at root} = 0$$

Myklestad applied this to systematic computation by making judicious new definitions and substitutions, as follows. The load and moment coefficients and centrifugal forces can be combined via the "Myklestad definitions," which simplify the above expressions:

$$\begin{aligned} A_n &= 1 + v_{Fn} a_n \\ B_n &= 1 + u_{Mn} a_n \\ C_n &= L_n + u_{Fn} a_n \\ D_n &= C_n a_n = (L_n + u_{Fn} a_n) a_n \end{aligned}$$

Then, a set of assumed linear functions is substituted in the set of four recurrence formulas for shear, moment, slope, and deflection. The assumed slope at the tip, φ , is carried as the independent variable; the assumed Myklestad amplitude coefficients are

$$\begin{aligned} \text{Shear:} \quad S_n &= -G_{\varphi n} \varphi + G_n \\ \text{Moment:} \quad M_n &= H_{\varphi n} \varphi - H_n \\ \text{Slope:} \quad \alpha_n &= h_{\varphi n} \varphi - h_n \\ \text{Deflection:} \quad z_n &= -g_{\varphi n} \varphi + g_n \end{aligned}$$

Substituting these linear functions into the four original recurrence formulas and equating coefficients yields simple recurrence formulas for the amplitude coefficients now suitable for systematic computation. [Note: The classical Myklestad derivation is used here to avoid the ambiguity, which is seen in sign definitions in later works.]

$$\begin{aligned} G_{\varphi(n+1)} &= G_{\varphi n} + M_{n+1} \omega^2 g_{\varphi(n+1)} \\ G_{(n+1)} &= G_n + M_{(n+1)} \omega^2 g_{(n+1)} \\ H_{\varphi(n+1)} &= B_n H_{\varphi n} + D_n h_{\varphi n} + C_n G_{\varphi n} \\ H_{(n+1)} &= B_n H_n + D_n h_n + C_n G_n \\ h_{\varphi(n+1)} &= A_n h_{\varphi n} + v_{Mn} H_{\varphi n} + v_{Fn} G_{\varphi n} \\ h_{(n+1)} &= A_n h_n + v_{Mn} H_n + v_{Fn} G_n \\ g_{\varphi(n+1)} &= g_{\varphi n} + C_n h_{\varphi n} + u_{Mn} H_{\varphi n} + u_{Fn} G_{\varphi n} \\ g_{(n+1)} &= g_n + C_n h_n + u_{Mn} H_n + u_{Fn} G_n \end{aligned}$$

For the free tip case, the initial values are at the tip:

$$S_1 = M_1 \omega^2 \quad M_1 = 0 \quad \alpha_1 = \varphi \quad z_1 = 1.0$$

This yields the initial values of the amplitude coefficients:

$$\begin{aligned} G_{\varphi 1} &= 0 & G_1 &= M_1 \omega^2 \\ H_{\varphi 1} &= 0 & H_1 &= 0 \\ h_{\varphi 1} &= 1.0 & h_1 &= 0 \\ g_{\varphi 1} &= 0 & g_1 &= 1.0 \end{aligned}$$

Then the root, which is cantilevered, gives the condition (from deflection = 0)

$$\varphi = h_{r\text{root}} / h_{\varphi\text{root}}$$

The remainder in the program is the root deflection, which is the trial value; e.g., when the remainder is zero, the trial value of ω^2 was correct and a solution has been found:

$$Z_{\text{root}} = \text{remainder} = -g_{\varphi\text{root}} \varphi + g_{\text{root}} = 0 \quad (\text{for a root of the equations})$$

In general, this iterative method will give as many roots (frequencies) as there are lumped masses. The corresponding shear, moment, and deflection mode shapes are found for each root by simple substitution in the recurrence formulas. For most wind turbine blade dynamics, the lowest or fundamental frequency and mode shape are most important since they dominate the dynamic response (flap and lead-lag).

The computer program uses for input data the running mass of the blade and the section stiffness (for either flap or lead-lag) at the operating rpm. Another version of the program also allows the addition of a concentrated tip mass to assess the effect of a tip weight or mechanism (Appendix 10.3, 10.4).

4.5: BLADE TORSIONAL FREQUENCIES AND MODES

The blade torsional frequencies are also calculated using the Holzer table approach. The model assumed a shaft fixed at the root, with flywheels (I_i) at each station representing the lumped masses, and torsional stiffness (k_i) derived from the GJs:

$$k = \text{torsional stiffness} = GJ / L = GJ / \Delta r$$

where Δr is the segment length

$$I = \text{mass moment of inertia} = \int \int r^2 dm = \int \int (\rho dx) r^2 dA$$

where: r = chordwise coordinate
 x = spanwise coordinate
 ρ = mass density

[Note: the polar moment of inertia of a section is the ($r^2 dA$) portion.]

The $\rho r^2 dA$ integration must be performed in the section properties calculations since the density varies along the composite beam. Also, ballast weight must be assessed since it changes the section mass moment.

The question arises as to which value of section inertia to use, the minimum value (which occurs about the section cg), the value referenced to the shear center (about which section twisting occurs), or the value referred to the blade geometric axis (blade axis). In this treatment the elastic axis (shear center) is used as reference as explained here:

- For a uniform blade with a symmetrical cross section and uniform properties spanwise, the geometric, elastic, and mass axes coincide. Therefore, twisting in this case will occur in pure shear (twist) with the geometric center as its "axis of twisting."
- If the geometric axis is offset (chordwise) from the support at the root, the blade torsional motion still occurs on the mass/elastic axis of the blade. The offset support has just added another resultant degree of freedom: that of rigid body flexure and twisting about the support. [Note: this can be easily visualized by making a sketch.]
- Next for an unsymmetrical weight on the shaft (ballast), the new twisting motion will occur about the (old) elastic axis as before, even though it is not the geometric (from above) or mass axis anymore. Finally, for a complex geometry, the action of pure twisting is as close as possible to the elastic axis of the blade, even though this may not be a straight line. [Comment: This line of

reasoning, admittedly qualitative, was empirically verified in the modal analysis testing of the UTRC blade. The FFT software allowed computer graphics animation of the fundamental torsional motion, which clearly showed the twisting occurring about the elastic axis and not the mass axis.]

If the blade contains nonstructural or ballast weights, the mass centers are relocated off the elastic axis. This adds

- 1) torsional coupling to the bending modes (flutter),
- 2) bending to the torsional modes,
- 3) increasing mass moment of inertia to the section; thus lowering the natural torsional frequencies.

The solution is found using the same approach as before. The free body diagram is shown in Figure 4.8.

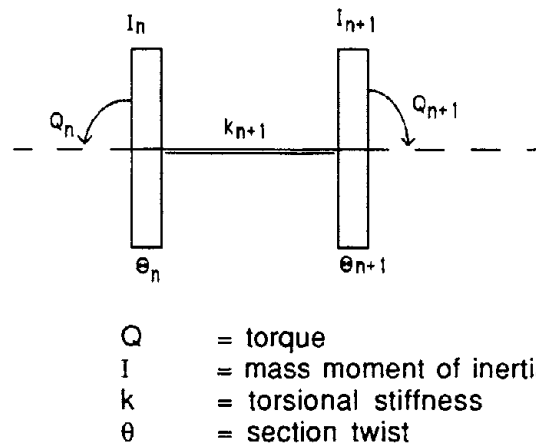


FIG. 4.8 FREE BODY DIAGRAM FOR TORSIONAL VIBRATION

The governing equations (recurrence formulas) are much simpler than before:

$$Q_{n+1} = Q_n + I_{n+1} \omega^2 (\theta_{n+1} - \theta_n)$$

$$\theta_{n+1} = \theta_n + (Q_{n+1} - Q_n) / (k_{n+1})$$

The boundary condition at the tip ($n = 1$) is $\theta = 1$; the remainder is the twist (θ) at the root which must equal zero for a solution. Actually this is precisely the method used by Holzer to calculate crankshaft frequencies by hand.

Again, there is provision in one version of this program to assess tip weights (inertias) on torsional frequency (Appendix 10.4).

5.0 DEVELOPMENT OF BLADE MASS/ELASTIC MODELS

5.1: TRIANGULATION OF MASS/ELASTIC PROPERTIES

The primary need in this phase was to verify the accuracy of the engineering constants used in the beam analysis. This involved comparing the static bending and frequency of vibration results from the experiments with the predictions based on assumed engineering constants. The static bending and frequency tests "bracket" the error bounds on uncertainty of the engineering constants and masses, since the static deflection results are more sensitive to the moduli, and the frequency results to the masses. Discrepancies must be evaluated and solutions iterated until a satisfactory semi-empirical set of engineering constants exists for each blade that satisfies both the theory and the tests (static and vibration). Additionally, the blade root attachments were evaluated to obtain the bending and torsional compliance of the attachments, primarily by referencing the root restraint view in the modal survey and by baseline spanwise inclinometer readings in the static tests. At the end of this phase a practical, accurate, and workable mathematical model existed for each blade/hub combination of the study.

5.2: UNIFORM FRP BLADE EXAMPLE: UTRC BLADE

The UTRC blade was the simplest structurally since it was a uniform pultrusion. The flexbeam/pendulum mechanism was not tested since the twisting kinematics were considered to be beyond the scope of this work. Clearly, the majority of the elastic twist of the flexbeam rotor is occurring in that portion of the blade that could be easily modeled elastically, which is proved in the results. This blade was a good choice for this study since:

- a. Blade is uniform pultrusion construction, which is representative of many rotor designs, and offers good cost reduction potential.
- b. Blade section properties are uniform and straightforward.
- c. Rotor has shown performance and flutter problems in the past due to blade flexibility (these were likely due to the flexbeam, however).
- d. Rotor performance was documented by AEI and SERI.

The UTRC blade is shown in Figure 1.3, at its full radius of 192 in., and with the root flexbeam portion comprising the inner 0.3 of the radius. The pultrusion was manufactured by Morrison Molded, and the flexbeam by UTRC. The pendulum weight and flexstrap pitching mechanism is not shown. The blade tested in this report was the outboard pultrusion, which has a cross-section as shown in Figure 5.1, from Ref. 18. This is a simple blade section, formed of a uniform laminate (constant thickness) skin with the airfoil shape and a solid spar in the leading edge, and a foam-filled aft section. The flexbeam portion was not tested but was actually very easily modeled since it was a simple rectangular cross section. This part of the report describes the development of a suitable aeroelastic model for this blade.

5.2.1: METHOD

To illustrate the derivation of purely theoretical section properties for a design, this discussion will begin by not including the test data, just the geometries and constituents. The airfoil shape NACA 23112 is determined by a table of offsets. The standard for the AEI program is 33 pairs of points (X,Y) beginning at the nose, proceeding around the upper surface to the tail, and returning to the nose on the lower surface. This is the data set format used in the program.

5.2.2: PREDICTION OF ENGINEERING MODULI AND DENSITIES

The skin laminate was comprised of four layers of reinforcement of two types: unidirectional (0°) and balanced ($\pm 45^\circ$), with two layers of each type. In this case we are not told the reinforcement cloth makeup; instead we have the thicknesses, which are

- .040" per layer for $\pm 45^\circ$ and
- .025" per layer for 0°

This gives a total (specified) skin thickness of .130 in. (Ref. 1). It remains to reduce this skin laminate to a single thickness with the proper overall density r_{skin} , longitudinal modulus E_{skin} , and shear modulus G_{skin} . The same must be done for the spar later.

NACA 23112 AIRFOIL

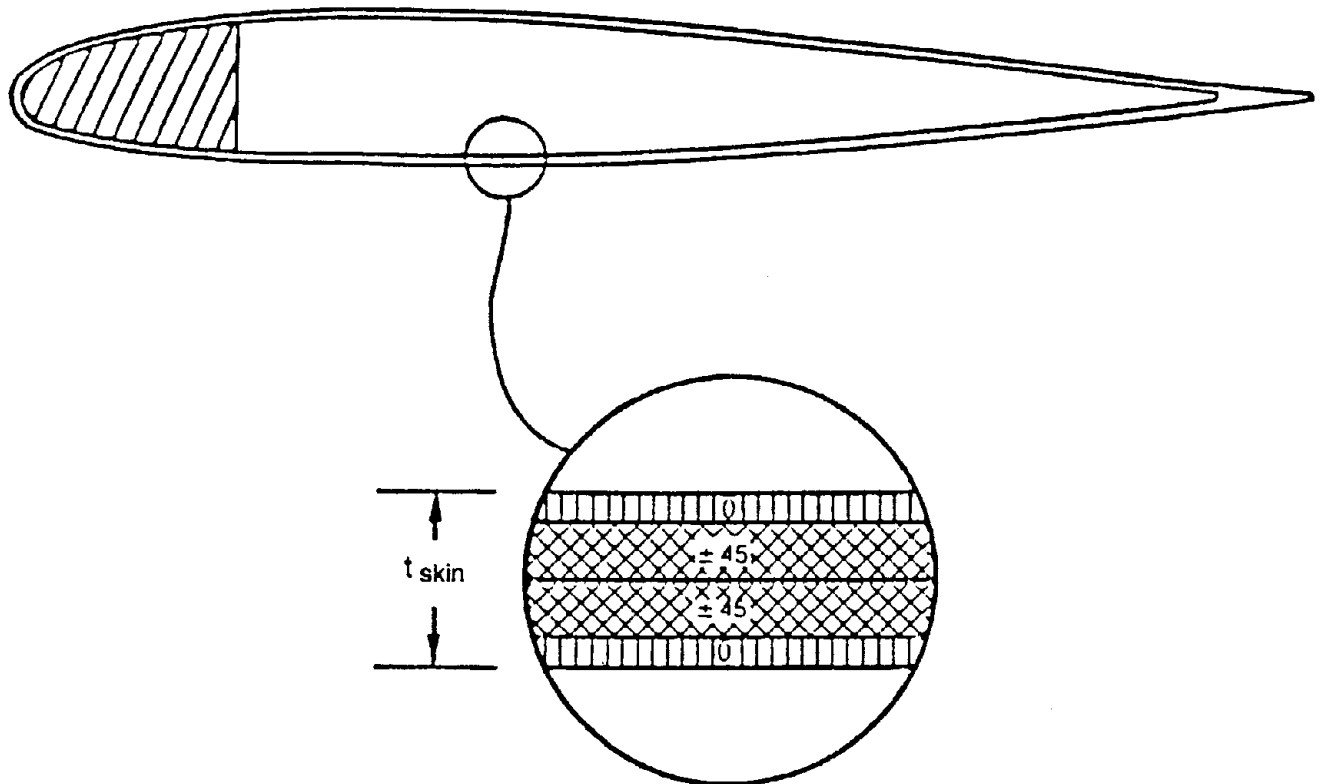
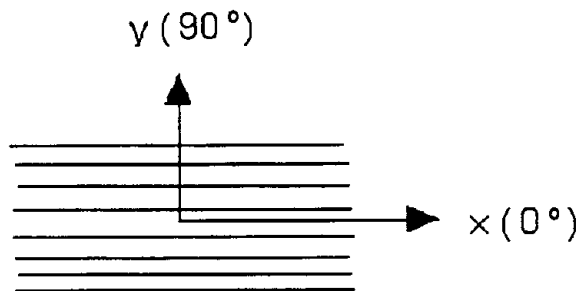


FIG. 5.1 UTRC CROSS SECTION

To calculate these values for the skin, first look at this unidirectional layer:



The properties of a unidirectional laminate are shown in Table 4.1, for glass reinforcement and epoxy or polyester resin matrix. If 50% volume is glass, the E-modulus is $5 \cdot 10^6$ psi and the density .0678 lb/in³, from first principles (see Table 4.1). For the transverse E-modulus, zero is normally assumed, as it is only the resin matrix. Practically speaking the G-modulus is also zero. We will estimate 50% fiber volume as representative of this pultrusion process, as it is for hand lay-up. This fiber ratio (or reinforcement to resin ratio) is very important in composites, as can be seen in this simple example. Fortunately, this ratio is easily found in an ASTM "burn-off" test, which simply burns off the resin under controlled conditions in a crucible to get an actual weight and thus a volume percentage.

For the $\pm 45^\circ$ layer we assume half the fibers run at 0° and the other half at 90° . This changes the above modulus value by cutting it in half, since half are now at 90° . This gives the $0^\circ/90^\circ$ properties, again based on a 50% volume, as

$$\begin{aligned} E (0^\circ/90^\circ) &= 2.5 \cdot 10^6 \text{ psi} \\ \rho &= .0678 \text{ lb/in.}^3 \end{aligned}$$

Since this layer is not oriented at 0° (but $\pm 45^\circ$), we must decrease the longitudinal E-values by 50% (because the fibers are not aligned with the E-direction), and also increase the G-value by 50% (because at 45° , the fibers are now along the principal transverse shear or twisting direction). This is shown in the "rule of thumb" directional properties engineering guide of Figure 5.2.

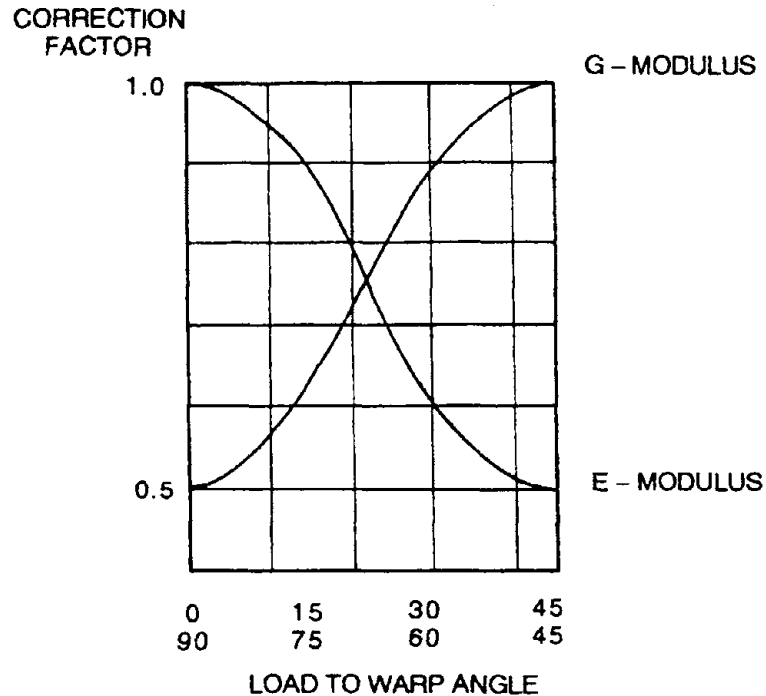


FIG. 5.2 RULE OF THUMB, FRP DIRECTIONAL PROPERTIES

Now we can calculate the total skin values of E and G by using the rule of mixtures. Analytically:

$$E_{\text{laminate}} = \frac{E_1 t_1 + E_2 t_2 + E_3 t_3 + \dots}{t_{\text{total}}}$$

This gives for the trial values:

$$E_{\text{skin}} = \frac{5 \cdot 10^6 \cdot .050 + 1.25 \cdot 10^6 \cdot .080}{.130} = 2.69 \cdot 10^6 \text{ psi}$$

$$G_{\text{skin}} = \frac{0.050 + 1.80 \cdot 10^6 \cdot .080}{.130} = 1.10 \cdot 10^6 \text{ psi}$$

Going on, the spar is a unidirectional bundle of rovings (fibers) in a compact resin matrix. We can expect a higher fiber volume (say 60%) and thus higher values than the less compact skin above:

$$E_{\text{spar}} = 6 \cdot 10^6 \text{ psi}$$

$$\rho_{\text{spar}} = .0727 \text{ lb/in.}^3$$

$$G_{\text{spar}} = 1.10 \cdot 10^6 \text{ psi (for simplicity)}$$

5.2.3: PREDICTION OF SPAR PROPERTIES

The skin contribution will be based on the composite values of E, G, and ρ determined above, integrated around the airfoil shape as input via the offsets. Also the spar values of E, G, and ρ have been estimated, but not the geometry. The program has the capability of assessing a D-spar in the leading edge and further spars or webs. Here we do not have a D-shape, so that calculation is eliminated. We treat this "spar" as a "web," so must describe it as a collection of moments of inertia and weights at its centroid. The program then adds its contribution simply by using the parallel-axis theorem. A graphic calculation of the spar yields

$$A_{\text{spar}} = 2.7076 \text{ in.}^2$$

$$I_x = 0.3831 \text{ in.}^4$$

$$I_y = 0.9887 \text{ in.}^4$$

$$I_{xy} = 0$$

$$(x,y)_{\text{centroid}} = (1.4796, 0.1843) \text{ in.}$$

The contribution of the foam is ignored.

5.2.4: INITIAL PREDICTED RESULTS

The above predictions yield section properties which we can only expect to be in "the ballpark." These are listed in Table 5.1 which also shows the 1) manufacturer's specifications (which have been used up to now), 2) the values from the blade sectioning, and 3) the final semi-empirical values after they have been verified by both testing and predictions. Obviously it is wise in most cases to perform the tests, including blade sectioning, because the blade model is much more accurate, as can be seen in the table.

TABLE 5.1: UTRC BLADE SECTION PROPERTIES COMPARED TO MANUFACTURERS' SPECIFICATIONS AND SECTIONAL VALUES

	Geometry (in.)				Mass and Stiffness							
	c	t _{AF}	t _{skin}	d _{spar}	m (lb/in.)	EI _{flap}	EI _{lag}	GJ	y _{cg} (%c)	EA(%c)	I	
Mfr's Spec	15	1.8	.130	2.5	.352	4.15	292	3.75	26.0	37	6.14	
Init. Pred.	(15)	1.8	(.130)	2.5	.431	6.57	377	7.43	31.8	33	8.59	
Measured	14.9	1.95	.118	2.56	.422	---	---	---	31.6	---	---	
Final Model	15	1.95	.118	2.56	.422	5.87	32.4	5.58	31.3	46.2	8.38	

5.2.5: SPECIFICATION VERIFICATION BY BLADE SECTIONING

By close blade inspection, and specifically by cutting a blade into sections, some of the predictions of the above approach can be much more accurate. Referring to Table 5.1, the specified skin thickness of 0.130 in. turned out to be 0.118 in. and the running mass went from 0.352 lb/in. to 0.422 lb/in. Clearly the manufacturer's specifications were wrong; thus the initial predictions above gave section properties that were also wrong; (see Table 5.1). At this stage, though, no direct measurement of EI and GJ can be made, nor have the mass and section inertia values been verified.

What can be concluded is that blade inspection is important to verify a manufacturer's claims. Here all the structural and mass properties are different. The aerodynamic properties as well are different; with an actual section thickness of 13% (1.95/15), this airfoil will behave differently than the specified 12% (NACA 23112).

5.2.6: VERIFICATION BY DEFLECTION TESTS

The key load case was 175 lb total load, evenly distributed along the span applied to the quarter chord. Figure 5.3 shows the predicted flapwise deflection of the UTRC blade for the EI's from both the initial calculation (above) and for the corrected semi-empirical values of EI. The test data are also shown. When the correct values (e.g., of skin thickness, airfoil thickness), have been determined from sectioning, the E-modulus in this case is simply varied until the static deflection matches the theoretical prediction for test, as shown.

There is more to this part, however. A test load case will generally produce flapwise, chordwise, and torsional deflections, and they all must be correctly predicted by the theoretical model. Therefore, it is not enough to match just the flapwise deflection; the chordwise deflection and twist must also match. In the AEI approach the loading is applied at the blade quarterchord. If the principal axes of the section are not exactly horizontal in the laboratory, some chordwise deflection will also occur. Also, if the section shear center (elastic axis) is off the quarterchord, twist will occur. For this load case (175 lb) 10.47 in. flapwise and 0.08 in. chordwise deflection, and roughly 0.1° (nose up) twist were measured. (Note: the blade was mounted lifting surface down to mimic airload.)

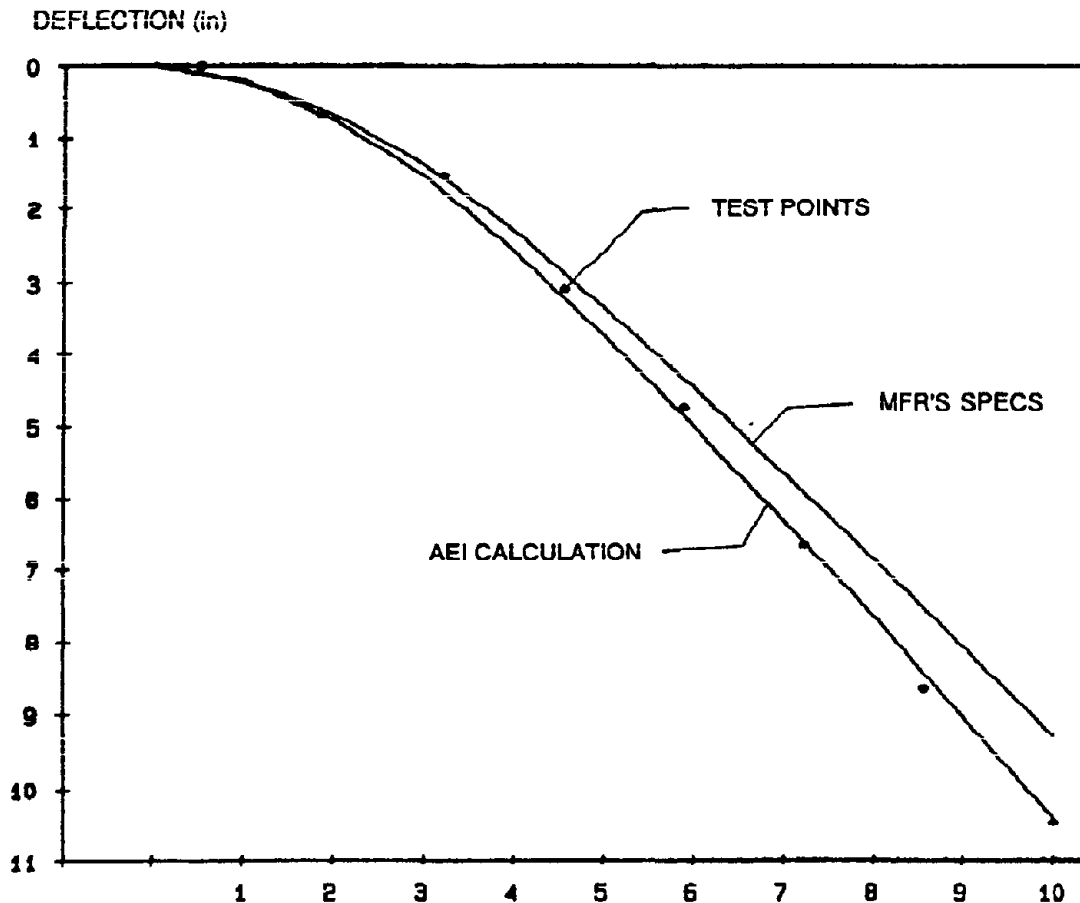


FIG. 5.3 UTRC FLAP DEFLECTION, TEST VS. CALCULATION VS. SPECIFICATIONS

Looking at the section properties, the principal axes of the blade are 0.42° (nose up). This is the effect which causes the inplane deflection. When this value of 0.42° is used as a simple pitch input in the calculation, the flap and lead-lag deflections are reproduced accurately. The resulting values of modulus are

		<u>initial values</u>
E_{skin}	$= 2.05 \cdot 10^6$ psi	$(2.67 \cdot 10^6)$
E_{spar}	$= 5.70 \cdot 10^6$ psi	$(6.0 \cdot 10^6)$
G_{skin}	$= 1.34 \cdot 10^6$ psi	$(1.1 \cdot 10^6)$
G_{spar}	$= 0.50 \cdot 10^6$ psi	$(1.1 \cdot 10^6)$

The initial guesses are also shown. So far the values of EI (flap and lag) have been verified, but the value of GJ has not, since there was so little elastic twist in this 175 lb load case. Also, the masses and moments of inertia have not been verified; these remain for the frequency tests. Using the torsional test case (23 ft-lb @ STA 0.9) the predicted twist is 0.24° (at the tip) compared to the test value of 0.37° . Either the GJ is too high or the elastic axis is slightly off. Rather than modify the G's to achieve congruence at this point, which could be done easily by reducing G_{skin} and G_{spar} equally until 0.37° is reached, we will depend on the frequency test still to come.

Note: A blade analysis could stop at this point, depending on the static weights for mass distribution and the information gained so far for section cg and section inertia. However, these parameters are crucial to the elastic twist on a "flying" blade, which has far more centrifugal force than the loads that can be conveniently applied in the lab, so further verification is warranted.

5.2.7: VERIFICATION BY FREQUENCY TESTS

The flapwise and first torsional frequencies and modes are shown in Figures 5.4 and 5.5.

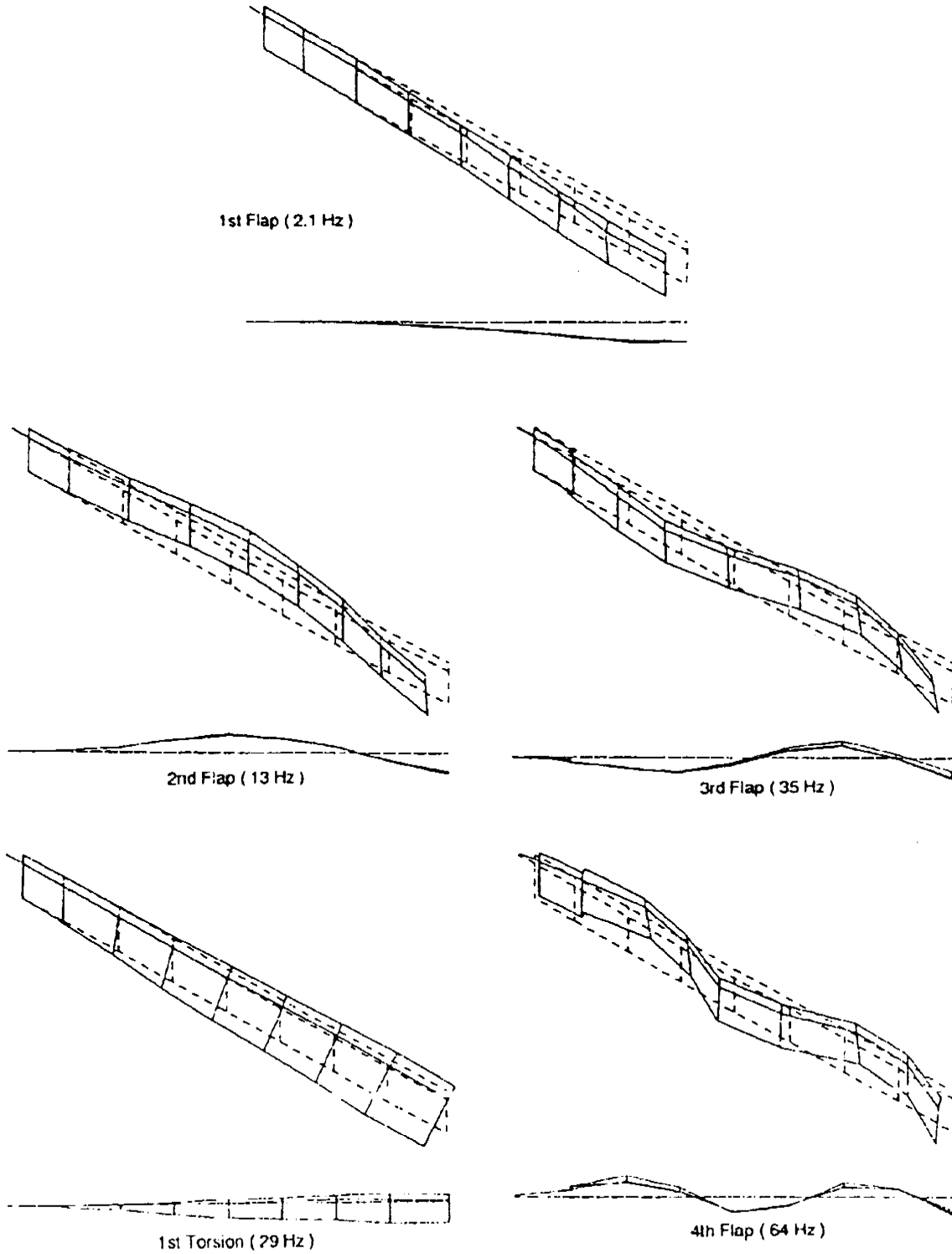


FIG. 5.4 UTRC VIBRATION MODES, TEST (MODAL SURVEY)

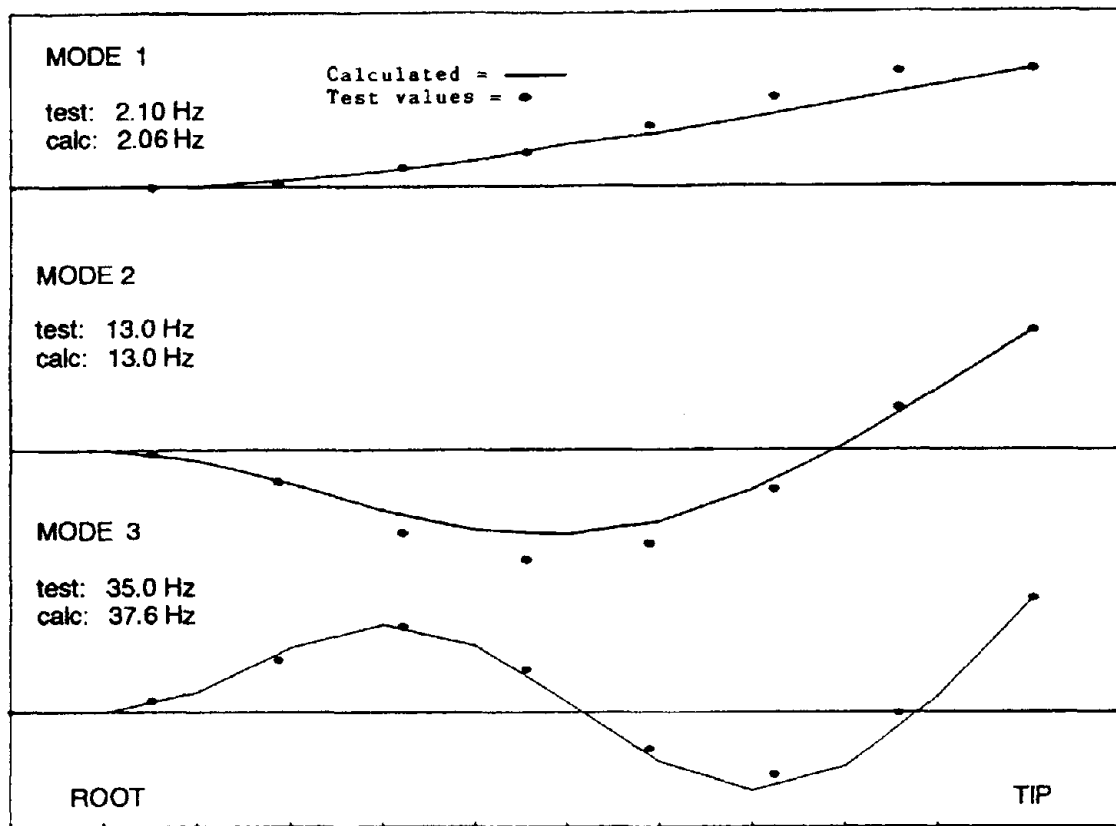


FIG. 5.5 UTRC FLAPWISE VIBRATION MODES, PREDICTED VS. TEST

The frequency program results are shown in Table 5.2 below:

TABLE 5.2: UTRC FREQUENCY OF VIBRATION RESULTS: TEST VS. PREDICTION

Mode	Test	Prediction
1st flap	2.1 Hz	2.06 Hz
2nd flap	13.0 Hz	12.99 Hz
3rd flap	35.0 Hz	37.60 Hz
1st lag	13.0 Hz	15.22 Hz
2nd lag	78.0 Hz	96.0 Hz

The principal axis stiffnesses were used since they are the decoupled modes on a uniform blade. It can be seen that the flapwise values are in excellent agreement, verifying the EI_{flap} and masses. The lag frequencies are different from the test presumably because the root fixity softened the cantilever, thus driving the lag frequencies down. In a sense the lag value is inconsequential anyway, because it is much higher than the first flap frequency.

The torsional values are shown below. Clearly the test data have confirmed the blade values of GJ and I ; final GJ is $6.2 \cdot 10^6$ psi with $G_{spar} = 0.1 \cdot 10^6$ and $G_{spar} 0.9 \cdot 10^6$.

Mode	Test	Prediction
1st Torsion	29.0 Hz	28.39 Hz
2nd Torsion	84.0 Hz	84.71 Hz

In the torsional frequency calculation the blade is assumed to oscillate about its elastic axis (see 4.5 under Theoretical Method), which has a larger mass moment of inertia than about the section cg (10.51 vs. 8.38).

Going back to the flexure tests will "close the loop" on the GJ. The G's have been matched by the frequency tests, and the predicted static torsion of the load test above (23 ft-lb @ STA .9) now matches the test value of 0.37°. This is a good indication that the blade masses and stiffness are correct since they match static and dynamic test values simultaneously.

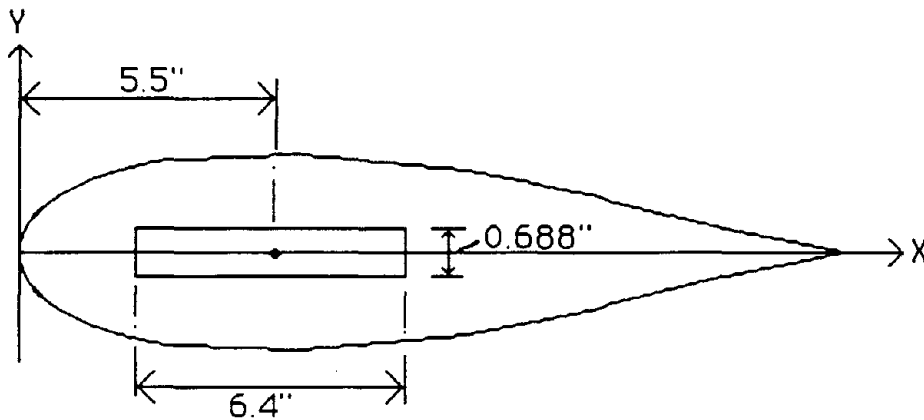
5.2.8: FINAL AEROELASTIC MODEL

At this stage the pultruded UTRC blade has been successfully modeled; the following conditions have been met by the predictions matching the tests:

- 1) total blade weight
- 2) blade flexural deflection, flap and lag
- 3) blade static torsional deflection
- 4) blade flapwise frequencies and modes
- 5) blade lag frequencies and modes
- 6) blade torsional frequencies and modes

It remains to construct the total aeroelastic model by adding the flexbeam portion and creating the appropriate blade properties file for the elastic twist calculations.

The flexbeam is attached at STA 0.3 and has the rectangular geometry shown below from inspection:



The input data for the flexbeam are derived from the manufacturer's specifications and the inspection values (the stiffnesses were not tested):

$$\begin{aligned}
 A_{I\text{-beam}} &= 4.4032 \text{ in.}^2 \\
 I_x &= 24.04 \text{ in.}^4 \\
 I_y &= 0.2404 \text{ in.}^4 \\
 I_{xy} &\cong 0.0 \\
 (x,y)_c &= (5.5, 0.0)
 \end{aligned}$$

also

$$\begin{aligned}
 EI_{\text{flap}} &= 1.37 \cdot 10^6 \\
 EI_{\text{lag}} &= 137 \cdot 10^6 \\
 GJ &= 0.24 \cdot 10^6
 \end{aligned}$$

The final values for the UTRC aeroelastic blade model are shown in Table 5.3.

TABLE 5.3: FINAL UTRC BLADE MODEL ANALYSIS

	STA	r/R	(in.) r	(lb/in.) m _o	(*10 ⁶) EI _{flap}	(*10 ⁶) EI _{lag}	(*10 ⁶) GJ	(in.) Y _{cg}	(in.) YEA	I _{cg}
Root	11	0	0	.32	1.37	137	0.24	5.50	5.50	1.74
	10	.1	19.2	.32	1.37	137	0.24	5.50	5.50	1.74
	9	.2	38.4	.32	1.37	137	0.24	5.50	5.50	1.74
	8	.3	57.6	.426	5.867	324	5.58	4.69	6.93	8.38
	7	.4	76.8	.426	5.867	324	5.58	4.69	6.93	8.38
	6	.5	96.0	.426	5.867	324	5.58	4.69	6.93	8.38
	5	.6	115.2	.426	5.867	324	5.58	4.69	6.93	8.38
	4	.7	134.4	.426	5.867	324	5.58	4.69	6.93	8.38
	3	.8	153.6	.426	5.867	324	5.58	4.69	6.93	8.38
	2	.9	172.8	.426	5.867	324	5.58	4.69	6.93	8.38
Tip	1	1.0	192.0	.426	5.867	324	5.58	4.69	6.93	8.38

5.3: WOOD-EPOXY BLADE EXAMPLE: GOUGEON ESI 54 BLADE

The ESI/Gougeon blade was chosen for this study for the following reasons:

- The blade is of composite wood construction, with good material and arrangement quality control and very good documentation by the manufacturer.
- The airfoil is NACA 230XX series, used for helicopter rotors and other wind turbines, and has substantial available test data.
- Field test data were available and accurate.
- Compared to other composite wood blades, this blade has relatively high torsional stiffness but still measurable elastic twist.
- The root attachment is indicative of a standard teetering approach with high stiffness and potential for pitch/flap coupling.
- The blade radius is midway between the 10- and 20-meter diameters of the other blades in the test program.
- The blade planform represents a "tip speed ratio tolerant" design, with slight taper and twist, indicative of most constant RPM wind turbines.
- Tip brake/weight complicates the loads analysis for both structural and aerodynamic loads and was thought to be responsible for possible elastic twist observed in the field.
- Because this blade uses a low pitching moment airfoil, inertial elastic twist (especially from the tip brake) is likely to dominate the results.
Note: this prediction turned out to be true.

5.3.1: ESI BLADE DESCRIPTION

The ESI 54 blade, which is manufactured by Gougeon Brothers Inc. (Bay City, Mich.) is shown in Fig 1.2. It is of a wood-epoxy construction, and uses a web-reinforced shell with constant skin thickness at each span station and no D-spar. Also, it is not highly twisted and has no ballast weights or mechanisms. And probably most important, it has no structural discontinuities as do both the UTRC and Carter 300 blades. The root fixture is an oval bolted-stud pattern of relatively large dimension, so the root fixity in the test rig was relatively rigid. The blade was much stiffer than the Carter, and there was consequently much less static gravity deflection, on the order of a few inches only, which simplified the testing. The blade was tested and analyzed bare without the tip flaps which were used in the ESI 54 turbine. The elastic twist due to the tip mechanisms, as seen later, dominates this blade's elastic twist.

The manufacturer, GBI, was very cooperative, and provided a complete engineering description of this blade, along with drawings, section coordinates, section properties, and analyses (Ref. 21). These data are held proprietary to GBI, but the comparisons of the section properties are presented since they are

important. Clearly this test program verified both AEI's and GBI's section analysis programs, with the exception of the shear center calculation(see below).

5.3.2: DERIVE INITIAL SECTION PROPERTIES

The section property code input file for this blade is much easier than for a glass-reinforced composite blade. Geometrically, the only values needed for each station are the offsets (airfoil shape), skin thickness, and web values. This blade uses a modified NACA 230XX airfoil, so the standard profiles could not be used. Instead the GBI specifications were verified by sectioning, and these actual values were used.

The engineering moduli for Douglas fir vary over a rather wide range; however, the constituents and methods used by GBI are reliable and repeatable. Therefore, we had high confidence in the initial design spec:

$$\begin{aligned} E_{\text{grain}} &= 1.57 \cdot 10^6 \text{ psi} \\ G &= 0.15 \cdot 10^6 \text{ psi} \\ \rho &= 0.025 \text{ lb/in.}^3 \end{aligned}$$

It remains to find the skin thickness and web location for each blade station, either from the specs or from the tests, and then to calculate the section properties. The much more effective engineering approach factors in the test results from the very beginning, as illustrated here.

The blade inspection (sectioning) yielded measured values of running mass, section Cg location, shear web location, and skin thickness. These values plus the estimated moduli and density (above) provided a set of section properties calculated by AEI (Figs. 5.6 to 5.11). As can be seen, the two predictions (AEI and GBI) are in close agreement, with the notable exceptions of the shear center and GJ (which are discussed further below). Clearly the GBI program has included the contribution of the plywood web differently (and as shown below, more accurately) than the AEI program. It remains to validate all the section data by comparison with flexural and dynamic test data.

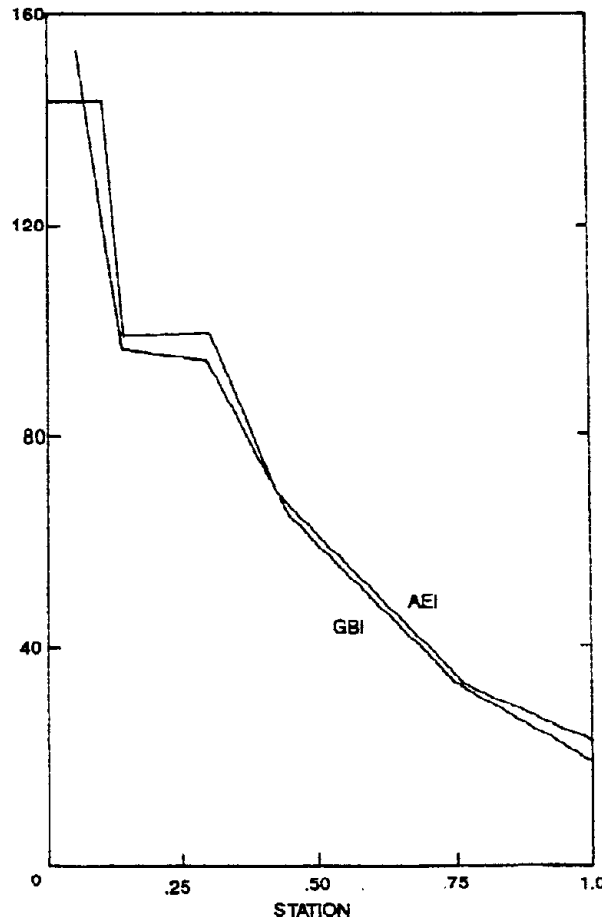


FIG. 5.6 ESI SECTION EA: AEI CALCULATION VS. GBI SPECIFICATION

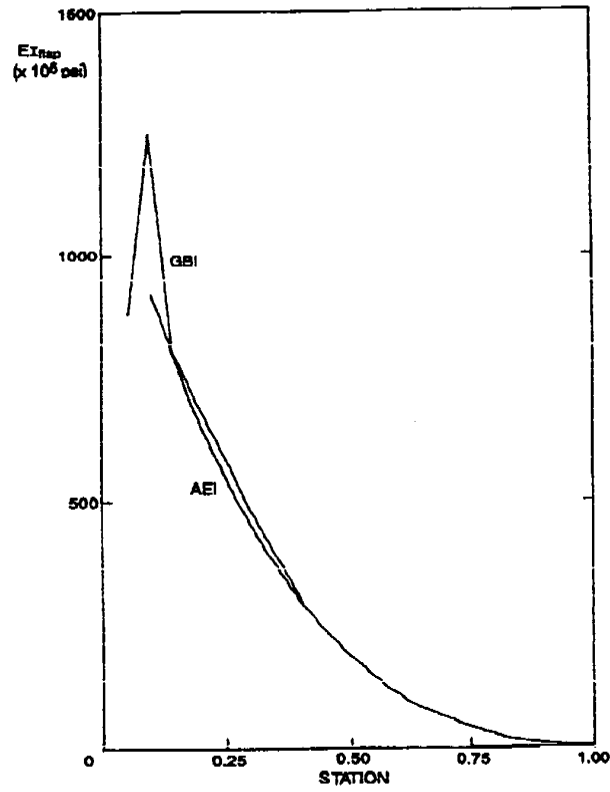


FIG. 5.7 ESI SECTION EI_{FLAP} : AEI CALCULATION VS. GBI SPECIFICATION

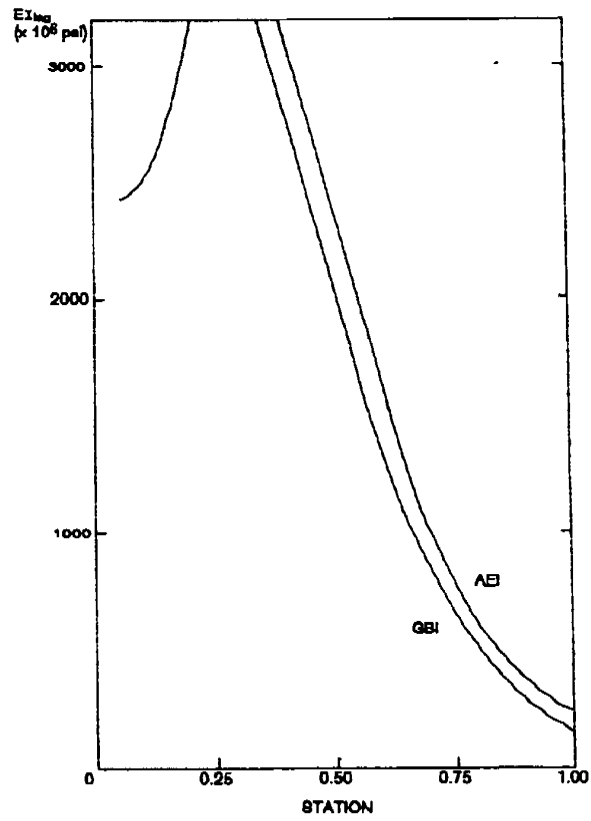


FIG. 5.8 ESI SECTION EI_{LAG} : AEI CALCULATION VS. GBI SPECIFICATION

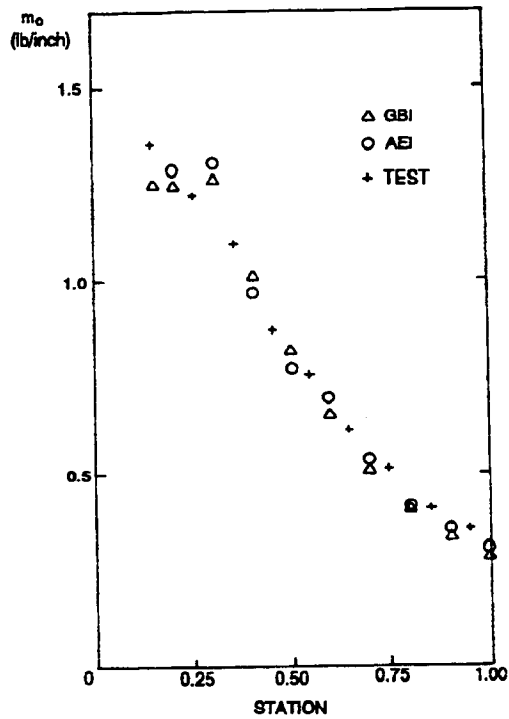


FIG. 5.9 ESI BLADE RUNNING MASS: AEI CALCULATION VS. GBI SPECIFICATION

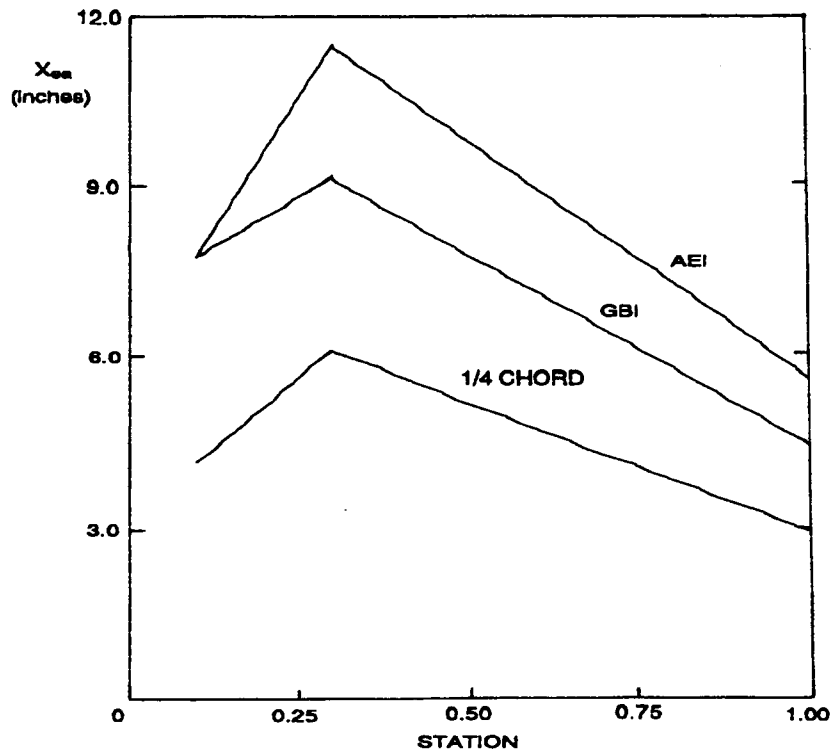


FIG. 5.10 ESI SECTION SHEAR CENTER: AEI CALCULATION VS. GBI SPECIFICATION

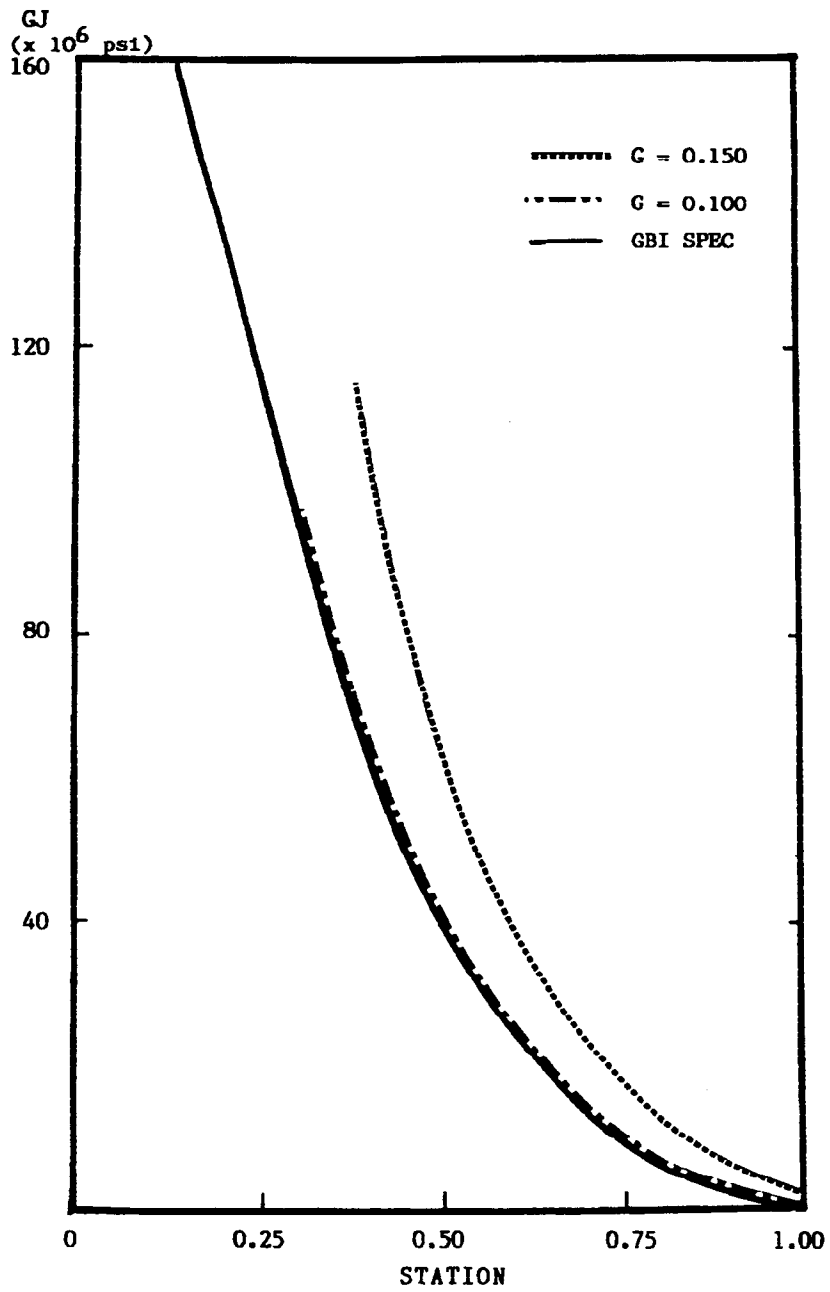


FIG. 5.11 ESI SECTION GJ: AEI CALCULATION VS. GBI SPECIFICATION

5.3.3: VERIFICATION BY FLEXURAL TESTS

The ESI blade flexural data (e.g., deflections vs. load) were different than for the UTRC blade. First, the ESI blade was so stiff in torsion, owing to the large wall thickness for the wood epoxy blade, that practically no measurable ($\pm 6'$ arc) elastic twist resulted from the bending, even for 450 lb. of distributed load (tip deflection of 5 in.). Second, the flap deflection values had to be corrected for the root elasticity; that is, when the blade was loaded, the steel root studs stretched allowing the blade root slope to change. This slope change amounted to 7 minutes of arc at 225 lb load and 16 minutes of arc at 450 lb. (Note: this introduced substantial tip errors of 0.66 in. and 1.51 in. in the raw distributions.)

The corrected (and uncorrected) blade deflection is compared to both the AEI and GBI section predictions in Figure 5.12. The agreement is very good especially at the tip where it is most important. Therefore, the original input values were considered to be correct for this blade in flexure.

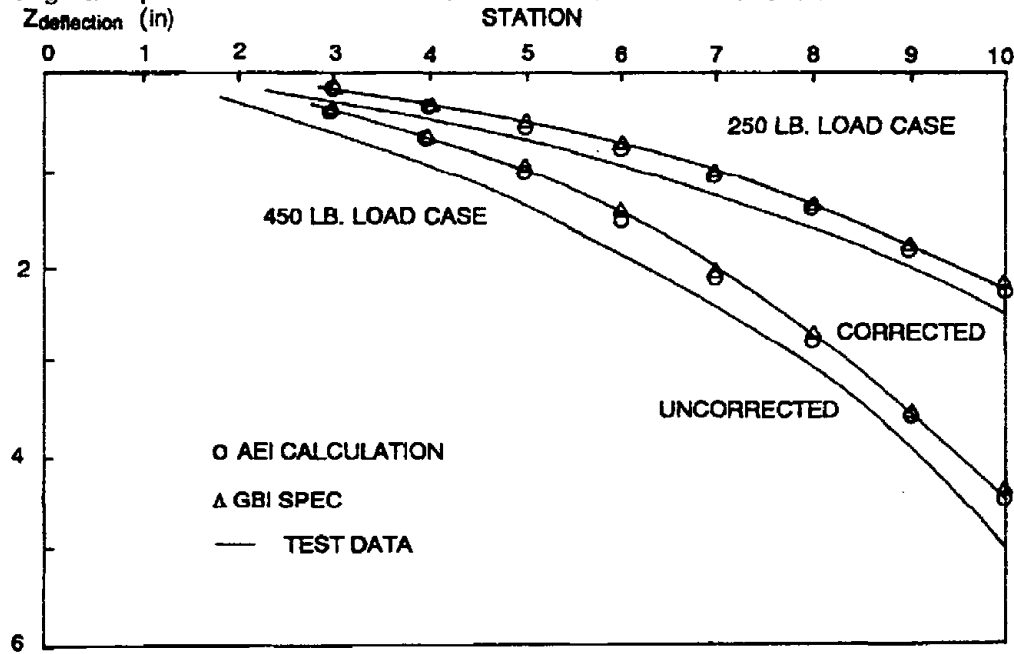


FIG. 5.12 ESI BLADE DEFLECTION: PREDICTIONS VS. TEST

5.3.4: VERIFICATION OF STATIC TWIST

The static twist of the distributed load cases (225 lb, 450 lb) was too small to be useful (see Fig 5.13). The torsion load case applied 50 lb. concentrated load at STA 0.9 of the test blade, offset 12.4" forward of the blade quarterchord. The test result is shown in Figure 5.14; it is a small but very believable distribution with an accuracy at least as good as 0.1 deg ($6'$ of arc). Therefore, there was good confidence in both test results, even though the amplitudes were very low.

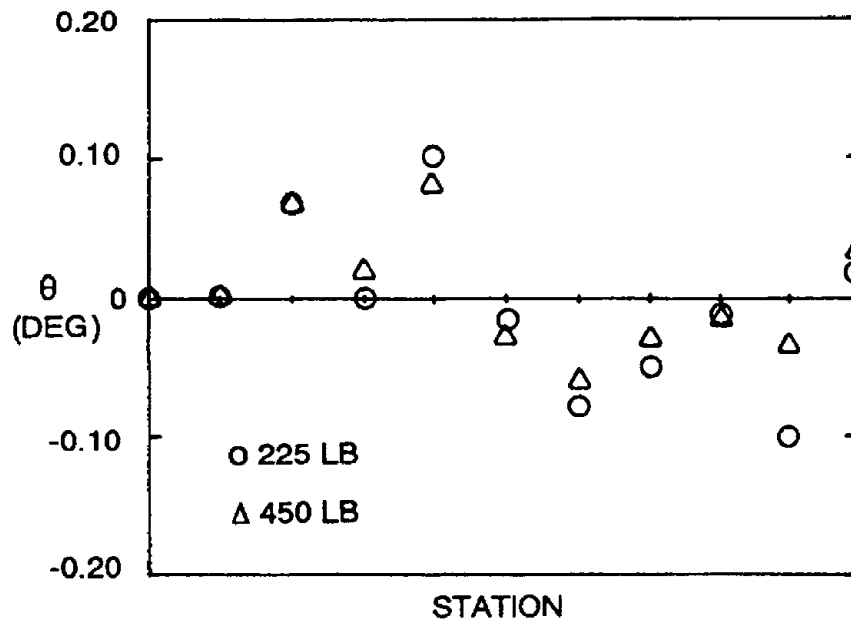


FIG. 5.13 ESI BLADE STATIC TWIST

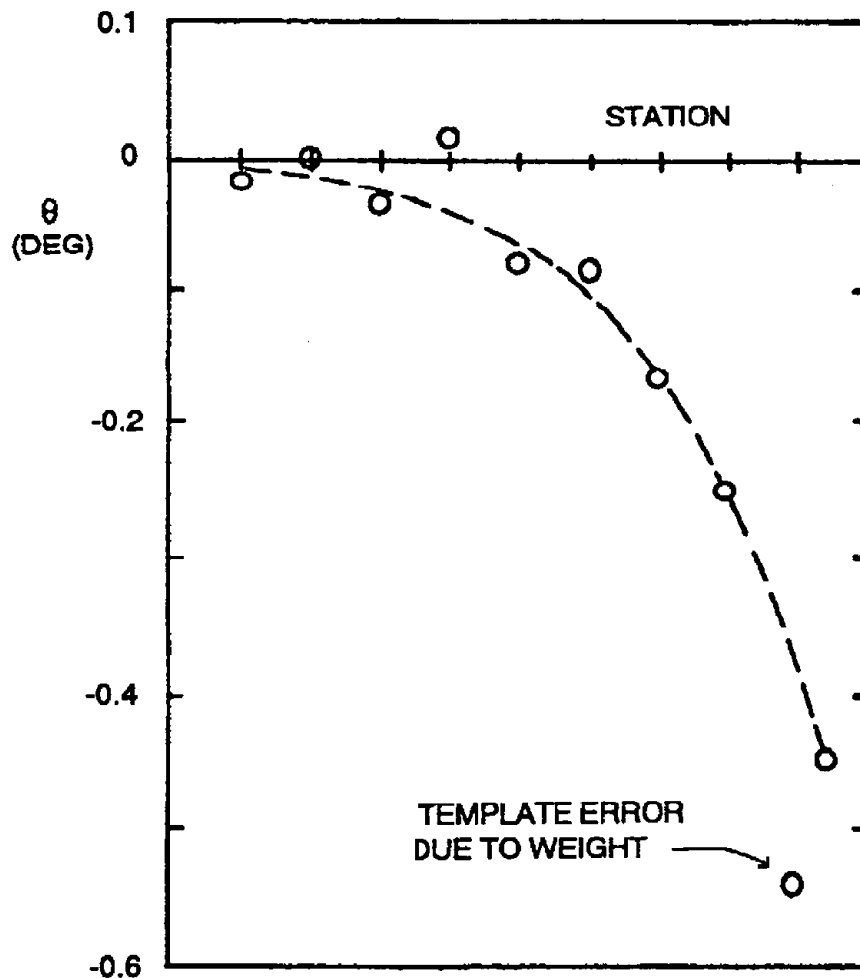


FIG. 5.14 ESI BLADE STATIC TWIST (50 lb TORSION LOAD CASE)

The original distributions are shown in the GJ calculation in Figure 5.11; the AEI prediction used $G = 0.15$. Lowering G to 0.10 gives the final GJ, which agrees well with the GBI specification. However, when these two distributions are used to predict the static torsion above, it can be seen in Figure 5.15 that the tip twist varies by about 15%, even though both distributions are close to the test values. The blade GJ can not

be considered to be verified yet though, since the load case was only a single static moment applied to the quarterchord. [As seen before for the UTRC blade, the torsional frequency is a much more accurate determinant of GJ].

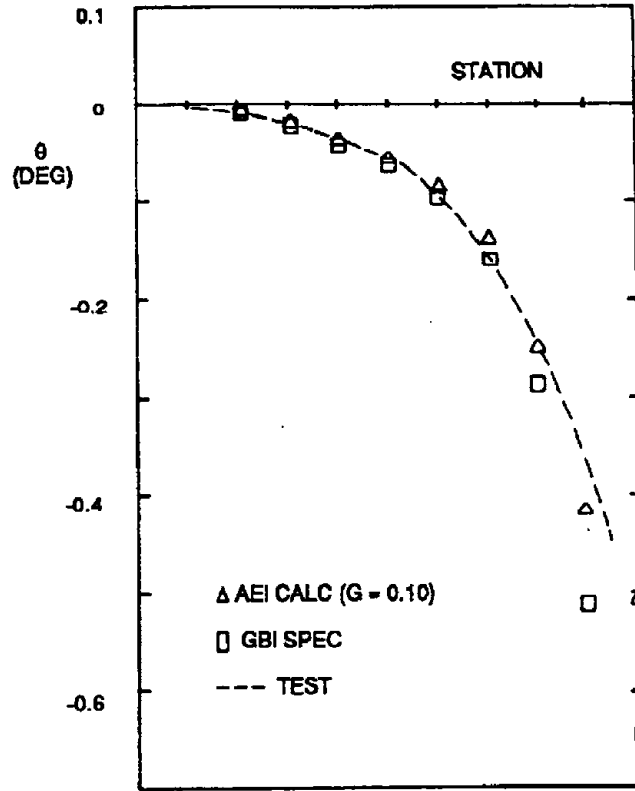


FIG. 5.15 ESI BLADE STATIC TWIST: PREDICTIONS VS. TEST

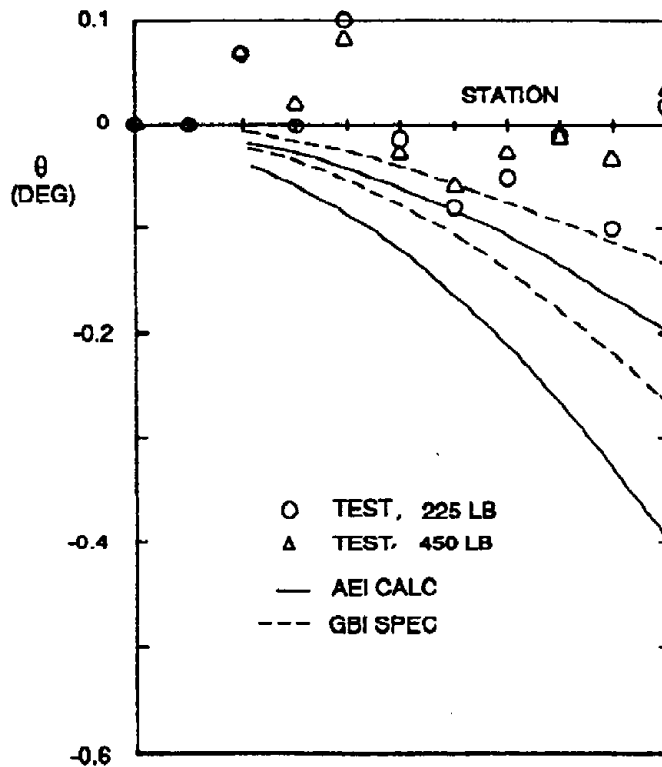


FIG. 5.16 ESI BLADE TWIST: PREDICTIONS VS. TEST (FLEXURE LOAD CASES)

A further check on twist can be made, to compare the twist of the dead load cases and "bracket" the solution. These are shown overlaid with the test results in Figure 5.16; clearly, they do not match the test data. Since there is high confidence in the above range (0.1 - 0.15) of GJ, the error must be in the predictions of shear center (both AEI and GBI). These predictions are for an aft shear center (see Fig 5.10) which produces a section moment when loaded at the quarterchord. The error in the predicted twist in Figure 5.16 is attributable to the difference in shear center prediction in Figure 5.10, as described here:

The test data clearly show no effective elastic twist of the ESI blade when loaded at the quarterchord (See Fig 5.16). This is the requirement for the definition of shear center. Therefore, it must be concluded that the actual shear center of the blade is the quarterchord and both predictions are in slight error. Thus for the blade aeroelastic model, an empirical value for shear center must be used: when this value is input for the two high load cases the resulting twist is zero, as in the test data. This illustrates the necessity of doing both static tests; flexural and torsional, otherwise this important result would have been overlooked.

5.3.5: VERIFICATION BY FREQUENCY TEST

The total blade weight was measured and predicted to be 260 lb., which lends credence to the calculated mass distributions. The first three flap bending modes and the first torsion mode are shown in Figure 5.17. The frequency results are shown in Table 5.4 for the test, the GBI specifications, and the AEI calculation.

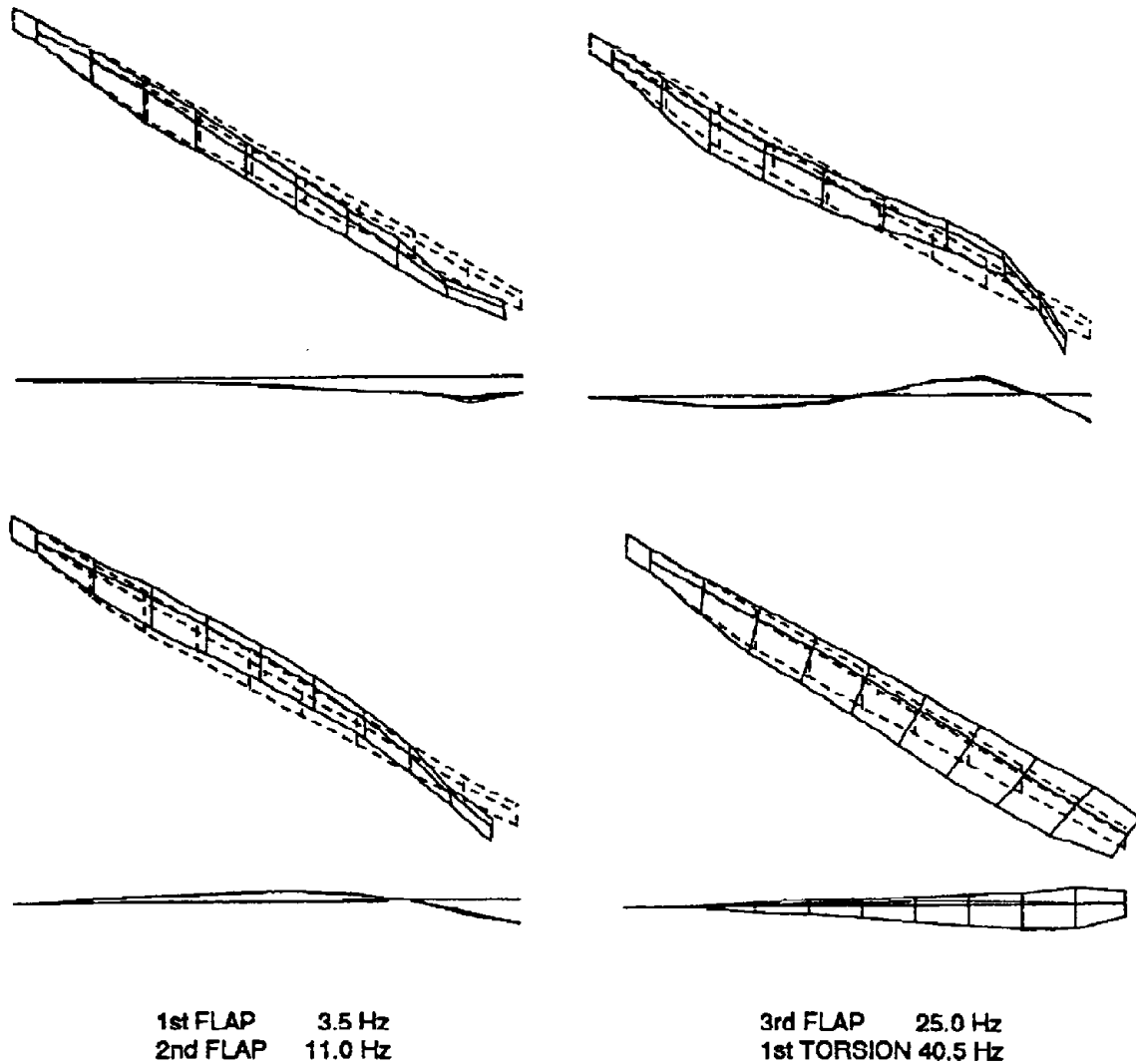


FIG. 5.17 ESI BLADE FREQUENCY MODES

TABLE 5.4 ESI BLADE FREQUENCIES OF VIBRATION: PREDICTION VS. TEST

Mode	Test Frequency	AEI Prediction	GBI Prediction
1st flap	3.5 Hz	3.2-4.0 Hz	3.4 Hz
2nd flap	11.0	10.2-12.4	10.2
3rd flap	25.0	22.8- 27.5	23.0
1st lag	8.0	9.2	8.2
2nd lag	33.0	35.0	31.4
3rd lag	80.0	88.0	81.0
1st torsion	40.5	35.1 (40.5)	32.6
2nd torsion	68.0	60.8 (70.0)	54.0

NOTES: (1) The range of the AEI predictions indicates the limits from the true flap and lag directions to the principal axis direction.

(2) Torsional frequencies in parentheses indicate G modification (0.100 to 0.133 *10⁶).

The range in the AEI predictions derives from taking the EI values in the true lag and flap directions or in the principal axis direction which yield a softer "flapwise" frequency. The blade is not highly twisted (8 deg) so the principal axes are definable in blade terms. Also from Table 5.4, since the flap test frequencies are higher than the GBI frequencies, it can be concluded that the GBI values are conservative. For lead-lag the same is probably true since non-ideal root fixity usually depresses the lag frequency test data. This effect is clear with the AEI predictions, but not the GBI values. This also illustrates how sensitive the frequencies are to EI values; Figure 5.7 would not lead one to suspect such a large variation in frequency.

The torsion values clearly show the predicted GJ's to be too low. Recall that these GJ's were based on the static twist results (G = 0.10) and very low values of twist (<18 minutes of arc). To bring the calculated frequencies up to the test values (shown in parentheses), the value of G was increased from 0.100 to 0.133. This is understandable since the original prediction was 0.150. The resulting final GJ is shown in Figure 5.18. This value of GJ matches the static data less well (but still within the experimental accuracy of ±0.1°), but the dynamic data are matched exactly; this is more believable since fixity problems are eliminated.

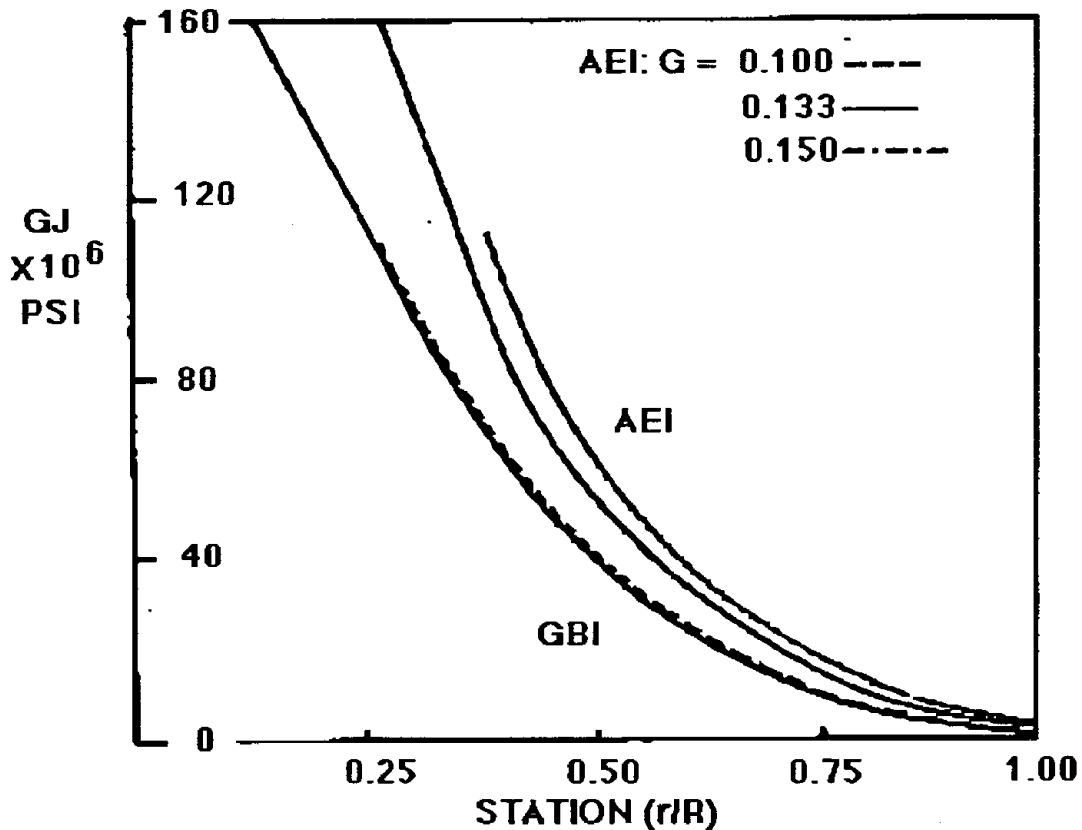


FIG. 5.18 ESI BLADE: FINAL GJ VALUES

5.3.6: FINAL AEROELASTIC MODEL

The final values of the ESI aeroelastic blade model are shown in Table 5.5. The blade twist was determined by geometric calculation and verified by measurement.

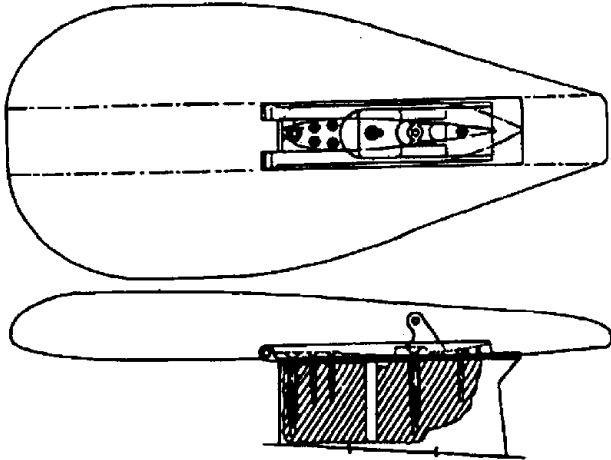
TABLE 5.5: ESI BLADE: FINAL AEROELASTIC MODEL

STA	r/R	(in.) r	(deg) β_0	(slugs/in.) m_0	10^6 psi EI flap	10^6 psi EI lag	10^6 psi GJ	(in.) Y_{cg}	(in.) YEA	I_{cg}
Root	11	0	0	.07823	941.0	4259	349.3	7.63	4.20	67.76
	10	.1	32.4	2.13	.07823	941.0	4259	349.3	7.63	67.76
	9	.2	64.8	2.13	.04012	607.4	3736	166.8	9.67	56.6
	8	.3	97.2	2.13	.04084	520.4	5204	152.3	11.48	74.6
	7	.4	129.6	2.45	.03021	303.7	3327	86.8	10.70	47.3
	6	.5	162.0	2.87	.02417	185.0	2233	53.3	9.85	31.5
	5	.6	194.4	3.38	.02175	116.8	1673	35.9	9.06	23.3
	4	.7	226.8	4.04	.01679	81.0	1036	23.4	8.09	14.6
	3	.8	259.2	4.94	.01291	30.4	614	9.4	7.21	8.4
	2	.9	291.7	6.12	.01115	13.5	416	4.9	6.45	5.60
Tip	1	1.0	324.1	8.18	.00956	4.9	252	2.1	5.54	3.34

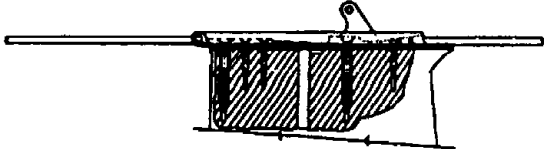
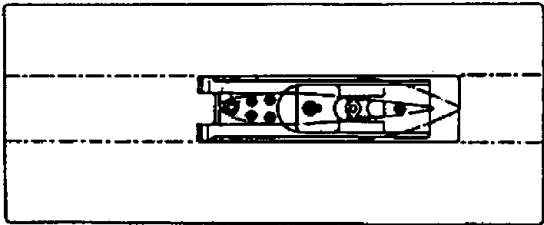
5.3.7: ESI TIP FLAPS

The operational ESI blade had tip flaps that were attached to a mount (see Fig. 5.19) and had the function of centrifugal overspeed brake. There were two versions of the tips: the "regular tip" and the "whisper tip," the latter incorporating a fiberglass-foam aerodynamic fairing. From tests and inspections of the two tips, the weights and Cg's of the assemblies (e.g., brackets, bolts, swing arms, springs, and bash

plates) were analyzed to correctly account for the added weight and inertia. Given here are free-body diagrams and the resulting values of the equipollent system, which gives the correct mass locations.



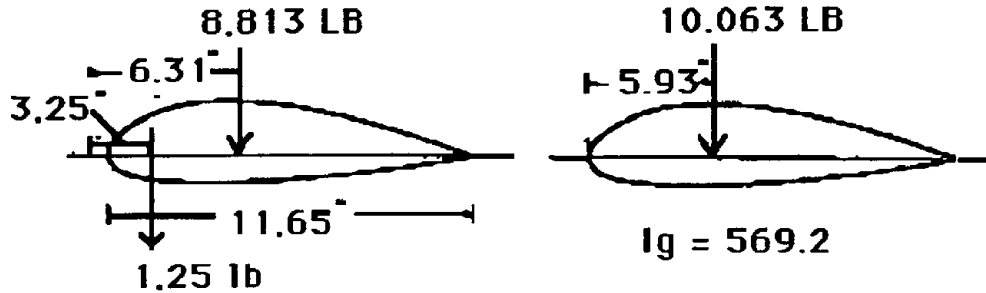
WHISPERTIP



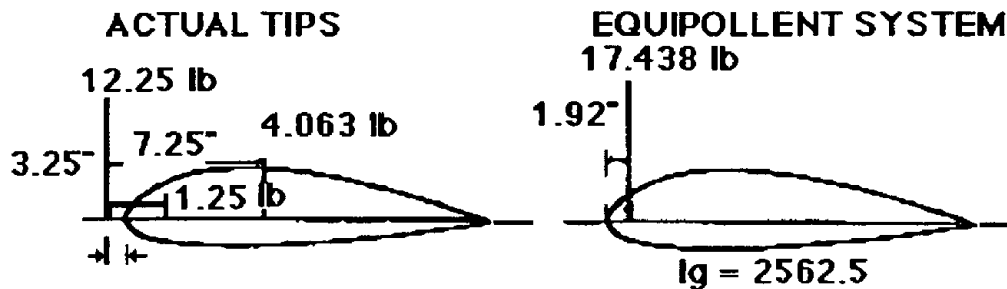
REGULAR TIP

FIG. 5.19 ESI TIP FLAPS

Regular Tip:



Whisper Tip:



These are treated as tip ballast in the calculations of section properties in the elastic twist computer runs. These values were verified with some (unpublished) test data taken by ESI on the blades in the field and made available for this report (Ref. 26); Table 5.6 shows how the added tip weight decreases the flap frequency, and the added tip inertia greatly decreases the torsional frequency.

TABLE 5.6: ESI VIBRATION FREQUENCIES WITH TIPS (HZ)

Mode	No Tip (Prediction)	Regular Tip (Prediction)	Whisper Tip Prediction	Whisper Tip Test
1st flap	3.5	2.69	2.41	2.40
1st torsion	40.5	23.60	12.51	12.50

5.4: COMPLEX COMPOSITE FRP EXAMPLE: CARTER 300 BLADE

The 10-meter FRP Carter blade was a good candidate for this study for the following reasons:

- The relatively high tip speed, blade radius, and chord give high Reynolds number and power output.
- The good quality control of the laminate molding and arrangement by the manufacturer yielded consistent material properties.
- The relatively low bending stiffness led to measurable structural twist coupling (due to elastic axis offset from loading axis).
- The large ballast mass added to the outboard blade caused substantial elastic twist.
- The highly twisted, tapered planform is indicative of high-performance wind turbine rotors.
- The LS(1) series airfoil was used; it is indicative of new wind turbine airfoil families, and possesses large aerodynamic pitching moment.
- The root attachment is indicative of underslung, non-pitch/flap coupled teetering hub.

5.4.1: CARTER BLADE DESCRIPTION

The Carter blade was manufactured by Carter Wind Systems (Burkburnett, Tex.); it has a composite shell with a reinforced leading edge (D-spar) plus webs (see Figs. 1.1 and 5.20). It has a radius of over 10 m (33.6 ft) and in a teetering rotor (2-bladed) configuration powers the Carter 300 kW wind turbine.

If the entire blade had a structure as in Figure 5.20 the structural analysis would proceed in the usual manner as for the above blades. However, the Carter blade incorporates an innovative internal pitching mechanism ("pitch-up snubber") which provides overspeed protection for the wind turbine by causing a pitch change (of roughly 14 deg nose up) past airfoil stall, by the action of an electrically released toggle (see Figs. 3.5 and 5.21). The release of the toggle allows a preloaded overcenter spring in the root to produce a large mechanical control moment between the (moving) blade shell and the (stationary) root I-beam. The I-beam is a high quality (high glass/resin ratio) unidirectional beam comprising the principal loading path in the inner 35% of the blade as shown. By its I-beam section, it has relatively stiff bending properties and relatively soft torsional rigidity; thus it acts as a torsional spring against the control actuator. In operation, once the blade is pitched up, it is gradually returned to the running pitch position by a spring/damper actuator. This mechanism, unmodified, looks like a one-way spring damper to torsional motion of the blade, thereby drastically altering the fundamental torsional frequency of the blade and the torsional static deflections (via the relatively soft spring/damper). Consequently, for the testing this control arrangement was replaced by a rigid turnbuckle, so the blade could be set at the correct (run) pitch angle and the true blade structural statics and dynamics could be measured.

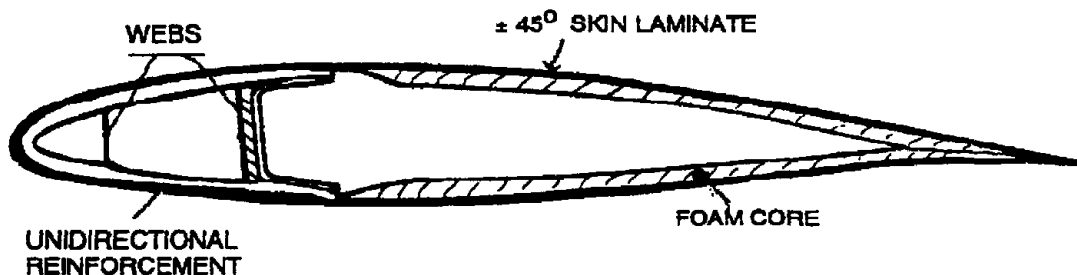


FIG. 5.20 CARTER BLADE INTERNAL STRUCTURE

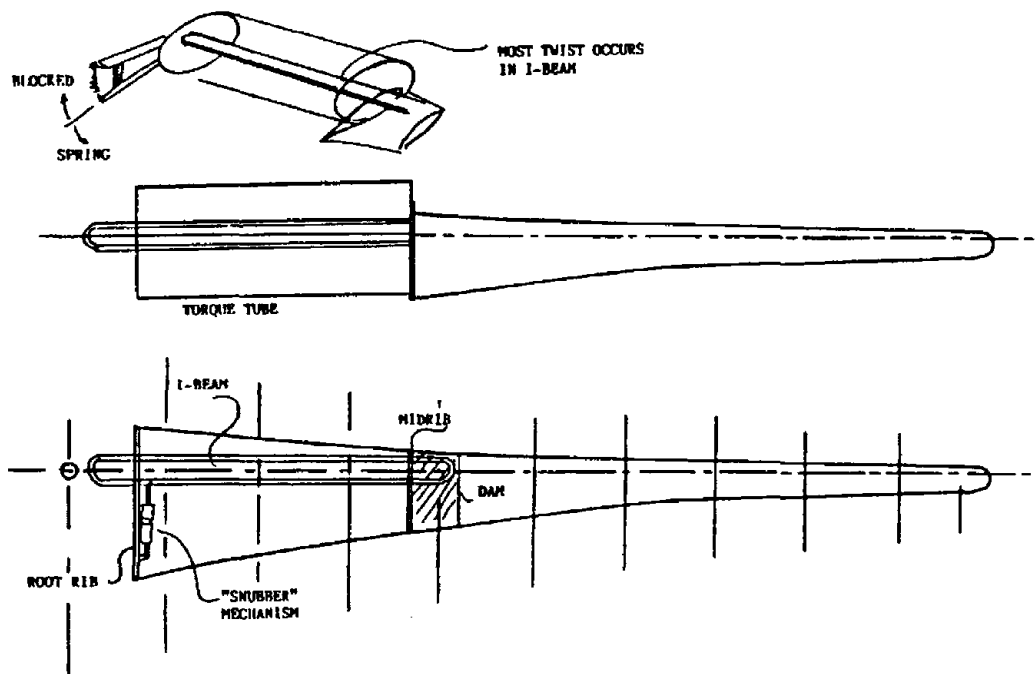


FIG. 5.21 CARTER BLADE FREE BODY DIAGRAM

The free body diagram in torsion of this blade is also shown in Figure 5.21. The block/spring was replaced by the turnbuckle during testing. The inboard blade consists of a shell or "torque-tube" fixed around the I-beam, which is attached to the hub and a midrib at STA 0.37 radius, and "potted" there in an epoxy resin compound. When the blade is pitched, the twist occurs primarily in the (torsionally soft) I-beam and very little in the torque tube. It was the goal of this project to establish the torsional and flexural properties of both working in concert. This mechanical arrangement, of course, greatly complicated the testing and analysis, but could be adequately handled in the computer programs as seen below.

The blade was further complicated by the addition of 138.8 lb of lead weight ballast to the leading edge for mass balancing on the outboard third of the blade. This added weight (the basic unballasted blade with mechanisms was only 700 lb) caused a large static gravity deflection of about 30" when cantilevered, and also created a large section mass moment of inertia outboard. The latter doesn't affect the static elastic twist but does drastically modify the torsional frequencies and modes, as well as the flexural frequencies.

Further structural coupling occurs due to the geometry of the blade. Figure 1.4 illustrates in three views the control and blade axes. The blade shell is geometrically constructed (e.g., twisted and tapered) about the blade axis which is at about 40% chord; but the blade is fixed (root) and acts about the control axis which is at the quarterchord, since the I-beam is placed there. The mass axis (not shown) is also roughly at or ahead of the quarterchord, and is definitely not a straight line. The intermodal coupling is best illustrated in the frequency mode shapes in the modal survey testing part (Sec 5.4.10).

5.4.2: ASCERTAIN BLADE GEOMETRY

The blade data furnished by Carter Wind Systems were meager in comparison to that furnished by GBI, in view of the information needed for this complex blade. The first task was to ascertain the actual blade twist, taper, and arrangement geometry.

The blade radius and chord lengths were found accurately by inspection, as was the thickness distribution. Finding the twist was complicated by the gravity component due to the ballast weight. This effect was eliminated by measuring the dead load twist for the blade in normal position and inverted position. The six individual dead load cases are given in Figure 5.22, and the resulting twist distribution in Figure 5.23; the Carter specification is also shown. Clearly the STA 0.8 template was about 1° off (which could have been the operator error referred to in 3.9.1); but otherwise the twist measurements were very repeatable. Figure 5.24 is an error scatter plot and shows the variation in cases to be subject to only 0.1% experimental error.

The final blade geometry is given in Table 5.7 compared to the specifications. The actual blade length was only 0.75" longer than the spec. (376.25 vs. 375.50 in.). As can be seen, the values agree extremely well, verifying the tooling and assembly method of the Carter 300 blade. [Note: blade sectioning is not actually required for this task, but is a great help; it is required for later verifications.]

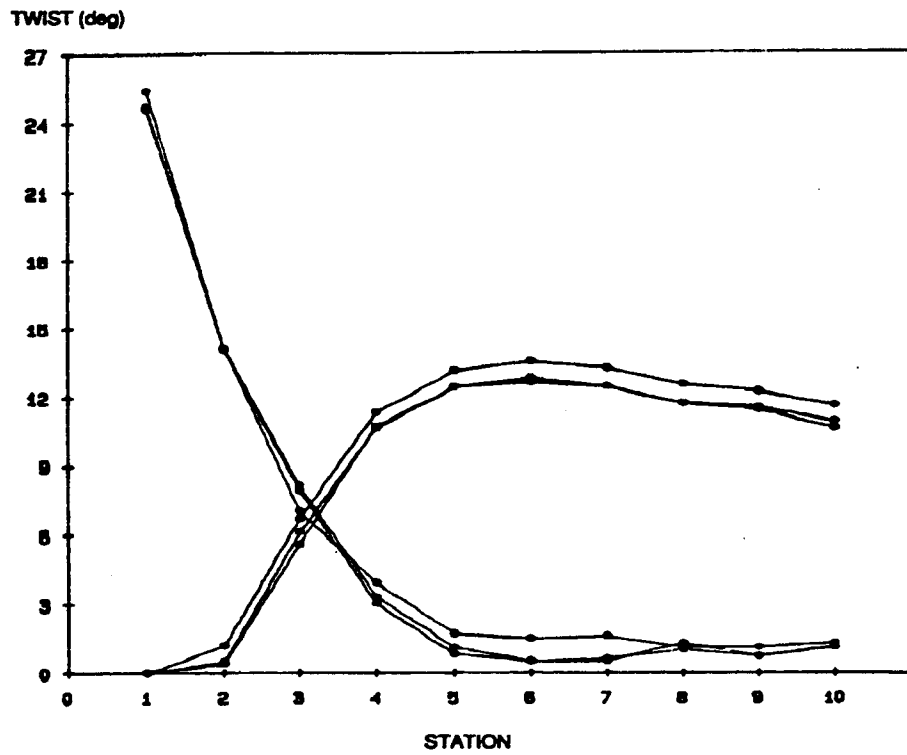


FIG. 5.22 CARTER DEAD LOAD TWIST

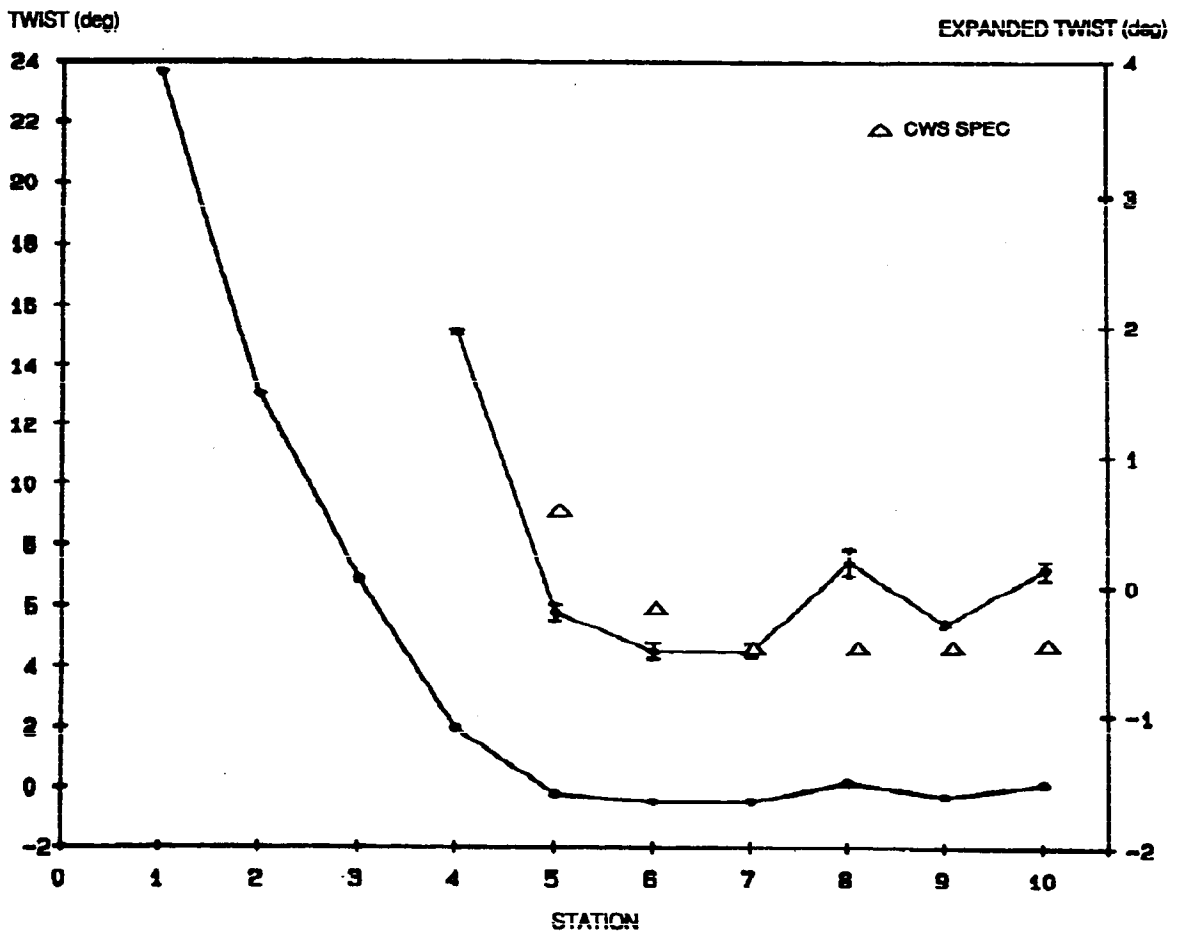


FIG. 5.23 CARTER BLADE: TWIST FROM MEASUREMENT

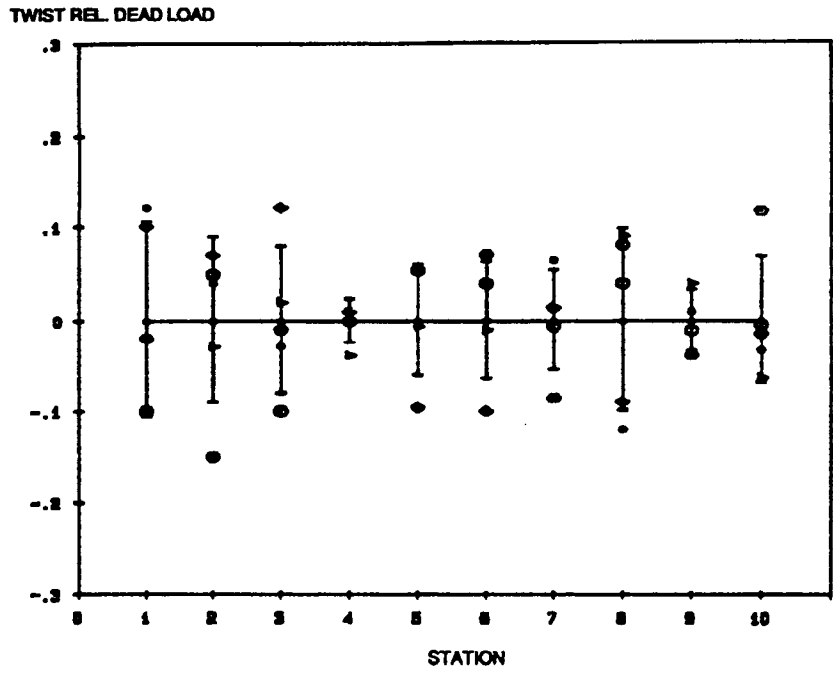


FIG. 5.24 CARTER BLADE: ERROR SCATTER PLOT, TWIST DATA

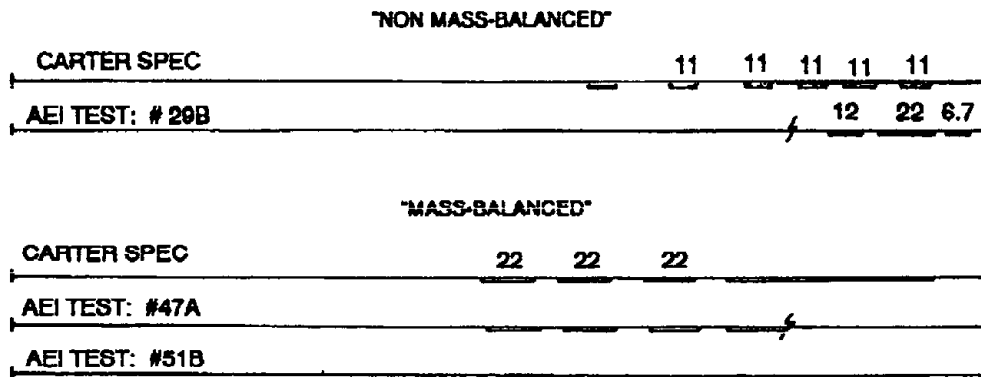
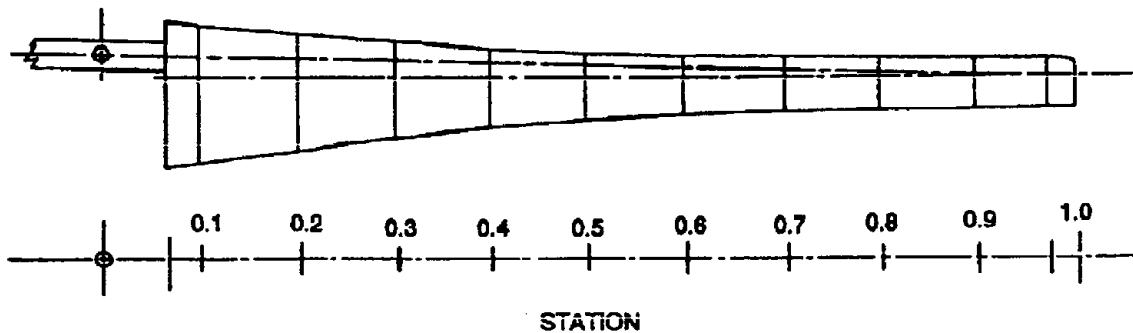
TABLE 5.7: CARTER BLADE GEOMETRY, MEASURED VS. CARTER SPECIFICATIONS

	STA	r/R	r (in.)	chord (in.)		thickness (in.)		twist (°)	
				meas.	spec.	meas.	spec.	meas.	spec.
root	11	0	0	- -	- -	- -	- -	- -	- -
	10	.1	40.28	66.75	66.80	14.06	14.25	26.1	27.09
	9	.2	80.55	52.81	52.80	11.06	11.40	15.5	15.96
	8	.3	120.83	42.50	42.88	8.19	8.56	9.4	9.7
	7	.4	161.10	34.56	34.99	5.25	5.70	4.5	5.67
	6	.5	201.38	28.97	29.50	3.81	4.00	2.3	2.93
	5	.6	241.65	25.56	26.00	3.28	3.39	2.0	2.19
	4	.7	281.93	23.44	23.73	2.97	3.09	2.0	2.01
	3	.8	322.20	22.19	22.45	2.78	2.92	2.0	2.00
	2	.9	362.48	21.41	21.65	- - - -	2.82	2.2	2.00
tip	1	1.0	402.75	21.00	21.00	- - - -	2.73	2.6	2.00

5.4.3: ASCERTAIN BALLAST AND WEIGHT DISTRIBUTION

Ballast weight distribution information was furnished by Carter, as requested (see App. 10). Still, all the weights and ballast locations had to be accurately measured and verified, since mass distribution in the blade is crucial to its properties.

The AEI sectioned blade actually consisted of portions of two separate blades (Carter #29B and #47A), identical in makeup except for the amount and locations of the lead ballast. There were two types of CARTER 300 blade: "mass-balanced" and "non-mass-balanced." The AEI flexural test blade (Carter #51B), was "mass-balanced," and represented the majority of blades in service, so that configuration was chosen for the analysis. The ballast weights consist of molded lead inserts for the leading edge, with mass of 0.8636 lb/in. and of various lengths. The nondestructive test necessary for finding the ballast locations in the test blade successfully used a standard metal detector. Results are given in Figure 5.25. The "mass-balanced" blade is very close to the Carter specification, but the "non mass-balanced" series does not match the specification, and has further additional weight at the tip. The chordwise ballast cg location was determined by measurement. The weight distribution summary is given in Table 5.8 for the Carter "mass balanced" 300 kW blade.



NOTES: (1) Ballast weight is 0.836 lb/in, (2) ballast length indicated in inches.

FIG. 5.25 CARTER BLADE BALLAST SUMMARY

TABLE 5.8: CARTER BLADE WEIGHTS SUMMARY

STA	Panel	(lb)			(in.) X _{cg}	(lb) Ballast ³	ballast position	
		Spar	Skin	Total			X _g	Y _g
0 - 1	1	-	-	281.251	-	0		
1 - 2	2	33.75	58.0	91.75	-	0		
2 - 3	3	28.0	47.75	75.75	-	0		
3 - 4	4	-	-	75.25	10.87	0		
4 - 5	5	-	-	66.75	8.43	19.0	1.63	0.25
5 - 6	6	-	-	55.75	6.54	19.0	1.63	0.25
6 - 7	7	-	-	51.0	5.39	19.0	1.63	0.25
7 - 8	8	-	-	45.38	4.80	38.0	1.63	0.25
8 - 9	9	-	-	49.772	4.092	38.0	1.63	0.25
9 - 10	10	-	-	42.80	4.57	5.8	1.63	0.25
				835.454	138.8			

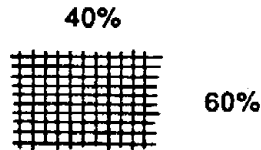
NOTES:

1. includes root rib and mechanisms
2. adjusted to "mass balanced blade":
$$\frac{(27.5 \cdot 5.98) + (22.27 \cdot 1.75)}{47.77} = 4.09 \text{ in.}$$
3. the ballast is centered at the station (i.e., 1/2 contribution for each adjacent panel) and slightly adjusted spanwise for convenience.
4. Carter Wind Systems estimate 700 lb.

5.4.4: DERIVE CARTER DESIGNED COMPOSITE VALUES

Carter Wind Systems did not perform a stiffness analysis for this blade design; they relied on allowable ultimate strength values for a specified laminate and load, and performed spot checks on the blade, using prescribed loads (unknown cases) and the trapezoidal approximation to section moment of inertia, (Ref. 27). The AEI study required a stiffness analysis and section properties; therefore, a blade section analysis had to be done based on the manufacturer's specified laminate design. This section covers the derivation of the code input.

The Carter laminate, even though the blade is quite complex, consists of only four types of glass reinforcement; derivations of their expected as-built properties follows.



1) Style 60/40, 7.5 oz./yd² cloth:

This woven cloth reinforcement has 40% of the glass fibers on the 0° direction (y) and 60% in the 90°(x). Assuming a layer (ply) thickness of 0.015" (CARTER estimate), a 1-yard square has a volume of (36)*(36)*(0.015) = 19.44 in.³. The glass in the X-direction amounts to 60% of 7.5 oz. = 4.5 oz. Taking the density of E-glass of 0.09838 lb/in.³ (or 1.57408 oz/in.³) and the modulus of 10*10⁶psi gives 2.8588 in.³ of E-glass in the X-direction (90° orientation) for our "standard" square yard. The E-modulus of the laminate is easily found by the volume percentage of glass in that direction:

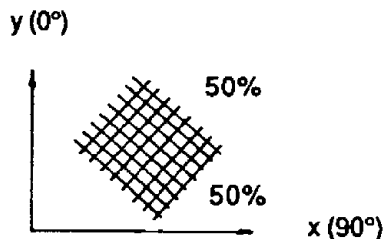
	<u>glass</u>	<u>per yd²</u>	<u>E-modulus</u>
90° or x direction:	4.5 oz =	2.8588 in. ³	$E_1 = 10 \cdot 10^6 [2.8588/19.44] = 1.47 \cdot 10^6$
0° or y direction:	3.0 oz =	1.9059 in. ³	$E_2 = 10 \cdot 10^6 [1.9059/19.44] = 0.98 \cdot 10^6$

The glass content of this ply is:

$$\text{glass content by volume} = \frac{4.7647 \text{ in.}^3 \text{ (glass)}}{19.49 \text{ in.}^3 \text{ (total)}} = .2451\%$$

From the chart (Table 4.1) this gives a ply weight density of 0.05715 lb/in.³. Lastly the G-modulus is estimated from the engineering chart (Fig. 5.2) and rules of thumb:

$$G \cong 1.47 \cdot 10^6 \cdot 0.6 \text{ (conversion factor for orientation)} = 0.882 \cdot 10^6 \text{ psi}$$



2) Style ± 45°, 10 oz. cloth:

This bonded ply fiber reinforcement ("Knytex") has balanced orthogonal properties oriented ±45°. Carter assumed a layer thickness of 0.020" for this reinforcement. The calculation proceeds as before, except the orientation factor decreases the E-modulus by 50% and increases the G-modulus by 2:

$$\text{Volume} = (36)(36)(.020) = 25.92 \text{ in.}^3$$

	<u>glass</u>	<u>per yard²</u>	<u>E-modulus</u>	
90° or x direction:	5 oz.	= 3.1765 in. ³	$E_{1,2} = \frac{3.1765}{25.92}$	$(10 \cdot 10^6)(.5) = 0.613 \cdot 10^6$
0° or y direction:	5 oz.	= 3.1765 in. ³	$E_{1,2} = \frac{3.1765}{25.92}$	$(10 \cdot 10^6)(.5) = 0.613 \cdot 10^6$

where .5 is the orientation correction factor (± 45 deg)

$$\text{glass content by volume} = \frac{6.353}{25.92} = .2451\%$$

$$\text{weight density (from chart)} = .05715 \text{ lb / in.}^3$$

$$G_{45^\circ} \equiv (0.613 \cdot 10^6)(1/0.5)(0.6) = 0.7356 \cdot 10^6 \text{ psi}$$

$$G_{0^\circ} \equiv (.7356 \cdot 10^6)(1/0.5) = 1.4712 \cdot 10^6 \text{ psi}$$

where 0.6 and 0.5 are the G-correction factor and the orientation factor (Fig. 5.2), respectively.

3) Unidirectional Tape, 13 oz. (Knytex A-130):

This ply is the principal spanwise reinforcement in the blade; this is a convenient way to mold unidirectional properties in the hand lay-up process. This reinforcement weighs 13 oz./square yard, and Carter assumed a ply thickness of 0.015 in. The properties are calculated easily in the above manner.

4) Style woven roving, 24 oz:

Woven roving consists of E-glass rovings (yarns) in an open weave pattern, with balanced orthogonal fibers, e.g., 50/50. Carter estimated a ply thickness of 0.026 in. for this reinforcement.

The summary predicted properties for the as-built plies are given in Table 5.9.

TABLE 5.9: CARTER REINFORCEMENTS, AS-BUILT SUMMARY

STYLE	(in.) t	(*10 ⁶ psi) E ₀	(*10 ⁶ psi) G	(lb/in. ³) ρ	glass %
60/40	.015	1.47	0.882	.05715	24.5
±45	.020	0.613	1.471	.05715	24.5
uni	.015	4.25	0.400	.06677	42.5
WR	.026	2.263	1.358	.06814	45.2

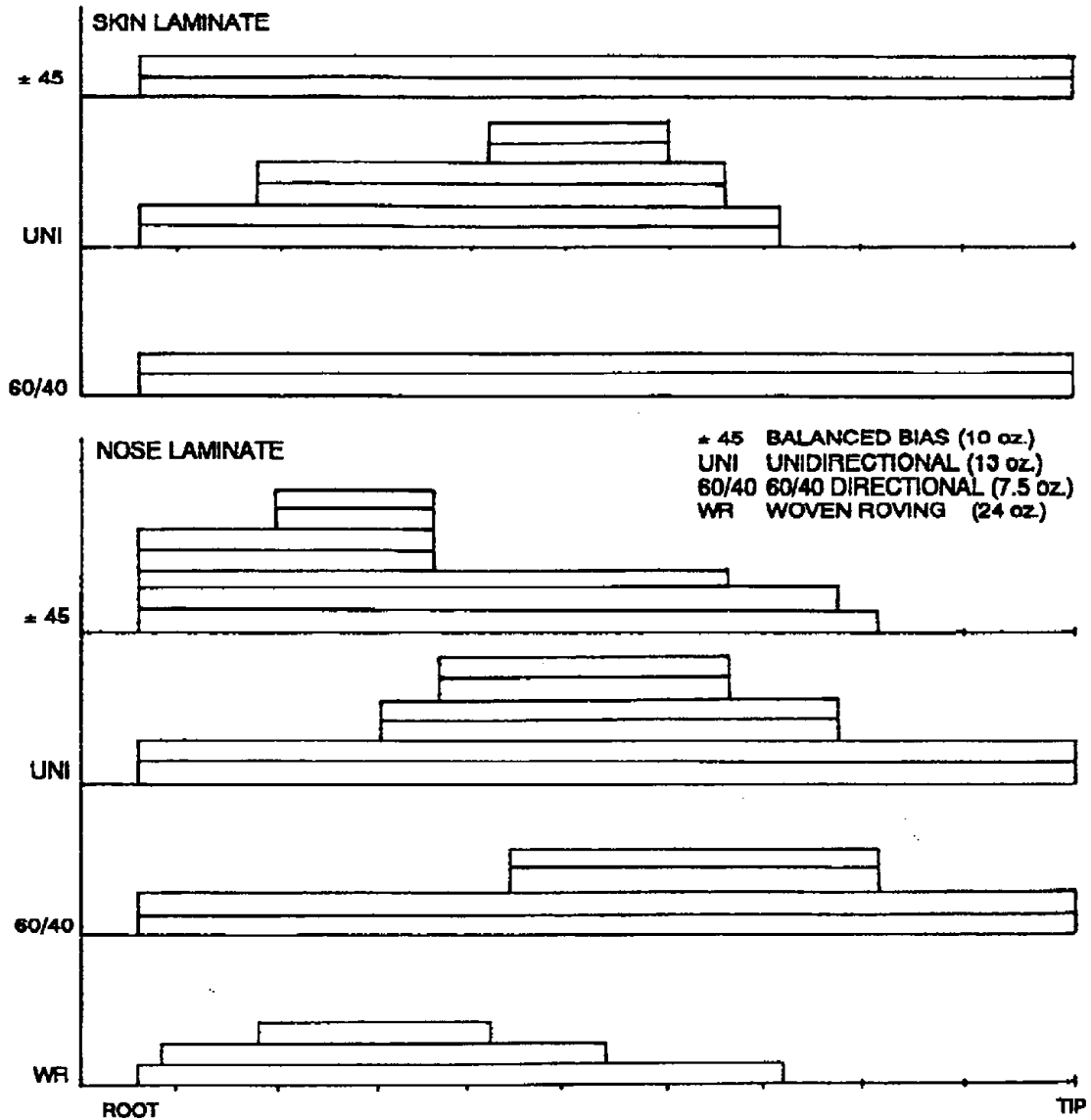


FIG. 5.26 CARTER BLADE LAMINATE SCHEDULE

Now given the laminate schedule for the Carter blade (Fig. 5.26), an estimate can be made for the properties, thicknesses, and other inputs to the section calculation code. For this blade the PVC foam coring (in the aft upper and lower skin panels and the spar web) was ignored, and the laminates at each section were assembled ply by ply and then reduced to skin, D-spar, and web contributions.

Clearly, as the laminate makeup changes, so also do the aggregate properties E , G and ρ . Therefore, an average value for the section properties code had to be chosen. One refinement to the section properties code would allow separate engineering values for each station. An example of this variation is shown in Figure 5.27 which plots the weight density distribution for the spar portion. Table 5.10 gives the skin and spar input values for the Carter design schedule.

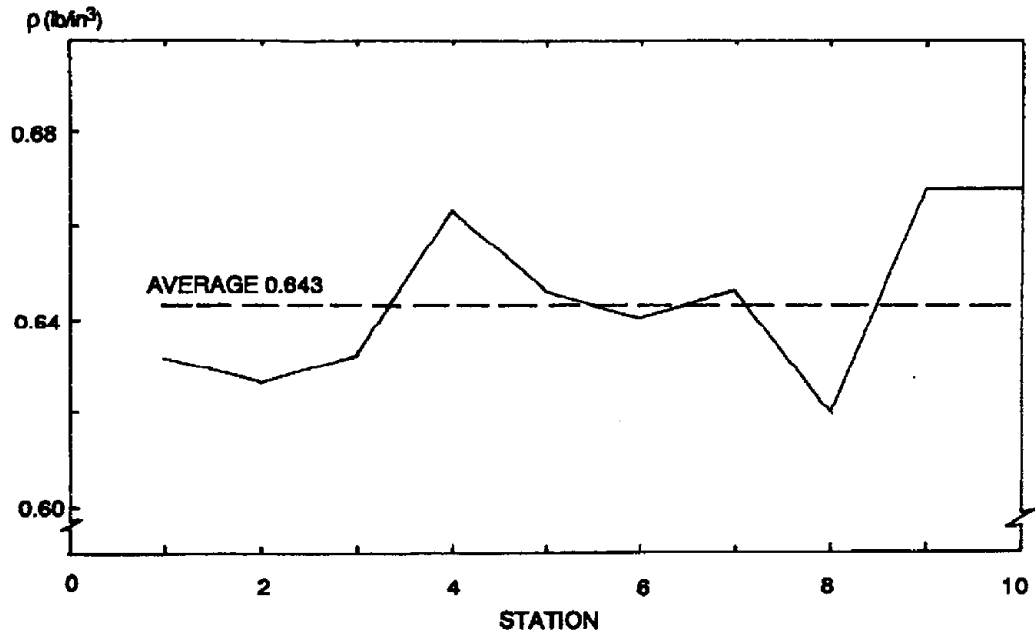


FIG. 5.27 CARTER SPANWISE WEIGHT DENSITY (SPAR PORTION)

The unidirectional reinforcement in the skin and the nose webs were included separately. These values for each station were estimated as tilted rectangles ($I = bh^3/12$) and summed using the parallel axis theorem. This resulted in the web input given in Table 5.10. Another refinement to the code would integrate these multi-"webs" as it does for the skin and spar rather than assume rectangles.

TABLE 5.10: CARTER BLADE LAMINATE DESIGN SUMMARY

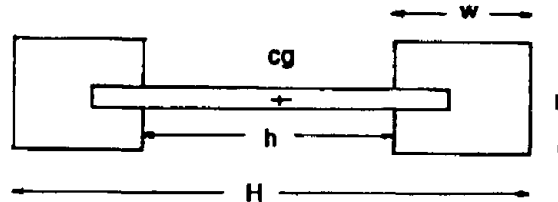
STA	r/R	SKIN		SPAR			WEBS						
		t (in.)	t (in.)	ρ	E	G	A _T	X _g	Y _g	I _{xy}	I _{yg}		
Root	11	0	.070	---	---	---	---	---	---	---	---	---	---
	10	.1	.070	.142	.0632	1.986	1.203	3.916	31.41	2.72	162	1255	
	9	.2	.070	.208	.0627	1.756	1.274	5.312	28.19	2.04	151	869	
	8	.3	.070	.238	.0632	2.071	1.164	4.237	23.22	1.54	69.8	421.6	
	7	.4	.070	.188	.0663	3.039	0.911	3.346	19.40	1.01	24.8	202.1	
	6	.5	.070	.192	.0646	2.899	.846	3.986	16.98	.70	15.51	53.6	
	5	.6	.070	.166	.0641	3.000	.766	3.510	15.07	.58	9.59	104.1	
	4	.7	.070	.116	.0646	3.086	.739	1.277	12.72	.57	2.68	33.9	
	3	.8	.070	.060	.0620	2.860	.641	.213	6.00	.73	.125	0	
	2	.9	.070	.030	.0668	4.250	.400	.210	5.38	.71	.120	0	
Tip	1	1.0	.070	.030	.0668	4.250	.400	.202	5.00	.68	.108	0	

	SKIN	SPAR	WEBS
ρ_e (lb/in. ³)	.0572	.0644	.065
\bar{E} (*106 psi)	.980	3.0000	3.500
\bar{G} (*106 psi)	1.219	0.8000	.500

5.4.5: DERIVE CONTRIBUTION OF I-BEAM

Lastly, the I-beam must be accounted for in the section properties code. The Carter I-beam cross section is shown below in Table 5.11; the beam portion is highly compacted filament-wound rovings, so it should have a very high glass ratio (in fact, the highest readily achievable in any glass-resin laminate) of 90%. The test data and geometric data must be used to find the area, weight, and moments of inertia for the I-beam.

TABLE 5.11: CARTER I-BEAM SECTION DATA FROM TEST



STA	(r/R)	(1/4 chord)		(in.)			
		X _g	Y _g	H	h	b	w
1	.1	16.69	0	(data lost)			1.562
2	.2	13.20	0	13.88	10.56	2.38	1.562
3	.3	10.63	0	11.76	8.63	2.38	1.562
4	.4	8.64	0	9.76	6.63	1.97	1.562

From Carter data, the reinforcement of the unidirectional beams consists of 135 wraps of 40-strand rovings of E-glass. Therefore, the total number of ends are $135 \times 40 = 5400$; so in each beam there are 5400 rovings. The measured beam area is $1.562 \times 2.380 = 3.718 \text{ in.}^2$. Taking the standard weight of roving as 450 yards/lb allows a calculation of glass percentage,

For 450 yards/lb roving the yarn size is $= 0.62765 \times 10^3 \text{ in.}^2/\text{roving}$ (from $\rho_{E\text{-glass}} = 0.09838 \text{ lb/in.}^3$)
 then: A (5400 ends) $= (5400) (.62765 \times 10^3) = 3.389 \text{ in.}^2$
 so: glass content $= 3.389/3.718 = 91.15\%$

This verifies the original assumption of 90% glass in the I-beam. The engineering values for this laminate are, then, from Figure 4.1:

$$\begin{aligned} E_{\text{uni}} &= 9.115 \times 10^6 \text{ psi} \\ G_{\text{uni}} &= 0.200 \times 10^6 \text{ psi} \\ \rho_{\text{uni}} &= 0.09351 \text{ lb/in.}^3 \end{aligned}$$

Adding the small shear web contribution, consisting of mainly $\pm 45^\circ$ plies, changes the aggregate values of the I-beam to

$$\begin{aligned} E_{\text{I-beam}} &= 9.0 \times 10^6 \text{ psi} \\ G_{\text{I-beam}} &= 0.420 \times 10^6 \text{ psi} \\ \rho_{\text{I-beam}} &= 0.0935 \text{ lb/in.}^3 \end{aligned}$$

At this point it is a good idea to make a quick check on this value from a weight comparison.

$$\begin{aligned} \text{I-beam length} &= 194 \text{ in.} \\ \text{weight of uni spars} &= 2 \times 194 \times .09351 = 135 \text{ lb.} \\ \text{weight of } \pm 45^\circ \text{ web} &= 194 \times 6 \times .260 \times .05715 = \underline{17.3 \text{ lb.}} \\ \text{total weight} &= 152.3 \text{ lb.} \end{aligned}$$

This compares favorably with the measured value of 165 lb, allowing for gussets and fill at the root end.

Next the moments of inertia must be found, since the I-beam principal axis is not placed on the chord line because the blade's twisted. The chordline values of I are simply determined from the principal axis values of I_{max} and I_{min} , using a standard Mohr's circle solution (Ref. 12). This gives the moments of inertia (Table 5.12) of the I-beam at each station, in the section (X-Y or airfoil) coordinate system.

TABLE 5.12: CARTER I-BEAM MOMENTS OF INERTIA

r/R	β_0	A (in. ²)	(X _g , Y _g) (in.)	I ₀	I _{min}	I _x	I _y	I _{xy}
					(in. ⁴)			
0.1	25°	8.75	16.69,0	313	4.17	58	258	-118
0.2	14°	8.00	13.20,0	220	4.17	18	205	-50
0.3	7.5°	7.44	10.63,0	195	3.51	5	190	-25
0.4	3.7°	6.15	8.64,0	105	2.00	2	104	-7

5.4.6: AEI MEASUREMENTS FOR COMPOSITE SECTION PROPERTIES

Along with the theoretical design input files developed above from composite engineering analysis, an additional input file was constructed on the basis of the blade sectioning measurements. Practically speaking, the sectioning allows an independent check of the above predictions (for example, cg, laminate thickness, weights) but also will quickly indicate where the actual blade differed from the design (e.g., more plies than specified, presence of defects). The section inspection values are listed in Table 5.13 along with the specified design values.

TABLE 5.13: CARTER BLADE: SECTION MEASUREMENTS VS. CARTER SPECIFICATIONS (INCHES)

r/R	h _w (airfoil thickness)		d _w (nose to web)		t _{le}		t _{web}		t _{skin}	
	meas.	spec.	meas.	spec.	meas.	spec.	meas.	spec.	meas.	spec.
0	-	-	-	-	-	-	-	-	-	-
.1	14.06	14.30	5.25	9.50	-	.212	-	.040	-	.100
.2	11.06	11.30	3.74	9.50	-	.278	-	.040	-	.130
.3	8.19	8.56	1.88	9.00	-	.308	-	.040	-	.130
.4	5.25	5.70	-	8.50	.394	.258	-	.040	.167	.130
.5	3.81	4.05	6.81	8.25	.421	.262	.141	.040	.189	.160
.6	3.28	3.39	6.69	7.60	.385	.236	.084	.040	.155	.160
.7	2.97	3.09	5.63	7.20	.356	.186	.086	.040	.112	.100
.8	2.78	2.92	4.81	6.00	.254	.110	.101	.040	.113	.070
.9	-	2.82	4.90	5.38	.243	.080	.093	.040	.203	.070
1.0	-	2.73	-	-	-	.080	-	.040	-	.070

Note: Blanks are missing data or missing component (e.g., web = 0.1)

Conclusions that can be drawn from Table 5.13 are

- (1) Difference in d_w is evident since Carter assumed a distance from junction of the nose to the skin, and the actual web distance is inset somewhat.
- (2) Difference in leading edge thickness is because actual blade has a much higher resin content than predicted, since it was formed in a deep concave mold with no vacuum-bagging or other compacting.
- (3) The actual spar web is twice as thick (less PVC core) since the number of ±45° plies was apparently doubled over the design value.
- (4) The airfoil thickness is very close to the prediction, indicating good tooling and assembly.
- (5) The skin thickness is very close to the prediction, since it was vacuum-bagged (pressure formed).

This information now allows the blade section input values to be ascertained from the test values:

- (1) The new thicknesses and centroid locations (web) are the test values.
- (2) The new E- and G- modulus will be volume weighted to the new thickness since the reinforcement hasn't changed but the resin has.
- (3) The resin weight densities are volume-weighted to the measured values of glass content.

In conclusion, the specified laminate schedule is probably close to the actual blade laminate, but there is no way to tell for sure until both AEI and Carter section properties are calculated and the deflections compared with the test values. The I-beam contribution is the same, since measured and predicted values agreed (see above).

The initial (trial) values for the AEI blade model are

$$\begin{array}{ll}
 E_{\text{skin}} = 4.0 \cdot 10^6 \text{ psi} & E_{\text{spar}} = 4.0 \cdot 10^6 \text{ psi} \\
 G_{\text{skin}} = 0.5 \cdot 10^6 \text{ psi} & G_{\text{spar}} = 0.75 \cdot 10^6 \text{ psi} \\
 \rho_{\text{skin}} = 0.05 \text{ lb/in.}^3 & \rho_{\text{spar}} = 0.05 \text{ lb/in.}^3
 \end{array}$$

The initial values for the inboard I-beam portion are

STA	EI(xp)	EI(yp)	β
0.0	37.53 * 106	2817 * 106	25°
0.1	37.53	2817	25°
0.2	37.53	1980	14°
0.3	31.59	1755	7.5°

where:

- (1) $E_{\text{I-beam}} = 9 \cdot 10^6 \text{ psi}$
- (2) The skin is assumed to contribute no bending stiffness from the root to 0.35 STA, by Carter design intent.
- (3) The I-beam values are carried to the origin.
- (4) The twist angle will affect the values in the true chordwise X-Y system, but the flexural tests are really in the principle axes of the I-beam.

5.4.7: VERIFICATION BY FLEXURAL TEST

The three bending tests used represent two distributed load cases (total load = 225 lb and 325 lb), and one torsional load case (50 lb @ STA 0.9 in torsion). This section discusses the bending tests only. Again, the test loads were placed on the blade quarterchord. The major deflection was in the flapping direction. As seen in Appendix 6, the inplane (chordwise) deflections were very small, indicating very little principal axis coupling. However, elastic twists were substantial, indicating the elastic axis was not on the quarterchord.

Two corrections had to be made in the (flapwise) data:

- 1) the dead load droop had to be eliminated, and
- 2) the root compliance (teeter angle) had to be considered.

The dead load droop was easily found from (averaged) dead load cases. The root angle was measured with the clinometer, with a repeatability better than three minutes of arc. The three flapwise distributions and the corresponding elastic teeter values are shown in Figure 5.28.

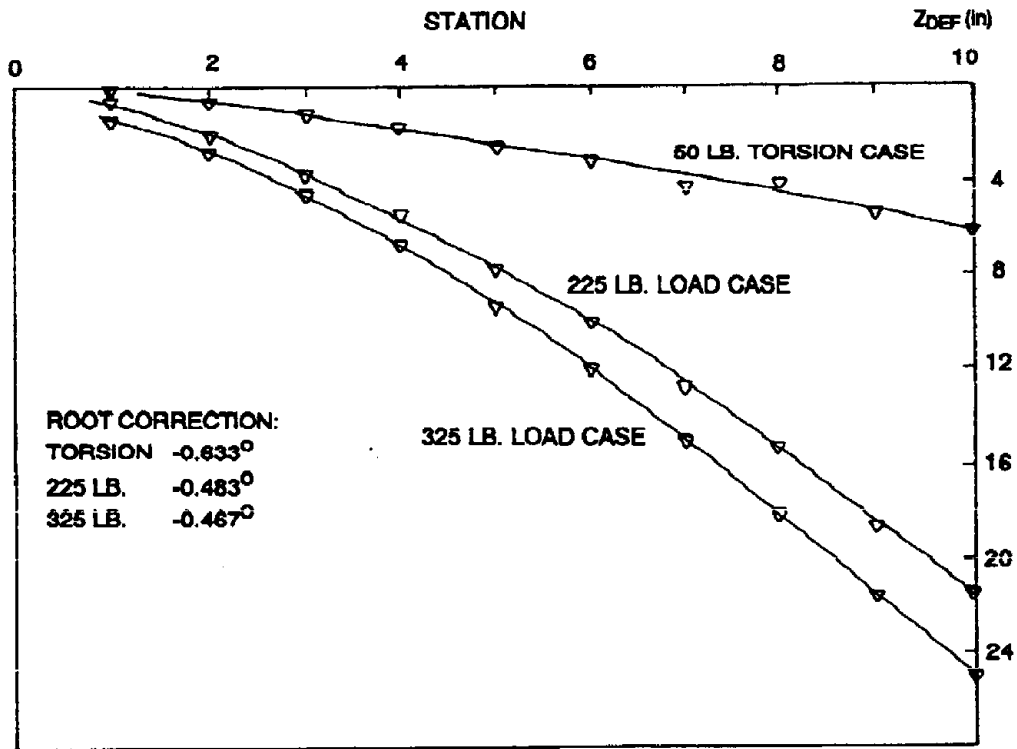


FIG. 5.28 CARTER STATIC DEFLECTION: 3 LOAD CASES (CORRECTED)

As stated above, the flexural stiffness inboard was initially assumed to be provided only by the I-beam, as conceived by the Carter Wind Systems design. This resulted in predicted blade deflections that were much too large. However, by increasing the inboard section modulus in the flapwise direction by a "fudge-factor" of 2.67, all three bending cases matched the test data, as can be clearly seen in Figure 5.29.

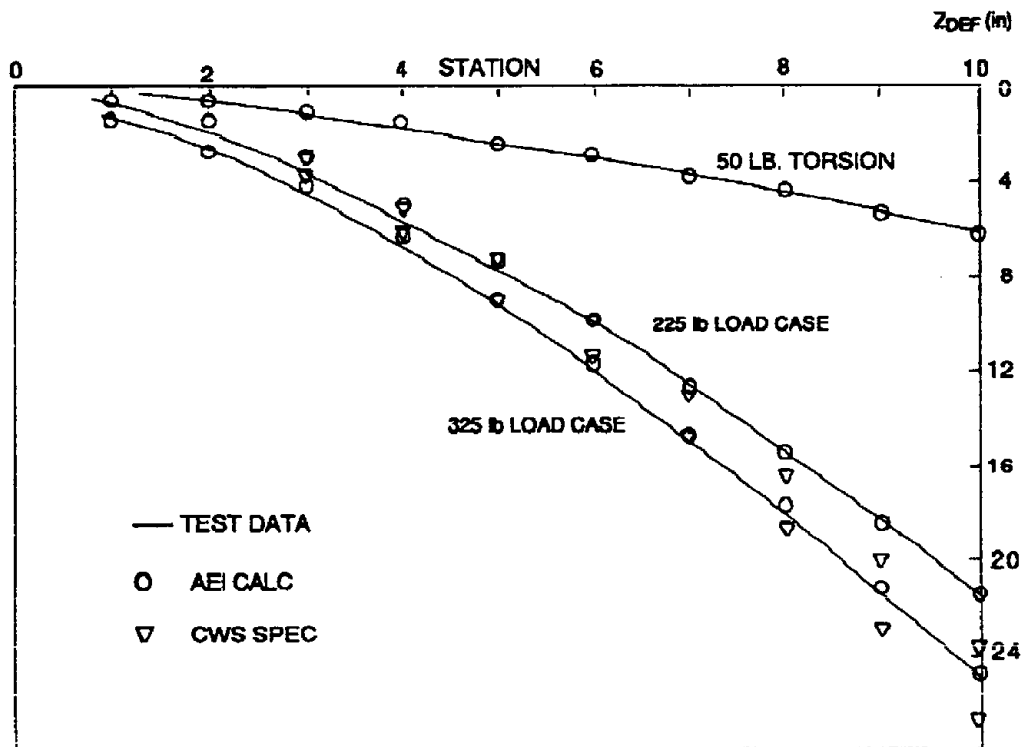


FIG. 5.29 CARTER STATIC DEFLECTION: AEI CALCULATION VS. MEASURED (CORRECTED)

This is a very interesting result, and indicates that the blade shell is contributing after all to the flexural stiffness inboard, actually more than doubling the stiffness of the I-beam alone. This is evidently caused by some transfer of bending moment through the root rib I-beam attachment, allowing some of the moment to be carried in the skin. For point of reference, if the root rib were rigidly attached to the skin and the I-beam, the flexural stiffness would have been on the order of 50 times the I-beam stiffness, rather than only 2.67. In any case, the root rib joint has to be designed with this additional load in mind.

Another conclusion from Figure 5.29 is that the Carter design laminate and AEI observed laminate curves are very close, verifying the overall Carter laminate specification.

5.4.8: VERIFICATION OF STATIC TWIST

The corresponding static twist associated with the two bending cases (225 lb and 325 lb) above are shown in Figures 5.30 and 5.31, along with the Carter and AEI laminate predictions. Note the increased scatter of the test data inboard, where the skin profiles were being warped by shear lag, causing the templates to drift. However, the match is still very good, verifying the AEI and Carter laminate models.

A further static torsion test case was also used. The results are shown in Figure 5.32. In this test load a point load at 0.70 STA was used, offset from the quarterchord to produce a point moment. This figure shows the marked difference in the AEI and Carter design laminate, whereas the distributed load cases above really did not. Clearly the AEI laminate values are more accurate than the Carter specs. Note the divergence of the test point right on the loaded template, presumably due to local warping amounting to six minutes of arc.

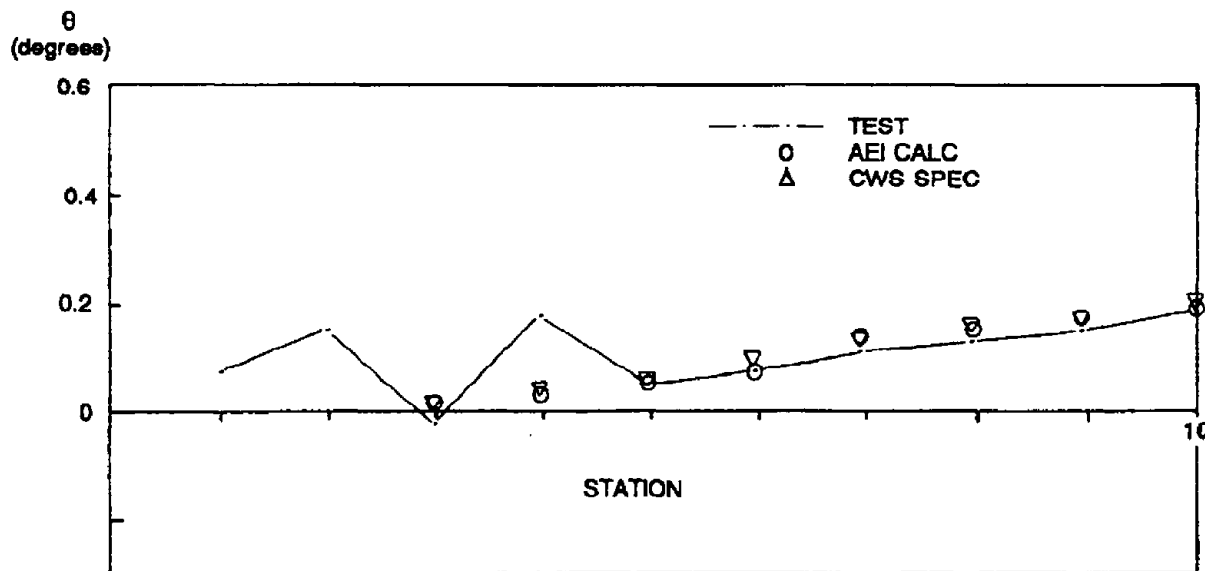


FIG. 5.30 CARTER MEASURED TWIST: (225 POUND TEST) PREDICTED VS. AEI CALCULATION VS. SPECIFICATION

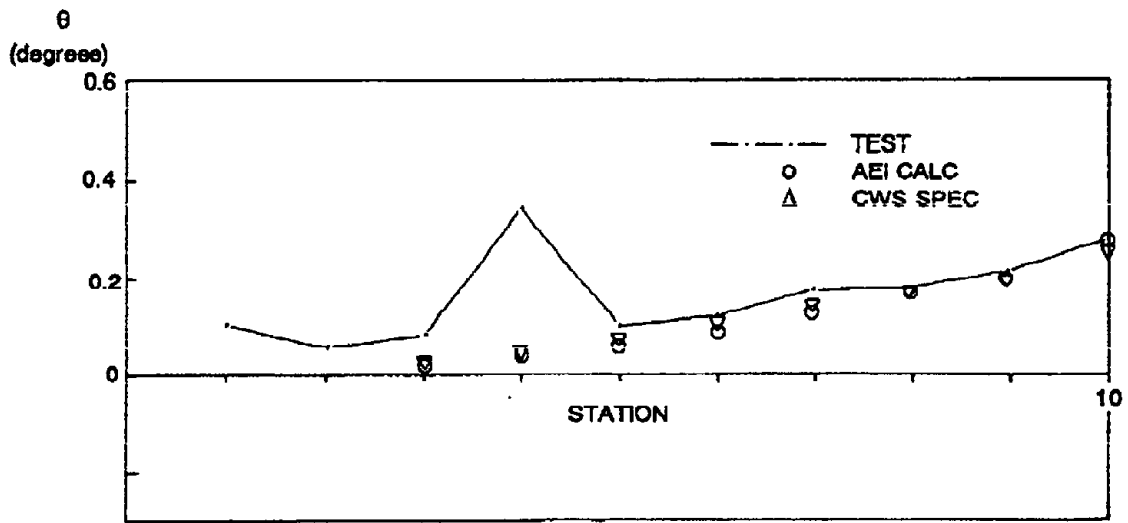


FIG. 5.31 CARTER MEASURED TWIST: 325 POUND TEST

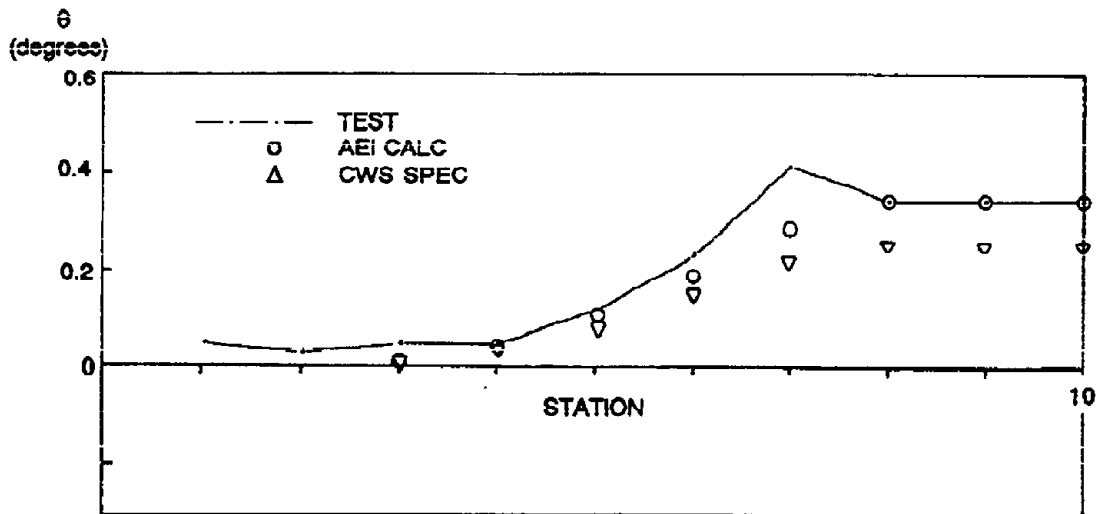


FIG. 5.32 CARTER MEASURED TWIST: TORSION LOAD CASE

5.4.9: COMPARISONS OF RUNNING MASS DISTRIBUTIONS

With the estimated weight densities above, the blade running mass distributions for the Carter design laminate and AEI measured laminate are compared with the actual running masses from the sectioning tests (Table 5.14 and Figure 5.33).

TABLE 5.14: CARTER BLADE RUNNING MASS COMPARISON (LB)

STA	AEI Calculation	CARTER design laminate	AEI Test
0.0	2.11	1.74	0.838*
0.1	2.11	1.74	6.983*
0.2	1.69	1.72	2.278*
0.3	1.42	1.50	1.868*
0.4	1.14	1.16	1.868*
0.5	1.27	1.19	1.657
0.6	1.07	1.08	1.384
0.7	0.93	0.86	1.266
0.8	1.30	1.19	1.127
0.9	1.42	1.16	1.236
1.0	0.61	0.35	1.063
Total	559.9 lb	512.7 lb	835.4 lb

NOTES:

- 1) The total blade weight of 835.4 lb is from test; the (*) items contain mechanisms, counterweights, root rib, mid-rib, an amount of potting resin to attach the I-beam, and an unknown amount of undocumented extra laminate, all adding up to over 200 lb of weight.
- 2) The two calculations contain the outboard ballast weights, but not the inboard ballast weights, which were unknown and impossible to determine nondestructively.
- 3) The two calculations have weights from STA 0.0 to 0.4 which are artificially low since no weight density correction was used for the (much denser) I-beam portion.

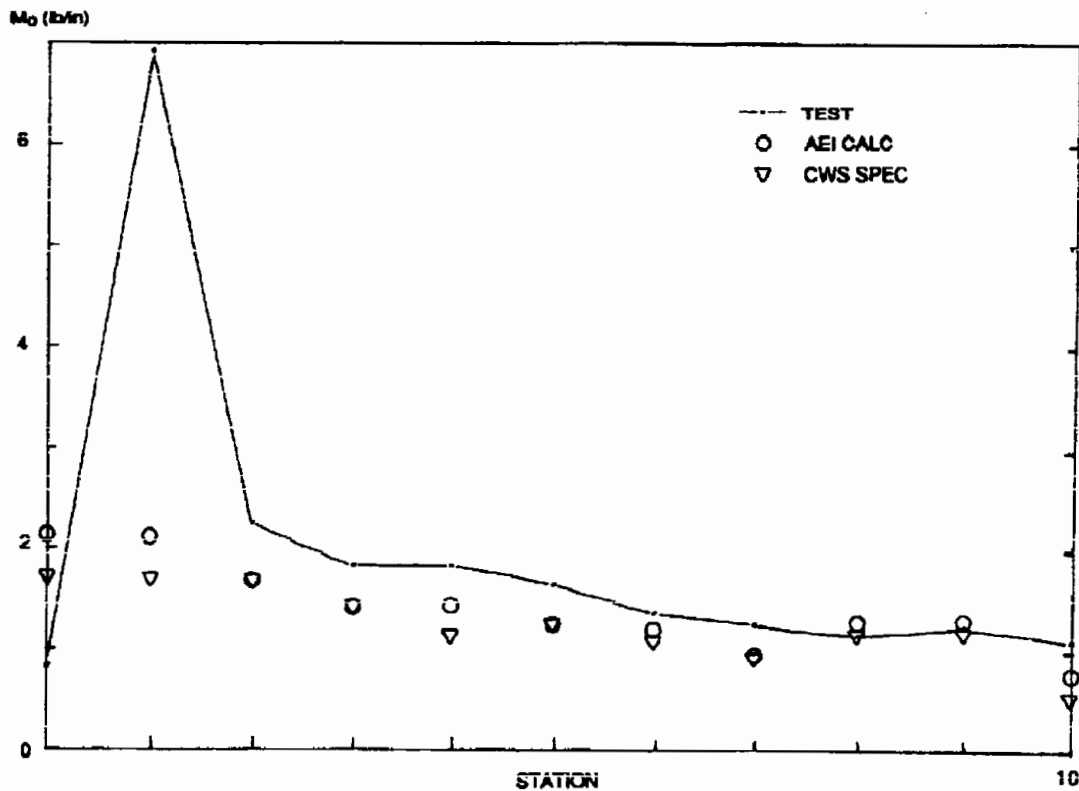


FIG. 5.33 CARTER BLADE RUNNING MASS COMPARISONS

However, the only accurate method for determining mass accuracy is in the frequency test comparisons below.

5.4.10: VERIFICATION BY FREQUENCY OF VIBRATION TESTS

The frequency of vibration tests resulted in the first three fundamental modes in flap and lead-lag, well-separated; two fundamental torsional modes were also produced: one called "root torsion" in which the blade performs as a rigid body about the I-beam root segment at the root rib, and the next called "outboard torsion" in which the outboard blade oscillates about the midblade, under the major influence of the outboard ballast weights (See Figure 3.15).

The frequency comparisons are given in Table 5.15 for

- 1) The Carter design laminate and weights.
- 2) The AEI calculated value.
- 3) The AEI calculated value with no ballast.
- 4) The AEI calculated laminate and measured weights.
- 5) The test results.

As can be seen, the following conclusions can be made about the flap values:

- 1) All the calculated values had lower masses than the actual blade and thus had higher frequencies of vibration.
- 2) The calculated value with ballast weight omitted ("AEI-No ballast") had the highest frequencies of all, as expected; this illustrates the degradation of blade natural frequency by adding ballast weights.
- 3) With the correct (measured) mass distribution, which includes over-specified resin, mechanisms, and so on the flap frequencies are correctly predicted.

TABLE 5.15: CARTER BLADE FLAPWISE FREQUENCY COMPARISON

Mode	Test	CARTER design	AEI calc	AEI no ballast	AEI test weights
1st flap	0.59	0.76	0.63	0.98	0.60
2nd flap	2.73	3.13	2.90	5.54	2.59
3rd flap	7.40	8.43	8.49	13.45	7.42
1st lag	3.00	3.08	3.16	-	3.03
2nd lag	16.0	12.63	19.79	-	17.56
3rd lag	40.4	33.01	51.78	-	43.74
root torsion only	13.25	-	-	-	-
1st outboard torsion	19.74	19.76	15.3	-	14.89

The conclusions about the lead-lag values are similar; the test weights correctly predict the frequencies.

For the torsional frequencies, these conclusions can be drawn:

- 1) The root torsion (rigid body mode) was not predicted since root torsional rigidity GJ was not reduced to the I-beam value in the code. This was not considered realistic for the blade study since this motion is really a change in root pitch and not an elastic twist. However, this frequency (degree of freedom) with the Carter dashpot snubber in place will cause the root pitch to change slightly when the blade is in operation. Actually this vibration mode directly affects the root pitch setting and the operation of the pitch-up snubber mechanism in the field, and is bound to be troublesome to the turbine on both counts.
- 2) The torsional frequency is correctly predicted by the Carter design specification, but not by the actual blade weights. The error is in the actual placement of the "extra mass" seen in the section test weights. They were assumed to be added to the section cg, but in actuality were close to the elastic axis, thus decreasing section mass moment of inertia. This is not considered to be a large mismatch or error since all the other torsional values do check with the data. A refinement in the Carter 300 model would iterate with the "extra masses" to find a new cg (and section mass moment of inertia) to raise the torsional frequency slightly.

The final blade weights (cumulative), section running mass, and section cg's are given in Table 5.16.

TABLE 5.16: CARTER BLADE: FINAL MASSES AND SECTION CG'S

	r/R	STA	Cumulative Weight		Running Mass		Section cg	
			AEI model	Test	AEI model	Test	AEI model	Test
root	0.0	11	920.43	835.45	2.11	0.838	23.89	-
	0.1	10	702.58	554.20	6.98	6.983	18.86	-
	0.2	9	486.82	462.45	2.28	2.278	17.32	-
	0.3	8	404.65	386.70	1.87	1.868	13.90	10.87
	0.4	7	330.76	311.45	1.87	1.868	10.79	8.43
	0.5	6	259.51	244.70	1.66	1.657	6.31	6.54
	0.6	5	198.0	188.95	1.38	1.384	5.16	5.39
	0.7	4	144.5	137.95	1.27	1.266	4.00	4.80
	0.8	3	96.09	92.57	1.13	1.126	3.88	4.09
tip	0.9	2	47.36	42.80	1.24	1.236	4.87	4.57
	1.0	1	-	-	1.06	1.063	5.35	-

5.4.11: FINAL AEROELASTIC MODEL

The final values of the Carter 300 aeroelastic blade model are shown in Table 5.17.

TABLE 5.17: CARTER BLADE: FINAL AEROELASTIC BLADE MODEL

	STA	r	β	m_0	EI_{flap}	EI_{lag}	GJ	Y_{cg}	Y_{ea}	I_{cg}
		(in.)	(deg)	(slug/in.)	(10^6 psi)	(10^6 psi)	(10^6 psi)			
root	11	0	-	2.11	100.10	2817	518.30	23.89	21.92	704.6
	10	40.3	26.1	6.98	100.10	2817	518.30	18.86	21.92	781.9
	9	80.6	15.5	2.28	100.10	1980	251.80	17.32	17.20	366.8
	8	120.8	9.4	1.87	84.24	1756	128.40	13.90	13.71	194.9
	7	161.1	4.5	1.87	781.10	7846	63.50	10.79	11.16	106.4
	6	201.4	2.3	1.66	260.70	4608	41.01	6.31	10.48	98.3
	5	241.7	2.0	1.38	140.70	2699	23.34	5.16	8.91	56.9
	4	281.9	2.0	1.27	89.14	1650	13.36	4.00	7.41	33.8
	3	322.2	2.0	1.13	63.98	1244	10.76	3.88	8.02	28.2
tip	2	362.5	2.2	1.24	78.42	1527	15.20	4.87	9.82	40.3
	1	402.8	2.6	1.06	87.98	1506	14.98	5.35	9.81	37.8

6.0 ELASTIC TWIST CALCULATION

6.1: GENERAL

After the blade structural models were developed, the next step was to calculate the expected elastic twist under normal operating wind turbine conditions. This was assumed to be steady-state (e.g., gravity and cyclic loads were not considered). This task consisted of calculating the blade section pitching moments and forces, and then calculating the elastic twist due to those applied loads. These loads are both inertial and aerodynamic. This was the primary theoretical goal of the project and had to be theoretical since no flight testing was done. Briefly, the inertial and aerodynamic loads were used with the structural deflection program described above to obtain the elastic twist due to each component of loading in order to ascertain design significance; that is, to point out where most of the elastic twist in flight originates.

6.2: CALCULATION OF BLADE CENTRIFUGAL TENSION AND MOMENTS

The inertial loads on the blade sections are caused by centrifugal forces (and moments) acting on the mass elements. This section derives the centrifugal loads on a flying blade (see also Ref. 28, for example). Up to now the analysis has been static, with beam mass and stiffness properties expressed, calculated and verified. Now the mass properties are combined with the operating condition to give the inertial blade loads.

Consider the discrete lumped mass blade shown in Figure 6.1, which is the easiest way to express centrifugal forces. The figure shows an undeflected lumped mass blade at precone angle β_{pc} , rotating at angular speed Ω about the z-axis. The root reactions are F_0 and M_0 , or centrifugal force (tension) and moment. Mass elements at radius r_n and flap deflection z_n have an inertial centrifugal force of $m_n r_n \Omega^2$, as shown.

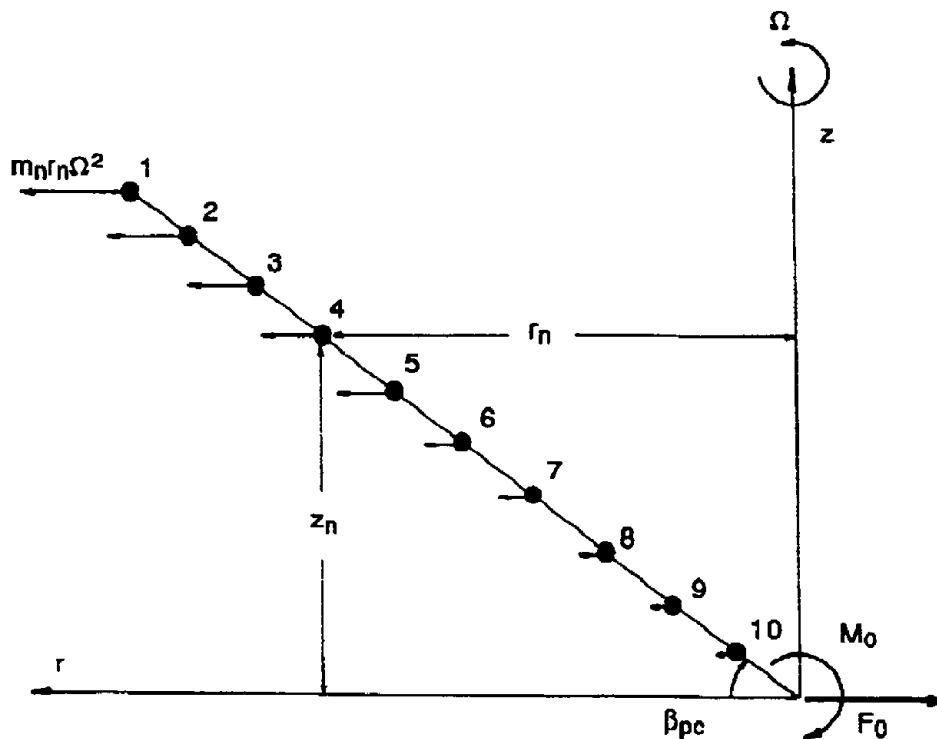


FIG. 6.1 DISCRETE LUMPED MASS CASE

Basic static theory states

$$F_{\text{centrifugal } i} = m_i r_i \Omega^2$$

$$F_o = \sum_{n=1}^N m_n r_n \Omega^2$$

$$M_o = \sum_{n=1}^N m_n r_n z_n \Omega^2$$

and the centrifugal moment at any mass element n is

$$M_n = \sum_{i=1}^{n-1} F_{\text{cent } i} [z_i - z_n]$$

$$= \sum_{i=1}^{n-1} F_{\text{cent } i} z_i - \sum_{i=1}^{n-1} F_{\text{cent } i} z_n$$

These expressions can be generalized to a continuum form, just by imagining many lumped masses, (which in the limit are differential elements) and by starting the integration at the blade tip rather than the root.
Note: here $x = R - r$ and is zero at the tip.)

$$\text{Blade Tension at } x = \int_0^x m r \Omega^2 dx$$

$$\text{Blade Moment at } x = \int_0^x m r \Omega^2 z dx - z(x) * [\text{Blade Tension}]$$

The above expressions can be easily checked by hand with a simple blade, such as the uniform UTRC blade. Referring to Figure 6.1, it can be seen that the calculation algorithm does not change if the blade shape is curved, as it will be when under the combined action of section forces and moments. The calculation program starts with an undeflected (straight) blade and then allows it to deflect in flap, lag, and twist until a loop convergence is reached (Appendix 10.5).

6.3: DERIVATION OF SECTION TORSION

There are, in general, five significant points on the blade section:

- | | |
|--------------------|---|
| mass center | - cg |
| shear center | - point about which no section torsional deflection is incurred with bending |
| aerodynamic center | - quarterchord |
| control axis | - axis defined by root fixation or pitch bearing |
| blade axis | - axis defined by original layout (placement and twist) of the blade airfoil shape (usually determined by construction considerations). |

Clearly the first three above are not necessarily straight line loci on the blade, but the last two are (Fig. 1.4). For purposes of calculating the elastic twist the control axis was used. It is defined by the centroid of the blade root fixture as it would be if a pitching bearing were used. Then the elastic twist can be directly added or subtracted from the pitch angle changes. Thus, the four points that must be known or determined for each blade section are cg, shear center, aerodynamic center, and control axis. The cg's and shear centers are calculated in the section program above. The aerodynamic center is placed at the quarterchord, and the control axis is specified by the blade fixture. The section airfoil offsets, which locate these centers relative to each other are also input to the program.

The section torsion that causes elastic twist is most easily calculated relative to the shear center of the section. This would be the "elastic axis" of a uniform blade. The above is true because torsion can be produced by a section force only if the line of action of the force does not pass through the shear center. Section moments then are all added together and must be resisted by an equal and opposite elastic moment at that section's shear center. The calculation then proceeds, introducing the section torsional stiffness GJ , which then gives the differential elastic twist for that section. To determine the total elastic twist, these section values must be integrated starting at the blade root.

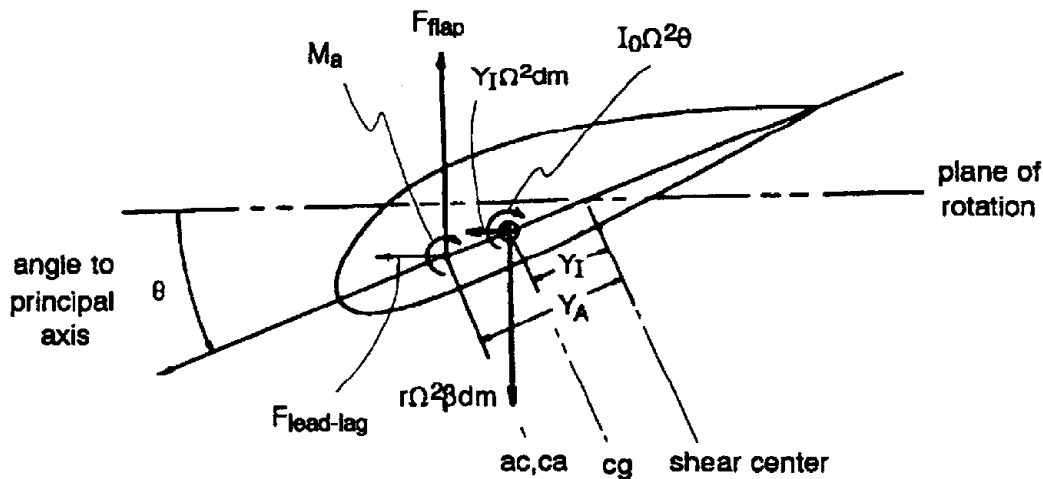
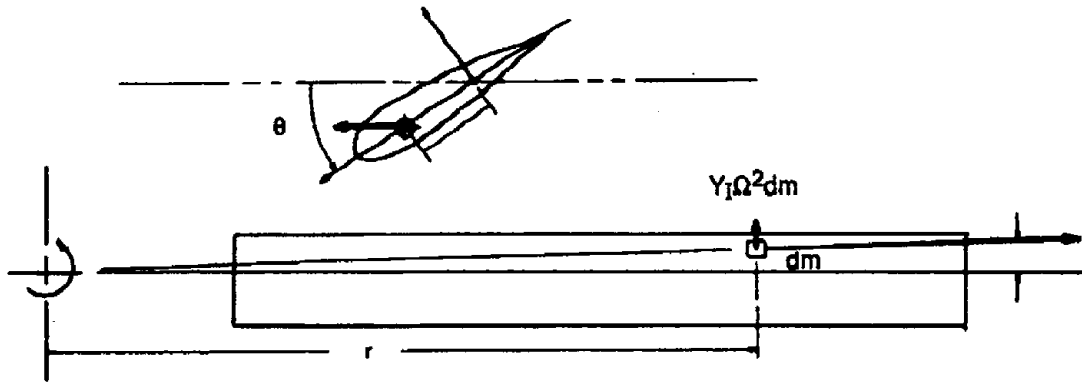


FIG. 6.2 BLADE SECTION FREE BODY DIAGRAM

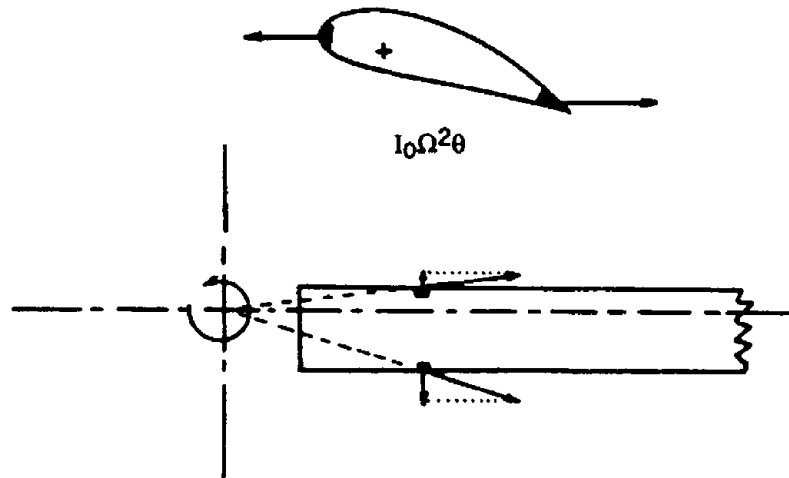
The differential section steady torsional moments are shown in Figure 6.2, the section free body diagram (see Ref. 2 for a complete derivation): (Note: the offsets Y_I and Y_A have been simplified for clarity.)

1. Aerodynamic forces in flapping (F_{flap}) and lead-lag ($F_{lead-lag}$) act at the aerodynamic center, with a moment arm to the shear center, $Y_A \cos\theta$.
2. Aerodynamic moment (M_a) acts at the aerodynamic center and is directly translated to the shear center.

3. The primary centrifugal moment is due to the vertical force component (flapping deflection) and is simply the product of the section force $r \Omega^2 \beta \, dm$ times its moment arm from the shear center $Y_I \cos \theta$.
4. The secondary centrifugal moment is due to the horizontal centrifugal force component, and comes from the vertical distribution of mass around the elastic axis; this is shown as $Y_I \Omega^2 \, dm$ in the free body diagram. This force is depicted in the following sketch:



5. The third part of the centrifugal moment is the "tennis racket" or propeller moment, $I_o \Omega^2 \theta$, where I_o is the section pitching mass moment of inertia about the cg and θ is the angle from the plane of rotation to the section's principal mass axis. Tennis racket moment is simply caused by a pitch (or a twist) rotation of the section mass away from the plane of rotation, which produces a centrifugal force couple on the mass elements above and below the plane of rotation tending to reduce pitch. This moment is zero when $\theta = 0$, or when the plane of rotation passes through the principal (major) mass axis of the blade (which is also referred to as the section principal axis of inertia). See the following sketch:



These are the steady torsional moments acting on the section to produce local twist. These torsional moments are calculated in the second part of the STRESS ELASTIC program, and the deflections are then found and integrated to obtain the elastic twist distribution of the blade. The simplified moment arms Y_I and Y_A from Figure 6.2 are not used in the program, which includes instead all the section moment arm contributions (e.g. x,y components).

The section ballast weight contributions are done as separate calculations in the program to allow separate assessment of their effects by the blade designer. A ballast weight in the leading edge, for instance, adds to the tennis racket moment and has both the horizontal and vertical centrifugal components as above, again on their (new) moment arms to the shear center. [The basic tennis racket moment, if transposed to the shear center using the parallel axis theorem, confirms the horizontal centrifugal component described above (see Ref. 2).]

Each elastic twist contribution is listed in the program output to assess its individual significance:

1. Aerodynamic Contribution includes torsion due to lift force, inplane aero force, and aerodynamic moment.
2. Centrifugal Contribution includes the basic section torsion due to horizontal and vertical centrifugal force (less ballast).
3. Tennis Racket Contribution includes the basic section tennis racket torsion (less ballast).
4. Ballast Contribution includes all the contributions of ballast weight, including centrifugal (horizontal and vertical) and additional tennis racket moment.

6.4: AERODYNAMIC LOADS

The aerodynamic section loads were calculated with a strip theory similar to the PROPPC code (Ref. 7 and next section) since PROPPC has no section pitching moment calculation. The loads were output (expressed) as distributed load at the aerodynamic center (e.g. lb/in. or in.-lb/in.) along the span. The airfoil C_L/C_d section data were corrected by the aerodynamic consultant (S.J. Miley) to reflect airfoil deviations noted in the laboratory inspection of the blade airfoil section (Appendix 10.6). At and above stall no reliable theory exists; SERI had anticipated this and specified post-stall routines for the code (see Sec. 7.1).

6.5

GENERAL DISCUSSION: OPERATING BLADE ELASTIC DEFLECTIONS

Figure 6.3 shows the output of a calculation for the UTRC blade, giving all the flexural output.

```

UTRC.BLADE.PLUS.FLEXBEAM.1-21-88      ** TCORR = 1.083 ** OMEGA = 11.310 ** PITCH = 0.00 ** CONING = 5.00
UTRC.BLADE.30MPH.LOAD.1-21-88
# STATIONS: 11 E-SKIN: 2050000.00 E-LONG: 5699999.00 # DIVISIONS: 500 GS: 899999.75 GL: 99999.97 WS: 0.064 ML:

      TIP
STATION      1      2      3      4      5      6      7      8      9      10
-----
STA POS      0.00    19.20   38.40   57.60   76.80   96.00  115.20  134.40  153.60  172.80
X-LOAD       0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00
Y-LOAD      -0.87    -0.89    -0.72    -0.44    -0.34    -0.18    -0.04     0.03     0.04     0.03
Z-LOAD       5.17     4.51     4.00     2.99     2.17     1.57     1.13     0.81     0.61     0.47
P-MOMENT     5.48     3.97     3.09     3.03     2.49     2.06     1.70     1.65     1.50     1.49
EA           0.2286E+08 0.2286E+08 0.2286E+08 0.2286E+08 0.2286E+08 0.2286E+08 0.2286E+08 0.2286E+08 0.2510E+08 0.2510E+08 0.2510E+08
EIYP        0.5867E+07 0.5867E+07 0.5867E+07 0.5867E+07 0.5867E+07 0.5867E+07 0.5867E+07 0.5867E+07 0.1372E+07 0.1372E+07 0.1372E+07
EI ZP       0.3240E+09 0.3240E+09 0.3240E+09 0.3240E+09 0.3240E+09 0.3240E+09 0.3240E+09 0.3240E+09 0.1370E+09 0.1370E+09 0.1370E+09
GJ          0.5580E+07 0.5580E+07 0.5580E+07 0.5580E+07 0.5580E+07 0.5580E+07 0.5580E+07 0.5580E+07 0.2400E+06 0.2400E+06 0.2400E+06
SCALE       15.00    15.00    15.00    15.00    15.00    15.00    15.00    15.00    15.00    15.00
TSCALE      1.00     1.00     1.00     1.00     1.00     1.00     1.00     1.00     1.00     1.00
-----
Y-BCENT      3.40     3.40     3.40     3.40     3.40     3.40     3.40     3.40     5.50     5.50
Z-BCENT      0.18     0.18     0.18     0.18     0.18     0.18     0.18     0.18     0.00     0.00
Y-SNRCTR     6.93     6.93     6.93     6.93     6.93     6.93     6.93     6.93     5.50     5.50
Z-SNRCTR     0.17     0.17     0.17     0.17     0.17     0.17     0.17     0.17     0.00     0.00
-----
WEIGHT       0.43     0.43     0.43     0.43     0.43     0.43     0.43     0.43     0.32     0.32
Y-CG         4.69     4.69     4.69     4.69     4.69     4.69     4.69     4.69     5.50     5.50
Z-CG         0.18     0.18     0.18     0.18     0.18     0.18     0.18     0.18     0.00     0.00
I-EA        10.51    10.51    10.51    10.51    10.51    10.51    10.51    10.51    1.74     1.74
I-CG         8.38     8.38     8.38     8.38     8.38     8.38     8.38     8.38     1.74     1.74
-----
BALLAST      0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00
TOTAL WT     0.43     0.43     0.43     0.43     0.43     0.43     0.43     0.43     0.32     0.32
NEW Y-CG     4.69     4.69     4.69     4.69     4.69     4.69     4.69     4.69     5.50     5.50
NEW Z-CG     0.18     0.18     0.18     0.18     0.18     0.18     0.18     0.18     0.00     0.00
NEW I-CG     8.38     8.38     8.38     8.38     8.38     8.38     8.38     8.38     1.74     1.74
BLD-WGHT    0.00     8.18    16.36    24.54    32.72    40.90    49.08    57.44    64.74    70.80
-----
V-AERO       0.00    92.68   174.18   241.06   290.36   325.99   351.64   370.06   383.43   394.27
BH-AERO      0.00   907.62  3487.33  7501.78 12631.62 18564.69 25085.99 32022.73 39264.74 46740.13 5
Y-AEROD     -0.20    -0.19    -0.18    -0.18    -0.16    -0.15    -0.12    -0.09    -0.04    -0.01
Z-AEROD     292.85   257.55   222.31   187.30   152.77   119.03   86.46   55.13   26.49   6.96
Z-CONING    16.73   15.06   13.39   11.71   10.04   8.37   6.69   5.02   3.35   1.67
-----
SECTION TORSION DUE TO: (+ = NOSE DOWN)
P-MOMENT     5.48     3.97     3.09     3.03     2.49     2.06     1.70     1.65     1.50     1.49
LIFT        -16.54    -14.46   -12.82   -9.58    -6.95    -5.01    -3.60    -2.56    -3.18    -2.46
TOT-AERO    -11.06   -10.49   -9.73   -6.55    -4.46    -2.95    -1.90    -0.91    -1.68    -0.97
-----
SECTION ALONE:
TNS-RCKT     0.05     0.05     0.05     0.05     0.05     0.05     0.05     0.05     0.00     0.00
FLAP-DEF     9.59     8.52     7.45     6.39     5.33     4.29     3.28     2.31     0.00     0.00
LD-LAG       0.01     0.01     0.01     0.01     0.01     0.01     0.01     0.01     0.00     0.00
TOTAL        9.65     8.58     7.51     6.44     5.38     4.35     3.33     2.37     0.00     0.00
-----
BALLAST ALONE:
TNS-RCKT     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00
CENTRIF     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00
TOTAL        0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00     0.00
-----
TOTAL SECTION + BALLAST:
TNS-RCKT     0.05     0.05     0.05     0.05     0.05     0.05     0.05     0.05     0.00     0.00
FLAP-DEF     9.59     8.52     7.45     6.39     5.33     4.29     3.28     2.31     0.00     0.00
LD-LAG       0.01     0.01     0.01     0.01     0.01     0.01     0.01     0.01     0.00     0.00
TOTAL        9.65     8.58     7.51     6.44     5.38     4.35     3.33     2.37     0.00     0.00
SECTION     -1.42    -1.91    -2.23    -0.11     0.93     1.40     1.43     1.45    -1.67    -0.97
-----
TORSION      0.00   -32.26  -72.29  -92.96  -83.37  -60.35  -32.48   0.26   3.18  -22.56
EL TWIST    -0.251  -0.248  -0.238  -0.221  -0.203  -0.189  -0.179  -0.176  -0.182  -0.132
TENSION     0.00   493.97  935.94 1325.91 1663.89 1949.88 2183.87 2368.11 2486.89 2544.70
V-IMPL      0.00   -17.18  -32.90  -43.79  -51.05  -56.04  -58.17  -58.15  -57.36  -56.64

```

FIG. 6.3 CODE OUTPUT, UTRC BLADE, 30 mph CASE

This is shown as an example to describe the format:

Heading- "UTRC blade @ 30 mph load"
rotor speed (Ω) = 11.31 rad/sec = 108 rpm
pitch angle = 0 degrees
coning angle = 5 degrees

The output by station is listed from tip to root, station 1 to 11; this divides the blade into 10 equal segments. The top line shows the radius value measured from the tip to each station (thus the root value is the radius of the rotor: 192 in. or 16 ft).

Blade section aerodynamic load is listed next, in X, Y, and Z directions (tension, lead-lag and flap) and pitching moment, all referenced to the quarterchord. These loads are the output of the aerodynamic strip theory. The remainder of this part gives the section stiffness and geometric properties. Bending and shear centers (elastic axis) are given next. It can be seen that this is a uniform blade; the values are constant for the outer 70% of the blade, which represents the pultrusion portion. The inner 30%, for the UTRC rotor, consists of a flexbeam with different properties. Weights, cg's, and moments of inertia are followed by a ballast weight section (for this blade the ballast is zero, as can be seen). Finally, the running "blade weight" line ends with the total blade weight (76.9 lb), which was verified in the bench test.

The aerodynamic flapping shear and bending moments are given next. [Note: these must be integrated from the tip, as shown.] The root values are the aerodynamic hub shear (400 lb) and bending moment (54,367 inch-lb) for this condition. Next, the simple blade deflection is given, that would occur with the aerodynamic load alone (e.g., -0.2 in. inplane and 292 in. in flap) to compare with actual blade deflections. The final line of this part gives the flap deflection due to coning angle (e.g. 16.7 in. at the tip). These are the initial trial values of blade flap deflection. [Note: In this case the trial value is not realistic: 292 in. of flap deflection is twice the rotor radius! The linear beam assumptions have clearly been violated by the large airload at 30 mph; as seen below, the aeroelastic moment is largely counteracted by the centrifugal moment bringing the algorithm back into its assumption bounds.]

The next four parts of the output list all the blade section pitching moments (in in.-lb, positive nose-down):

- 1 - Aerodynamic pitching moment.
 - 2 - Aerodynamic moment due to lift offset from shear center (total aerodynamic moment).
 - 3 - Inertial centrifugal moment due to tennis racket effect.
 - 4 - Inertial moment due to flap deflection.
 - 5 - Inertial moment due to lead-lag deflection (total section inertial moment).
 - 6 - 7 - Inertial moments due to ballast.
 - 8 - 10 - Total section inertial moments due to tennis racket, flap deflection, lead-lag deflection.
 - 11 - Total inertial moment.
 - 12 - Total section moment, or the sum of the inertial and aerodynamic contributions
- [Note: on this blade at the tip, the aerodynamic moment is 11 in.-lb nose up and the inertial moment 9.65 in.-lb nose down, yielding 1.4 in.-lb nose up net section moment.]

The torsion is just the integration of the section moment distribution and appears next. Following is the elastic twist in degrees. [Note: this case, 30 mph, has only 0.25 degrees, nose up, elastic twist at the tip.]

The following five lines give the blade structural reactions at each station:

- tension (lb)
- lead-lag shear (lb)
- lead-lag bending moment (in.-lb)
- flap shear (lb)
- flap bending moment (in.-lb)

The root values constitute the steady hub loading specification for this operating condition. This example shows the UTRC hub experiencing 2564 lb of centrifugal force (tension), 56 lb of lead-lag shear, 8847 in.-lb of torque, 400 lb of flapping shear, and 4880 in.-lb of flap bending moment.

Following the principal axis angle and the M/EI values, the blade flexural deflections are given in inches, for lead-lag and flapping. This blade has 0.52 in. of inplane deflection and 30 in. of flap deflection.

The elastic twist denotes the torsional deflection of this blade. It can be seen that the bulk of the twist occurs in the softer flexbeam portion. This is more obvious in the other UTRC cases that experience greater elastic twist.

Clearly, the centrifugal moments of this blade have reduced the tip deflection via centrifugal relief, which was the design intent. It should be noted that the flap deflection is not a straight line, since it represents the equilibrium position of a complex blade operating under complex aerodynamic and centrifugal loads.

6.6 UTRC BLADE DEFLECTIONS

The simplest blade is a uniform pultrusion connected to a torsionally flexible beam at its root (.30 STA). The shear centers (elastic axis) and mass centers are collinear and likewise the inertial terms. The flap deflection and elastic twist for the UTRC blade are given in the Tables 6.1 and 6.2 below and plotted in Figures 6.4 and 6.5 for wind speeds from 10-70 mph (pitch = 0 degrees, coning = 5 degrees).

TABLE 6.1: UTRC FLAP DEFLECTION (in.)

V ₀ (mph)	Root	Tip
10	0	1.5	2.6	3.4	4.1	4.8	5.5	6.2	6.8	7.5	8.2
20	0	1.9	4.0	6.2	8.4	10.5	12.6	14.6	16.6	18.6	20.6
30	0	2.2	5.0	8.0	11.1	14.2	17.3	20.5	23.6	26.8	30.0
40	0	2.3	5.3	8.6	12.0	15.4	18.8	22.3	25.8	29.3	32.8
50	0	2.4	5.6	9.1	12.7	16.3	19.9	23.5	27.1	30.7	34.4
60	0	2.6	6.1	9.9	13.8	17.8	21.9	26.0	30.1	34.3	38.5
70	0	2.8	6.6	11.0	15.5	20.0	24.7	29.3	34.0	38.7	43.4

TABLE 6.2: UTRC ELASTIC TWIST (DEGREES)

V ₀ (mph)	Root	Tip
10	0	1.25	2.53	3.10	3.06	3.11	3.16	3.21	3.25	3.27	3.28
20	0	.25	.55	.70	.70	.70	.71	.73	.75	.77	.78
30	0	-.13	-.18	-.18	-.18	-.19	-.20	-.22	-.24	-.25	-.25
40	0	.10	.32	.46	.44	.45	.44	.43	.42	.40	.39
50	0	.47	1.11	1.45	1.41	1.44	1.45	1.45	1.45	1.45	1.45
60	0	.90	2.03	2.61	2.55	2.60	2.63	2.65	2.65	2.66	2.66
70	0	1.52	3.34	4.23	4.15	4.22	4.28	4.31	4.34	4.35	4.35

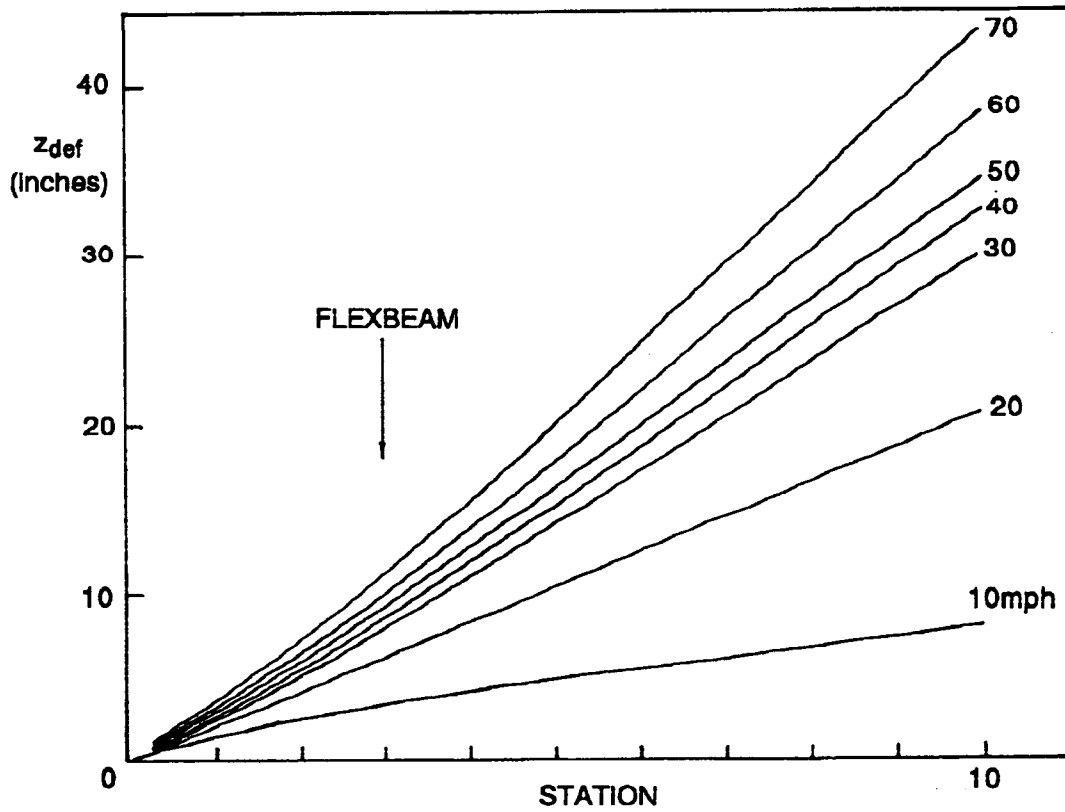


FIG. 6.4 UTRC FLAP DEFLECTION (10-70 mph)

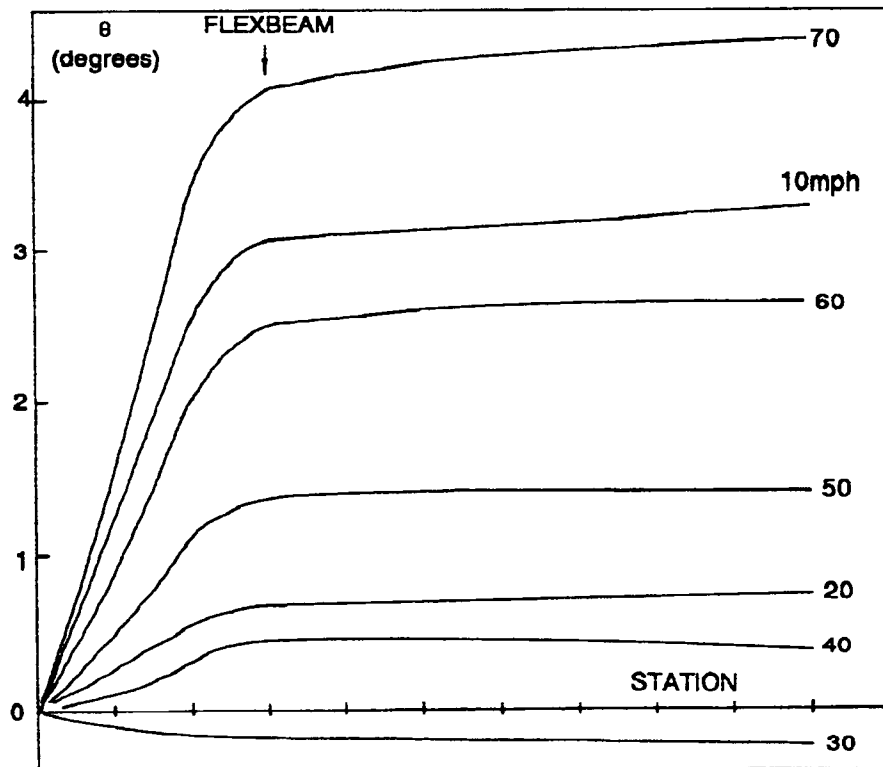


FIG. 6.5 UTRC ELASTIC TWIST (10-70 mph)

The flap deflections (and lead-lag deflections not shown) are easy to understand. The aerodynamic bending moment increases with increasing wind speed, and the centrifugal moment does not. Therefore, the tip deflection increases as the aerodynamic moment increases. The static tip deflection due to straight coning angle is 16.7 inches (5 degrees); the 10 mph values above are less than this, indicating the centrifugal moment is dominating the aerodynamic moment at 10 mph. This is borne out by examining the root flap moment for these cases, reproduced here from the printouts:

<u>V_o (mph)</u>	<u>Root Flap Moment (in-lb)</u>
10	-1843
20	1380
30	4280
40	5460
50	6670
60	8790
70	11190

The elastic twist is more complex. Not only do the gradients change along the blade, but also with wind speed; the tip elastic twist is large at 10 and 70 mph, but nearly nothing at 30 mph. To discover the sources of this twist, each section moment must be examined over the range 10-70 mph. Referring to Tables 6.1 and 6.2:

at 10 mph:

The 16.7 in. of static flap coning are reduced to 8.2 in. by centrifugal relief. Section twisting moments are dominated by the airfoil pitching moment (NACA 23012) and the centrifugal moment due to flapping, both nose down. About one-third of this is offset by a nose-up lift moment due to aerodynamic center offset. The other section moments are negligible (tennis racket, lead-lag). The net nose-down moment amounts to 290 in.-lb of torsion at 0.30 STA, twisting the flexbeam about 3.1 degrees. An additional 0.2 degrees of twist occurs in the much stiffer pultrusion portion of the blade.

at 30 mph: (see Fig. 6.3):

The airfoil pitching moment has not changed appreciably, but the lift offset moment is much greater since lift is much greater. Thus the net aerodynamic moment is now large and nose up. This is nearly offset by a large nose-down centrifugal moment, now made larger than above since the flap deflection is four times as great. Tennis racket and lead-lag are again negligible. The net elastic twist is slightly (0.25 degrees) nose up.

at 60 mph:

The aerodynamic pitching moments have not changed drastically from above. But, the centrifugal flap-deflection moment has increased since there is greater flap bending. Now the net moment is again nose down, amounting to 290 in.-lb at 0.30 STA and yielding about 2.6 degrees of elastic twist. The distributions are not linear in any case since the net torsion is the sum of individual nonlinear values (e.g., lift, pitching moment, flap bending).

6.7 ESI BLADE DEFLECTIONS

The ESI blade was of vacuum-bagged, wood (Douglas fir), veneer/epoxy internal construction and was easily the stiffest blade tested. This rotor blade was in service at two operating speeds: 77 and 90 rpm, corresponding to 54 and 85 kW rating; the rpm difference greatly affects the deflections via the inertial and aerodynamic loads. No ballast was used in either approach; however, two versions of the overspeed tip mechanism (tip flap) were used. These are depicted in Figure 5.19: the "regular tip" was a

simple steel alloy flat plate on a hinged spring/release mechanism, and the "whisper tip" was a much larger fiberglass fairing over a larger baseplate and release mechanism. The regular tip weighed 10.06 lb and the whisper tip 17.44 lb, with the cg of the latter being forward of the former (1.22 in. vs. 5.93 in. aft of LE; see Fig. 5.19).

TABLE 6.3: ESI TIP FLAP DEFLECTION

V _o (mph)		10	20	30	40	50	60	70
77 rpm:	no tip	26.7	31.5	35.8	38.8	40.5	41.1	41.9
	regular tip	25.5	29.9	33.9	36.7	38.3	38.9	39.6
	whisper tip	24.7	28.9	32.7	35.4	36.9	37.4	38.1
90 rpm:	no tip	25.4	30.7	35.7	39.9	42.7	44.3	44.9
	regular tip	24.0	28.8	33.4	37.2	39.7	41.1	41.7
	whisper tip	23.3	27.6	31.9	35.5	37.9	39.2	39.7

TABLE 6.4: ESI TIP ELASTIC TWIST

V _o (mph)		10	20	30	40	50	60	70
77 rpm:	no tip	.11	.06	.00	-.03	-.02	.10	.20
	regular tip	.00	-.08	-.15	-.20	-.18	-.07	.03
	whisper tip	.25	.22	.19	.17	.19	.31	.42
90 rpm:	no tip	.17	.10	.02	-.05	-.10	-.05	.09
	regular tip	.02	-.08	-.18	-.27	-.33	-.29	-.15
	whisper tip	.35	.31	.27	.24	.21	.27	.41

No Tip: This case was run to separate the tip weight effects on the deflections. The static coning of 5° gave 28.2 in. of flap deflection; at 10 mph the centrifugal relief again dominates, giving a negative root bending moment and a tip deflection less than the coned value. As wind speed increases, aerodynamic load increases, faster for the 90 rpm case. The flap deflections bear this out. The 90 rpm case has lower angles of attack at 10 - 30 mph, and as wind speed increases, the 77 rpm version stalls first and sheds airload. Thus the 90 rpm version has a higher eventual tip deflection (44.9 in.). However, this is still a small number since almost half of this value is due to static coning. The root bending moment for this 70 rpm case is a huge 257,000 in.-lb (i.e. this is a 2067 lb flap shear load). This certainly must be a very stiff blade; for this highest load case, the root laminate strain is less than 1400 microstrain! The root tension (centrifugal force) is 8100 lb at 90 rpm. This is a further indication that the blade is not heavy.

The elastic twist is always negligible for this blade and is dominated by the simple airfoil pitching moment (NACA 230XX variation airfoil). As would be expected the twist increases rapidly in stall, which can be clearly seen in the 77 rpm case at 60-70 mph. [Note: these twists are so small that a difference of only 1 in. of tip deflection reduces it by 70%; compare the 77 and 90 rpm, 10-mph case]

Regular Tip: Adding only 10 lb to the tip at a chordwise position of 5.93 in. and some section moment of inertia are the only differences from the above values. As would be expected the centrifugal force at 90 rpm (root tension) goes up from 8100 to 8840 lb and the root bending moments decrease (e.g. 232,000 vs. 257,000 in.-lb). Again the flap deflections are always less than the runs without tip weight. [Note: this is the traditional reason to add tip weight to helicopter blades -- to reduce tip deflection.] The root laminate strains are also less than before since the flexure is less than with no tip weight.

The elastic twist is more complex. First, adding the weight to the tip is only part of the story; the additional section moment of inertia also depends on the chordwise distribution of the weights (e.g., tip parts; see Sec. 5.3.7). So each mechanism part had to be calculated independently to get the "ballast" moment of inertia. The elastic twist values are now clearly dominated by the tip ballast via centrifugal twisting moments. As can be seen in Figure 6.6, the elastic twist occurs almost entirely in the outer 20% of the blade. This is because the "ballast" has been added as a point load at the tip. This means that the tip twist is only an approximate indication of the overall blade effect, even though the highest airload occurs at the blade tip. Still these twists are small (0.33 degree or less).

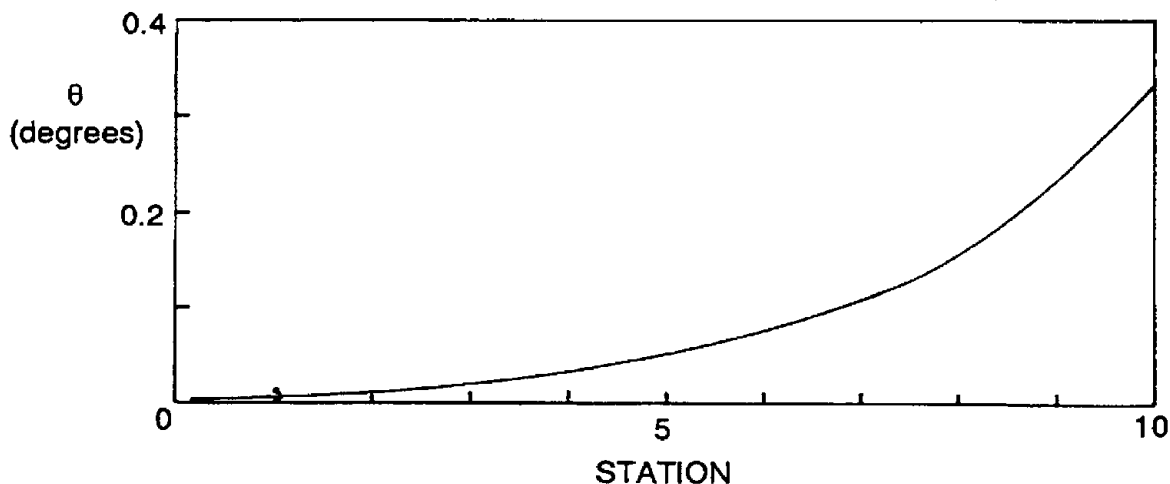


FIG. 6.6 ESI ELASTIC TWIST DISTRIBUTION

The effect of stall can also be seen, as the 77 rpm (more stalled) blade at 60-70 mph has a nose-down twist compared to the 90 rpm (less stalled) blade.

Whisper Tip: The whisper tip is about 7.5 lb heavier than the regular tip. This increased tip weight reduces the flap deflection even more, and increases the root tension slightly (8100 lb no tip, 8840 lb regular, 9390 lb whisper tip, at 90 rpm). The root bending moments are also reduced by the various tip weights (171,000 to 148,000 to 133,000 in.-lb).

The other important difference is the cg location of the whisper tip. The additional weight is forward of the section cg and shear center; in fact it is almost on the leading edge (1.92 in. from LE). One would expect a far-forward tip mass to produce a large nose-down pitching moment with flap deflection. This can be easily seen to be the case by looking at the elastic twist results in Table 6.4. All the twist is now nose down, and amounts to about 0.33 degree. Still this is a small value and would not be expected to affect performance very much. It does, however, produce section torsion which increases the laminate (veneer) shear strain.

Some unpublished flight test strip chart data were available from ESI tests on the 90 rpm, whisper tip version (Ref. 26). They showed an average torsion at 0.90 STA to be about 50 ft-lb (600 in.-lb) nose-down at 20-25 mph. The calculations show for this condition: 90 rpm, whisper tip, 0.90 STA, 20 mph, a blade torsion of 36.9 ft-lb (443 in.-lb) nose-down, which compares very favorably with the data.

6.8: CARTER BLADE DEFLECTIONS

The Carter blade was by far the most complicated of the test series. This blade was made up of various laminates of vacuum-bagged glass reinforcements in an epoxy matrix. The makeup and detailed structural arrangement of the blade are fully described in Section 4.4. The flap and lag deflections and elastic twist for wind speeds of 10 - 70 mph are given below in Tables 6.5, 6.6 and 6.7 for the operating

value of 2.0°. [Note: all deflections were corrected to the control axis (quarterchord) of the blade. Also the 50, 60, and 70 mph cases must be considered somewhat academic since the turbine does not operate at those wind speeds.]

The clearest result seen is the increase of blade bending, both flapwise and edgewise, with increasing wind speed. The negative inplane deflections denote forward bending, i.e. positive torque on the rotor shaft. Realistically, with tip deflections of 6 ft, the blade can no longer be considered an actuator disk (e.g. as in PROPPC code) and some aerodynamic load relief due to flap bending must be occurring. This probably tends to reduce angles of attack, inplane loads, and the power output. So the power output calculations must all be suspect for this highly flexible blade above 30 mph. This discussion will, therefore, focus on the design and elasticity of the structure rather than the power output.

TABLE 6.5: CARTER FLAP DEFLECTION (IN.)

V _o (mph)	Root											Tip
10	0	2.4	4.3	5.9	7.3	8.7	10.0	11.3	12.6	13.8	15.1	
20	0	3.1	6.5	10.2	14.1	18.1	22.1	26.1	30.0	33.9	37.8	
30	0	3.7	8.7	14.4	20.6	27.1	33.8	40.6	47.4	54.2	61.0	
40	0	4.0	9.7	16.5	24.0	31.9	40.2	48.7	57.4	66.2	75.0	
50	0	4.1	10.2	17.4	25.4	33.9	43.0	52.5	62.4	72.4	82.6	
60	0	4.1	10.2	17.4	25.3	33.8	42.7	52.1	61.9	71.9	82.0	
70	0	4.2	10.3	17.6	25.6	34.1	42.9	52.2	61.9	71.7	81.6	

TABLE 6.6: CARTER LEAD-LAG DEFLECTION (IN.)

V _o (mph)	Root											Tip
10	0	0.0	0.0	0.0	0.1	0.1	0.1	0.1	0.2	0.2	0.2	
20	0	0.0	-.1	-.2	-.3	-.5	-.6	-.8	-.9	-1.1	-1.2	
30	0	-.1	-.3	-.8	-1.3	-1.9	-2.6	-3.2	-3.9	-4.6	-5.3	
40	0	-.2	-.6	-1.3	-2.3	-3.4	-4.6	-5.8	-7.1	-8.4	-9.7	
50	0	-.2	-.8	-1.7	-3.0	-4.4	-5.9	-7.5	-9.1	-10.8	-12.6	
60	0	-.2	-.8	-1.8	-3.1	-4.7	-6.3	-8.0	-9.8	-11.7	-13.6	
70	0	-.2	-.8	-1.8	-3.1	-4.6	-6.1	-7.8	-9.6	-11.4	-13.3	

TABLE 6.7: CARTER ELASTIC TWIST (DEGREES)

V _o (mph)	Root											Tip
10	0	.03	.07	.14	.28	.49	.79	1.23	1.71	1.98	2.05	
20	0	.02	.06	.12	.25	.46	.76	1.22	1.69	1.96	2.02	
30	0	.01	.03	.08	.19	.38	.66	1.08	1.50	1.71	1.75	
40	0	.01	.03	.08	.18	.35	.59	.92	1.22	1.35	1.37	
50	0	.01	.04	.09	.21	.38	.61	.87	1.04	1.08	1.08	
60	0	.02	.06	.13	.27	.48	.74	1.05	1.27	1.33	1.33	
70	0	.04	.09	.19	.36	.62	.94	1.33	1.64	1.75	1.76	

Figure 6.7 shows the spanwise elastic twist and Figure 6.8 the pitching moment distributions of this blade for 10 and 30 mph; the corresponding distributions of section moment and the major constituents are also given in Table 6.8.

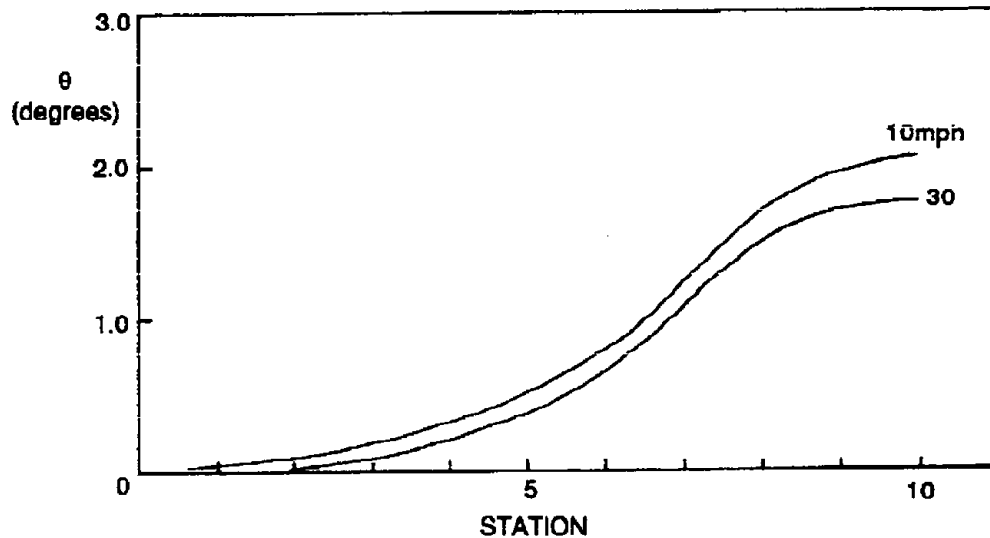


FIG. 6.7 CARTER ELASTIC TWIST (10 AND 30 mph CASES)

TABLE 6.8: CARTER BLADE SECTION TORSION COMPONENTS

$V_o = 10$ mph	Root	Tip
AERO PITCHING MOMENT	0	7.5	11.3	13.3	13.3	14.5	15.2	16.6	18.9	21.6	21.6
LIFT OFFSET	0	1.9	-.2	-2.3	-1.6	-4.2	-3.8	-2.4	-4.1	-8.7	-8.9
TENNIS RACKET	0	-.1	-.1	-.1	-1.3	-.6	-.4	-.3	-.2	-.4	-.4
FLAP DEFLECTION	0	6.4	-.2	-.4	.7	9.1	7.8	7.4	8.9	12.8	10.8
TOTALS:	-	9.8	10.8	10.6	11.2	18.4	18.7	21.1	23.3	24.9	22.9
$V_o = 30$ mph	Root	Tip
AERO PITCHING MOMENT	0	14.5	14.3	14.3	14.0	13.8	14.5	16.9	19.5	22.1	22.1
LIFT OFFSET	0	-26.5	-23.9	-21.8	-20.2	-29.7	-25.2	-17.4	-26.9	-52.4	-50.0
TENNIS RACKET	0	0	0	0	-1.3	-.6	-.4	-.3	-.2	-.4	-.4
FLAP DEFLECTION	0	9.7	-.5	-.8	2.1	28.2	26.4	26.3	33.4	50.2	43.8
TOTALS:	-	-8.4	10.2	-8.4	-5.4	11.3	15.1	25.4	25.5	19.2	15.3

10 mph Case: The tip deflections are small for this case; the flap value is 15.1 in., which is less than the 28 in. from coning, so the flap root moment is negative. The lead-lag moment is positive, or aft, indicating negative power output.

Surprisingly, the elastic twist is the highest for any wind speed case: over 2° at the tip. Referring to Table 6.8 allows a closer look at the moment components. The angles of attack are in the lifting range, and there is substantial (nose-down) aeroelastic pitching moment all along the blade, since its large root chord and twist keep the aerodynamic values high. This indicates a well-designed blade, or one where all the blade sections operate close to the same angles of attack (i.e. "work together"). The next line in the table is moment due to lift offset, or quarterchord to elastic axis. This offset is appreciable, with the shear

center being aft of the quarterchord (see Sec. 4.4). However this moment is small at 10 mph owing to a relatively small lift value.

Inertially (centrifugally) speaking, the only appreciable moment arises from flap coupling. Tennis racket and inplane coupling are small and can be neglected here (see line 3). The fourth line in Table 6.8 shows that the flap coupling term is slightly nose-up inboard; this is due to the section cg being aft of the shear centers inboard. Outboard the situation is drastically reversed, since substantial lead ballast has been added to the leading edge (see Sec. 4.4). This has brought the cg forward of the shear center, which creates a nose-down flap coupling moment. The net result of all the terms is a substantial nose-down moment, which causes substantial elastic nose-down elastic twist.

All the section centers and the forces and moments are shown as free body diagrams in Figure 6.8 for station 0.90. Referring to the figure: the small lift (2.0 lb) acts through the aeroelastic offset (4.4 in.) but is not enough to counter the aeroelastic pitching moment (21.6 in.-lb) and the centrifugal flap coupling moment (12.8 in.-lb), both of which are nose-down. The net result is a relatively smooth moment distribution which is always nose down (Fig. 6.9).

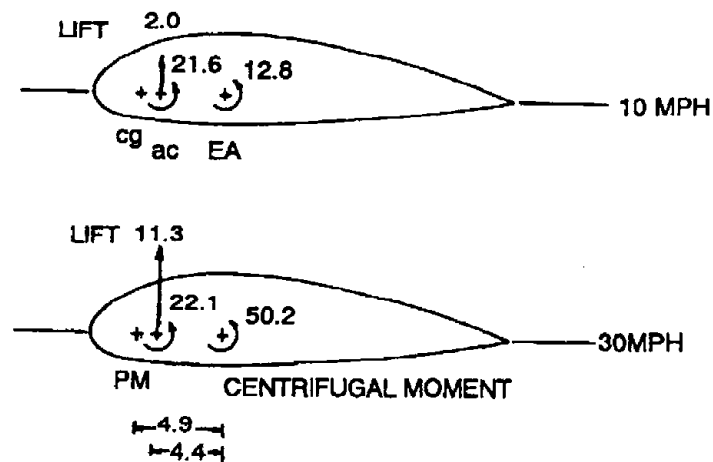


FIG. 6.8 CARTER SECTION FREE BODY DIAGRAMS

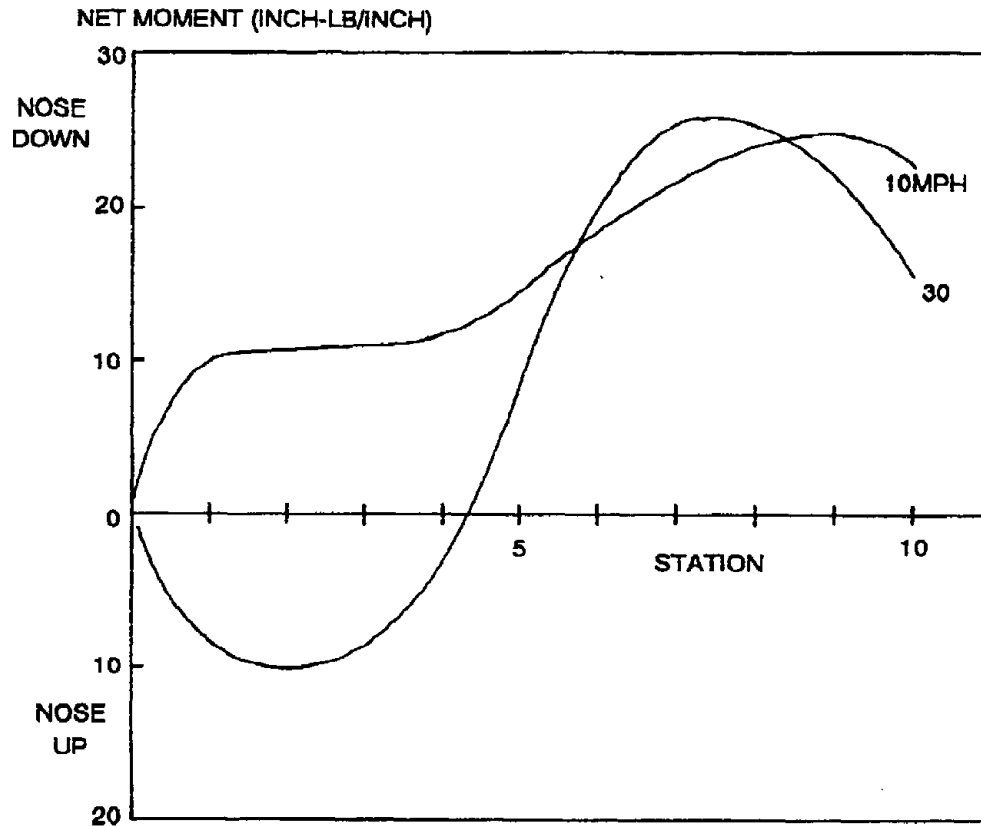


FIG. 6.9 CARTER PITCHING MOMENT DISTRIBUTION(10 AND 30 mph CASES)

30 mph Case: For this case the tip deflections are much greater; this is a load case near the rated output of the turbine, and the tip deflection is over 5 feet. The root flap moment is 10,558 ft-lb; the flap shear is 2990 lb. The root lead-lag shear is 1824 lb, and the inplane moment (which gives rotor torque) is about twice the size of the flap moment. Here is an illustration of the advantage of a flexible blade: this blade is flexible in the flap direction, so the loading is largely relieved by centrifugal force; the inplane stiffness is high, so the inplane deflection is low. Very little root load is relieved.

Surprisingly, the elastic twist for this case is less than at 10 mph. The aerodynamic loads were much less. The reason can be seen in Figure 6.8, the lift force is much greater, thus the lift offset term is large. The flap coupling term is also larger (50.2 in.-lb) but it is practically canceled by the offset term. The net moment distribution can be seen in Fig 6.9; the moment relief due to lift is easily seen.

Effect of Ballast: The effect of removing the ballast on the elastic twist can be seen in Figure 6.10, which shows the net section moment and elastic twist for the same blade but now with no lead ballast outboard. The elastic twist is now nose up rather than nose down as above.

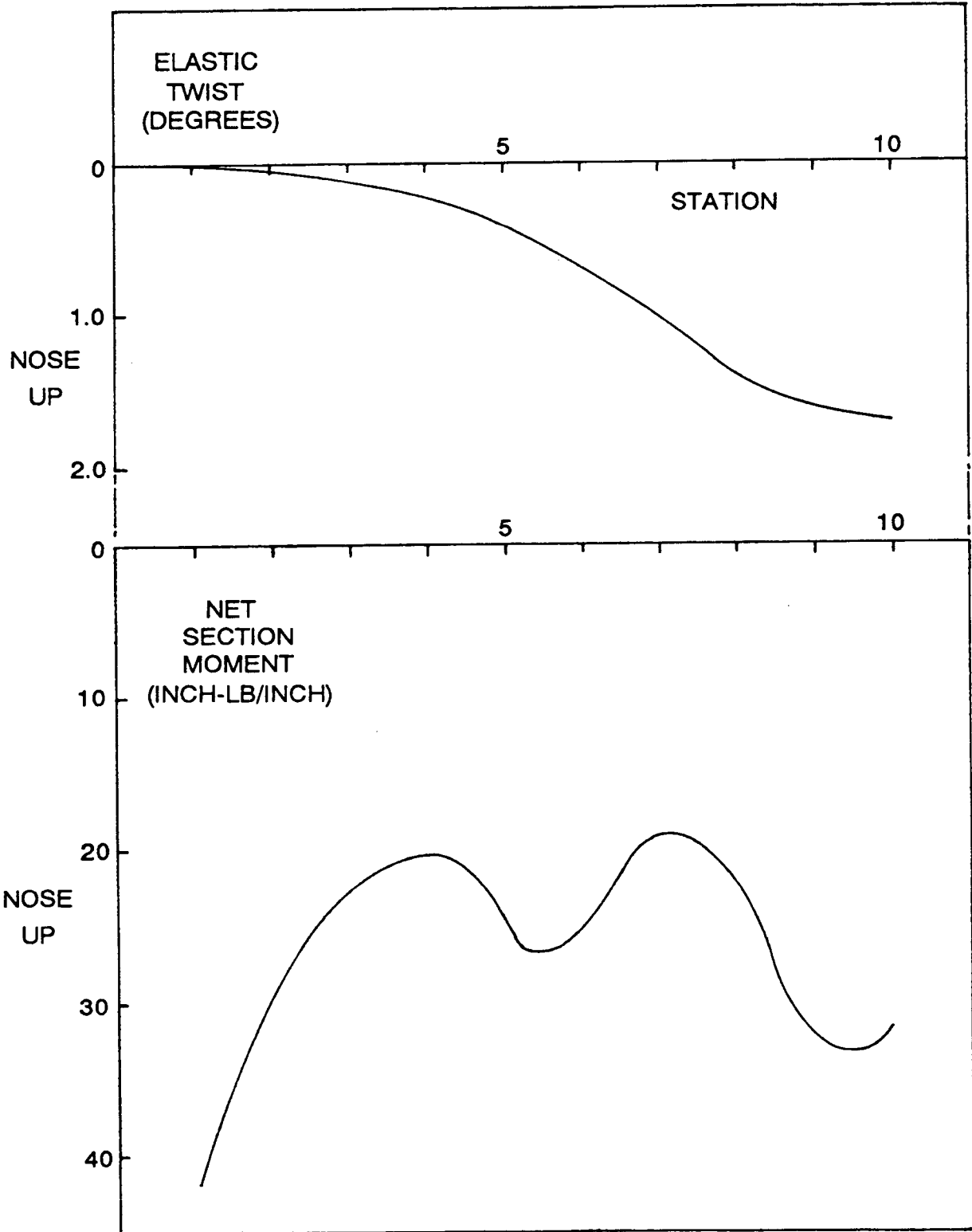


FIG. 6.10 CARTER ELASTIC TWIST AND SECTION MOMENT, NO BALLAST (30 mph)

This is because the centrifugal moment is now much less because the weight is less and the cg is much farther aft (closer to the shear center). Referring to Figure 6.8, the free body diagram for 30 mph looks the same except now the centrifugal moment is only 2.9 in.-lb/in. rather than 50.2 in.-lb/in. (with ballast). With no ballast the nose-up lift offset dominates and the result is nose-up elastic twist.

It is clear that the chordwise position of the ballast can be adjusted (slightly aft) to completely zero the elastic twist, if that becomes the wish of the designer. [Note: this would only be strictly true at one wind speed, since the lift vector changes and the centrifugal moment does not (except for flap deflection contribution)]

Finally, the major designer's reason for the lead ballast can be easily seen in the flap deflection results. Without the ballast this flexible blade, under the 30 mph loading, would have a tip deflection of over 10 feet; by adding the ballast this tip deflection is reduced to 61 in. A secondary effect of reducing the flap deflection is to also reduce the flap coupling term above.

7.0 PERFORMANCE CALCULATIONS AND RESULTS

The last phase of the project used the adjusted twist distributions of the blades to determine the adjusted blade aerodynamic loading and wind turbine performance. We used the current PROPPC.FOR computer code and airfoil C_L/C_D data sets specified by SERI and the consultation on aerodynamic corrections due to airfoil shape errors documented in the first phase. Adjusted rotor performance and loads were compared to unadjusted loads, and the aerodynamic loading set used to generate the elastic twist was adequately verified.

7.1: AERODYNAMIC CALCULATIONS

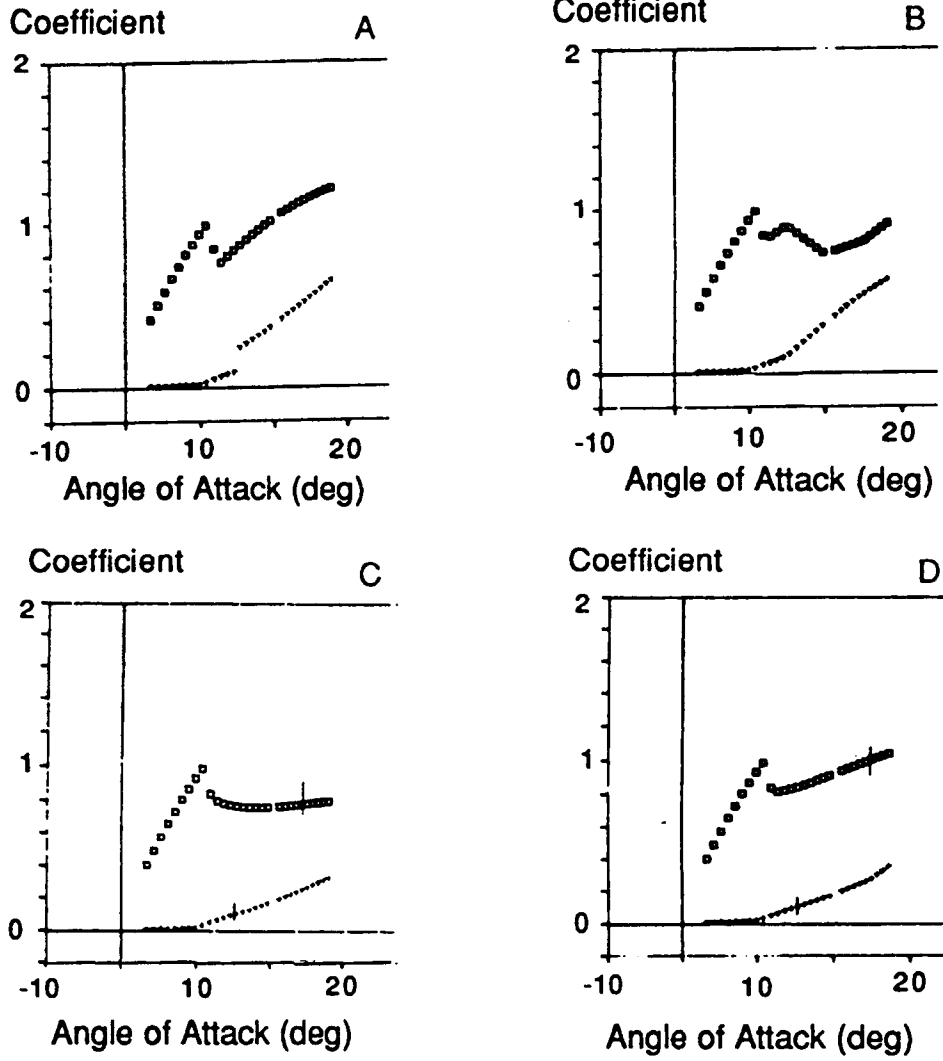
This phase finally applied the elastic twist results to rotor performance calculations to modify the wind turbine performance and assess the effect of elastic twist. Four post-stall airfoil C_L/C_D characteristic options were used, as specified by SERI (Fig 7.1 and Refs. 29-31):

- A. Standard 2-D table data with flat plate stall algorithm.
- B. Standard 2-D table data with NACA 0012 stall values from wind tunnel test.
- C. Viterna post-stall synthesization routine.
- D. Viterna/Tangler post-stall routine.

Each operating case was run for each of these, for each rotor, to determine the baseline performance. The cases were then rerun using the adjusted elastic twist results. Comparison of the post-stall synthesization routines can be seen in Figures 7.2, 7.3, and 7.4. The experimental 2-D lift and drag data which were input to PROPPC are shown in Table 7.1

TABLE 7.1: AIRFOIL LIFT AND DRAG DATA INPUT TO PROPPC

Carter 300:	STA 1-5 NACA LS(1)-0417	STA 6-10 NACA LS(1)-0413
ESI:	STA 1-7 NACA 23024	STA 8 NACA 23021 STA 9 NACA 23018 STA 10 NACA 23015
UTRC 8:	STA 3-10 NACA23112	(most of STA 3 is part of the flexbeam)



□ Lift Coefficient + Drag Coefficient
 KEY: A FLAT Plate, B NACA 0012, C Viterna, D Tangler

FIG. 7.1 2-D SECTION DATA COMPARISON: STANDARD VS. VITERNA VS. TANGLER

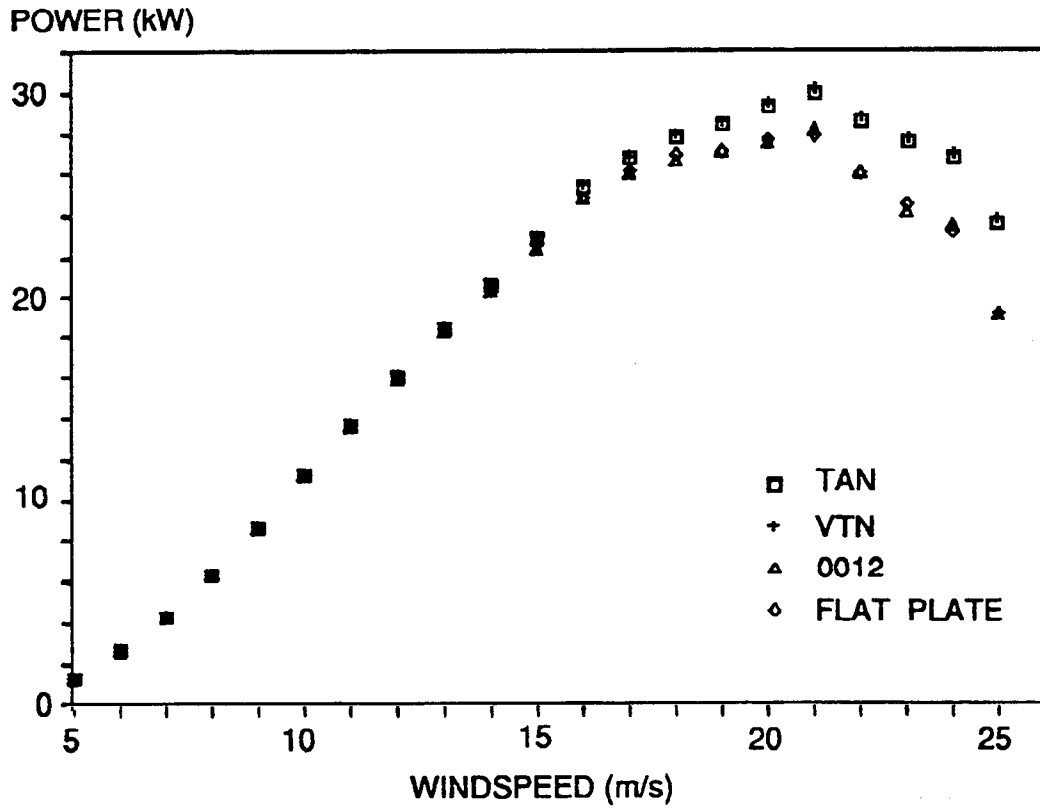


FIG. 7.2 UTRC: COMPARISON OF POST-STALL ROUTINES

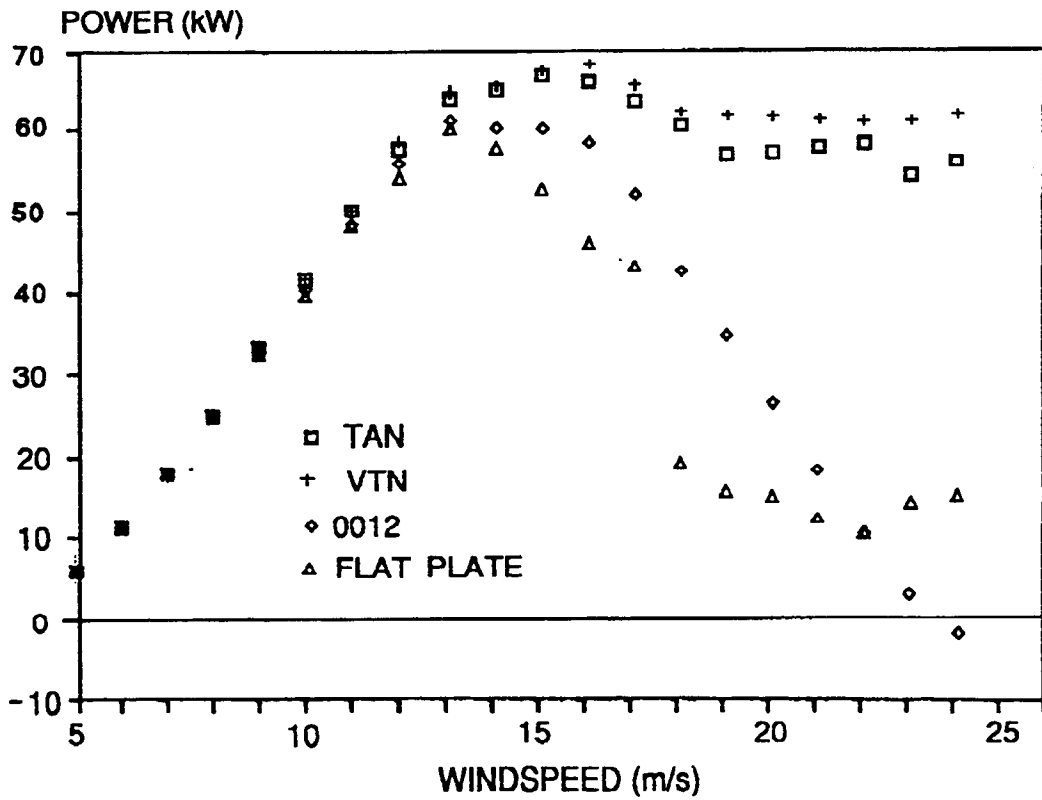


FIG. 7.3 ESI: COMPARISON OF POST-STALL ROUTINES

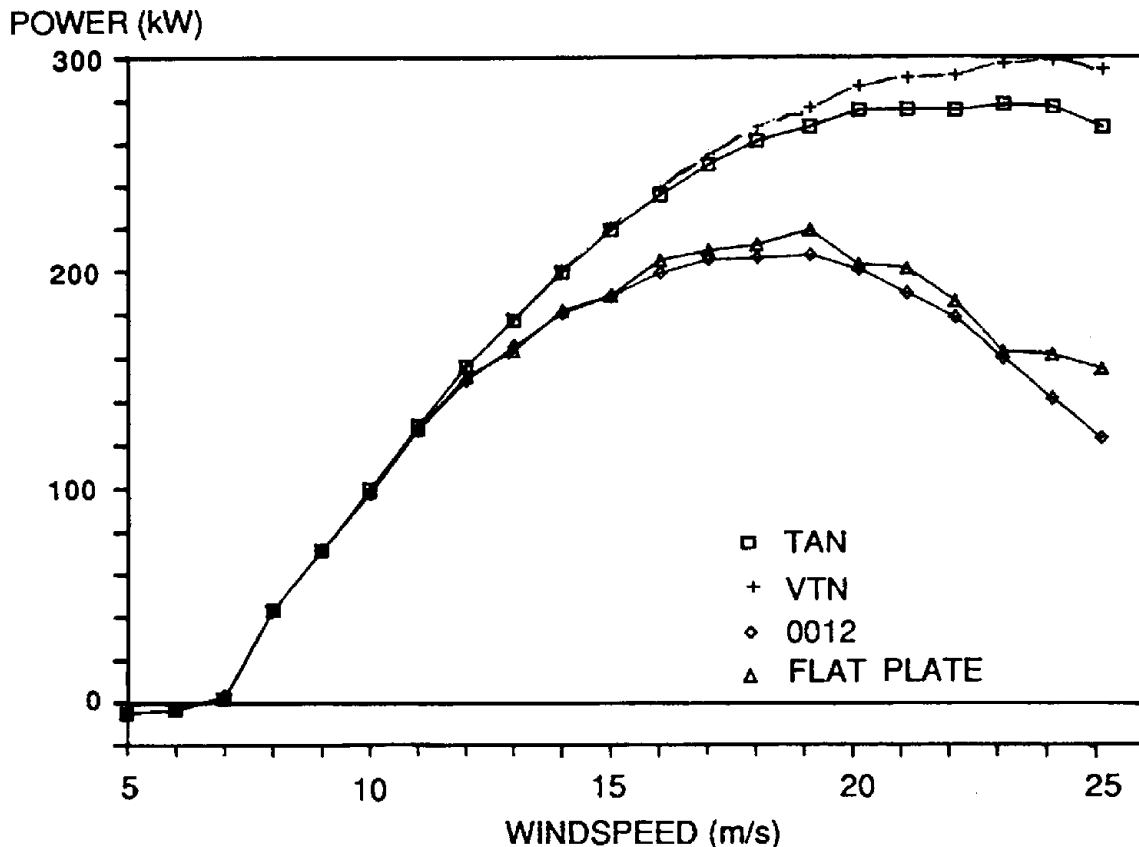


FIG. 7.4 CARTER: COMPARISON OF POST-STALL ROUTINES

For the Carter 300, lift and drag data for the LS(1)-0417 and -0413 were used to approximate the sections, which varied from -0421 at the root to -0413 at the tip. The ESI 54 has a different airfoil (NACA 230XX) at each station, and was approximated by four airfoils. The problem here was that the inboard sections had up to 44% thickness and no experimental data were available; the thickest airfoil in the available literature, 24%, was used. Since the UTRC 8 was a constant chord, zero twist blade (NACA 23112), the lift and drag data were simpler. When the lift and drag data determined by Miley were used, section data from STA 6-10 were replaced for the Carter 300, 7-10 for the ESI, and all sections for the UTRC 8. Lift and drag coefficients from the static experimental data were used corresponding to the Reynolds numbers for all those stations. Tip and hub losses were included. All units were operated at essentially constant rpm.

For each wind turbine, predicted power curves were obtained with the four different post-stall routines: flat plate, Viterna, Tangler, and 0012 experimental data. The output at each station (angles of attack, lift and drag coefficients, thrust, torque moments and power) were obtained for 5, 10, 15, 20, and, 25 m/s.

The Tangler and Viterna methods gave similar power outputs at high wind speeds for all three blades. Both methods predicted higher power than the flat plate and attached 0012 data, which is consistent with power curves obtained in the field. Even though the flat plate and attached 0012 data are included in the graphs, the data should be considered suspect in the post-stall region. None of the curves is smooth at high wind speeds because of the finite number of blade sections allowed in PROPPC. Twenty sections were used for the UTRC 8 and ten for the Carter 300 and ESI 54.

7.2: OPERATIONAL CASES CONSIDERED

The performance results of the elastic twist calculations were finally compared with available field test results. These data were acquired from published sources and manufacturers' literature (see below). The field data were selected by reviewing key accuracy elements in field test such as

- a. Anemometer locations and type (cup vs. prop vane)
- b. Wind turbulence measurements.
- c. Power or load transducers used.
- d. Binning and summing techniques.
- e. Averaging time.
- f. Number of readings taken.
- g. Drivetrain and generator loss model.

The UTRC data came from two sources, a final report from Rocky Flats testing (Ref. 18) and the cumulative data collected at the AEI/Borger Stripper Well site (Ref. 32). The Carter data came from a manufacturer's bulletin describing the Carter 300 (Ref. 33). The ESI data were extracted from a final report by Pacific Wind Energy, Inc. describing the Whiskey Run Windfarm (Refs. 34, 35). The data were all power curves with a known anemometer height and corrected to a standard atmospheric density. All curves were converted to metric units for both wind speed and power output.

7.3: TURBINE POWER CURVE PERFORMANCE RESULTS

Comparisons of field data and the computer models are given in Figures 7.5 - 7.7.

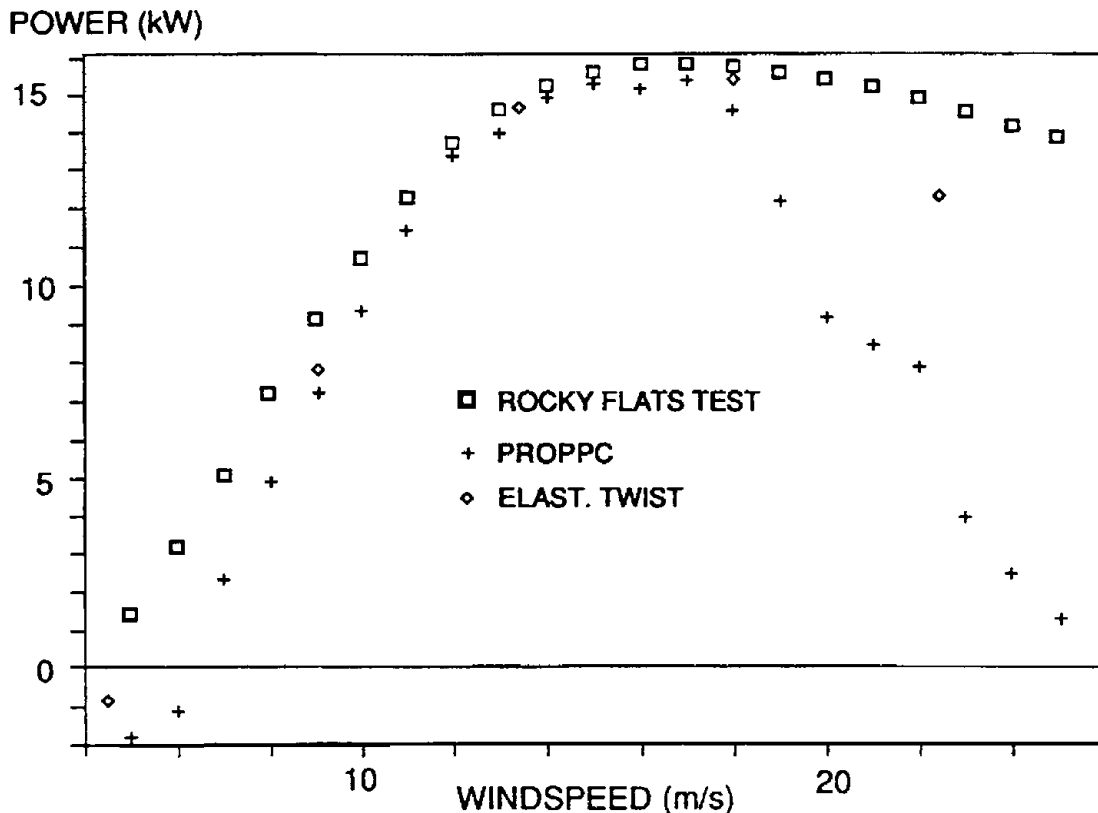


FIG. 7.5 UTRC POWER OUTPUT: COMPARISON OF FIELD TEST, PROPPC PREDICTIONS, AND ELASTIC TWIST

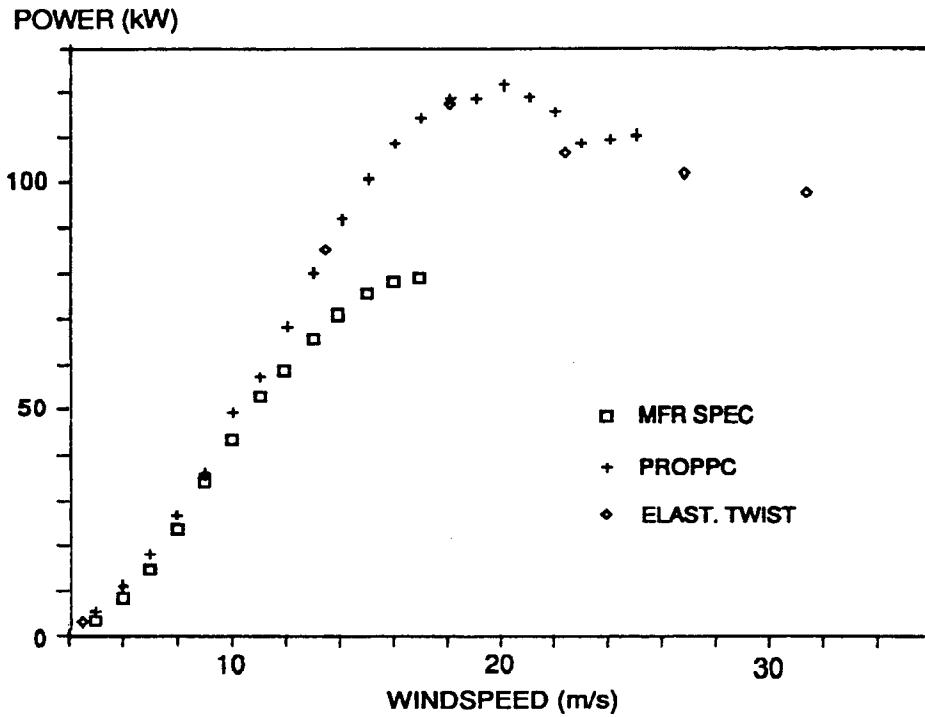


FIG. 7.6 ESI 54 POWER OUTPUT: COMPARISON OF MANUFACTURER DATA, PROPPC PREDICTION, AND ELASTIC TWIST

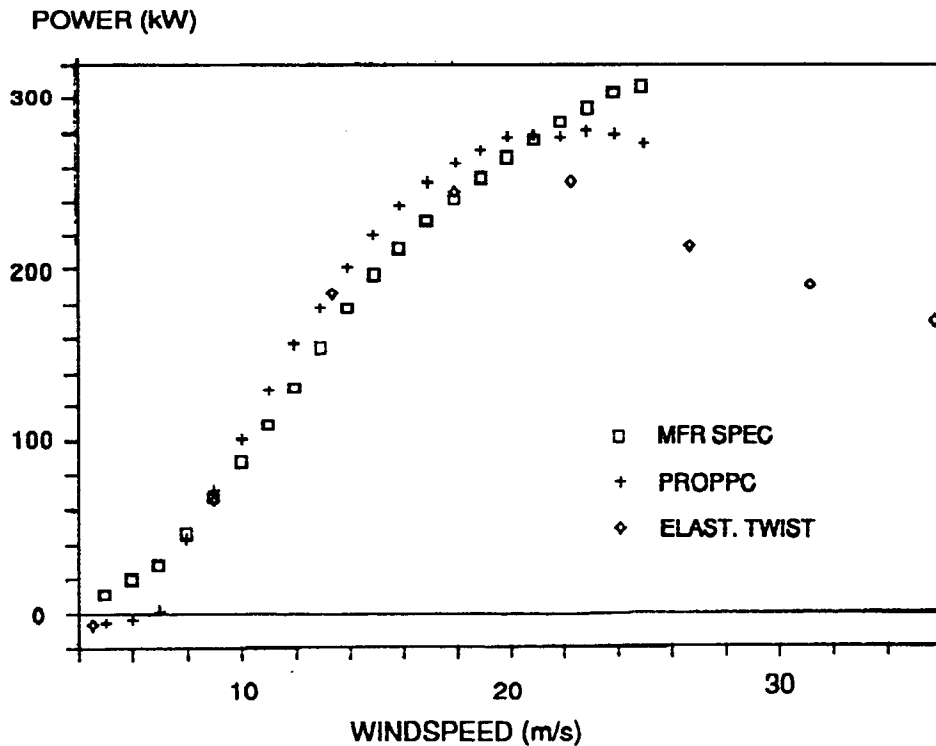


FIG. 7.7 CARTER 300 POWER OUTPUT: COMPARISON OF MANUFACTURER DATA, PROPPC PREDICTION, AND ELASTIC TWIST

7.3.1: UTRC 8 kW BLADE

There is a large difference at high wind speeds for the UTRC 8 as predicted power output is around 30 kW (Fig. 7.5). Also notice that the Viterna and Tangler post-stall calculations predict decreasing power at high wind speeds, in contrast to the fairly flat power curve from atmospheric testing. The lift and drag would have to be changed drastically to reduce the peak power to 15 kW, or the pitch needs to be changed. The lift and drag from the EPPLER Code (Appendix 10.6) over predicts the power output due to optimistic lift curve slopes at medium angles of attack, even at low wind speeds.

Clearly the elastic twist of the UTRC 8 blade affects the power curve at wind speeds above 12 m/s (27 mph), which helps to explain the discrepancy between tests and prediction. The elastic twist is nose down (Table 6.2), reducing angle of attack. The pendulum/flexstrap root attachment was modeled in the study as a constant root pitch of 5° nose up, which is its operating equilibrium at rpm.

7.3.2: ESI 54 S BLADE

Power curves were obtained for two rotor speeds, 77 and 90 rpm (ESI 54S). The power curve from field data for the ESI 54S (90 rpm) is below the predicted output. Data obtained from the EPPLER Code (Appendix 10.6) also over predicts the power output. The lift and drag data were used for STA 7-8 (.75 radius) and STA 9-10 (.95 radius), while STA 1-6 were left unchanged. The large discrepancy at 16 m/s in the smoothness of the Miley curve is from the artificial connection of EPPLER Code data through angles of attack of -4° to 16°, then the experimental data to 20° (same as before), and then the Tangler post-stall method.

Figure 7.6 shows the ESI data for 90 rpm with the regular tip mechanism. The discrepancy between the PROPPC predictions and test data is still quite large. The effect of the elastic twist is negligible among all the cases of 77 and 90 rpm and the two tip mechanisms. Part, but by no means all, of the difference can be attributed to drivetrain and tip drag losses, which were not accounted for in this case.

7.3.3: CARTER 300 BLADE

Predicted power vs. wind speed (Fig. 7.4) shows the characteristic leveling off at high wind speeds for the Tangler and Viterna methods. When the Tangler-predicted output is compared to the manufacturer's data, the field data show higher power output at low and high wind speeds. The predicted power using the lift and drag coefficients obtained from the EPPLER Code agree quite well with the calculated power curve using the experimental data. The Carter 300 blades closely matched the LS (1) airfoil offset specifications.

Figure 7.7 shows the Carter 300 power curve comparisons, the manufacturer's test data, the PROPPC prediction (zero live twist), the prediction with elastic twist, and the power curve that would be obtained with zero ballast weight but including elastic twist. With no ballast weight the elastic twist is large and nose-up (e.g. 2.75° at 27 m/s [60 mph]) since the blade shear center (elastic axis) is at roughly 48% chord, outboard. This results in a large nose-up section moment due to quarterchord (aeroelastic lift) offset. Adding the ballast weight counteracts this with inertial moments resulting in 1.3° nose-down (Table 6.7). The effect of this much smaller elastic twist on the Carter power curve is to reduce the angle of attack at higher wind speeds. Still, in Figure 7.7, this effect cannot be clearly seen against the test curve. Again the drivetrain losses have not been accounted for, and presumably the predictions still do not match the test because of some other aerodynamic effect that is delaying stall and/or enhancing the output near the airfoil 2-D stall angles.

7.4: DISCUSSION

The PROPPC Code output by station shows how much each part of the blade contributes to the performance. All the blades stall progressively from the root as the wind speed increases, even the Carter 300 which has a large twist (approaching a Glauert approximation). Looking more closely at the post-stall methods, some observations can be made.

First, for a given airfoil, different sections (blades) have a range of angles of attack depending on the wind speed. Combining these into a power curve, a smooth curve of lift and drag coefficients is obtained over a wide range of angles of attack, except for the flat plate method.

Second, the lift and drag data are static data; the post-stall modifications are based on an averaging process. If those routines are attached at different terminal angles of attack of the static data, then the predicted power output changes. Both the Tangler and Viterna models should be investigated from angles of attack at the onset of stall rather than only at the end of the data input, which may be a fairer way of comparing stall.

The PROPPC Code is sensitive to the input parameters. Lift and drag data were not available for all the airfoils for the range of Reynolds numbers needed, and also for the thick airfoils at the root. If the actual airfoil is different from the airfoil specified, and/or the pitch or twist is incorrect, then the predicted output will be different from the measured field data, even for the pre-stall region of windspeeds. At 10 m/s, the differences between the predicted (Tangler post-stall) and the atmospheric power are 13% for the Carter 300, 7% for the ESI 54S, and 10% for the UTRC 8. Even if the input data are consistent with the actual blade characteristics, the prediction of power still must be considered suspect in the post-stall region.

8.0 CONCLUSIONS

8.1: COMPARISON OF PREDICTED AND EXPERIMENTAL RESULTS

Power output and blade loading are affected by the live "elastic twist" on flexible wind turbine blades. The aeroelastic analysis method developed during this project predicts substantial live twist which helps explain discrepancies observed between calculated and measured power curves. Still the discrepancies between predictions and test curves on stall-regulated turbines are significant, and presumably are due to some other aerodynamic phenomenon.

8.2: USE OF METHOD FOR ANALYSIS OF COMPOSITE BLADES

A secondary goal of this project was to develop a practical method of analyzing composite blades: one within the means and abilities of most wind turbine blade manufacturers. This necessarily means appropriately practical theoretical methods and complementary bench tests for verification, refinement, and quality control. This process was not straightforward since most composite blades have complex geometry (twist and taper), internal structure (laminate reinforcement and spars), and root attachment.

8.3: USE OF METHOD FOR BLADE QUALITY CONTROL FOR MANUFACTURERS

The experimental method developed at AEI has been presented in detail in this report. A conventional cantilever stand was used for all blades, and linear measurements were referenced to a simple laser-defined baseplane. Section angles were taken with a sensitive but readily obtainable mechanical clinometer, and results indicate a routine achievable accuracy of 0.1 degree. A key part of the method was sectioning a blade to check and verify the internal laminate geometry and ballast locations. For a properly constructed blade, the design specifications may be very close to the bench test. Frequencies and modes of vibration were measured on the test stand with SERI assistance, but even this could have been done in house with better instrumentation.

In summary, the experimental methods developed in the project allow for practical verification of the predicted mass and stiffness properties of the blade, including the static bending deflections, static elastic twist, fundamental frequencies, and modes of vibration in flapping, lead-lag, and torsion. The aeroelastic model thus has the same static deflections, twist, modes, and frequencies of vibration as the actual blade. The blade testing procedure and codes permit practical standardized quality control, verification of blade design codes, and assessment of the static effects of ballast and tip mechanisms. For example, the actual mass and stiffness properties of the UTRC blade differed from the manufacturer's specifications by 21% (on running mass) to 49% (on torsional rigidity), for the ESI and Carter blades the differences were generally less than 10%.

8.4: DESIGN IMPLICATIONS OF RESULTS

The elastic twist is greatly affected, via centrifugal moments, by the addition of ballast or tip weights. However, these effects may be beneficial if used correctly. The methodology used in this study enables designers of composite blades to assess the effect of adding structural elements, ballast, and tip mechanisms. Ultimately this could allow the tailoring of longitudinal and torsional section properties to exploit live twist to either enhance performance (e.g., delay stall) or reduce loading (e.g., promote stall). Also, the elastic axis of a composite blade can be changed dramatically by varying the type and amount of reinforcement, thereby changing the ratio of longitudinal modulus to shear modulus. These features may be used to advantage by FRP blade designers. Progress is being made in Great Britain in power limiting of vertical axis wind turbines by using "mirror" or "spiral" patterns of glass reinforcement (GRP) in just this way (Ref. 36).

8.5: RECOMMENDATIONS FOR FURTHER WORK AND HAWT IMPROVEMENT

Future beneficial work on elastic twist would "close the loop" by recalculating the aerodynamic loads with the elastic twist corrections in place, measuring the actual torsion on operating blades, and parametrically investigating the beneficial effects of live twist, particularly for the Carter-type blade. Other work could examine the calculated laminate flexural and shear strains, which must not be allowed to exceed allowable limits for a specified fatigue life, and which are particularly important at joints of dissimilar structural elements like root hubs. Finally, dynamic elastic twist and its effect on transient loads could be studied.

9.0 REFERENCES

- 1 J. L. Tangler and D. M. Somers, "Status of the Special-Purpose Airfoil Families", Proceedings, AWEA Conference, "Windpower '87", San Francisco, CA, Oct. 1987.
- 2 D. M. Eggleston and F. S. Stoddard, Wind Turbine Engineering Design, Van Nostrand Reinhold, New York, 1987.
- 3 F. S. Stoddard, "Structural Dynamics, Stability and Control of High Aspect Ratio Wind Turbines", UMass Alternative Energy Program, NTIS RFP-3027/67925/3533/79-7, Amherst, MA, 1979.
- 4 L. J. Broutman and H. Kroch, Eds., Modern Composite Materials, Addison-Wesley, New York, 1967.
- 5 G. Lubin, Ed., Handbook of Composites, Van Nostrand Reinhold, New York, 1982.
- 6 Y. Stavsky and N. J. Hoff, "Mechanics of Composite Structures", in Composite Engineering Laminates, A.G.H. Dietz, Ed., MIT Press, Cambridge, MA, 1969.
- 7 J. L. Tangler, "A HAWT Performance Prediction Code for PCs: PROPPC", User's Guide, SERI, Golden, CO, Jan., 1987.
- 8 K. L. Starcher, et al, "Modal Analysis of Six HAWT Blades", Alternative Energy Institute, WTSU, AEI 87-2, Canyon, TX, Sept. 1987.
- 9 L. J. Broutman, "Mechanical Behavior of Fiber-Reinforced Plastics", in Composite Engineering Laminates, op. cit.
- 10 S. Jarosch, "Fatigue Properties and Test Procedures of GRP Plastic Rotor Blades", Proceedings, American Helicopter Society 25th Annual Forum, Washington D.C., May, 1969.
- 11 S. P. Timoshenko and J. M. Goodier, Theory of Elasticity, McGraw-Hill, New York, 1951.
- 12 J. P. Den Hartog, Advanced Strength of Materials, McGraw-Hill, New York, 1952.
- 13 R. Rivello, Theory and Analysis of Flight Structures, Addison-Wesley, New York, 1975.
- 14 J. P. Den Hartog, Mechanical Vibrations, 4th Ed., McGraw-Hill, New York, 1956.
- 15 N. O. Myklestad, "A New Method of Calculating Natural Modes of Uncoupled Bending Vibrations of Airplane Wings and other Types of Beams", Journal of Aeronautical Sciences, 15:5, 1944.
- 16 W. T. Thomsen, Vibration Theory and Applications, Prentice-Hall, 1965.
- 17 M. A. Prohl, "A General Method For Calculating Critical Speeds of Flexible Rotors", ASME Transactions, A-142, Sept. 1945.
- 18 R. B. Taylor and M. C. Cheney, "UTRC 8KW Wind System: Phase II Test Technical Report", USDOE/Rocky Flats RFP-3232/2, Golden, CO, Feb. 1981.
- 19 I. H. Abbott and A. E. Von Doenhoff, Theory of Wing Sections, Dover, New York, 1959.

- 20 J. M. Salkind, et al., "Composite Materials: Testing and Design", ASTM STP497, Amer. Soc. Testing and Materials., 1972.
- 21 R. Monroe, et al., Personal Communications, Calculations and Engineering Drawings for ESI 54 Blade, Gougeon Bros., Inc., Bay City, MI, July 1986.
- 22 M. D. Zuteck, "Development and Manufacture of Wood Composite Wind Turbine Rotors", Proceedings, DOE/NASA Workshop on Large HAWT, NASA Conf-810752, Cleveland, OH, July 1981.
- 23 S. Lieblein, et al., "Design and Evaluation of Low-Cost Laminated Wood Composite Blades for Wind Turbines: Blade Design, Fabrication Concept and Cost Analysis", DOE/NASA/0101-1, NASA CR-165463, Nov. 1982.
- 24 R. F. Lark, et al., "Fabrication of Low-Cost MOD-0A Wood Composite Wind Turbine Blades", DOE/NASA/20320-45, NASA-TM-83323, Feb. 1983.
- 25 R. H. Monroe, "Design and Development of Large Wood/Epoxy Wind Turbine Blades", Proceedings, AWEA Conference, "Windpower '87", San Francisco, CA, Oct. 1987.
- 26 C. P. Butterfield, et al., Personal Communication and Raw Field Test Data, ESI, Inc., Boulder, CO, 1986.
- 27 J. Carter, Jr., Personal Communications and Engineering Calculations, Carter Wind Systems, Burkburnett, TX, 1987.
- 28 A.R.S. Bramwell, Helicopter Dynamics, John Wiley and Sons, New York, 1976.
- 29 L. A. Viterna and R. D. Corrigan, "Fixed Pitch Rotor Performance of Large HAWT", Proceedings, DOE/NASA Workshop on Large HAWT, NASA Conference-810752, Cleveland, OH, July 1981.
- 30 J. L. Tangler and C. Ostowari, "HAWT Post Stall Airfoil Characteristics Synthesization", DOE No. DE-ACO4-76DP03533, Rocky Flats Test Center, Golden, CO, 1984.
- 31 S. F. Hoerner, Fluid-Dynamic Drag, Hoerner Fluid Dynamics, P.O. Box 342, Brick Town, NJ, 08723, 1975.
- 32 E. H. Gilmore, "Windpower to Assist in Stripper (Oil) Well Pumping", AEI/WTSU Report 85-1, Canyon, TX, Feb. 1985.
- 33 D. M. Prossner, "Carter Wind Systems: Performance Results", Carter Wind Systems Press Release, Burkburnett, TX, May 1985.
- 34 "ESI 54S Manufacturer's Specifications", ESI Inc., Boulder, CO, Aug. 1983.
- 35 S. N. Walker and J. E. Wade., "Effects of Precipitation on Wind Turbine Performance", Pacific Wind Energy, SERI/STR-217-3287, March 1988.
- 36 N. M. Karaolis, et al., "Passive Aerodynamic Control Using Composite Blades", Proceedings, BWEA Workshop on use of Composite Materials for Wind Turbines, ETSU-N-109, Harwell, UK, Nov. 1987.

10.0 APPENDICES

10.1: COMPUTER CODE: DETERMINATION OF COMPOSITE BLADE SECTION PROPERTIES

10.1.1: BRIEF DISCUSSION

The computer program used to calculate the blade section properties uses the algorithms explained in Chapter 4. In summary, the program has the following general characteristics:

- Uses unidirectional composite elasticity theory.
- Has provision for arbitrary skin, spar, shell, shear web, and afterbody geometry and laminate makeup.
- Has provision for addition of ballast weights or other nonstructural elements.
- Uses either the actual laminate thickness or the weighted-area method for determining blade section values.
- Employs second-order curve-fit routines to calculate 33 differential section element contributions and spanwise integration over 10 span stations.
- Provides the following section properties:
 1. principal axes.
 2. moments and products of inertia.
 3. polar moments of inertia.
 4. bending centroid (area centroid).
 5. torsional centroid (shear center).
 6. section bending stiffness.
 7. section torsional rigidity.
 8. mass center.

10.1.2 PROGRAM LISTING, STRESSM

The program STRESSM is listed here in its entirety. This version is intended for static loading conditions such as a laboratory bench test. Part 1 contains the beam section calculations and Part 2 the deflection calculations. Following that is a summary section and the aeroelastic results. The working of the program is fully explained in the comment lines, as are the input and output files, forms and units. In the elastic twist analysis programs, the deflection calculations of Part 2 and the summaries at the end differ; these are explained below in further Appendices (STRESSDATA and STRESSELASTIC).

```
C PROGRAM ELASTIC TWIST
C
C .....
C ..... STRESSM .....
C ..... JAN 1988 .....
C .....
C ..... ALTERNATIVE ENERGY INSTITUTE .....
C ..... WEST TEXAS STATE UNIVERSITY .....
C ..... CANYON TX 79016 806-656-2295 .....
C .....
C
C THIS PROGRAM CALCULATES THE SECTION MASS AND STIFFNESS PROPERTIES
C OF AN ARBITRARY EXTERNAL GEOMETRY (TWIST, TAPER) AND SPECIFIED
C INTERNAL SPAR-SHELL STRUCTURAL GEOMETRY, COMPOSITE WIND TURBINE
C BLADE, AND THEN FINDS THE DEFLECTIONS AND STRAINS IN THE LAMINATE
C FOR A SPECIFIED EXTERNAL LOAD.
C
C THIS VERSION OF THE PROGRAM ("STRESSM") INCLUDES ALL THE BENDING
C AND TORSIONAL CALCULATIONS, AND IS INTENDED FOR STATIC LOADING
C CONDITIONS SUCH AS FOR A LABORATORY BENCH TEST.
C
C THE PROGRAM WAS ORIGINALLY WRITTEN BY E.VAN DUSEN (COMPOSITE
C ENGINEERING, INC., 277 BAKER AVE., CONCORD MA 01742 617-371-3132)
C FOR COMPOSITE RACING SHELLS AND HAS BEEN EXTENSIVELY REWRITTEN BY
C F.S. STODDARD (AEI, BOX 248 WTSU, CANYON TX 79016) FOR WIND TURBINE
C BLADES.
C
C .....
C .....
C PROGRAM DESCRIPTION:
C
C PART 1 OF THIS PROGRAM CALCULATES THE SECTION PROPERTIES OF AN
C ARBITRARY COMPOSITE BEAM, USING EITHER ACTUAL THICKNESSES AND
C MODULI OF PLIES, OR USING MODULUS-WEIGHTING (THE LATTER ROUTINE
C HAS BEEN SUPPRESSED FOR THE CALCULATION OF ELASTIC TWIST). THE
C BEAM (BLADE) IS MODELED BY A SHELL OF CONSTANT THICKNESS WITH
C ARBITRARY LONGITUDINAL MODULUS AND SHEAR MODULUS, AND A SPAR
C CONSISTING OF A NOSE SECTION EMBEDDED NEXT TO THE SHELL IN THE
C BLADE LEADING EDGE, AND A WEB (OR WEBS) LOCATED INTERNALLY. THE
C SPAR ALSO HAS CONSTANT THICKNESS AND ARBITRARY LONGITUDINAL AND
C SHEAR MODULI. THE WEB IS DESCRIBED ONLY BY ITS MOMENTS OF INERTIA
C AND CENTROID IN THE SECTION AXIS SYSTEM. MULTI-WEB BEAM DESIGNS ARE
C HANDLED BY INCLUDING ALL THE ACTUAL WEBS INTO THIS ONE MODEL WEB.
C
C THE SECTION AIRFOIL SHAPES ARE DETERMINED BY SCALING AN INPUT
C TABLE OF OFFSETS. THE CHORD SCALING FACTOR (SCALE) DETERMINES THE
C AIRFOIL CHORD AND THE THICKNESS SCALING FACTOR (TSCALE) DETERMINES
C THE SECTION THICKNESS PROPORTIONAL TO THE TABLE OF OFFSETS.
C ALL THESE INPUT DATA ARE INCLUDED IN A DATAFILE GIVING THE GEOMETRY
C AND STRUCTURAL DESIGN OF THE BLADE OF INTEREST.
C
C IN PART 1 PARABOLIC SEGMENTS ARE FITTED THROUGH EACH GROUP OF THREE
C SECTION OFFSET POINTS. THE I OF THE SKIN+SPAR+WEB IS CALCULATED BY
C DIVIDING THE GROUP OF POINTS INTO (DIV) STRAIGHT LINE SEGMENTS OF
C ROUGHLY EQUAL SIZE AND SUMMING THE APPROPRIATE VARIABLES.
C THE SECTION DETAILS ARE PRINTED FOR EACH STATION, INCLUDING THE ACTUAL
C AIRFOIL OFFSETS WHICH CAN BE USED FOR TEMPLATE CONSTRUCTION.
```



```

GTHK1 = TSKIN + TSPAR * (GL/GS)
GTHK2 = TSKIN
C
131 CONTINUE
C
C -----PRINT OUT RESULTS-----
C
PRINT 930,IST,THK1,THK2,XTHK,TSPAR,TSKIN,TSCALE(IST)
PRINT 925, (X(I),I=1,NPT)
PRINT 925, (Y(I),I=1,NPT)
TXSM = 0
TXSMS = 0
TXSML = 0
TYSM = 0
TYSMS = 0
TYSML = 0
TXSI = 0
TXSIS = 0
TXSIL = 0
TYSI = 0
TYSIS = 0
TYSIL = 0
TXYSI = 0
TXYSIS = 0
TXYSIL = 0
TAREA = 0
TSA = 0
TLA = 0
TAA = 0
TDST = 0
C
J1 = NPT - 2
C
DO 340 J = 1,J1,2
C
C -----
C *****CALCULATION OF SKIN MOMENT OF INERTIA AND CENTROID*****
C *****FROM CHORD LINE*****
C
CALL LCOE (COE1,COE2,COE3,J)
C
C -----SELECTION OF DIVISIONS BY DIVIDING A STRAIGHT-----
C -----LINE BETWEEN POINTS INTO EQUAL SEGMENTS-----
C
DX1 = (X(J+1)-X(J))/DIV
DX2 = (X(J+2)-X(J+1))/DIV
I5 = DIV + 1
DO 210 ID = 1,I5
XC(ID) = X(J) + (ID-1) * DX1
XC(DIV + ID) = X(J+1) + (ID-1) * DX2
210 CONTINUE
I3 = 2 * DIV + 1
DO 230 ID = 1,I3
YC(ID) = COE1 * ((XC(ID)**2) + COE2 * XC(ID) + COE3
YC(ID+DIV) = COE1 * ((XC(ID+DIV)**2) + COE2 * XC(ID+DIV) + COE3
230 CONTINUE
TA = 0
TSKINA = 0
TSPARA = 0
XSM = 0
XSMS = 0
XSML = 0
YSM = 0
YSMS = 0
YSML = 0
XSI = 0
XSIS = 0
XSIL = 0
YSI = 0
Y SIS = 0
YSIL = 0
XYSI = 0
XYSIS = 0
XYSIL = 0
SDST = 0
SDA = 0
C
C -----CORRECT FOR SHELL THICKNESS; OUTER SKIN GEOMETRY IS-----
C -----GIVEN; XLEV,YLEV ARE COORDINATES OF MIDPOINT OF DAREA---
C
DO 305 ID = 2,I3
XLV = (XC(ID-1) + XC(ID))/2.0
YLV = (YC(ID-1) + YC(ID))/2.0
C
C -----CHANGE VALUES IF SPAR HAS ENDED-----
C -----NOTE: THIS VERSION OF THE PROGRAM USES THE ACTUAL SKIN AND

```

```

C -----SPAR THICKNESSES FOR CALCULATION OF THE SECTION PROPERTIES
AND
C -----ELASTIC TWIST SINCE THE E-MODULUS AND G-MODULUS MUST BE
C -----INDEPENDENT. TO USE MODULUS-WEIGHTING FOLLOWING STATEMENTS
C -----MUST BE CHANGED, EG. THK, GTHK, ETC.
C
IF ((XLV-XTHK) .GE. 0.0) GO TO 250
C
THK = THK1
GTHK = GTHK1
THK = TSKIN + TSPAR
GTHK = TSKIN + TSPAR
TSPARI = TSPAR
GO TO 260
250 CONTINUE
THK = THK2
GTHK = GTHK2
THK = TSKIN
GTHK = TSKIN
TSPARI = 0
260 CONTINUE
C
DYC = YC(ID) - YC(ID-1)
DXC = XC(ID) - XC(ID-1)
HYP = SQRT (DYC ** 2 + DXC ** 2)
DX = THK * DYC / (2.0 * HYP)
DY = THK * DXC / (2.0 * HYP)
XLEV = XLV + DX
YLEV = YLV - DY
DAREA = THK * HYP
DSAREA = TSKIN * HYP
DLAREA = TSPARI * HYP
DST = HYP / GTHK
DA = DXC * YLV
C
C -----CALCULATE TOTAL MODULUS-WEIGHTED AREA-----
C
TA = TA + DAREA
C
C -----CALCULATE TOTAL SKIN AREA-----
C
TSKINA = TSKINA + DSAREA
C
C -----CALCULATE TOTAL SPAR AREA-----
C
TSPARA = TSPARA + DLAREA
C
C -----FIRST MOMENT ABOUT X-AXIS-----
C
XSM = XSM + YLEV * DAREA
XSMS = XSMS + YLEV * DSAREA
XSML = XSML + YLEV * DLAREA
C
C -----FIRST MOMENT ABOUT Y-AXIS-----
C
YSM = YSM + XLEV * DAREA
YSMS = YSMS + XLEV * DSAREA
YSML = YSML + XLEV * DLAREA
C
C -----X MOMENT OF INERTIA-----
C
XSI = XSI + DAREA * (YLEV ** 2)
XSIS = XSIS + DSAREA * (YLEV**2)
XSIL = XSIL + DLAREA * (YLEV**2)
C
C -----Y MOMENT OF INERTIA-----
C
YSI = YSI + DAREA * (XLEV ** 2)
YSIS = YSIS + DSAREA * (XLEV**2)
YSIL = YSIL + DLAREA * (XLEV**2)
C
C -----PRODUCT OF INERTIA-----
C
XYSI = XYSI + DAREA * XLEV * YLEV
XYSIS = XYSIS + DSAREA * XLEV * YLEV
XYSIL = XYSIL + DLAREA * XLEV * YLEV
C
C -----CALCULATE SHEAR FLOW INTEGRALS FOR SKIN ---
C
SDST = SDST + DST
SDA = SDA + DA
305 CONTINUE
C
C -----CALCULATE RUNNING TOTALS-----
C
C -----AREAS---

```

```

C
TAREA = TAREA + TA
TSA = TSA + TSKINA
TLA = TLA + TSPARA
C
-----1ST MOMENT OF AREA ABOUT X-AXIS---
C
TXSM = TXSM + XSM
TXSMS = TXSMS + XSMS
TXSML = TXSML + XSML
C
-----1ST MOMENT OF AREA ABOUT Y-AXIS----
C
TYSM = TYSM + YSM
TYSMS = TYSMS + YSMS
TYSML = TYSML + YSML
C
-----X-MOMENT OF INERTIA--
C
TXSI = TXSI + XSI
TXSIS = TXSIS + XSIS
TXSIL = TXSIL + XSIL
C
-----Y-MOMENT OF INERTIA--
C
TYSI = TYSI + YSI
TYSIS = TYSIS + YSIS
TYSIL = TYSIL + YSIL
C
-----PRODUCT OF INERTIA--
C
TXYSI = TXYSI + XYSI
TXYSIS = TXYSIS + XYSIS
TXYSIL = TXYSIL + XYSIL
C
-----SHEAR FLOW INTEGRALS--
C
TAA = TAA + SDA
TDST = TDST + SDST
340 CONTINUE
C
*****
C
**CALCULATE SECTION VALUES BASED ON MODULUS-WEIGHTING*****
C
-----MODULUS-WEIGHTED BENDING CENTROID----
C
SXB = TYSM / TAREA
SYB = TXSM / TAREA
C
-----MODULUS-WEIGHTED E'S-----
C
EXSI = TXSI * EL
EYSI = TYSI * EL
C
-----READ WEB INFORMATION - CROSS-SECTIONAL AREA (AL)-----
C
-----IX,IY ALL ABOUT WEB CENTROID, XL,YL-----
C
READ (5,*) AL,XLI,YLI,Xyli,YL,YL
C
-----PRINT OUT THESE NUMBERS-----
C
PRINT 940,AL,XLI,YLI,Xyli,YL,YL
C
-----1ST AREA MOMENTS OF WEB-----
C
XLM = AL * YL
YLM = AL * XL
C
-----E'S OF WEB-----
C
EXLI = EL * XLI
EYLI = EL * YLI
C
-----CALCULATION OF CENTROID AND EI FOR TOTAL COMPOSITE
STRUCTURE
C
EA = EL * TAREA + EL * AL
EA1(IST) = EA
C
-----TOTAL COMPOSITE STRUCTURE BENDING CENTROID BASED ON
-----MODULUS-WEIGHTING-----
C
XB = (EL * TYSM + EL * YLM) / EA
YB = (EL * TXSM + EL * XLM) / EA
YCB(IST) = XB

```

```

ZCB(IST) = YB
C
-----FIND EIX,EIY,EIXY FOR TOTAL STRUCTURE ABOUT ITS CENTROID
----- (XB,YB) BY USING THE PARALLEL AXIS THEOREM IN SEQUENCE,
C
-----WORKING BACKWARDS-----
C
EXI = EXSI+EXLI-EL*TAREA*(SYB**2-(SYB-YB)**2)+EL*AL*((YL-YB)**2)
EYI = EYSI+EYLI-EL*TAREA*(SXB**2-(SXB-XB)**2)+EL*AL*((XL-XB)**2)
EXYI = TXYSI*EL+XYLI*EL-EL*TAREA*(SXB*SYB-(SXB-XB)*(SYB-YB))
+ AL*EL*(XL-XB)*(YL-YB)
C
-----FIND PRINCIPAL AXIS OF INERTIA BY USING MOHR'S CIRCLE--
C
SUM = (EXI+EYI)/2.0
DIF = (EYI-EXI)/2.0
PP = ATAN(EXYI/DIF)
PHI = PP/2.0
PANG(IST) = PHI
RAD = SQRT(EXYI**2 + DIF**2)
C
----PRINCIPAL EIX AND EIY----
C
PXI = SUM - RAD
PYI = SUM + RAD
EIXP(IST) = PXI
EIZP(IST) = PYI
C
-----PRINCIPAL RADIUS OF GYRATION-----
C
PRX = SQRT(PXI/EA)
PRY = SQRT(PYI/EA)
C
-----PRINT OUT RESULTS-----
C
PRINT 945,XB,YB,TAREA,PHI,PXI,PYI,PRX,PRY,EA
C
-----CALCULATE POLAR MOMENT OF INERTIA: J = IX + IY-----
C
-----IF G-SKIN & E-SKIN ARE DIFFERENT THE TORSIONAL CENTROID--
C
-----WILL DIFFER FROM THE BENDING CENTROID-----
C
GA = GS * TAREA*(ES/EL) + GL * AL
C
-----CALCULATE SHEAR CENTER (TORSIONAL CENTROID) BASED ON
-----MODULUS-WEIGHTED VALUES-----
C
XBT = (GS * TYSM + GL * YLM) / GA
YBT = (GS * TXSM + GL * XLM) / GA
YCT(IST) = XBT
ZCT(IST) = YBT
C
-----CALCULATE TORSIONAL STIFFNESS-----
C
GJ1N = GS*TXSI+GL*XLI-GS*TAREA*(ES/EL)*(SYB**2-(SYB-YBT)**2) +
+ GL*AL*(YL-YBT)**2 + GS*TYSI + GL*YLI - GS*TAREA*(ES/EL)*
+ (SXB**2-(SXB-XBT)**2) + GL*AL*((XL-XBT)**2)
GJ1(IST) = GJ1N
C
-----CALCULATE ALTERNATE TORSIONAL STIFFNESS BASED ON SHEAR FLOW--
C
GJ2N = GS* 4.0 * (TAA**2) / TDST
GJ2(IST) = GJ2N
C
-----PRINT FURTHER RESULTS-----
C
PRINT 950,XBT,YBT,GJ1N,GJ2N
C
PRINT ' , '
PRINT ' , '
C
*****
C
**CALCULATE SECTION VALUES BASED ON ACTUAL THICKNESSES*****
C
-----SKIN AND SPAR BENDING CENTROIDS-----
C
SXBS = TYSMS / TSA
SXBL = TYSML / TLA
SYBS = TXSMS / TSA
SYBL = TXSML / TLA
C
-----CALCULATION OF CENTROID AND EI FOR TOTAL COMPOSITE STRUCTURE
C
EA = ES * TSA + EL * TLA + EL * AL
EA1(IST) = EA
C
-----TOTAL COMPOSITE STRUCTURE BENDING CENTROID BASED ON
-----ACTUAL THICKNESSES-----

```

```

C
XB = ( ES * TYSMS + EL * TYSML + EL * YLM ) / EA
YB = ( ES * TXSMS + EL * TXSML + EL * XLM ) / EA
YCB(IST) = XB
ZCB(IST) = YB

C
-----FIND EIX,EIY,EIXY FOR TOTAL STRUCTURE ABOUT ITS CENTROID
C
----- (XB,YB) BY USING THE PARALLEL AXIS THEOREM IN SEQUENCE,
C
-----WORKING BACKWARDS, FOR THE SKIN, SPAR, AND WEB-----
C
EXI = ES * TXSIS - ES * TSA * (SYBS**2) + ES * TSA * ((SYBS
+
- YB)**2) + EL * TXSIL - EL * TLA * (SYBL**2) + EL * TLA
+
* ((SYBL - YB)**2) + EL * XLI + EL * AL * ((YL - YB)**2)
EYI = ES * TYSIS - ES * TSA * (SXBS**2) + ES * TSA * ((SXBS
+
- XB)**2) + EL * TYSIL - EL * TLA * (SXBL**2) + EL * TLA
+
* ((SXBL - XB)**2) + EL * YLI + EL * AL * ((XL - XB)**2)
EXYI = ES * TXYSIS - ES * TSA * SYBS * SXBS + ES * TSA
+
* (SYBS - YB) * (SXBS - XB) + EL * TXYSIL - EL * TLA
+
* SYBL * SXBL + EL * TLA * (SYBL - YB) * (SXBL - XB)
+
+ EL * XYLI + EL * AL * (XL - XB) * (YL - YB)

C
-----FIND PRINCIPAL AXES OF INERTIA BY USING MOHR'S CIRCLE---
C
SUM = (EXI+EYI)/2.0
DIF = (EYI-EXI)/2.0
PP = ATAN(EXYI/DIF)
PHI = PP/2.0
PANG(IST) = PHI
RAD = SQRT(EXYI**2 + DIF**2)

C
----PRINCIPAL EIX AND EIY----
C
PXI = SUM - RAD
PYI = SUM + RAD
EIYP(IST) = PXI
EIZP(IST) = PYI

C
----PRINCIPAL RADIUS OF GYRATION----
C
PRX = SQRT(PXI/EA)
PRY = SQRT(PYI/EA)

C
-----PRINT OUT RESULTS-----
C
PRINT 946,XB,YB,TSA,TLA,PHI,PXI,PYI,PRX,PRY,EA

C
-----CALCULATE POLAR MOMENT OF INERTIA: J = IX + IY-----
C
-----IF G-SKIN & E-SKIN ARE DIFFERENT THE TORSIONAL CENTROID--
C
-----WILL DIFFER FROM THE BENDING CENTROID-----
C
-----IT WILL ALSO INCLUDE CONTRIBUTIONS FROM SKIN,SPAR&WEB--
C
GA = GS * TSA + GL * TLA + GL * AL

C
----CALCULATE SHEAR CENTER (TORSIONAL CENTROID) BASED ON
C
----ACTUAL THICKNESSES OF LAMINATES----
C
XBT = ( GS * TYSMS + GL * TYSML + GL * YLM ) / GA
YBT = ( GS * TXSMS + GL * TXSML + GL * XLM ) / GA
YCT(IST) = XBT
ZCT(IST) = YBT

C
-----CALCULATE TORSIONAL STIFFNESS-----
C
GJ1N = GS * TXSIS - GS * TSA * (SYBS**2) + GS * TSA * ((SYBS
+
- YBT)**2) + GL * TXSIL - GL * TLA * (SYBL**2) + GL
+
* TLA * ((SYBL - YBT)**2) + GL * XLI + GL * AL * ((YL
+
- YBT)**2) + GS * TYSIS - GS * TSA * (SXBS**2) + GS
+
* TSA * ((SXBS - XBT)**2) + GL * TYSIL - GL * TLA
+
* (SXBL**2) + GL * TLA * ((SXBL - XBT)**2) + GL * YLI
+
+ GL * AL * ((XL - XBT)**2)
GJ1(IST) = GJ1N

C
-----PRINT FURTHER RESULTS-----
C
PRINT 950,XBT,YBT,GJ1N,GJ2N

C
-----MASS CALCULATIONS-----
C
-----SECTION MASS CALCULATIONS CAN NOW BE DONE USING THE ACTUAL
C
-----THICKNESSES-----
C
-----TOTAL RUNNING WEIGHT IS THE SUM OF ALL COMPONENTS-----

WA = WS * TSA + WL * TLA + WL * AL
WA1(IST) = WA

C
-----1ST WEIGHT MOMENTS-----
C
XBG = ( WS * TYSMS + WL * TYSML + WL * YLM ) / WA
YBG = ( WS * TXSMS + WL * TXSML + WL * XLM ) / WA
YGB(IST) = XBG
ZGB(IST) = YBG

C
-----SECTION MASS MOMENTS OF INERTIA ABOUT ORIGIN (NOSE)---
C
IXMO = WS * TXSIS + WL * TXSIL + WL * (XLI + AL * (YL**2))
IYMO = WS * TYSIS + WL * TYSIL + WL * (YLI + AL * (XL**2))
IYO(IST) = IXMO
IZO(IST) = IYMO

C
-----MASS MOMENTS NOW MOVED TO THE SECTION CG (XBG,YBG)----
C
-----NOTE: THIS VALUE WILL BE THE MINIMUM VALUE OF J ---
C
IXMG = IXMO - ( WA * (YBG**2) )
IYMG = IYMO - ( WA * (XBG**2) )
IYG(IST) = IXMG
IZG(IST) = IYMG

C
-----ALSO MOVED TO THE ELASTIC AXIS (XBT,YBT)-----
C
IXMT = IXMG + WA * ( (YBT - YBG) **2)
IYMT = IYMG + WA * ((XBT - XBG)**2)
IYT(IST) = IXMT
IZT(IST) = IYMT

C
-----THE SECTION MASS MOMENTS OF INERTIA (PER UNIT LENGTH) ARE NOW
C
-----THE SUMS OF IX, IY, FOR BOTH THE ELASTIC AXIS AND SECTION CG---
C
IBMT = IXMT + IYMT
IBMG = IXMG + IYMG
IMT(IST) = IBMT
IMG(IST) = IBMG

C
-----PRINT SECTION MASS RESULTS-----
C
PRINT 955,WL,WS,WA,XBG,YBG,IBMT,IBMG

C
-----READ BALLAST INFORMATION: RUNNING BALLAST WEIGHT---
C
-----IN LB/INCH AND CHORDWISE POSITION---
C
READ (5,*) WB1,XBAL1,YBAL1

C
WB(IST) = WB1
YBAL(IST) = XBAL1
ZBAL(IST) = YBAL1

C
-----TOTAL RUNNING WEIGHT IS THE SUM---
C
WT1 = WA + WB1
WT(IST) = WT1

C
-----1ST WEIGHT MOMENTS: NEW CG POSITION WITH BALLAST---
C
XBGB = (( WA * XBG ) + (WB1 * XBAL1)) / WT1
YBGB = ((WA * YBG) + (WB1 * YBAL1)) / WT1
YGBB(IST) = XBGB
ZGBB(IST) = YBGB

C
-----NOW CALCULATE NEW MASS MOMENT OF INERTIA ABOUT OLD CG
C
-----USING THE PARALLEL AXIS THEOREM-----
C
IMBB = IBMG + WB1 * ((XBG-XBAL1)**2 + (YBG-YBAL1)**2)

C
-----NOW MOVE TO NEW CG THEREBY FINDING NEW MINIMUM
C
-----SECTION MASS MOMENT OF INERTIA, INCLUDING BALLAST
C
IMBG = IMBB - WT1 * ((XBG-XBGB)**2 + (YBG-YBGB)**2)
IMGB(IST) = IMBG

C
-----PRINT NEW SECTION MASS RESULTS-----
C
PRINT 956, WB1,XBAL1,YBAL1,WT1,XBGB,YBGB,IMBB,IMBG
PRINT ' '
PRINT ' '

C
570 CONTINUE
C

```

```

C *****
C **** P A R T 2 : ELASTIC TWIST AND DEFLECTION PART OF PROGRAM **
C *****
C -----READ THE HEADING FOR THE BLADE RUN-----
C
C READ(5,994) HEAD2
C
C -----PRINT 1ST LINES OF OUTPUT-----
C
C PRINT *, *****
C PRINT *, *****
C PRINT 994,HEAD2
C PRINT 998,HEADING,TCORR,PITCH1
C PRINT 910,STA,ES,EL,DIV,GS,GL,WS,WL
C PRINT *,'
C PRINT 964
C PRINT 965,(I,I=1,STA)
C PRINT 973
C PRINT 970, 'STA POS',STX
C PRINT 970, 'X-LOAD',LX
C PRINT 970, 'Y-LOAD',LY
C PRINT 970, 'Z-LOAD',LZ
C PRINT 992, 'EA',EA1
C PRINT 992, 'EYP',EYP
C PRINT 992, 'EIZP',EIZP
C PRINT 992, 'GJ',GJ2
C PRINT 970, 'SCALE',SCALE
C PRINT 970, 'TSCALE',TSCALE
C PRINT 973
C PRINT *,'
C PRINT 996, (I,I=1,NSO)
C PRINT 972
C PRINT 975, 'Y STR.OF',YSO
C PRINT 975, 'Z STR.OF',ZSO
C PRINT 972
C PRINT *,'
C PRINT 965, (I,I=1,STA)
C PRINT 973
C PRINT 970, 'Y-BCENT',YCB
C PRINT 970, 'Z-BCENT',ZCB
C PRINT 970, 'Y-SHRCTR',YCT
C PRINT 970, 'Z-SHRCTR',ZCT
C PRINT 973
C PRINT 970, 'WEIGHT',WA1
C PRINT 970, 'Y-CG',YGB
C PRINT 970, 'Z-CG',ZGB
C PRINT 970, 'EA',EAT
C PRINT 970, 'ICG',IMG
C PRINT 973
C PRINT 970, 'BALLAST',WB
C PRINT 970, 'TOTAL WT',WT
C PRINT 970, 'NEW Y-CG',YGBB
C PRINT 970, 'NEW Z-CG',ZGBB
C PRINT 970, 'NEW I-CG',IMGB
C CALL INTEG(STX,WT,BWT,1,STA)
C PRINT 970, 'BLD-WGHT',BWT
C PRINT 973
C
C -----NOTE: THE TORSION CALCULATED HERE IS FOR THE STATIC BENCH
C -----TEST, AND INCLUDES APPLIED Z-LOAD (FLAP DIRECTION) AT THE
C -----1/4-CHORD, AND APPLIED STATIC MOMENT ABOUT 1/4-CHORD-----
C
C -----CALCULATE TORSION; SECTION TORQUE = L*(0.25 CHORD-
C -----Y SHEAR CENTER) + PITCHING MOMENT, POSITIVE-----
C -----NOSEDOWN. ENTER CHORD LENGTH USED TO CALCULATE SCALE-----
C -----FACTOR. REMEMBER: AIRFOIL NOSE IS ORIGIN OF SECTION-----
C -----AXIS SYSTEM FOR TORSION CALCULATION-----
C
C DO 640 I = 1,STA
C TANG(I) = (TANG(I) + PITCH1) / 57.29577951
C AANG(I) = PANG(I) + TANG(I)
C Q(I) = LZ(I)*(SIN(TANG(I))*ZCT(I)-COS(TANG(I))*(YCT(I) -
C + (0.25*SCALE(I)))) +
C + LY(I)*(SIN(TANG(I))*(YCT(I)-(0.25*SCALE(I))) +
C + COS(TANG(I))*ZCT(I))
C QS(I) = Q(I) + PM(I)
C
C WA1(I) = WA1(I)/32.2
C WT(I) = WT(I)/32.2
C WB(I) = WB(I)/32.2
C 640 CONTINUE
C
C CALL INTEG(STX,QS,QT,1,STA)

```

```

C -----PRINT TORSION FIGURES-----
C
C PRINT 970, 'TORSION',QT
C
C -----TWIST1 = TORSION*TORK/GJ
C
C -----TORK IS A VALUE FROM HANDBOOKS, 2.0 IS APPROX FOR AIRFOIL
C -----SKIN SHAPES-----
C
C TORK = 2.0
C
C -----TWIST2 = TWIST CALCULATED USING SHEAR FLOW-----
C -----TWIST2 = TORSION / GJ2-----
C
C -----REVERSE THE INDEX TO PERFORM INTEGRATION STARTING-----
C -----AT ROOT END-----
C
C DO 705 I = 1,STA
C IR = STA + 1 - I
C ELAST1(I) = QT(IR)*TORK/GJ1(IR)
C ELAST2(I) = QT(IR) / GJ2(IR)
C 705 CONTINUE
C
C CALL INTEG(STX,ELAST1,TLAST1,1,STA)
C CALL INTEG(STX,ELAST2,TLAST2,1,STA)
C
C DO 725 I = 1,STA
C IR = STA + 1 - I
C
C -----RENUMBER THE INDEX-----
C
C TELAST1(I) = TLAST1(IR)
C TELAST2(I) = TLAST2(IR)
C 725 CONTINUE
C
C -----PRINT TWIST, RENUMBERED-----
C
C PRINT 993, 'TORSDEF1',TELAST1
C
C DO 800 I = 1,STA
C TELAST2(I) = TELAST2(I)*57.29578
C 800 CONTINUE
C
C PRINT 995, 'EL.TWIST',TELAST2
C
C DO 820 I = 1,STA
C TELAST2(I) = TELAST2(I)/57.29578
C 820 CONTINUE
C
C
C ***** BENDING DEFLECTIONS SECTION *****
C
C -----
C
C CALCULATE BEAM SHEARS AND BENDING MOMENTS IN REFERENCE AXES,
C WHICH ARE THE Y AND Z DIRECTIONS. BLADE SHEARS AND MOMENTS
C ARE SUMMED FROM THE FREE END OF THE BEAM.-----
C
C FLAPPING LOADS (LZ), SHEARS (VZ), AND MOMENTS (MFLAP) ARE ALL
C POSITIVE IN THE FLAPPING DIRECTION (SUCH AS TO INCREASE FLAPPING)
C LAGGING LOADS (LY), SHEARS (VY), AND MOMENTS (MLAG) ARE ALL
C POSITIVE IN THE LAGGING DIRECTION (SUCH AS TO RETARD BLADE TORQUE).
C
C -----
C
C CALL INTEG(STX,LX,FX,1,STA)
C CALL INTEG(STX,LY,VY,1,STA)
C CALL INTEG(STX,VY,MLAG,1,STA)
C
C -----PRINT TENSION-----
C
C PRINT 970, 'TENSION',FX
C
C -----PRINT INPLANE SHEAR-----
C
C PRINT 970, 'V-INPL',VY
C
C -----PRINT INPLANE MOMENT-----
C
C PRINT 970, 'BM-INPL',MLAG
C CALL INTEG(STX,LZ,VZ,1,STA)
C CALL INTEG(STX,VZ,MFLAP,1,STA)
C
C -----PRINT FLAPPING SHEAR-----
C
C PRINT 970, 'V-FLAP',VZ

```

```

C
C
C -----PRINT FLAPPING MOMENT-----
C
C PRINT 970, 'BM-FLAP ',MFLAP
C
C -----PRINCIPAL AXIS ANGLE (AANG) = TWIST ANGLE (TANG) +
C -----CHORD TO PRINCIPAL AXIS ANGLE OF SECTION (PANG) +
C -----ELASTIC TWIST ANGLE (TELAST 1 OR TELAST 2)-----
C
C *****
C ***** STRESS CALCULATIONS *****
C *****
C
C NOTE: THIS PROGRAM DOES NOT PRINT OUT LAMINATE STRESSES.
C
C -----CALCULATE THE STRESS DUE TO AXIAL TENSION. RESOLVE-----
C -----BENDING MOMENT TO PRINCIPAL AXES. FIND NEUTRAL AXIS-----
C -----RELATIVE TO DISC.-----
C
C DO 870 I=1,STA
C TANG(I) = TANG(I) + TELAST2(I)
C AANG(I) = PANG(I) + TANG(I)
C SXTN(I)=FX(I) * ES/EA1(I)
C
C -----NOTE: ROTATION OF AXES FROM SECTION COORDINATES TO-----
C -----PRINCIPAL AXES COORDINATES IS THROUGH POSITIVE ANGLE----
C -----PANG (COUNTERCLOCKWISE).
C
C MFLAPP(I) = MFLAP(I)*COS(AANG(I))-MLAG(I)*SIN(AANG(I))
C MLAGP(I) = MLAG(I)*COS(AANG(I))+MFLAP(I)*SIN(AANG(I))
C
C 870 CONTINUE
C
C -----PRINT PRINCIPAL AXIS ANGLE AFTER CONVERTING TO DEGREES--
C
C DO 900 I=1,STA
C AANG(I) = AANG(I) * 57.29577951
C 900 CONTINUE
C
C PRINT 970, 'PR A A ',AANG
C
C -----MUST CHANGE PRICIPAL AXIS ANGLE BACK TO RAD FOR CALCS--
C
C DO 901 I=1,STA
C AANG(I) = PANG(I) + TANG(I)
C 901 CONTINUE
C
C -----PRINT X-STRESS TENSION-----
C
C PRINT 970, 'SX TENS.',SXTN
C
C -----CALCULATE STRESSES DUE TO BENDING. STRESS IS THE SUM----
C -----OF BENDING IN PLANES OF PRINCIPAL AXES.-----
C
C DO 1005 K=1,NSO
C
C DO 885 J=1,STA
C
C -----TRANSLATE YSO AND ZSO TO COORDINATES THROUGH CENTROID--
C
C YK = YSO(K)*SCALE(J)-YCB(J)
C ZK = ZSO(K)*SCALE(J)-ZCB(J)
C
C -----C'S ARE DISTANCES TO STRESSED FIBER
C -----IN PRINCIPAL AXIS SYSTEM, FOR THE SELECTED STRESS OFFSETS
C -----CALCULATED ABOVE-----
C
C CY = YK*COS(PANG(J))+ZK*SIN(PANG(J))
C CZ = ZK*COS(PANG(J))-YK*SIN(PANG(J))
C
C -----NEGATIVE SIGN DENOTES FIBER IN COMPRESSION.-----
C -----THE FLEXURE FORMULA IS: BEND STRESS =M*C/I-----
C -----POSITIVE MLAGP MOMENT IS IN LAGGING DIRECTION,---
C -----AND GIVES TRAILING EDGE COMPRESSION. POSITIVE MFLAPP MOMENT
C -----IS IN FLAPPING DIRECTION, AND GIVES UPPER----
C -----SURFACE COMPRESSION.-----
C -----BOTH HAVE BEEN RESOLVED INTO THE PRINCIPAL AXES OF EACH
C -----SECTION, SO THEY ARE NOT EXACTLY IN THE FLAP AND LAG
C -----DIRECTIONS (Z,Y).-----
C
C SX(J) = -MFLAPP(J)*CZ*EL/EIYP(J)-MLAGP(J)*CY*EL/EIZP(J)
C 885 CONTINUE
C

```

```

C -----PRINT BEAM TENSILE STRESS DUE TO BENDING MOMENT--
C
C PRINT 970, 'SX-BM ', SX
C
C DO 890 J=1,STA
C SX(J) = SX(J) + SXTN(J)
C
C 890 CONTINUE
C
C -----PRINT BEAM TENSILE STRESS TOTAL -----
C
C PRINT 970, 'SX TOTAL', SX
C
C 1005 CONTINUE
C
C -----CALCULATE BENDING DEFLECTION. DEFINE CURVE M/EI IN-----
C -----REVERSE ORDER TO INTEGRATE FROM FIXED END-----
C
C DO 1110 I=1,STA
C IR = STA +1-I
C RX(IR) = STX(STA)-STX(I)
C MYP(I) = MFLAPP(I)/EIYP(I)
C MZP(I) = MLAGP(I)/EIZP(I)
C MY(IR) = (MYP(I)*COS(AANG(I))) + (MZP(I)*SIN(AANG(I)))
C MZ(IR) = (MZP(I)*COS(AANG(I))) - (MYP(I)*SIN(AANG(I)))
C
C 1110 CONTINUE
C
C -----REVERSE THE ORDER-----
C
C DO 1120 I=1,STA
C IR = STA+1-I
C MYE(I) = MY(IR)
C MZE(I) = MZ(IR)
C LYC(I) = LY(I)
C LZC(I) = LZ(I)
C
C 1120 CONTINUE
C
C -----PRINT THE CURVE M/EI-REVERSE, Y-DIRECTION, INTEGRATED FROM
C -----THE FIXED END-----
C
C PRINT 992, 'M.EIYP ',MYP
C
C -----PRINT THE CURVE M/EI-REVERSE, Z-DIRECTION, INTEGRATED FROM
C -----THE FIXED END-----
C
C PRINT 992, 'M.EIZP ',MZP
C PRINT 973
C
C -----FIND DEFLECTION BY INTEGRATION: D2Y/DX2 = M/EI-----
C -----WHERE LYC = DY/DX AND LZC = DZ/DX ARE THE ROOT SLOPES
C -----NOTE: THIS SECTION IS SIMILAR TO THE SUBROUTINE "INTEG"
C
C LYC(1) = 0.0
C LZC(1) = 0.0
C DEFY(1) = 0.0
C DEFZ(1) = 0.0
C I6 = STA - 1
C DO 1115 JJ=1,I6,2
C J = JJ
C IF(J.EQ. I6) J = J-1
C CALL LCOE1(C1,C2,C3,J,RX,MZ)
C CALL LCOE1(D1,D2,D3,J,RX,MY)
C IF(JJ.EQ. I6) J = J+1
C C4 = LYC(J)-C1*RX(J)**3/3-C2*RX(J)**2/2-C3*RX(J)
C D4 = LZC(J)-D1*RX(J)**3/3-D2*RX(J)**2/2-D3*RX(J)
C C5 = DEFY(J)-C1*RX(J)**4/12-C2*RX(J)**3/6-C3*RX(J)**2/2-C4*RX(J)
C D5 = DEFZ(J)-D1*RX(J)**4/12-D2*RX(J)**3/6-D3*RX(J)**2/2-D4*RX(J)
C K = J+1
C LYC(K) = C1*RX(K)**3/3+C2*RX(K)**2/2+C3*RX(K)+C4
C LZC(K) = D1*RX(K)**3/3+D2*RX(K)**2/2+D3*RX(K)+D4
C DEFY(K)=C1*RX(K)**4/12+C2*RX(K)**3/6+C3*RX(K)**2/2+C4*RX(K)+C5
C DEFZ(K)=D1*RX(K)**4/12+D2*RX(K)**3/6+D3*RX(K)**2/2+D4*RX(K)+D5
C IF(JJ.EQ. I6) GO TO 1115
C K = J+2
C LYC(K) = C1*RX(K)**3/3+C2*RX(K)**2/2+C3*RX(K)+C4
C LZC(K) = D1*RX(K)**3/3+D2*RX(K)**2/2+D3*RX(K)+D4
C DEFY(K)=C1*RX(K)**4/12+C2*RX(K)**3/6+C3*RX(K)**2/2+C4*RX(K)+C5
C DEFZ(K)=D1*RX(K)**4/12+D2*RX(K)**3/6+D3*RX(K)**2/2+D4*RX(K)+D5
C
C 1115 CONTINUE
C
C -----REVERSE ORDER-----
C
C DO 1215 I=1,STA

```

```

IR = STA + 1 - I
DFY(IR) = DEFY(I)
DFZ(IR) = DEFZ(I)
C
1215 CONTINUE
C
C -----PRINT Y DEFLECTION-----
C
PRINT 970, 'Y-DEF. ', DFY
C
C -----PRINT Z DEFLECTION-----
C
PRINT 970, 'Z-DEF. ', DFZ
C
DO 1300 I=1, STA
EIFLAP(I) = EIYP(I)*COS(AANG(I))+EIZP(I)*SIN(AANG(I))
EILAG(I) = -EIYP(I)*SIN(AANG(I))+EIZP(I)*COS(AANG(I))
C
1300 CONTINUE
C
C -----PRINT THE FLAP AND LAG STIFFNESSES-----
C
PRINT 992, 'EI-FLAP', EIFLAP
PRINT 992, 'EI-LAG ', EILAG
C
PRINT 973
C
C ***** STRAIN CALCULATIONS *****
C
C FIND STRAINS DUE TO AXIAL TENSION (CENTRIFUGAL FORCE)
C AND BENDING MOMENT.
C
C STRAIN DUE TO TENSION (SXTN) IS JUST TENSION AT A
C STATION DIVIDED BY THE COMPOSITE EA OF THE STATION:
C
DO 1500 I = 1, STA
SXTN(I) = FX(I) / EA1(I)
AANG(I) = PANG(I) + TANG(I)
1500 CONTINUE
C
C CALCULATE STRAINS DUE TO BENDING. STRAIN IS THE SUM
C OF THE BENDING STRAIN IN THE PLANES OF THE PRINCIPAL
C AXES.
C
DO 1600 K = 1, NSO
DO 1550 J = 1, STA
C
C FIRST TRANSLATE Y AND Z STRAIN OFFSETS (YSO,ZSO) GIVEN
C IN UNITY SECTION COORDINATES, TO COORDINATES REFERRED TO
C THE SECTION CENTROID. THIS CALCULATION INCLUDES THE TWO
C SCALE FACTORS FOR THE BLADE SIMILAR SECTIONS (SCALE,
C TSCALE).
C
YK = YSO(K) * SCALE(J) - YCB(J)
ZK = ZSO(K) * SCALE(J) * TSCALE(J) - ZCB(J)
C
C THE C'S ARE THE DISTANCES TO STRESSED FIBER IN THE
C PRINCIPAL AXIS SYSTEM, FOR THE SELECTED STRAIN OFFSETS
C CALCULATED ABOVE:
C
CY = YK * COS(-AANG(J)) + ZK * SIN(-AANG(J))
CZ = ZK * COS(-AANG(J)) - YK * SIN(-AANG(J))
C
C THE NEGATIVE SIGN DENOTES FIBER IN COMPRESSION. THE
C FLEXURE FORMULA FOR STRAIN IS: BENDING STRAIN =
C M * C / EI.
C
C NOTE: THE POSITIVE MLAGP MOMENT IS IN THE LAGGING
C DIRECTION, AND GIVES TRAILING EDGE COMPRESSION. POSITIVE
C MFLAPP MOMENT IS IN THE FLAPPING DIRECTION, AND GIVES
C UPPER SURFACE COMPRESSION. BOTH HAVE BEEN RESOLVED INTO
C THE PRINCIPAL AXES OF EACH SECTION, SO THEY ARE NOT
C EXACTLY IN THE FLAP AND LAG DIRECTIONS (Z, Y).
C
C CALCULATE BENDING STRAIN:
C
SX(J) = -MFLAPP(J) * CZ/EIYP(J) -MLAGP(J)*CY/EIZP(J)
C
C CALCULATE TOTAL STRAIN AND MICROSTRAIN:
C
SX(J) = SX(J) + SXTN(J)
SX(J) = SX(J) * 1000000.0

```

```

C
1550 CONTINUE
C
C PRINT TOTAL STRAIN:
C
PRINT 971, 'M-STRAIN', K, SX
C
1600 CONTINUE
PRINT 973
C
C
2000 CONTINUE
C
C ***** FORMAT STATEMENTS *****
C
905 FORMAT(5X, I2, 2F6.2, I4, 2F6.2)
910 FORMAT('# STATIONS: ', I2, 2X, 'E-SKIN: ', F10.2, 2X, 'E-LONG: ',
+ F10.2, ' # DIVISIONS: ', I4, ' GS: ', F10.2, ' GL: ', F10.2,
+ ' WS: ', F5.3, ' WL: ', F5.3)
915 FORMAT(5X, I2, 2X, F6.4)
920 FORMAT(5X, 9F6.4, /, 5X, 8F6.4, /, 5X, 8F6.4)
925 FORMAT(5X, 9F8.4, /, 3(5X, 8F8.4, /))
930 FORMAT(' OFFSETS FOR STATION: ', I2, ' THK1: ', F6.4,
+ ' THK2: ', F6.4, ' XTHK: ', F7.3, /, 29X, 'TSPAR: ',
+ F6.4, ' TSKIN: ', F6.4, ' TSCALE: ', F6.4)
933 FORMAT(5X, 6F6.4)
935 FORMAT(5X, 4F6.3)
940 FORMAT(' WEB DATA - AL, IX, IY, IX, XL, YL: ', 6(3X, F8.3), /)
945 FORMAT(' POSITION OF CENTROID FROM BASE LINE X: ', F8.2,
+ ' Y: ', F8.2, /, ' SKIN AREA: ', F10.4, /, ' ANGLE OF PRINCI',
+ ' PAL AXIS: ', F6.2, ' EIX: ', E12.5, ' EY: ', E12.5, /, ' RX: ',
+ F6.2, ' RY: ', F6.2, ' EA: ', E12.5, /)
946 FORMAT(' POSITION OF CENTROID FROM BASE LINE X: ', F8.2,
+ ' Y: ', F8.2, /, ' SKIN AREA: ', F10.4, ' SPAR AREA: ', F10.4,
+ /, ' ANGLE OF PRINCIPAL AXIS: ', F6.2, ' EIX: ', E12.5, ' EY: ',
+ E12.5, /, ' RX: ', F6.2, ' RY: ', F6.2, ' EA: ', E12.5, /)
950 FORMAT(' POSITION OF TORSIONAL CENTROID X: ', F8.2, ' Y: ',
+ F8.2, /, 25X, ' GJ1: ', F16.2, ' GJ2: ', F16.2, /)
955 FORMAT(' SECTION MASS INFORMATION WSPAR: ', F6.4, ' WSKIN: ',
+ F6.4, /, 10X, ' SECTION RUNNING WEIGHT: ', F8.4, ' XCG: ', F8.2,
+ ' YCG: ', F8.2, /, 10X, ' ELASTIC AXIS MOMENT OF INERTIA: ', F12.4, /,
+ 10X, ' SECTION CG MOMENT OF INERTIA: ', F12.4, /)
956 FORMAT(' SECTION BALLAST INFORMATION WBALLAST: ', F6.4, ' XBALL: ',
+ F8.2, ' YBALL: ', F8.2, /, 10X, ' NEW SECTION RUNNING WEIGHT: ',
+ F8.4, ' XCG: ', F8.2, ' YCG: ', F8.2, /, 10X, ' OLD SECTION CG
+ MOMENT OF INERTIA: ', F12.4, /, 10X, ' NEW SECTION CG MOMENT OF
+ INERTIA: ', F12.4, /)
963 FORMAT(5X, I3)
964 FORMAT(19X, ' TIP: 106X, / ROOT)
965 FORMAT(' STATION ', 11(9X, I2))
970 FORMAT(' ', A8, 1X, 11F11.2)
971 FORMAT(' ', A8, I2, 11(F10.0, 1X))
972 FORMAT('-----',
+ '-----')
973 FORMAT('-----',
+ '-----',
+ '-----')
975 FORMAT(' ', A8, 3X, 6F12.2)
980 FORMAT(5X, 8F8.2, /, 5X, 3F8.2)
981 FORMAT(6X, 8F8.3, /, 6X, 3F8.3)
983 FORMAT(7X, 6F8.4)
985 FORMAT(' ', A8, 11, 11F11.2)
990 FORMAT(5X, 8F8.4, /, 5X, 3F8.4)
992 FORMAT(' ', A8, 1X, 11(E10.4, 1X))
993 FORMAT(' ', A8, 1X, 11F11.6)
994 FORMAT(1A80)
995 FORMAT(' ', A8, 1X, 11F11.3)
996 FORMAT(' STRESS OFFSETS: ', 10(I2, 10X))
998 FORMAT(' ', 1A40, ' ** TCORR = ', F6.3, ' ** PITCH = ', F6.3, /)
C
C -----END OF PROGRAM-----
C
CLOSE(5)
CLOSE(6)
END
C
C *****
C
C ----- SUBROUTINE INTEG -----
C
SUBROUTINE INTEG(X, Y, XS, IS, IE)
C
C *****

```

C THIS ROUTINE INTEGRATES THE AREA UNDER A CURVE OF HEIGHT Y AND
 C SPACING X BY FITTING A SECOND ORDER EQUATION THROUGH THE POINTS.
 C XI IS THE AREA OF EACH SEGMENT, AND XS IS THE SUM UP TO THAT
 C POINT. IT STARTS AT X(IE) AND STOPS AT X(IE).

C -----
 C DIMENSION X(12),Y(12),XI(12),XS(12)
 C XI(1) = 0.0
 C XS(1) = 0.0
 C I2 = IS + 1

C -----MAIN INTEGRATION LOOP-----
 C

C DO 1070 JJ = I2,IE,2
 C J = JJ
 C IF(JJ.EQ. IE) J = J - 1
 C CALL LCOE1(CO1,CO2,CO3,J-1,X,Y)
 C K = J+1
 C XI(K) = CO1*(X(K)**3-X(J)**3)/3+CO2*(X(K)**2-X(J)**2)/2+CO3*(X(K)
 C +X(J))
 C IF(JJ.EQ. IE) GO TO 1070
 C K = J-1
 C XI(J) = CO1*(X(J)**3-X(K)**3)/3+CO2*(X(J)**2-X(K)**2)/2+CO3*(X(J)
 C +X(K))

1070 CONTINUE
 C XS(IS+1) = XI(IS+1)
 C IF(IS.GE. IE-1) GO TO 1310
 C IS = IS+2
 C DO 1305 J = IS,IE
 C XS(J) = XI(J)+XS(J-1)
 1305 CONTINUE
 1310 CONTINUE

C RETURN
 C END

C -----
 C

C ----- SUBROUTINE LCOE -----
 C

C SUBROUTINE LCOE(CO1,CO2,CO3,J)
 C

C -----
 C

C THIS SUBROUTINE FITS A SECOND ORDER CURVE THROUGH EACH SET OF 3
 C POINTS, STARTING AT J. X IS THE ABSCISSA, AND Y THE ORDINATE.
 C VIZ: Y = CO1*X**2+CO2*X+CO3.(THE PARABOLIC AXIS IS VERTICAL).

C -----
 C

C COMMON X(33),Y(33)
 C CO1 = 0.0
 C CO2 = 0.0
 C CO3 = 0.0
 C C1N=X(J)*Y(J+2)-Y(J+1)+X(J+1)*Y(J)-Y(J+2)+X(J+2)*Y(J+1)-Y(J)
 C C2N=(X(J)**2)*Y(J+1)-Y(J+2)+(X(J+1)**2)*Y(J+2)-Y(J)+(X(J+2)**2
 C +Y(J)-Y(J+1))
 C C3N=X(J)**2*(X(J+1)*Y(J+2)-X(J+2)*Y(J+1))+X(J+1)**2*(X(J+2)*Y
 C +J)-X(J)*Y(J+2)+(X(J+2)**2)*X(J)*Y(J+1)-X(J+1)*Y(J))
 C DET = X(J)*((X(J+2)**2)-(X(J+1)**2))+X(J+1)*((X(J)**2)-(X(J+2)**2
 C +)+X(J+2)*((X(J+1)**2)-(X(J)**2))

C3N=(X(J)**2)*(X(J+1)*Y(J+2)-X(J+2)*Y(J+1))+X(J+1)**2*(X(J+2)*Y
 C +J)-X(J)*Y(J+2)+(X(J+2)**2)*X(J)*Y(J+1)-X(J+1)*Y(J))
 C DET = X(J)*((X(J+2)**2)-(X(J+1)**2))+X(J+1)*((X(J)**2)-(X(J+2)**2
 C +)+X(J+2)*((X(J+1)**2)-(X(J)**2))
 C CO1 = C1N/DET
 C CO2 = C2N/DET
 C CO3 = C3N/DET

C RETURN
 C END

C -----
 C

C ----- SUBROUTINE LCOE1 -----
 C

C SUBROUTINE LCOE1(CO1,CO2,CO3,J,X,Y)
 C

C -----
 C

C THIS SUBROUTINE FITS A SECOND ORDER CURVE THROUGH EACH SET OF 3
 C POINTS, STARTING AT J. THIS IS SIMILAR TO SUBROUTINE LCOE, EXCEPT
 C THAT IT IS FOR PARAMETERS FROM THE STRESS PROGRAM RATHER THAN
 C THE EIS PROGRAM.

C -----
 C

C DIMENSION X(12),Y(12)
 C CO1 = 0.0
 C CO2 = 0.0
 C CO3 = 0.0

C1N=X(J)*Y(J+2)-Y(J+1)+X(J+1)*Y(J)-Y(J+2)+X(J+2)*Y(J+1)-Y(J)
 C C2N=(X(J)**2)*Y(J+1)-Y(J+2)+(X(J+1)**2)*Y(J+2)-Y(J)+(X(J+2)**2
 C +Y(J)-Y(J+1))
 C C3N=X(J)**2*(X(J+1)*Y(J+2)-X(J+2)*Y(J+1))+X(J+1)**2*(X(J+2)*Y
 C +J)-X(J)*Y(J+2)+(X(J+2)**2)*X(J)*Y(J+1)-X(J+1)*Y(J))
 C DET = X(J)*((X(J+2)**2)-(X(J+1)**2))+X(J+1)*((X(J)**2)-(X(J+2)**2
 C +)+X(J+2)*((X(J+1)**2)-(X(J)**2))
 C CO1 = C1N/DET
 C CO2 = C2N/DET
 C CO3 = C3N/DET

C RETURN
 C END

10.1.3 SAMPLE RUN, CARTER BLADE

The sample input files are for the Carter blade (CARTBL.AEIFINAL) and zero applied loading (CARTLD.ZERO) to illustrate the calculation of the section values for a complex composite blade. Note: for this run the Carter I-beam was not included, as this complication must be handled by the next version of the code (STRESSDATA) as shown below.

INPUT BLADE FILE: CARTBL.AEIFINAL

```

11 4.00 4.00 500 0.500 0.750
33 1.000 .050 .050
.0000 .0125 .0250 .0500 .0750 .1000 .1500 .2000 .2500
.3500 .5000 .6000 .7000 .8000 .9000 .9500 1.0000
.9500 .9000 .8000 .7000 .6000 .5000 .3500 .2500
.2000 .1500 .1000 .0750 .0500 .0250 .0125 .0000
.0000 .0244 .0342 .0476 .0564 .0629 .0722 .0783 .0822
.0850 .0787 .0685 .0542 .0368 .0174 .0070 .0000
-.0012 -.0026 -.0112 -.0232 -.0344 -.0415 -.0452 -.0436
-.0415 -.0382 -.0333 -.0299 -.0255 -.0191 -.0145 .0000
006
.0000 .2500 .5000 1.0000 .5000 .2500
.0000 .3042 .0787 .0000 -.0415 -.0436
2.6 2.2 2.0 2.0 2.0 2.3
4.5 9.4 15.5 26.1 26.1
21.41 0.995 0.040 0.200 5.90
0.153 0.038 0.0001 0.000 4.500 0.680
0.594 1.630 0.250

```

```

21.41 0.995 0.040 0.203 5.90
0.158 0.038 0.0001 0.000 4.630 0.680
0.763 1.630 0.250
22.19 0.996 0.141 0.113 6.31
0.177 0.045 0.0001 0.000 4.810 0.700
0.773 1.630 0.250
23.44 0.997 0.244 0.112 7.00
0.145 0.035 0.0001 0.000 5.630 0.740
0.812 1.630 0.250
25.56 0.999 0.230 0.155 7.75
0.179 0.068 0.0001 0.000 6.690 0.820
0.782 1.630 0.250
28.97 1.001 0.232 0.189 8.94
0.353 0.184 0.0001 0.000 6.810 0.950
0.862 1.630 0.250
34.56 1.019 0.227 0.167 10.00
6.150 2.000 104.0 -7.00 8.64 0.00
0.730 8.64 0.00
42.50 1.055 0.227 0.167 13.25
7.440 5.000 190.0 -25.00 10.63 0.00

```


0.450 10.63 0.00
 52.81 1.070 0.227 0.167 16.00
 8.000 18.00 205.0 -50.00 13.20 0.00
 .590 13.20 0.00
 66.75 1.071 0.227 0.167 21.88
 8.750 58.00 258.0 -118.0 16.69 0.00
 4.87 16.69 0.00
 66.75 1.071 0.227 0.167 21.88
 8.750 58.00 258.0 -118.0 16.69 0.00
 0.000 16.69 0.00
 CARTER.BLADE.AEI.DATA.ALL.TEST.WTS.5-14-88

001
 00.00 40.28 80.55 120.83 161.10 201.38 241.65 281.93
 322.20 362.48 402.75
 00.00 00.00 00.00 00.00 00.00 00.00 00.00
 00.00 00.00 00.00
 00.00 00.00 00.00 00.00 00.00 00.00 00.00
 00.00 00.00 00.00
 00.00 00.00 00.00
 00.00 00.00 00.00
 00.00 00.00 00.00
 0.00 0.00 0.00 0.00 0.00 0.00 0.00
 0.00 0.00 0.00
 0.00 402.75 0.00 0.00
 CARTER.BLADE.ZERO.LOAD.TEST.12-11-87

INPUT LOAD FILE: CARTLD.ZERO

[Note: This output gives the values for each blade section, starting with the blade tip. The airfoil shape offsets appear first. These can be used for making templates.]

TERMINAL INPUT AND OUTPUT: STRESSM

ENTER INPUT FILE NAME FOR SECTION GEOMETRY:
 CARTBL.AEIFINAL
 ENTER INPUT FILE NAME FOR APPLIED LOADING:
 CARTLD.ZERO
 CARTER.BLADE.ZERO.LOAD.TEST.12-11-87 ** TCORR = 1.000 ** PITCH = 0.000

STATIONS: 11 E-SKIN: 3999999.50 E-LONG: 3999999.50 # DIVISIONS: 500 GS: 499999.94 GL: 749999.88 WS: 0.050 WL: 0.050

OFFSETS FOR STATION: 1 THK1: 0.2400 THK2: 0.2000 XTHK: 5.900
 TSPAR: 0.0400 TSKIN: 0.2000 TSCALE: 0.9950
 0.0000 0.2676 0.5352 1.0705 1.6057 2.1410 3.2115 4.2820 5.3525
 7.4935 10.7050 12.8460 14.9870 17.1280 19.2690 20.3395 21.4100
 20.3395 19.2690 17.1280 14.9870 12.8460 10.7050 7.4935 5.3525
 4.2820 3.2115 2.1410 1.6057 1.0705 0.5352 0.2676 0.0000

 0.0000 0.5198 0.7286 1.0140 1.2015 1.3400 1.5381 1.6680 1.7511
 1.8108 1.6765 1.4593 1.1546 0.7839 0.3707 0.1491 0.0000
 -0.0256 -0.0554 -0.2386 -0.4942 -0.7328 -0.8841 -0.9629 -0.9288
 -0.8841 -0.8138 -0.7094 -0.6370 -0.5432 -0.4069 -0.3089 0.0000

SECTION MASS INFORMATION WSPAR: 0.0500 WSKIN: 0.0500
 SECTION RUNNING WEIGHT: 0.4772 XCG: 10.05 YCG: 0.32
 ELASTIC AXIS MOMENT OF INERTIA: 19.5401
 SECTION CG MOMENT OF INERTIA: 19.5155

SECTION BALLAST INFORMATION WBALLAST: 0.7630 XBAL: 1.63 YBAL: 0.25
 NEW SECTION RUNNING WEIGHT: 1.2402 XCG: 4.87 YCG: 0.28
 OLD SECTION CG MOMENT OF INERTIA: 73.6074
 NEW SECTION CG MOMENT OF INERTIA: 40.3281

WEB DATA - AL,IX,IY,IXY,XL,YL
 0.153 0.038 0.000 0.000 4.500 0.680
 POSITION OF CENTROID FROM BASE LINE X: 10.04 Y: 0.32
 SKIN AREA: 8.7520 SPAR AREA: 0.5023
 ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.31945E+08 EIY: 0.15086E+10
 RX: 0.92 RY: 6.33 EA: 0.37629E+08

POSITION OF TORSIONAL CENTROID X: 9.81 Y: 0.32
 GJ1: 200538112.00 GJ2: 14980016.00

SECTION MASS INFORMATION WSPAR: 0.0500 WSKIN: 0.0500
 SECTION RUNNING WEIGHT: 0.4704 XCG: 10.04 YCG: 0.32
 ELASTIC AXIS MOMENT OF INERTIA: 19.2822
 SECTION CG MOMENT OF INERTIA: 19.2574

SECTION BALLAST INFORMATION WBALLAST: 0.5940 XBAL: 1.63 YBAL: 0.25
 NEW SECTION RUNNING WEIGHT: 1.0644 XCG: 5.35 YCG: 0.28
 OLD SECTION CG MOMENT OF INERTIA: 61.3052
 NEW SECTION CG MOMENT OF INERTIA: 37.8391

OFFSETS FOR STATION: 2 THK1: 0.2430 THK2: 0.2030 XTHK: 5.900
 TSPAR: 0.0400 TSKIN: 0.2030 TSCALE: 0.9950
 0.0000 0.2676 0.5352 1.0705 1.6057 2.1410 3.2115 4.2820 5.3525
 7.4935 10.7050 12.8460 14.9870 17.1280 19.2690 20.3395 21.4100
 20.3395 19.2690 17.1280 14.9870 12.8460 10.7050 7.4935 5.3525
 4.2820 3.2115 2.1410 1.6057 1.0705 0.5352 0.2676 0.0000

 0.0000 0.5198 0.7286 1.0140 1.2015 1.3400 1.5381 1.6680 1.7511
 1.8108 1.6765 1.4593 1.1546 0.7839 0.3707 0.1491 0.0000
 -0.0256 -0.0554 -0.2386 -0.4942 -0.7328 -0.8841 -0.9629 -0.9288
 -0.8841 -0.8138 -0.7094 -0.6370 -0.5432 -0.4069 -0.3089 0.0000

OFFSETS FOR STATION: 3 THK1: 0.2540 THK2: 0.1130 XTHK: 6.310
 TSPAR: 0.1410 TSKIN: 0.1130 TSCALE: 0.9960
 0.0000 0.2774 0.5547 1.1095 1.6642 2.2190 3.3285 4.4380 5.5475
 7.7665 11.0950 13.3140 15.5330 17.7520 19.9710 21.0805 22.1900
 21.0805 19.9710 17.7520 15.5330 13.3140 11.0950 7.7665 5.5475
 4.4380 3.3285 2.2190 1.6642 1.1095 0.5547 0.2774 0.0000

0.0000 0.5393 0.7559 1.0520 1.2465 1.3902 1.5957 1.7305 1.8167
 1.8786 1.7394 1.5139 1.1979 0.8133 0.3846 0.1547 0.0000
 -0.0265 -0.0575 -0.2475 -0.5127 -0.7603 -0.9172 -0.9990 -0.9636
 -0.9172 -0.8443 -0.7360 -0.6608 -0.5636 -0.4221 -0.3205 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL
 0.177 0.045 0.000 0.000 4.810 0.700
 POSITION OF CENTROID FROM BASE LINE X: 8.71 Y: 0.34
 SKIN AREA: 5.1252 SPAR AREA: 1.8902
 ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.27781E+08 EIY: 0.12456E+10
 RX: 0.98 RY: 6.58 EA: 0.28770E+08

POSITION OF TORSIONAL CENTROID X: 8.02 Y: 0.34
 GJ1: 175248992.00 GJ2: 10754780.00

SECTION MASS INFORMATION WSPAR: 0.0500 WSKIN: 0.0500
 SECTION RUNNING WEIGHT: 0.3596 XCG: 8.71 YCG: 0.34
 ELASTIC AXIS MOMENT OF INERTIA: 16.0909
 SECTION CG MOMENT OF INERTIA: 15.9172

SECTION BALLAST INFORMATION WBALLAST: 0.7730 XBAL: 1.63 YBAL: 0.25
 NEW SECTION RUNNING WEIGHT: 1.1326 XCG: 3.88 YCG: 0.28
 OLD SECTION CG MOMENT OF INERTIA: 54.6805
 NEW SECTION CG MOMENT OF INERTIA: 28.2251

WEB DATA - AL,IX,IY,IXY,XL,YL
 0.158 0.038 0.000 0.000 4.630 0.680
 POSITION OF CENTROID FROM BASE LINE X: 10.05 Y: 0.32
 SKIN AREA: 8.8833 SPAR AREA: 0.5023
 ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.32305E+08 EIY: 0.15289E+10
 RX: 0.92 RY: 6.33 EA: 0.38174E+08

POSITION OF TORSIONAL CENTROID X: 9.82 Y: 0.32
 GJ1: 203117408.00 GJ2: 15195294.00

OFFSETS FOR STATION: 4 THK1: 0.3560 THK2: 0.1120 XTHK: 7.000
 TSPAR: 0.2440 TSKIN: 0.1120 TSCALE: 0.9970
 0.0000 0.2930 0.5860 1.1720 1.7580 2.3440 3.5160 4.6880 5.8600
 8.2040 11.7200 14.0640 16.4080 18.7520 21.0960 22.2680 23.4400
 22.2680 21.0960 18.7520 16.4080 14.0640 11.7200 8.2040 5.8600
 4.6880 3.5160 2.3440 1.7580 1.1720 0.5860 0.2930 0.0000

0.0000 0.5702 0.7992 1.1124 1.3180 1.4700 1.6873 1.8298 1.9210
1.9864 1.8392 1.6008 1.2666 0.8600 0.4066 0.1636 0.0000
-0.0280 -0.0608 -0.2617 -0.5422 -0.8039 -0.9698 -1.0563 -1.0189
-0.9698 -0.8927 -0.7782 -0.6988 -0.5959 -0.4464 -0.3389 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL

0.145 0.035 0.000 0.000 5.630 0.740
POSITION OF CENTROID FROM BASE LINE X: 8.22 Y: 0.36
SKIN AREA: 5.3662 SPAR AREA: 3.6181
ANGLE OF PRINCIPAL AXIS: 0.00 EIX: 0.38537E+08 EIY: 0.16515E+10
RX: 1.03 RY: 6.72 EA: 0.36517E+08

POSITION OF TORSIONAL CENTROID X: 7.41 Y: 0.36
GJ1: 233931296.00 GJ2: 13356832.00

SECTION MASS INFORMATION WSPAR: 0.0500 WSKIN: 0.0500
SECTION RUNNING WEIGHT: 0.4565 XCG: 8.22 YCG: 0.36
ELASTIC AXIS MOMENT OF INERTIA: 21.4275
SECTION CG MOMENT OF INERTIA: 21.1251

SECTION BALLAST INFORMATION WBALLAST: 0.8120 XBAL: 1.63 YBAL: 0.25
NEW SECTION RUNNING WEIGHT: 1.2685 XCG: 4.00 YCG: 0.29
OLD SECTION CG MOMENT OF INERTIA: 56.4041
NEW SECTION CG MOMENT OF INERTIA: 33.8205

OFFSETS FOR STATION: 5 THK1: 0.3850 THK2: 0.1550 XTHK: 7.750
TSPAR: 0.2300 TSKIN: 0.1550 TSCALE: 0.9990
0.0000 0.3195 0.6390 1.2780 1.9170 2.5560 3.8340 5.1120 6.3900
8.9460 12.7800 15.3360 17.8920 20.4480 23.0040 24.2820 25.5600
24.2820 23.0040 20.4480 17.8920 15.3360 12.7800 8.9460 6.3900
5.1120 3.8340 2.5560 1.9170 1.2780 0.6390 0.3195 0.0000

0.0000 0.6230 0.8733 1.2154 1.4401 1.6061 1.8436 1.9993 2.0989
2.1704 2.0096 1.7491 1.3840 0.9397 0.4443 0.1787 0.0000
-0.0306 -0.0664 -0.2860 -0.5924 -0.8784 -1.0597 -1.1542 -1.1133
-1.0597 -0.9754 -0.8503 -0.7635 -0.6511 -0.4877 -0.3702 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL

0.179 0.068 0.000 0.000 6.690 0.820
POSITION OF CENTROID FROM BASE LINE X: 9.74 Y: 0.39
SKIN AREA: 8.0987 SPAR AREA: 3.7737
ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.60330E+08 EIY: 0.27016E+10
RX: 1.12 RY: 7.49 EA: 0.48206E+08

POSITION OF TORSIONAL CENTROID X: 8.91 Y: 0.39
GJ1: 381029376.00 GJ2: 23335424.00

SECTION MASS INFORMATION WSPAR: 0.0500 WSKIN: 0.0500
SECTION RUNNING WEIGHT: 0.6026 XCG: 9.74 YCG: 0.39
ELASTIC AXIS MOMENT OF INERTIA: 34.9366
SECTION CG MOMENT OF INERTIA: 34.5241

SECTION BALLAST INFORMATION WBALLAST: 0.7820 XBAL: 1.63 YBAL: 0.25
NEW SECTION RUNNING WEIGHT: 1.3846 XCG: 5.16 YCG: 0.31
OLD SECTION CG MOMENT OF INERTIA: 85.9536
NEW SECTION CG MOMENT OF INERTIA: 56.9064

OFFSETS FOR STATION: 6 THK1: 0.4210 THK2: 0.1890 XTHK: 8.940
TSPAR: 0.2320 TSKIN: 0.1890 TSCALE: 1.0010
0.0000 0.3621 0.7242 1.4485 2.1727 2.8970 4.3455 5.7940 7.2425
10.1395 14.4850 17.3820 20.2790 23.1760 26.0730 27.5215 28.9700
27.5215 26.0730 23.1760 20.2790 17.3820 14.4850 10.1395 7.2425
5.7940 4.3455 2.8970 2.1727 1.4485 0.7242 0.3621 0.0000

0.0000 0.7076 0.9918 1.3804 1.6355 1.8240 2.0937 2.2706 2.3837
2.4649 2.2822 1.9864 1.5717 1.0672 0.5046 0.2030 0.0000
-0.0348 -0.0754 -0.3248 -0.6728 -0.9976 -1.2035 -1.3108 -1.2644
-1.2035 -1.1078 -0.9657 -0.8671 -0.7395 -0.5539 -0.4205 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL

0.353 0.184 0.000 0.000 6.810 0.950
POSITION OF CENTROID FROM BASE LINE X: 11.37 Y: 0.45
SKIN AREA: 11.1935 SPAR AREA: 4.3878
ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.10277E+09 EIY: 0.46142E+10
RX: 1.27 RY: 8.51 EA: 0.63737E+08

POSITION OF TORSIONAL CENTROID X: 10.48 Y: 0.45
GJ1: 648485632.00 GJ2: 41007072.00

SECTION MASS INFORMATION WSPAR: 0.0500 WSKIN: 0.0500
SECTION RUNNING WEIGHT: 0.7967 XCG: 11.37 YCG: 0.45
ELASTIC AXIS MOMENT OF INERTIA: 59.5947
SECTION CG MOMENT OF INERTIA: 58.9623

SECTION BALLAST INFORMATION WBALLAST: 0.8620 XBAL: 1.63 YBAL: 0.25
NEW SECTION RUNNING WEIGHT: 1.6587 XCG: 6.31 YCG: 0.34
OLD SECTION CG MOMENT OF INERTIA: 140.7870
NEW SECTION CG MOMENT OF INERTIA: 98.2643

OFFSETS FOR STATION: 7 THK1: 0.3940 THK2: 0.1670 XTHK: 10.000
TSPAR: 0.2270 TSKIN: 0.1670 TSCALE: 1.0190
0.0000 0.4320 0.8640 1.7280 2.5920 3.4560 5.1840 6.9120 8.6400
12.0960 17.2800 20.7360 24.1920 27.6480 31.1040 32.8320 34.5600
32.8320 31.1040 27.6480 24.1920 20.7360 17.2800 12.0960 8.6400
6.9120 5.1840 3.4560 2.5920 1.7280 0.8640 0.4320 0.0000

0.0000 0.8593 1.2044 1.6763 1.9862 2.2151 2.5426 2.7575 2.8948
2.9934 2.7715 2.4123 1.9087 1.2960 0.6128 0.2465 0.0000
-0.0423 -0.0916 -0.3944 -0.8170 -1.2115 -1.4615 -1.5918 -1.5354
-1.4615 -1.3453 -1.1727 -1.0530 -0.8980 -0.6726 -0.5106 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL

6.150 2.000 104.000 -7.000 8.640 0.000
POSITION OF CENTROID FROM BASE LINE X: 12.17 Y: 0.38
SKIN AREA: 11.8070 SPAR AREA: 4.8286
ANGLE OF PRINCIPAL AXIS: 0.00 EIX: 0.18026E+09 EIY: 0.78827E+10
RX: 1.41 RY: 9.30 EA: 0.91142E+08

POSITION OF TORSIONAL CENTROID X: 11.16 Y: 0.35
GJ1: 1118653952.00 GJ2: 63499192.00

SECTION MASS INFORMATION WSPAR: 0.0500 WSKIN: 0.0500
SECTION RUNNING WEIGHT: 1.1393 XCG: 12.17 YCG: 0.38
ELASTIC AXIS MOMENT OF INERTIA: 101.9601
SECTION CG MOMENT OF INERTIA: 100.7866

SECTION BALLAST INFORMATION WBALLAST: 0.7300 XBAL: 8.64 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 1.8693 XCG: 10.79 YCG: 0.23
OLD SECTION CG MOMENT OF INERTIA: 110.0087
NEW SECTION CG MOMENT OF INERTIA: 106.4072

OFFSETS FOR STATION: 8 THK1: 0.3940 THK2: 0.1670 XTHK: 13.250
TSPAR: 0.2270 TSKIN: 0.1670 TSCALE: 1.0550
0.0000 0.5312 1.0625 2.1250 3.1875 4.2500 6.3750 8.5000 10.6250
14.8750 21.2500 25.5000 29.7500 34.0000 38.2500 40.3750 42.5000
40.3750 38.2500 34.0000 29.7500 25.5000 21.2500 14.8750 10.6250
8.5000 6.3750 4.2500 3.1875 2.1250 1.0625 0.5312 0.0000

0.0000 1.0940 1.5334 2.1343 2.5288 2.8203 3.2373 3.5108 3.6856
3.8112 3.5287 3.0714 2.4302 1.6500 0.7802 0.3139 0.0000
-0.0538 -0.1166 -0.5022 -1.0402 -1.5424 -1.8608 -2.0267 -1.9549
-1.8608 -1.7128 -1.4931 -1.3406 -1.1434 -0.8564 -0.6501 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL

7.440 5.000 190.000 -25.000 10.630 0.000
POSITION OF CENTROID FROM BASE LINE X: 14.94 Y: 0.50
SKIN AREA: 14.5394 SPAR AREA: 6.3910
ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.38680E+09 EIY: 0.14688E+11
RX: 1.85 RY: 11.38 EA: 0.11348E+09

POSITION OF TORSIONAL CENTROID X: 13.71 Y: 0.46
GJ1: 2091510272.00 GJ2: 128430672.00

SECTION MASS INFORMATION WSPAR: 0.0500 WSKIN: 0.0500
SECTION RUNNING WEIGHT: 1.4185 XCG: 14.94 YCG: 0.50
ELASTIC AXIS MOMENT OF INERTIA: 190.5984
SECTION CG MOMENT OF INERTIA: 188.4384

SECTION BALLAST INFORMATION WBALLAST: 0.4500 XBAL: 10.63 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 1.8685 XCG: 13.90 YCG: 0.38
OLD SECTION CG MOMENT OF INERTIA: 196.9089
NEW SECTION CG MOMENT OF INERTIA: 194.8689

OFFSETS FOR STATION: 9 THK1: 0.3940 THK2: 0.1670 XTHK: 16.000
TSPAR: 0.2270 TSKIN: 0.1670 TSCALE: 1.0700
0.0000 0.6601 1.3202 2.6405 3.9607 5.2810 7.9215 10.5620 13.2025
18.4835 26.4050 31.6860 36.9670 42.2480 47.5290 50.1695 52.8100
50.1695 47.5290 42.2480 36.9670 31.6860 26.4050 18.4835 13.2025
10.5620 7.9215 5.2810 3.9607 2.6405 1.3202 0.6601 0.0000

0.0000 1.3788 1.9325 2.6897 3.1870 3.5543 4.0798 4.4245 4.6448
 4.8031 4.4471 3.8707 3.0627 2.0794 0.9832 0.3955 0.0000
 -0.0678 -0.1469 -0.6329 -1.3110 -1.9438 -2.3450 -2.5541 -2.4637
 -2.3450 -2.1586 -1.8817 -1.6896 -1.4409 -1.0793 -0.8193 0.0000

POSITION OF TORSIONAL CENTROID X: 21.92 Y: 0.81
 GJ1: 7759856640.00 GJ2: 518288448.00

WEB DATA - AL,IX,IY,IXY,XL,YL

8.000 18.000 205.000 -50.000 13.200 0.000
 POSITION OF CENTROID FROM BASE LINE X: 18.76 Y: 0.65
 SKIN AREA: 18.0770 SPAR AREA: 7.7412
 ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.80271E+09 EIY: 0.27440E+11
 RX: 2.44 RY: 14.24 EA: 0.13527E+09

SECTION MASS INFORMATION WSPAR: 0.0500 WSKIN: 0.0500
 SECTION RUNNING WEIGHT: 2.1068 XCG: 23.89 YCG: 0.85
 ELASTIC AXIS MOMENT OF INERTIA: 712.7999
 SECTION CG MOMENT OF INERTIA: 704.5621

SECTION BALLAST INFORMATION WBALLAST: 4.8700 XBAL: 16.69 YBAL: 0.00
 NEW SECTION RUNNING WEIGHT: 6.9768 XCG: 18.86 YCG: 0.26
 OLD SECTION CG MOMENT OF INERTIA: 960.7367
 NEW SECTION CG MOMENT OF INERTIA: 781.9205

POSITION OF TORSIONAL CENTROID X: 17.20 Y: 0.60
 GJ1: 3895587328.00 GJ2: 251821952.00

SECTION MASS INFORMATION WSPAR: 0.0500 WSKIN: 0.0500
 SECTION RUNNING WEIGHT: 1.6909 XCG: 18.76 YCG: 0.65
 ELASTIC AXIS MOMENT OF INERTIA: 357.1907
 SECTION CG MOMENT OF INERTIA: 353.0326

OFFSETS FOR STATION: 11 THK1: 0.3940 THK2: 0.1670 XTHK: 21.880
 TSPAR: 0.2270 TSKIN: 0.1670 TSCALE: 1.0710
 0.0000 0.8344 1.6687 3.3375 5.0062 6.6750 10.0125 13.3500 16.6875
 23.3625 33.3750 40.0500 46.7250 53.4000 60.0750 63.4125 66.7500
 63.4125 60.0750 53.4000 46.7250 40.0500 33.3750 23.3625 16.6875
 13.3500 10.0125 6.6750 5.0062 3.3375 1.6687 0.8344 0.0000

SECTION BALLAST INFORMATION WBALLAST: 0.5900 XBAL: 13.20 YBAL: 0.00
 NEW SECTION RUNNING WEIGHT: 2.2809 XCG: 17.32 YCG: 0.48
 OLD SECTION CG MOMENT OF INERTIA: 371.5366
 NEW SECTION CG MOMENT OF INERTIA: 366.7502

0.0000 1.7443 2.4449 3.4029 4.0320 4.4967 5.1615 5.5976 5.8764
 6.0766 5.6262 4.8970 3.8747 2.6308 1.2439 0.5004 0.0000
 -0.0858 -0.1859 -0.8007 -1.6586 -2.4592 -2.9668 -3.2313 -3.1169
 -2.9668 -2.7309 -2.3806 -2.1375 -1.8230 -1.3654 -1.0366 0.0000

OFFSETS FOR STATION: 10 THK1: 0.3940 THK2: 0.1670 XTHK: 21.880
 TSPAR: 0.2270 TSKIN: 0.1670 TSCALE: 1.0710
 0.0000 0.8344 1.6687 3.3375 5.0062 6.6750 10.0125 13.3500 16.6875
 23.3625 33.3750 40.0500 46.7250 53.4000 60.0750 63.4125 66.7500
 63.4125 60.0750 53.4000 46.7250 40.0500 33.3750 23.3625 16.6875
 13.3500 10.0125 6.6750 5.0062 3.3375 1.6687 0.8344 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL
 8.750 58.000 258.000 -118.000 16.690 0.000
 POSITION OF CENTROID FROM BASE LINE X: 23.89 Y: 0.85
 SKIN AREA: 22.8496 SPAR AREA: 10.5372
 ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.17931E+10 EIY: 0.54572E+11
 RX: 3.26 RY: 17.99 EA: 0.16855E+09

POSITION OF TORSIONAL CENTROID X: 21.92 Y: 0.81
 GJ1: 7759856640.00 GJ2: 518288448.00

0.0000 1.7443 2.4449 3.4029 4.0320 4.4967 5.1615 5.5976 5.8764
 6.0766 5.6262 4.8970 3.8747 2.6308 1.2439 0.5004 0.0000
 -0.0858 -0.1859 -0.8007 -1.6586 -2.4592 -2.9668 -3.2313 -3.1169
 -2.9668 -2.7309 -2.3806 -2.1375 -1.8230 -1.3654 -1.0366 0.0000

SECTION MASS INFORMATION WSPAR: 0.0500 WSKIN: 0.0500
 SECTION RUNNING WEIGHT: 2.1068 XCG: 23.89 YCG: 0.85
 ELASTIC AXIS MOMENT OF INERTIA: 712.7999
 SECTION CG MOMENT OF INERTIA: 704.5621

WEB DATA - AL,IX,IY,IXY,XL,YL

8.750 58.000 258.000 -118.000 16.690 0.000
 POSITION OF CENTROID FROM BASE LINE X: 23.89 Y: 0.85
 SKIN AREA: 22.8496 SPAR AREA: 10.5372
 ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.17931E+10 EIY: 0.54572E+11
 RX: 3.26 RY: 17.99 EA: 0.16855E+09

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 16.69 YBAL: 0.00
 NEW SECTION RUNNING WEIGHT: 2.1068 XCG: 23.89 YCG: 0.85
 OLD SECTION CG MOMENT OF INERTIA: 704.5621
 NEW SECTION CG MOMENT OF INERTIA: 704.5620

CARTER.BLADE.AEI.DATA.ALL.TEST.WTS.5-14-88
 CARTER.BLADE.ZERO.LOAD.TEST.12-11-87 ** TCCORR = 1.000 ** PITCH = 0.000

STATIONS: 11 E-SKIN: 3999999.50 E-LONG: 3999999.50 # DIVISIONS: 500 GS: 499999.94 GL: 749999.88 WS: 0.050 WL: 0.050

STATION	TIP											ROOT										
	1	2	3	4	5	6	7	8	9	10	11	1	2	3	4	5	6	7	8	9	10	11
STA POS	0.00	40.28	80.55	120.83	161.10	201.38	241.65	281.93	322.20	362.48	402.75	0.00	40.28	80.55	120.83	161.10	201.38	241.65	281.93	322.20	362.48	402.75
X-LOAD	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Y-LOAD	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Z-LOAD	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
EA	0.3763E+08	0.3817E+08	0.2877E+08	0.3652E+08	0.4821E+08	0.6374E+08	0.9114E+08	0.1135E+09	0.1353E+09	0.1685E+09	0.1685E+09	0.3763E+08	0.3817E+08	0.2877E+08	0.3652E+08	0.4821E+08	0.6374E+08	0.9114E+08	0.1135E+09	0.1353E+09	0.1685E+09	0.1685E+09
EIYP	0.3195E+08	0.3231E+08	0.2778E+08	0.3854E+08	0.6033E+08	0.1028E+09	0.1803E+09	0.3868E+09	0.8027E+09	0.1793E+10	0.1793E+10	0.3195E+08	0.3231E+08	0.2778E+08	0.3854E+08	0.6033E+08	0.1028E+09	0.1803E+09	0.3868E+09	0.8027E+09	0.1793E+10	0.1793E+10
EIZP	0.1509E+10	0.1529E+10	0.1246E+10	0.1651E+10	0.2702E+10	0.4614E+10	0.7883E+10	0.1469E+11	0.2744E+11	0.5457E+11	0.5457E+11	0.1509E+10	0.1529E+10	0.1246E+10	0.1651E+10	0.2702E+10	0.4614E+10	0.7883E+10	0.1469E+11	0.2744E+11	0.5457E+11	0.5457E+11
GJ	0.1498E+08	0.1520E+08	0.1075E+08	0.1336E+08	0.2334E+08	0.4101E+08	0.6350E+08	0.1284E+09	0.2518E+09	0.5183E+09	0.5183E+09	0.1498E+08	0.1520E+08	0.1075E+08	0.1336E+08	0.2334E+08	0.4101E+08	0.6350E+08	0.1284E+09	0.2518E+09	0.5183E+09	0.5183E+09
SCALE	21.41	21.41	22.19	23.44	25.56	28.97	34.56	42.50	52.81	66.75	66.75	21.41	21.41	22.19	23.44	25.56	28.97	34.56	42.50	52.81	66.75	66.75
TSCALE	1.00	1.00	1.00	1.00	1.00	1.00	1.02	1.06	1.07	1.07	1.07	1.00	1.00	1.00	1.00	1.00	1.00	1.02	1.06	1.07	1.07	1.07
Y-BCENT	10.04	10.05	8.71	8.22	9.74	11.37	12.17	14.94	18.76	23.89	23.89	10.04	10.05	8.71	8.22	9.74	11.37	12.17	14.94	18.76	23.89	23.89
Z-BCENT	0.32	0.32	0.34	0.36	0.39	0.45	0.38	0.50	0.65	0.85	0.85	0.32	0.32	0.34	0.36	0.39	0.45	0.38	0.50	0.65	0.85	0.85
Y-SHRCTR	9.81	9.82	8.02	7.41	8.91	10.48	11.16	13.71	17.20	21.92	21.92	9.81	9.82	8.02	7.41	8.91	10.48	11.16	13.71	17.20	21.92	21.92
Z-SHRCTR	0.32	0.32	0.34	0.36	0.39	0.45	0.35	0.46	0.60	0.81	0.81	0.32	0.32	0.34	0.36	0.39	0.45	0.35	0.46	0.60	0.81	0.81
WEIGHT	0.47	0.48	0.36	0.46	0.60	0.80	1.14	1.42	1.69	2.11	2.11	0.47	0.48	0.36	0.46	0.60	0.80	1.14	1.42	1.69	2.11	2.11
Y-CG	10.04	10.05	8.71	8.22	9.74	11.37	12.17	14.94	18.76	23.89	23.89	10.04	10.05	8.71	8.22	9.74	11.37	12.17	14.94	18.76	23.89	23.89
Z-CG	0.32	0.32	0.34	0.36	0.39	0.45	0.38	0.50	0.65	0.85	0.85	0.32	0.32	0.34	0.36	0.39	0.45	0.38	0.50	0.65	0.85	0.85
EA	19.28	19.54	16.09	21.43	34.94	59.59	101.96	190.60	357.19	712.80	712.80	19.28	19.54	16.09	21.43	34.94	59.59	101.96	190.60	357.19	712.80	712.80
I-CG	19.26	19.52	15.92	21.13	34.52	58.96	100.79	188.44	353.03	704.56	704.56	19.26	19.52	15.92	21.13	34.52	58.96	100.79	188.44	353.03	704.56	704.56
BALLAST	0.59	0.76	0.77	0.81	0.78	0.86	0.73	0.45	0.59	4.87	0.00	0.59	0.76	0.77	0.81	0.78	0.86	0.73	0.45	0.59	4.87	0.00
TOTAL WT	1.06	1.24	1.13	1.27	1.38	1.66	1.87	1.87	2.28	6.98	2.11	1.06	1.24	1.13	1.27	1.38	1.66	1.87	1.87	2.28	6.98	2.11
NEW Y-CG	5.35	4.87	3.88	4.00	5.16	6.31	10.79	13.90	17.32	18.86	23.89	5.35	4.87	3.88	4.00	5.16	6.31	10.79	13.90	17.32	18.86	23.89

NEW Z-CG	0.28	0.28	0.28	0.29	0.31	0.34	0.23	0.38	0.48	0.26	0.85
NEW I-CG	37.84	40.33	28.23	33.82	56.91	98.26	106.41	194.87	366.75	781.92	704.56
BLD-WGHT	0.00	47.36	96.09	144.52	198.00	259.51	330.76	404.65	486.81	705.39	920.44

TORSION	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
ELTWIST	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
TENSION	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
V-INPL	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
BM-INPL	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
V-FLAP	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
BM-FLAP	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
PR A A	2.13	1.73	1.67	1.76	1.71	1.96	4.38	9.07	15.11	25.59	25.59
M.EIYP	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
M.EIZP	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00

Y-DEF.	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Z-DEF.	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
EI-FLAP	0.8797E+08	0.7841E+08	0.6398E+08	0.8914E+08	0.1407E+09	0.2607E+09	0.7811E+09	0.2699E+10	0.7929E+10	0.2519E+11	0.2519E+11
EI-LAG	0.1506E+10	0.1527E+10	0.1244E+10	0.1650E+10	0.2699E+10	0.4608E+10	0.7846E+10	0.1444E+11	0.2628E+11	0.4844E+11	0.4844E+11

M-STRAIN 1	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
M-STRAIN 2	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
M-STRAIN 3	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
M-STRAIN 4	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
M-STRAIN 5	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
M-STRAIN 6	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.

**** STOP

10.1.4 SAMPLE RUN, ESI BLADE, NO TIPS OR BALLAST

This sample run is for the ESI blade to illustrate a simpler blade structure, in this case wood-epoxy laminated veneers with no ballast or mechanisms.

INPUT BLADE FILE: ESIBL

```

11 1.88 1.88 500 0.133 0.133
33 01.00 0.0245 0.0245
.0000 .0004 .0020 .0040 .0080 .0199 .0399 .0798 .1196
.1994 .2792 .3589 .4786 .5982 .7179 .8375 1.0000
.8375 .7179 .5982 .4786 .3589 .2792 .1994 .1196
.0798 .0399 .0199 .0080 .0040 .0020 .0004 .0000
.0000 .0060 .0140 .0211 .0289 .0463 .0636 .0842 .0981
.1123 .1173 .1161 .1053 .0879 .0664 .0413 .0000
-.0377 -.0618 -.0828 -.0981 -.1065 -.1067 -.1013 -.0861
-.0730 -.0534 -.0389 -.0257 -.0182 -.0126 -.0060 .0000
006
.0000 .2792 .4786 1.0000 .4786 .2792
.0000 .1173 .1053 .0000 -.0981 -.1067
0.000 0.85 1.65 2.49 3.29 4.10 4.94 5.75 6.55 7.71 7.71
11.65 0.690 .00001 0.472 5.00
1.236 0.272 0.00 0.00 4.00 0.00
0.00 0.00 0.00
13.66 0.827 .00001 0.472 5.00
1.236 0.272 0.00 0.00 4.31 0.00
0.00 0.00 0.00
15.43 0.961 .00001 0.472 5.00
1.653 0.769 0.00 0.00 4.90 0.00
0.00 0.00 0.00
17.28 1.192 .00001 0.555 5.00
1.494 1.112 0.00 0.00 5.63 0.00
0.00 0.00 0.00
19.09 1.150 .00001 0.677 5.00
0.964 1.040 0.00 0.00 6.30 0.00
0.00 0.00 0.00
20.87 1.226 .00001 0.681 5.00
1.187 1.940 0.00 0.00 6.81 0.00
0.00 0.00 0.00
22.72 1.291 .00001 0.780 5.00
1.350 2.858 0.00 0.00 7.32 0.00
0.00 0.00 0.00
24.40 1.376 .00001 0.980 5.00
1.503 3.943 0.00 0.00 7.97 0.00
0.00 0.00 0.00
20.85 1.771 .00001 1.080 5.00
1.641 5.124 0.00 0.00 8.15 0.00
0.00 0.00 0.00
16.80 2.416 .00001 2.490 5.00
1.081 1.467 0.00 0.00 8.15 0.00
0.00 0.00 0.00
16.80 2.416 .00001 2.490 5.00
1.081 1.467 0.00 0.00 8.15 0.00
0.00 0.00 0.00
ESI.BLADE.1ST.TRY.MEAS.DATA.1-24-88

```

INPUT LOAD FILE: ESILDZERO

```

001
00.00 32.41 64.81 97.22 129.62 162.03 194.44 226.84
259.25 291.65 324.06
00.00 00.00 00.00 00.00 00.00 00.00 00.00 00.00
00.00 00.00 00.00
00.00 00.00 00.00 00.00 00.00 00.00 00.00 00.00
00.00 00.00 00.00
00.00 00.00 00.00 00.00 00.00 00.00 00.00 00.00
00.00 00.00 00.00
00.00 00.00 00.00
00.00 00.00 00.00
0.00 324.06 0.00 0.00
ESI.BLADE.ZERO.LOAD.1-24-88

```

TERMINAL INPUT AND OUTPUT: STRESSM

```

ENTER INPUT FILE NAME FOR SECTION GEOMETRY:
ESIBL
ENTER INPUT FILE NAME FOR APPLIED LOADING:
ESILDZERO
ESI.BLADE.ZERO.LOAD.1-24-88      ** TCORR = 1.000 ** PITCH = 0.000

```

```

# STATIONS: 11 E-SKIN: 1880000.00 E-LONG: 1879999.75 # DIVISIONS: 500 GS:
132999.97 GL: 132999.97 WS: 0.025 WL: 0.025

```

```

OFFSETS FOR STATION: 1 THK1: 0.4720 THK2: 0.4720 XTHK: 5.000
TSPAR: 0.0000 TSKIN: 0.4720 TSCALE: 0.6900
0.0000 0.0047 0.0233 0.0466 0.0932 0.2318 0.4648 0.9297 1.3933
2.3230 3.2527 4.1812 5.5757 6.9690 8.3635 9.7569 11.6500
9.7569 8.3635 6.9690 5.5757 4.1812 3.2527 2.3230 1.3933
0.9297 0.4648 0.2318 0.0932 0.0466 0.0233 0.0047 0.0000

0.0000 0.0482 0.1125 0.1696 0.2323 0.3722 0.5112 0.6768 0.7886
0.9027 0.9429 0.9333 0.8465 0.7066 0.5338 0.3320 0.0000
-0.3031 -0.4968 -0.6656 -0.7886 -0.8561 -0.8577 -0.8143 -0.6921
-0.5868 -0.4293 -0.3127 -0.2066 -0.1463 -0.1013 -0.0482 0.0000

```

```

WEB DATA - AL,IX,IY,IXY,XL,YL
1.236 0.272 0.000 0.000 4.000 0.000
POSITION OF CENTROID FROM BASE LINE X: 5.54 Y: 0.03
SKIN AREA: 11.3314 SPAR AREA: 0.0001
ANGLE OF PRINCIPAL AXIS: 0.00 EIX: 0.49275E+07 EIY: 0.25157E+09
RX: 0.46 RY: 3.26 EA: 0.23627E+08

```

POSITION OF TORSIONAL CENTROID X: 5.54 Y: 0.03
GJ1: 18146008.00 GJ2: 2125619.00

SECTION MASS INFORMATION WSPAR: 0.0245 WSKIN: 0.0245
SECTION RUNNING WEIGHT: 0.3079 XCG: 5.54 YCG: 0.03
ELASTIC AXIS MOMENT OF INERTIA: 3.3427
SECTION CG MOMENT OF INERTIA: 3.3427

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 0.00 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 0.3079 XCG: 5.54 YCG: 0.03
OLD SECTION CG MOMENT OF INERTIA: 3.3427
NEW SECTION CG MOMENT OF INERTIA: 3.3427

OFFSETS FOR STATION: 2 THK1:0.4720 THK2:0.4720 XTHK: 5.000
TSPAR:0.0000 TSKIN:0.4720 TSCALE:0.8270
0.0000 0.0055 0.0273 0.0546 0.1093 0.2718 0.5450 1.0901 1.6337
2.7238 3.8139 4.9026 6.5377 8.1714 9.8065 11.4402 13.6600
11.4402 9.8065 8.1714 6.5377 4.9026 3.8139 2.7238 1.6337
1.0901 0.5450 0.2718 0.1093 0.0546 0.0273 0.0055 0.0000

0.0000 0.0678 0.1582 0.2384 0.3265 0.5230 0.7185 0.9512 1.1082
1.2686 1.3251 1.3116 1.1896 0.9930 0.7501 0.4666 0.0000
-0.4259 -0.6981 -0.9354 -1.1082 -1.2031 -1.2054 -1.1444 -0.9727
-0.8247 -0.6033 -0.4394 -0.2903 -0.2056 -0.1423 -0.0678 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL
1.236 0.272 0.000 0.000 4.310 0.000
POSITION OF CENTROID FROM BASE LINE X: 6.45 Y: 0.04
SKIN AREA: 13.4188 SPAR AREA: 0.0001
ANGLE OF PRINCIPAL AXIS: 0.00 EIX: 0.13500E+08 EIY: 0.41608E+09
RX: 0.70 RY: 3.89 EA: 0.27551E+08

POSITION OF TORSIONAL CENTROID X: 6.45 Y: 0.04
GJ1: 30390596.00 GJ2: 4873743.00

SECTION MASS INFORMATION WSPAR: 0.0245 WSKIN: 0.0245
SECTION RUNNING WEIGHT: 0.3590 XCG: 6.45 YCG: 0.04
ELASTIC AXIS MOMENT OF INERTIA: 5.5983
SECTION CG MOMENT OF INERTIA: 5.5983

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 0.00 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 0.3590 XCG: 6.45 YCG: 0.04
OLD SECTION CG MOMENT OF INERTIA: 5.5983
NEW SECTION CG MOMENT OF INERTIA: 5.5983

OFFSETS FOR STATION: 3 THK1:0.4720 THK2:0.4720 XTHK: 5.000
TSPAR:0.0000 TSKIN:0.4720 TSCALE:0.9610
0.0000 0.0062 0.0309 0.0617 0.1234 0.3071 0.6157 1.2313 1.8454
3.0767 4.3081 5.5378 7.3848 9.2302 11.0772 12.9226 15.4300
12.9226 11.0772 9.2302 7.3848 5.5378 4.3081 3.0767 1.8454
1.2313 0.6157 0.3071 0.1234 0.0617 0.0309 0.0062 0.0000

0.0000 0.0890 0.2076 0.3129 0.4285 0.6865 0.9431 1.2485 1.4546
1.6652 1.7394 1.7216 1.5614 1.3034 0.9846 0.6124 0.0000
-0.5590 -0.9164 -1.2278 -1.4546 -1.5792 -1.5822 -1.5021 -1.2767
-1.0825 -0.7918 -0.5768 -0.3811 -0.2699 -0.1868 -0.0890 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL
1.653 0.769 0.000 0.000 4.900 0.000
POSITION OF CENTROID FROM BASE LINE X: 7.21 Y: 0.05
SKIN AREA: 15.3179 SPAR AREA: 0.0001
ANGLE OF PRINCIPAL AXIS: 0.00 EIX: 0.30363E+08 EIY: 0.61440E+09
RX: 0.98 RY: 4.39 EA: 0.31905E+08

POSITION OF TORSIONAL CENTROID X: 7.21 Y: 0.05
GJ1: 45613552.00 GJ2: 9385898.00

SECTION MASS INFORMATION WSPAR: 0.0245 WSKIN: 0.0245
SECTION RUNNING WEIGHT: 0.4158 XCG: 7.21 YCG: 0.05
ELASTIC AXIS MOMENT OF INERTIA: 8.4025
SECTION CG MOMENT OF INERTIA: 8.4025

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 0.00 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 0.4158 XCG: 7.21 YCG: 0.05
OLD SECTION CG MOMENT OF INERTIA: 8.4025
NEW SECTION CG MOMENT OF INERTIA: 8.4025

OFFSETS FOR STATION: 4 THK1:0.5550 THK2:0.5550 XTHK: 5.000
TSPAR:0.0000 TSKIN:0.5550 TSCALE:1.1920
0.0000 0.0069 0.0346 0.0691 0.1382 0.3439 0.6895 1.3789 2.0667

3.4456 4.8246 6.2018 8.2702 10.3369 12.4053 14.4720 17.2800
14.4720 12.4053 10.3369 8.2702 6.2018 4.8246 3.4456 2.0667
1.3789 0.6895 0.3439 0.1382 0.0691 0.0346 0.0069 0.0000

0.0000 0.1236 0.2884 0.4346 0.5953 0.9537 1.3100 1.7343 2.0206
2.3131 2.4161 2.3914 2.1689 1.8105 1.3677 0.8507 0.0000
-0.7765 -1.2729 -1.7055 -2.0206 -2.1937 -2.1978 -2.0866 -1.7735
-1.5036 -1.0999 -0.8013 -0.5294 -0.3749 -0.2595 -0.1236 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL
1.494 1.112 0.000 0.000 5.630 0.000
POSITION OF CENTROID FROM BASE LINE X: 8.09 Y: 0.07
SKIN AREA: 20.5731 SPAR AREA: 0.0001
ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.80980E+08 EIY: 0.10365E+10
RX: 1.40 RY: 5.00 EA: 0.41486E+08

POSITION OF TORSIONAL CENTROID X: 8.09 Y: 0.07
GJ1: 79055488.00 GJ2: 23382744.00

SECTION MASS INFORMATION WSPAR: 0.0245 WSKIN: 0.0245
SECTION RUNNING WEIGHT: 0.5406 XCG: 8.09 YCG: 0.07
ELASTIC AXIS MOMENT OF INERTIA: 14.5629
SECTION CG MOMENT OF INERTIA: 14.5629

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 0.00 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 0.5406 XCG: 8.09 YCG: 0.07
OLD SECTION CG MOMENT OF INERTIA: 14.5629
NEW SECTION CG MOMENT OF INERTIA: 14.5629

OFFSETS FOR STATION: 5 THK1:0.6770 THK2:0.6770 XTHK: 5.000
TSPAR:0.0000 TSKIN:0.6770 TSCALE:1.1500
0.0000 0.0076 0.0382 0.0764 0.1527 0.3799 0.7617 1.5234 2.2832
3.8065 5.3299 6.8514 9.1365 11.4196 13.7047 15.9879 19.0900
15.9879 13.7047 11.4196 9.1365 6.8514 5.3299 3.8065 2.2832
1.5234 0.7617 0.3799 0.1527 0.0764 0.0382 0.0076 0.0000

0.0000 0.1317 0.3073 0.4632 0.6345 1.0164 1.3962 1.8485 2.1536
2.4654 2.5751 2.5488 2.3117 1.9297 1.4577 0.9067 0.0000
-0.8276 -1.3567 -1.8178 -2.1536 -2.3380 -2.3424 -2.2239 -1.8902
-1.6026 -1.1723 -0.8540 -0.5642 -0.3996 -0.2766 -0.1317 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL
0.964 1.040 0.000 0.000 6.300 0.000
POSITION OF CENTROID FROM BASE LINE X: 9.06 Y: 0.08
SKIN AREA: 27.6210 SPAR AREA: 0.0001
ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.11679E+09 EIY: 0.16730E+10
RX: 1.47 RY: 5.58 EA: 0.53740E+08

POSITION OF TORSIONAL CENTROID X: 9.06 Y: 0.08
GJ1: 126615280.00 GJ2: 35927928.00

SECTION MASS INFORMATION WSPAR: 0.0245 WSKIN: 0.0245
SECTION RUNNING WEIGHT: 0.7003 XCG: 9.06 YCG: 0.08
ELASTIC AXIS MOMENT OF INERTIA: 23.3238
SECTION CG MOMENT OF INERTIA: 23.3238

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 0.00 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 0.7003 XCG: 9.06 YCG: 0.08
OLD SECTION CG MOMENT OF INERTIA: 23.3238
NEW SECTION CG MOMENT OF INERTIA: 23.3238

OFFSETS FOR STATION: 6 THK1:0.6810 THK2:0.6810 XTHK: 5.000
TSPAR:0.0000 TSKIN:0.6810 TSCALE:1.2260
0.0000 0.0083 0.0417 0.0835 0.1670 0.4153 0.8327 1.6654 2.4961
4.1615 5.8269 7.4902 9.9884 12.4844 14.9826 17.4786 20.8700
17.4786 14.9826 12.4844 9.9884 7.4902 5.8269 4.1615 2.4961
1.6654 0.8327 0.4153 0.1670 0.0835 0.0417 0.0083 0.0000

0.0000 0.1535 0.3582 0.5399 0.7395 1.1847 1.6273 2.1544 2.5100
2.8734 3.0013 2.9706 2.6943 2.2491 1.6990 1.0567 0.0000
-0.9646 -1.5813 -2.1186 -2.5100 -2.7250 -2.7301 -2.5919 -2.2030
-1.8678 -1.3663 -0.9953 -0.6576 -0.4657 -0.3224 -0.1535 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL
1.187 1.940 0.000 0.000 6.810 0.000
POSITION OF CENTROID FROM BASE LINE X: 9.85 Y: 0.09
SKIN AREA: 30.5815 SPAR AREA: 0.0001
ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.18505E+09 EIY: 0.22333E+10
RX: 1.76 RY: 6.11 EA: 0.59725E+08

POSITION OF TORSIONAL CENTROID X: 9.85 Y: 0.09
GJ1: 171082528.00 GJ2: 53306408.00

SECTION MASS INFORMATION WSPAR: 0.0245 WSKIN: 0.0245
SECTION RUNNING WEIGHT: 0.7783 XCG: 9.85 YCG: 0.09
ELASTIC AXIS MOMENT OF INERTIA: 31.5152
SECTION CG MOMENT OF INERTIA: 31.5152

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 0.00 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 0.7783 XCG: 9.85 YCG: 0.09
OLD SECTION CG MOMENT OF INERTIA: 31.5152
NEW SECTION CG MOMENT OF INERTIA: 31.5152

1.6638 0.8319 0.4149 0.1668 0.0834 0.0417 0.0083 0.0000
0.0000 0.2216 0.5170 0.7791 1.0671 1.7096 2.3485 3.1091 3.6224
4.1467 4.3313 4.2870 3.8882 3.2457 2.4518 1.5250 0.0000
-1.3921 -2.2820 -3.0574 -3.6224 -3.9325 -3.9399 -3.7405 -3.1793
-2.6956 -1.9718 -1.4364 -0.9490 -0.6720 -0.4653 -0.2216 0.0000

OFFSETS FOR STATION: 7 THK1: 0.7800 THK2: 0.7800 XTHK: 5.000
TSPAR: 0.0000 TSKIN: 0.7800 TSCALE: 1.2910
0.0000 0.0091 0.0454 0.0909 0.1818 0.4521 0.9065 1.8131 2.7173
4.5304 6.3434 8.1542 10.8738 13.5911 16.3107 19.0280 22.7200
19.0280 16.3107 13.5911 10.8738 8.1542 6.3434 4.5304 2.7173
1.8131 0.9065 0.4521 0.1818 0.0909 0.0454 0.0091 0.0000

0.0000 0.1760 0.4106 0.6189 0.8477 1.3580 1.8655 2.4697 2.8774
3.2939 3.4406 3.4054 3.0886 2.5782 1.9476 1.2114 0.0000
-1.1058 -1.8127 -2.4286 -2.8774 -3.1238 -3.1297 -2.9713 -2.5254
-2.1412 -1.5663 -1.1410 -0.7538 -0.5338 -0.3696 -0.1760 0.0000

WEB DATA - AL,IX,IY,IXY,XL,YL
1.641 5.124 0.000 0.000 8.150 0.000
POSITION OF CENTROID FROM BASE LINE X: 9.67 Y: 0.13
SKIN AREA: 51.0920 SPAR AREA: 0.0001
ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.60742E+09 EIY: 0.37357E+10
RX: 2.48 RY: 6.14 EA: 0.99138E+08

POSITION OF TORSIONAL CENTROID X: 9.67 Y: 0.13
GJ1: 307252352.00 GJ2: 166813920.00

SECTION MASS INFORMATION WSPAR: 0.0245 WSKIN: 0.0245
SECTION RUNNING WEIGHT: 1.2920 XCG: 9.67 YCG: 0.13
ELASTIC AXIS MOMENT OF INERTIA: 56.5991
SECTION CG MOMENT OF INERTIA: 56.5991

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 0.00 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 1.2920 XCG: 9.67 YCG: 0.13
OLD SECTION CG MOMENT OF INERTIA: 56.5991
NEW SECTION CG MOMENT OF INERTIA: 56.5990

WEB DATA - AL,IX,IY,IXY,XL,YL
1.350 2.858 0.000 0.000 7.320 0.000
POSITION OF CENTROID FROM BASE LINE X: 10.70 Y: 0.10
SKIN AREA: 38.3593 SPAR AREA: 0.0001
ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.30366E+09 EIY: 0.33267E+10
RX: 2.02 RY: 6.68 EA: 0.74654E+08

OFFSETS FOR STATION: 10 THK1: 2.4900 THK2: 2.4900 XTHK: 5.000
TSPAR: 0.0000 TSKIN: 2.4900 TSCALE: 2.4160
0.0000 0.0067 0.0336 0.0672 0.1344 0.3343 0.6703 1.3406 2.0093
3.3499 4.6906 6.0295 8.0405 10.0498 12.0607 14.0700 16.8000
14.0700 12.0607 10.0498 8.0405 6.0295 4.6906 3.3499 2.0093
1.3406 0.6703 0.3343 0.1344 0.0672 0.0336 0.0067 0.0000

0.0000 0.2435 0.5682 0.8564 1.1730 1.8793 2.5814 3.4176 3.9818
4.5581 4.7611 4.7124 4.2740 3.5678 2.6951 1.6763 0.0000
-1.5302 -2.5084 -3.3608 -3.9818 -4.3227 -4.3308 -4.1116 -3.4947
-2.9630 -2.1674 -1.5789 -1.0431 -0.7387 -0.5114 -0.2435 0.0000

POSITION OF TORSIONAL CENTROID X: 10.70 Y: 0.10
GJ1: 256825120.00 GJ2: 86833120.00

WEB DATA - AL,IX,IY,IXY,XL,YL
1.081 1.467 0.000 0.000 8.150 0.000
POSITION OF CENTROID FROM BASE LINE X: 7.63 Y: 0.15
SKIN AREA: 101.7326 SPAR AREA: 0.0002
ANGLE OF PRINCIPAL AXIS: -0.02 EIX: 0.94100E+09 EIY: 0.42585E+10
RX: 2.21 RY: 4.69 EA: 0.19329E+09

SECTION MASS INFORMATION WSPAR: 0.0245 WSKIN: 0.0245
SECTION RUNNING WEIGHT: 0.9729 XCG: 10.70 YCG: 0.10
ELASTIC AXIS MOMENT OF INERTIA: 47.3099
SECTION CG MOMENT OF INERTIA: 47.3099

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 0.00 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 0.9729 XCG: 10.70 YCG: 0.10
OLD SECTION CG MOMENT OF INERTIA: 47.3099
NEW SECTION CG MOMENT OF INERTIA: 47.3099

POSITION OF TORSIONAL CENTROID X: 7.63 Y: 0.15
GJ1: 367838144.00 GJ2: 349336960.00

SECTION MASS INFORMATION WSPAR: 0.0245 WSKIN: 0.0245
SECTION RUNNING WEIGHT: 2.5189 XCG: 7.63 YCG: 0.15
ELASTIC AXIS MOMENT OF INERTIA: 67.7597
SECTION CG MOMENT OF INERTIA: 67.7597

OFFSETS FOR STATION: 8 THK1: 0.9800 THK2: 0.9800 XTHK: 5.000
TSPAR: 0.0000 TSKIN: 0.9800 TSCALE: 1.3760
0.0000 0.0098 0.0488 0.0976 0.1952 0.4856 0.9736 1.9471 2.9182
4.8654 6.8125 8.7572 11.6778 14.5961 17.5168 20.4350 24.4000
20.4350 17.5168 14.5961 11.6778 8.7572 6.8125 4.8654 2.9182
1.9471 0.9736 0.4856 0.1952 0.0976 0.0488 0.0098 0.0000

0.0000 0.2014 0.4700 0.7084 0.9703 1.5545 2.1353 2.8270 3.2936
3.7704 3.9383 3.8980 3.5354 2.9512 2.2293 1.3866 0.0000
-1.2658 -2.0749 -2.7800 -3.2936 -3.5757 -3.5824 -3.4011 -2.8908
-2.4509 -1.7929 -1.3060 -0.8629 -0.6111 -0.4230 -0.2014 0.0000

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 0.00 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 2.5189 XCG: 7.63 YCG: 0.15
OLD SECTION CG MOMENT OF INERTIA: 67.7597
NEW SECTION CG MOMENT OF INERTIA: 67.7596

WEB DATA - AL,IX,IY,IXY,XL,YL
1.503 3.943 0.000 0.000 7.970 0.000
POSITION OF CENTROID FROM BASE LINE X: 11.48 Y: 0.12
SKIN AREA: 52.1716 SPAR AREA: 0.0001
ANGLE OF PRINCIPAL AXIS: -0.01 EIX: 0.52050E+09 EIY: 0.52037E+10
RX: 2.27 RY: 7.18 EA: 0.10091E+09

OFFSETS FOR STATION: 11 THK1: 2.4900 THK2: 2.4900 XTHK: 5.000
TSPAR: 0.0000 TSKIN: 2.4900 TSCALE: 2.4160
0.0000 0.0067 0.0336 0.0672 0.1344 0.3343 0.6703 1.3406 2.0093
3.3499 4.6906 6.0295 8.0405 10.0498 12.0607 14.0700 16.8000
14.0700 12.0607 10.0498 8.0405 6.0295 4.6906 3.3499 2.0093
1.3406 0.6703 0.3343 0.1344 0.0672 0.0336 0.0067 0.0000

POSITION OF TORSIONAL CENTROID X: 11.48 Y: 0.12
GJ1: 404957888.00 GJ2: 152297408.00

0.0000 0.2435 0.5682 0.8564 1.1730 1.8793 2.5814 3.4176 3.9818
4.5581 4.7611 4.7124 4.2740 3.5678 2.6951 1.6763 0.0000
-1.5302 -2.5084 -3.3608 -3.9818 -4.3227 -4.3308 -4.1116 -3.4947
-2.9630 -2.1674 -1.5789 -1.0431 -0.7387 -0.5114 -0.2435 0.0000

SECTION MASS INFORMATION WSPAR: 0.0245 WSKIN: 0.0245
SECTION RUNNING WEIGHT: 1.3150 XCG: 11.48 YCG: 0.12
ELASTIC AXIS MOMENT OF INERTIA: 74.5975
SECTION CG MOMENT OF INERTIA: 74.5975

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 0.00 YBAL: 0.00
NEW SECTION RUNNING WEIGHT: 1.3150 XCG: 11.48 YCG: 0.12
OLD SECTION CG MOMENT OF INERTIA: 74.5975
NEW SECTION CG MOMENT OF INERTIA: 74.5975

WEB DATA - AL,IX,IY,IXY,XL,YL
1.081 1.467 0.000 0.000 8.150 0.000
POSITION OF CENTROID FROM BASE LINE X: 7.63 Y: 0.15
SKIN AREA: 101.7326 SPAR AREA: 0.0002
ANGLE OF PRINCIPAL AXIS: -0.02 EIX: 0.94100E+09 EIY: 0.42585E+10
RX: 2.21 RY: 4.69 EA: 0.19329E+09

OFFSETS FOR STATION: 9 THK1: 1.0800 THK2: 1.0800 XTHK: 5.000
TSPAR: 0.0000 TSKIN: 1.0800 TSCALE: 1.7710
0.0000 0.0083 0.0417 0.0834 0.1668 0.4149 0.8319 1.6638 2.4937
4.1575 5.8213 7.4831 9.9788 12.4725 14.9682 17.4619 20.8500
17.4619 14.9682 12.4725 9.9788 7.4831 5.8213 4.1575 2.4937

POSITION OF TORSIONAL CENTROID X: 7.63 Y: 0.15
GJ1: 367838144.00 GJ2: 349336960.00

SECTION MASS INFORMATION WSPAR: 0.0245 WSKIN: 0.0245

SECTION RUNNING WEIGHT: 2.5189 XCG: 7.63 YCG: 0.15
 ELASTIC AXIS MOMENT OF INERTIA: 67.7597
 SECTION CG MOMENT OF INERTIA: 67.7597

SECTION BALLAST INFORMATION WBALLAST: 0.0000 XBAL: 0.00 YBAL: 0.00
 NEW SECTION RUNNING WEIGHT: 2.5189 XCG: 7.63 YCG: 0.15
 OLD SECTION CG MOMENT OF INERTIA: 67.7597
 NEW SECTION CG MOMENT OF INERTIA: 67.7596

 ESI.BLADE.1ST.TRY.MEAS.DATA.1-24-88
 ESI.BLADE.ZERO.LOAD.1-24-88 ** TCORR = 1.000 ** PITCH = 0.000

STATIONS: 11 E-SKIN: 1880000.00 E-LONG: 1879999.75 # DIVISIONS: 500 GS: 132999.97 GL: 132999.97 WS: 0.025 WL: 0.025

STATION	TIP										ROOT
	1	2	3	4	5	6	7	8	9	10	11
STA POS	0.00	32.41	64.81	97.22	129.62	162.03	194.44	226.84	259.25	291.65	324.06
X-LOAD	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Y-LOAD	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Z-LOAD	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
EA	0.2363E+08	0.2755E+08	0.3191E+08	0.4149E+08	0.5374E+08	0.5973E+08	0.7465E+08	0.1009E+09	0.9914E+08	0.1933E+09	0.1933E+09
EIYP	0.4928E+07	0.1350E+08	0.3036E+08	0.8098E+08	0.1168E+09	0.1850E+09	0.3037E+09	0.5205E+09	0.6074E+09	0.9410E+09	0.9410E+09
EIZP	0.2516E+09	0.4161E+09	0.6144E+09	0.1036E+10	0.1673E+10	0.2233E+10	0.3327E+10	0.5204E+10	0.3736E+10	0.4259E+10	0.4259E+10
GJ	0.2126E+07	0.4874E+07	0.9386E+07	0.2338E+08	0.3593E+08	0.5331E+08	0.8683E+08	0.1523E+09	0.1668E+09	0.3493E+09	0.3493E+09
SCALE	11.65	13.66	15.43	17.28	19.09	20.87	22.72	24.40	20.85	16.80	16.80
TSCALE	0.69	0.83	0.96	1.19	1.15	1.23	1.29	1.38	1.77	2.42	2.42
Y-BCENT	5.54	6.45	7.21	8.09	9.06	9.85	10.70	11.48	9.67	7.63	7.63
Z-BCENT	0.03	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15
Y-SHRCTR	5.54	6.45	7.21	8.09	9.06	9.85	10.70	11.48	9.67	7.63	7.63
Z-SHRCTR	0.03	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15
WEIGHT	0.31	0.36	0.42	0.54	0.70	0.78	0.97	1.32	1.29	2.52	2.52
Y-CG	5.54	6.45	7.21	8.09	9.06	9.85	10.70	11.48	9.67	7.63	7.63
Z-CG	0.03	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15
IEA	3.34	5.60	8.40	14.56	23.32	31.52	47.31	74.60	56.60	67.76	67.76
ICG	3.34	5.60	8.40	14.56	23.32	31.52	47.31	74.60	56.60	67.76	67.76
BALLAST	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
TOTAL WT	0.31	0.36	0.42	0.54	0.70	0.78	0.97	1.32	1.29	2.52	2.52
NEW Y-CG	5.54	6.45	7.21	8.09	9.06	9.85	10.70	11.48	9.67	7.63	7.63
NEW Z-CG	0.03	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15
NEW I-CG	3.34	5.60	8.40	14.56	23.32	31.52	47.31	74.60	56.60	67.76	67.76
BLD-WGHT	0.00	10.79	23.33	38.73	58.74	82.39	110.46	148.51	191.74	256.80	341.76
TORSION	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
EL TWIST	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
TENSION	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
V-INPL	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
BM-INPL	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
V-FLAP	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
BM-FLAP	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
PR A A	-0.18	0.63	1.39	2.14	2.93	3.72	4.53	5.31	5.92	6.66	6.66
M.EIYP	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
M.EIZP	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00	0.0000E+00
Y-DEF.	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Z-DEF.	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
EI-FLAP	0.4116E+07	0.1805E+08	0.4528E+08	0.1196E+09	0.2023E+09	0.3295E+09	0.5657E+09	0.9995E+09	0.9895E+09	0.1429E+10	0.1429E+10
EI-LAG	0.2516E+09	0.4159E+09	0.6135E+09	0.1033E+10	0.1665E+10	0.2217E+10	0.3292E+10	0.5133E+10	0.3653E+10	0.4121E+10	0.4121E+10
M-STRAIN 1	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
M-STRAIN 2	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
M-STRAIN 3	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
M-STRAIN 4	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
M-STRAIN 5	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
M-STRAIN 6	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.

**** STOP

10.2: COMPUTER CODE: BLADE DEFLECTIONS AND STRESSES UNDER LOADING

10.2.1 BRIEF DISCUSSION

The bending and twisting calculations are performed using the method described in Chapter 4. The program listed above in Appendix 10.1 (STRESSM) contains in Part 2 the static bending deflection and strain calculations. In summary, this code has the following characteristics:

Uses linear beam theory (plane sections remain plane).

Employs a second order curve-fit routine to integrate spanwise values.

Has provision for full three-dimensional applied loading and torques, although only three axis forces and the torsional moment are applied.

Boundary conditions are for a simple cantilever with arbitrary section properties.

Calculates deflections and twist relative to the local principal axes and shear center and transforms the values to the specified blade feathering (control) axis.

No small deflections are assumed, and the transverse shear strain terms are calculated and applied in the bending calculations.

The program provides simultaneous bending deflections in transverse planes (flapping and lead-lag) and torsional deflections (elastic twist).

10.2.2 PROGRAM LISTING, STRESSDATA

Program STRESSDATA is listed here. This code has provision for substitution of known (or test) beam section values prior to the deflection calculations. STRESSDATA includes the elastic twist calculations for a flying blade (e.g., including centrifugal force) which STRESSM does not, since it is for a static bench test. Note: portions which are identical to STRESSM (App. 10.1.2) and STRESSELASTIC (App.

10.5.1) have been excised ("...see [-]...").

```
C ..... STRESSDATA .....
C ..... ALTERNATIVE ENERGY INSTITUTE .....
C ..... WEST TEXAS STATE UNIVERSITY .....
C ..... CANYON TX 79016 806-656-2295 .....
C
C THIS PROGRAM CALCULATES THE SECTION MASS AND STIFFNESS PROPERTIES
C OF AN ARBITRARY EXTERNAL GEOMETRY (TWIST, TAPER) AND SPECIFIED
C INTERNAL SPAR-SHELL STRUCTURAL GEOMETRY, COMPOSITE WIND TURBINE
C BLADE, AND THEN FINDS THE BENDING DEFLECTIONS, ELASTIC TWIST, AND
C STRAINS IN THE LAMINATE FOR A SPECIFIED EXTERNAL LOAD AND ROTOR
C ROTATIONAL SPEED.
C
C THIS VERSION OF THE PROGRAM ("STRESSDATA") CONTAINS ALL THE ROUTINES
C AND PRINTS OUT THE TORSIONAL DEFLECTIONS (ELASTIC TWIST) DUE TO ALL
C INERTIAL AND APPLIED LOADS, AND DUE TO BLADE BENDING.
C
C THIS VERSION CALCULATES THE SECTION MASS AND AREA DATA, BUT USES
C ANOTHER
C INPUT FILE FOR THE SECTION STIFFNESS PROPERTIES. THIS ALTERNATE DATA
C IS ASSUMED TO BE FROM ANOTHER SOURCE FOR CHECKING.
C
C THIS PROGRAM WAS ORIGINALLY WRITTEN BY E.VAN DUSEN (COMPOSITE
C ENGINEERING, INC., 277 BAKER AVE., CONCORD MA 01742 617-371-3132)
C IN 1977 FOR COMPOSITE RACING SHELLS AND HAS BEEN EXTENSIVELY REWRITTEN
C BY F.S. STODDARD (AEI, BOX 248 WTSU, CANYON TX 79016 806-656-2295) FOR
C WIND TURBINE BLADES.
C
C GENERAL PROGRAM DESCRIPTION:
C
C PART 1 OF THIS PROGRAM CALCULATES THE SECTION PROPERTIES OF AN
C ARBITRARY COMPOSITE BEAM, USING EITHER ACTUAL THICKNESSES AND
C MODULI OF PLIES, OR USING MODULUS-WEIGHTING (THE LATTER ROUTINE
C HAS BEEN SUPPRESSED FOR THE CALCULATION OF ELASTIC TWIST). THE
C BEAM (BLADE) IS MODELED BY A SHELL OF CONSTANT THICKNESS WITH
C ARBITRARY LONGITUDINAL MODULUS AND SHEAR MODULUS, AND A SPAR
C CONSISTING OF A NOSE SECTION EMBEDDED NEXT TO THE SHELL IN THE
C BLADE LEADING EDGE, AND A WEB (OR WEBS) LOCATED INTERNALLY. THE
C SPAR ALSO HAS CONSTANT THICKNESS AND ARBITRARY LONGITUDINAL AND
C SHEAR MODULI. THE WEB IS DESCRIBED ONLY BY ITS MOMENTS OF INERTIA
C AND CENTROID IN THE SECTION AXIS SYSTEM. MULTI-WEB BEAM DESIGNS ARE
C HANDLED BY INCLUDING ALL THE ACTUAL WEBS INTO THIS ONE MODEL WEB.
C
C THE SECTION AIRFOIL SHAPES ARE DETERMINED BY SCALING AN INPUT
C TABLE OF OFFSETS. THE CHORD SCALING FACTOR (SCALE) DETERMINES THE
C AIRFOIL CHORD AND THE THICKNESS SCALING FACTOR (TSCALE) DETERMINES
C THE SECTION THICKNESS PROPORTIONAL TO THE TABLE OF OFFSETS.
C ALL THESE INPUT DATA ARE INCLUDED IN A DATAFILE GIVING THE GEOMETRY
C AND STRUCTURAL DESIGN OF THE BLADE OF INTEREST.
C
C IN PART 1 PARABOLIC SEGMENTS ARE FITTED THROUGH EACH GROUP OF THREE
C SECTION OFFSET POINTS. THE I OF THE SKIN+SPAR+WEB IS CALCULATED BY
C DIVIDING THE GROUP OF POINTS INTO (DIV) STRAIGHT LINE SEGMENTS OF
C ROUGHLY EQUAL SIZE AND SUMMING THE APPROPRIATE VARIABLES.
C THE OUTPUT OF PART 1 IS NOT PRINTED IN THIS VERSION OF THE PROGRAM.
C SECTION DETAILS ARE GIVEN IN THE PROGRAM VERSION "STRESSM".
C
C PART 2 OF THE PROGRAM SUBJECTS THE BLADE TO APPLIED LOADING AND TO
C CENTRIFUGAL (INERTIAL) LOADING. THE BLADE IS ASSUMED TO BE
C CANTILEVERED AT ITS ROOT (STATION 11) AND FREE AT THE TIP (STATION 1).
C THE PROGRAM THEN CALCULATES THE EQUILIBRIUM DEFLECTIONS OF THE
C ROTATING BLADE AND THE RESULTING STRAINS IN THE SKIN LAMINATE AT
C 6 SPECIFIED INPUT LOCATIONS AROUND THE SECTION FOR EACH BLADE STATION.
C
C THE DISTRIBUTED LOADING IS RESOLVED INTO X,Y, AND Z COMPONENTS.
C X LIES ALONG THE (UNDEFLECTED) AXIS OF THE BLADE WITH THE ORIGIN AT
C THE BLADE ROOT. Y-AXIS IS IN THE LAGGING DIRECTION AND Z-AXIS IS
C IN THE FLAPPING DIRECTION. THE FREE STREAM WIND IS IN THE POSITIVE
C Z-DIRECTION, AND ROTATION IS THUS CLOCKWISE LOOKING UPSTREAM FROM
C AFT OF THE ROTOR.
C
C THE BLADE STATION SPACING AND APPLIED LOADING (WHICH CONSISTS OF
C PREVIOUSLY CALCULATED AERODYNAMIC LOAD AND ANY OTHER EXTERNAL
C BLADE
C LOAD) ARE LISTED IN ANOTHER INPUT DATAFILE, WHICH ALSO INCLUDES
C ROTOR SPEED, BLADE RADIUS, BLADE PITCH ANGLE, AND BLADE CONING ANGLE.
C
C THE X-Y PLANE CONSTITUTES THE PLANE OF ROTATION OF THE ROTOR.
C THE Z-DEFLECTIONS ARE THE FLAPPING VALUES AND THE Y-DEFLECTIONS
C THE LEAD-LAG VALUES. TWIST AND PITCH ARE MEASURED POSITIVE AIRFOIL
C NOSE-DOWN, OR TENDING TO DECREASE ANGLE-OF-ATTACK.
C
C NOTE: IF THE BLADE OF INTEREST HAS A FLEXIBLE ROOT ATTACHMENT (IE
C FLEXBEAM) THE GJ OF THE ROOT CAN BE SPECIFIED AS THE MINIMUM IN THE
C GJ2N SECTION OF PART 1 OF THE PROGRAM. EI'S ARE HANDLED AS THE ROOT WEB.
C
C .....INITIALIZE VARIABLES.....
C
C DIMENSION XC(5000),YC(5000),R(4),XX(33),YY(33),DFZ2(11),DFZC(11)
```



```

DIMENSION RX(11),YSO(6),ZSO(6),DFY(11),DFZ(11),DFY1(11),DFZ1(11)
DIMENSION ZTOT(11),DFZT(11),ANGTOT(11),TANGTOT(11),AANG1(11)
REAL MLAG(11),MFLAP(11),MLAGP(11),MFLAPP(11),MCENT(11),RCENT(11)
REAL MCX(11),MC1(11),MFLAPA(11)
DIMENSION Q(11),QT(11),QS(11),TLAST1(11),TLAST2(11),FCENT(11)
+ELAST1(11),ELAST2(11),TELAST1(11),TELAST2(11),MCENT1(11),BWT(11)
COMMON X(33),Y(33)
DIMENSION G1(11),G2(11),G3(11),G4(11),G5(11),G6(11)
DIMENSION STX(11),EIYP(11),EIZP(11),FX(11),PANG(11)
+VY(11),VZ(11),EA1(11),SCALE(11),AANG(11),FXT(11),TSCALE(11)
+SXTN(11),SX(11),DEFY(11),DEFZ(11),YCB(11)
+ZCB(11),GJ1(11),YCT(11),ZCT(11),PM(11),TANG(11),TANGI(11)
+GJ2(11),EIFLAP(11),EILAG(11)
DIMENSION WA1(11),YGB(11),ZGB(11),YGBB(11),ZGBB(11),WB(11),
+YBAL(11),ZBAL(11),WT(11),RS(11),RB(11),RBAL(11)
DIMENSION QA(11),QC1(11),QC2(11),QTRM(11),QTI(11),QC1B(11),
+QC2B(11),QTRMB(11),QTRB(11),QCB(11),QB(11),QBTR(11),QL(11)
REAL IXMO,IYMO,IXMT,IYMT,IXMG,IYMG,IBMT,IBMG,IMBB,IMBG
REAL IZO(11),IYO(11),IZT(11),IYT(11),IZG(11),IYG(11)
REAL IMT(11),IMG(11),JMBG(11)
INTEGER STA,IST,NPT,I,J,DIV,ID,KC,JC,IC,J1,NSO,TOGGLE
REAL CY,LX(11),LY(11),LZ(11),MY(11),MZ(11),MPY(11),MPZ(11)
+NANG(11),MYE(11),MZE(11),MYP(11),MZP(11),LYC(11),LZC(11),
+LXI(11),LYI(11),LZI(11)
CHARACTER*40 INP1
CHARACTER*40 INP2
CHARACTER*40 INP3
CHARACTER*40 HEADING
CHARACTER*80 HEAD2
CHARACTER*80 HEAD3
DATA DFZ,DFZ1,DFZ2,ZTOT,DFZT/55*0.0/
DATA ANGTOT,TANGTOT,AANG1/33*0.0/
C
C -----READ INPUT FILES: INP1 IS THE BLADE SECTION GEOMETRY FILE
C -----AND INP2 IS THE APPLIED LOADING FILE. BOTH ARE READ FROM
C -----THE CURRENT FILE WORKSPACE UNDER THE NAMES INPUT FROM THE
C -----KEYBOARD VIA THE FOLLOWING COMMANDS:-----
C
WRITE(1,100)
100 FORMAT('ENTER INPUT FILE NAME FOR SECTION GEOMETRY:')
READ(1,101) INP1
101 FORMAT(1A40)
WRITE(1,102)
102 FORMAT('ENTER INPUT FILE NAME FOR APPLIED LOADING:')
103 FORMAT(1A80)
READ(1,101) INP2
C
C -----INP3 IS THE FILENAME FOR THE ALTERNATE SECTION STIFFNESS
C -----DATA TO BE LISTED LATER: FLAP AND LAG STIFFNESSES, TORSIONAL
C -----STIFFNESS, CHORDWISE SHEAR CENTER, AND EA PRODUCT.
C
WRITE(1,104)
104 FORMAT('ENTER INPUT FILE NAME FOR ALTERNATE SECTION PROPERTIES:')
READ(1,101) INP3
READ*, ECORR
READ*, GCORR
C
C -----OPEN INPUT TAPES-----
C
OPEN(UNIT=5,FILE=INP1)
OPEN(UNIT=6,FILE=INP2)
OPEN(UNIT=8,FILE=INP3)
C
C -----READ NUMBER OF CALCULATIONS-----
C
READ(6,*) NCAL
C
C -----RUN ENTIRE PROGRAM NCAL TIMES-----
C
DO 2000 I8=1,NCAL
C
C -----READ NUMBER OF STATIONS, E-SKIN, E-LONGITUDINAL,-----
C -----* DIVISIONS PER OFFSET FOR NUMERICAL INTEGRATION,-----
C -----G-SKIN, G-LONGITUDINAL-----
C
READ(5,*) STA,ES,EL,DIV,GS,GL
ES = ES * 10**6.0
EL = EL * 10**6.0
GS = GS * 10**6.0
GL = GL * 10**6.0
C
READ(8,*) (G1(I),I=1,STA)
READ(8,*) (G2(I),I=1,STA)
READ(8,*) (G3(I),I=1,STA)
READ(8,*) (G4(I),I=1,STA)

```

```

READ(8,*) (G5(I),I=1,STA)
READ(8,103) HEAD3
C
C -----READ NUMBER OF POINTS AND SKIN THICKNESS CORRECTION FOR RUN
C -----ALSO READ WEIGHT DENSITIES (WS) AND (WL) FOR SKIN & SPAR
C -----IN LB PER INCHES**3
C
READ(5,*) NPT,TCORR,WS,WL
C
C -----READ SECTION COORDINATES -X..., Y... -----
C -----NOTE: X-AXIS IS CHORD LINE-----
C
READ(5,*) (X(I),I=1,NPT)
READ(5,*) (Y(I),I=1,NPT)
DO 105 I = 1,NPT
XX(I) = X(I)
YY(I) = Y(I)*TCORR
105 CONTINUE
C
C -----READ NUMBER OF STRESS OFFSETS-----
C
READ(5,*) NSO
C
C -----READ STRESS OFFSETS-----
C
READ(5,*) (YSO(I),I=1,NSO)
READ(5,*) (ZSO(I),I=1,NSO)
C
C -----READ STATION POSITION FROM FREE END AND LOAD PER UNIT---
C -----LENGTH. X-LOAD IS TOWARDS TIP: Y-LOAD POSITIVE IN LAG---
C -----DIRECTION; Z-LOAD POSITIVE IN FLAP DIRECTION; PITCHING---
C -----MOMENT IS POSITIVE NOSEDOWN (INCH-LB); LOADS ARE IN---
C -----LBS. PER INCH.-----
C
READ(6,*) (STX(I),I=1,STA)
READ(6,*) (LX(I),I=1,STA)
READ(6,*) (LY(I),I=1,STA)
READ(6,*) (LZ(I),I=1,STA)
READ(6,*) (PM(I),I=1,STA)
C
C -----READ ROTATIONAL SPEED OMEGA (RAD/SEC), BLADE RADIUS (IN.),
C -----NOSEDOWN PITCH ANGLE, AND CONING ANGLE (BOTH IN DEGREES)-----
C
READ(6,*) OMEGA,RBL,PITCH1,CONING1
C
C -----CHANGE ANGLES FROM DEGREES TO RADIANS-----
C
PITCH = PITCH1/57.29577951
CONING = CONING1/57.29577951
C
C -----READ TWIST ANGLE. TWIST ANGLE IS IN DEGREES, POSITIVE NOSEDOWN--
C -----COUNTER-CLOCKWISE, LOOKING DOWN THE BLADE FROM THE TIP--
C
READ(5,*) (TANGI(I),I=1,STA)
C
C -----READ HEADING FOR THIS RUN-----
C
READ(6,101) HEADING
C
C -----
C ***** P A R T 1 : CALCULATIONS FOR EACH STATION *****
C
..... see STRESSM .....
570 CONTINUE
C
C NOTE: THIS SECTION INPUTS MANUFACTURER'S DATA WHICH IS BEING
C SUBSTITUTED FOR THE CALCULATED VALUES OF THE SECTION PROPERTIES
C ABOVE. THE REMAINDER OF THE SECTION PROPERTIES HAVE NOT BEEN
C CHANGED.
C
DO 600 I=1,STA
EIYP(I)=G1(I) * 1000000.*ECORR
EIZP(I)=G2(I) * 1000000.*ECORR
GJ2(I)=G3(I) * 1000000.*GCORR
YCT(I)=G4(I)
EA1(I)=G5(I) * 1000000.*ECORR
600 CONTINUE
C
PRINT 103, HEAD3
C
C ***** P A R T 2 : ELASTIC TWIST AND DEFLECTION PART OF PROGRAM *
C *****
..... ee STRESSELASTIC .....

```


Y-CG	10.04	10.05	8.71	8.22	9.74	11.37	12.17	14.94	18.76	23.89	23.89
Z-CG	0.32	0.32	0.34	0.36	0.39	0.45	0.38	0.50	0.65	0.85	0.85
FEA	19.28	19.54	16.09	21.43	34.94	59.59	101.96	190.60	357.19	712.80	712.80
I-CG	19.26	19.52	15.92	21.13	34.52	58.96	100.79	188.44	353.03	704.56	704.56

BALLAST	0.59	0.76	0.77	0.81	0.78	0.86	0.73	0.45	0.59	4.87	0.00
TOTAL WT	1.06	1.24	1.13	1.27	1.38	1.66	1.87	1.87	2.28	6.98	2.11
NEW Y-CG	5.35	4.87	3.88	4.00	5.16	6.31	10.79	13.90	17.32	18.86	23.89
NEW Z-CG	0.28	0.28	0.28	0.29	0.31	0.34	0.23	0.38	0.48	0.26	0.85
NEW I-CG	37.84	40.33	28.23	33.82	56.91	98.26	106.41	194.87	366.75	781.92	704.56
BLD-WGHT	0.00	47.36	96.09	144.52	198.00	259.51	330.76	404.65	486.82	705.40	920.43

V-AERO	0.00	458.86	896.87	1308.24	1694.70	2045.24	2352.78	2604.82	2798.65	2934.12	2988.52
BM-AERO	0.00	9311.17	36678.55	81172.98	141720.59	217187.94	305886.88	405930.56	514927.06	630665.00	750197.50
Y-AEROD	-7.62	-6.37	-5.15	-3.99	-2.96	-2.06	-1.31	-0.75	-0.34	-0.09	0.00
Z-AEROD	342.45	293.11	244.35	197.66	154.34	114.77	78.65	46.47	21.73	5.75	0.00
Z-CONING	28.09	25.28	22.48	19.67	16.86	14.05	11.24	8.43	5.62	2.81	0.00

SECTION TORSION DUE TO: (+ = NOSE DOWN)

P-MOMENT	22.14	22.14	19.49	16.87	14.47	13.77	13.98	14.31	14.26	14.46	0.00
LIFT	-50.00	-52.43	-26.92	-17.38	-25.19	-29.68	-20.17	-21.85	-23.90	-26.53	0.00
TOT-AERO	-27.86	-30.29	-7.43	-0.51	-10.72	-15.91	-6.19	-7.54	-9.64	-12.07	0.00

SECTION ALONE:

TNS-RCKT	-0.20	-0.18	-0.14	-0.16	-0.22	-0.37	-1.22	-0.04	-0.03	-0.02	0.00
FLAP-DEF	-1.00	-0.89	-1.78	-2.27	-2.53	-2.89	-3.57	-3.73	-3.32	-2.06	0.00
LD-LAG	0.00	0.00	0.00	0.00	0.00	0.00	0.02	0.06	0.18	0.57	0.57
TOTAL	-1.20	-1.07	-1.92	-2.42	-2.75	-3.26	-4.77	-3.71	-3.17	-1.52	0.57

BALLAST ALONE:

TNS-RCKT	0.05	0.05	0.03	0.03	0.07	0.15	0.05	0.14	0.40	5.19	0.00
CENTRIF	44.29	50.46	34.80	28.26	28.47	30.36	5.47	2.69	2.25	0.01	0.00
TOTAL	44.34	50.51	34.83	28.30	28.54	30.51	5.53	2.82	2.66	5.21	0.00

TOTAL SECTION + BALLAST:

TNS-RCKT	-0.40	-0.38	-0.24	-0.26	-0.36	-0.61	-1.29	-0.04	-0.04	-0.03	0.00
FLAP-DEF	43.82	50.18	33.39	26.30	26.35	28.18	2.06	-0.84	-0.47	9.72	0.00
LD-LAG	-0.28	-0.36	-0.23	-0.17	-0.20	-0.32	-0.02	0.00	-0.01	-6.00	0.57
TOTAL	43.14	49.44	32.92	25.87	25.79	27.25	0.76	-0.89	-0.51	3.69	0.57
SECTION	15.28	19.15	25.49	25.36	15.07	11.34	-5.43	-8.43	-10.15	-8.37	0.57

TORSION	0.00	685.08	1575.62	2633.99	3482.27	4057.93	4220.55	3937.09	3558.76	3161.66	2980.54
EL TWIST	1.746	1.705	1.495	1.079	0.661	0.378	0.190	0.082	0.034	0.012	0.000
TENSION	0.00	2725.27	5255.96	7462.13	9575.45	11625.98	13574.29	15160.52	16389.25	17994.87	19041.36
V-INPL	0.00	-83.77	-241.52	-412.89	-612.01	-818.13	-1030.90	-1252.01	-1475.09	-1705.16	-1824.68
BM-INPL	0.00	-1438.75	-7740.20	-20826.60	-41369.71	-70149.64	-107356.64	-153324.75	-208225.03	-272638.88	-344075.81
V-FLAP	0.00	458.86	896.87	1308.24	1694.70	2045.24	2352.78	2604.82	2798.65	2934.12	2988.52
BM-FLAP	0.00	411.17	-819.84	-5.95	2757.19	8146.78	15335.06	26055.31	44970.69	76994.38	126616.88
PR A A	3.88	3.43	3.16	2.84	2.37	2.34	4.57	0.08	0.03	0.01	0.00
M.EIYP	0.0000E+00	0.1537E-04	-1.1410E-04	0.2659E-04	0.7398E-04	0.1071E-03	0.1322E-03	0.3119E-03	0.4506E-03	0.7699E-03	0.1265E-02
M.EIZP	0.0000E+00	-9.232E-06	-6.241E-05	-1.260E-04	-1.526E-04	-1.512E-04	-1.342E-04	-8.730E-04	-1.052E-03	-9.678E-04	-1.221E-03

Y-DEF.	-5.34	-4.63	-3.93	-3.23	-2.56	-1.92	-1.31	-0.76	-0.34	-0.09	0.00
Z-DEF.	61.02	54.18	47.37	40.55	33.77	27.11	20.62	14.38	8.65	3.69	0.00
EI-FLAP	0.1338E+09	0.1238E+09	0.9642E+08	0.1202E+09	0.1718E+09	0.2911E+09	0.8072E+09	0.8675E+08	0.1013E+09	0.1007E+09	0.1001E+09
EI-LAG	0.1503E+10	0.1524E+10	0.1242E+10	0.1648E+10	0.2697E+10	0.4606E+10	0.7843E+10	0.1756E+10	0.1980E+10	0.2817E+10	0.2817E+10

M-STRAIN 1	0.	76.	117.	121.	109.	108.	165.	-1009.	-1556.	-1544.	-1725.
M-STRAIN 2	0.	-23.	248.	-6.	-392.	-763.	-1234.	-4342.	-7917.	-16675.	-27200.
M-STRAIN 3	0.	51.	219.	203.	115.	18.	-154.	-265.	-790.	-2651.	-4766.
M-STRAIN 4	0.	76.	273.	385.	420.	419.	264.	2682.	3983.	4905.	6428.
M-STRAIN 5	0.	90.	182.	280.	344.	393.	409.	1417.	2270.	3965.	6105.
M-STRAIN 6	0.	91.	143.	215.	270.	322.	394.	524.	939.	2468.	4257.

**** STOP

10.2.4 SAMPLE RUN, ESI BLADE, WITH TIP WEIGHT

The second sample run uses the ESI blade data. In this case only the location of the shear center had to be changed. This version of the ESI blade had the regular tip weight and ran at 90 rpm.

INPUT BLADE FILE: ESIBL.REGTIP

```

11 1.88 1.88 500 0.133 0.133
33 01.00 0.0245 0.0245
.0000 .0004 .0020 .0040 .0080 .0199 .0399 .0798 .1196
.1994 .2792 .3589 .4786 .5982 .7179 .8375 1.0000
.8375 .7179 .5982 .4786 .3589 .2792 .1994 .1196
.0798 .0399 .0199 .0080 .0040 .0020 .0004 .0000
.0000 .0060 .0140 .0211 .0289 .0463 .0636 .0842 .0981
.1123 .1173 .1161 .1053 .0879 .0664 .0413 .0000
-.0377 -.0618 -.0828 -.0981 -.1065 -.1067 -.1013 -.0861
-.0730 -.0534 -.0389 -.0257 -.0182 -.0126 -.0060 .0000
006
.0000 .2792 .4786 1.0000 .4786 .2792
.0000 .1173 .1053 .0000 .0981 -.1067
0.000 0.85 1.65 2.49 3.29 4.10 4.94 5.75 6.55 7.71 7.71
11.65 0.690 .00001 0.472 5.00
1.236 0.272 0.00 0.00 4.00 0.00
0.930 5.73 0.0
13.66 0.827 .00001 0.472 5.00
1.236 0.272 0.00 0.00 4.31 0.00
0.00 0.00 0.00
15.43 0.961 .00001 0.472 5.00
1.653 0.769 0.00 0.00 4.90 0.00
0.00 0.00 0.00
17.28 1.192 .00001 0.555 5.00
1.494 1.112 0.00 0.00 5.63 0.00
0.00 0.00 0.00
19.09 1.150 .00001 0.677 5.00
0.964 1.040 0.00 0.00 6.30 0.00
0.00 0.00 0.00
20.87 1.226 .00001 0.681 5.00
1.187 1.940 0.00 0.00 6.81 0.00
0.00 0.00 0.00
22.72 1.291 .00001 0.780 5.00
1.350 2.858 0.00 0.00 7.32 0.00
0.00 0.00 0.00
24.40 1.376 .00001 0.980 5.00
1.503 3.943 0.00 0.00 7.97 0.00
0.00 0.00 0.00
20.85 1.771 .00001 1.080 5.00
1.641 5.124 0.00 0.00 8.15 0.00
0.00 0.00 0.00

```

```

16.80 2.416 .00001 2.490 5.00
1.081 1.467 0.00 0.00 8.15 0.00
0.00 0.00 0.00
16.80 2.416 .00001 2.490 5.00
1.081 1.467 0.00 0.00 8.15 0.00
0.00 0.00 0.00
ESI.BLADE.MEAS.DATA.REGULAR.TIP.1-27-88

```

INPUT LOAD FILE: ES190LD.30MPH

```

001
00.00 32.41 64.81 97.22 129.62 162.03 194.44 226.84
259.25 291.65 324.06
00.00 00.00 00.00 00.00 00.00 00.00 00.00 00.00
00.00 00.00 00.00
-1.25 -1.63 -2.30 -2.71 -2.82 -1.93 -1.16 -.42 -.03 0.0 0.0
4.79 5.47 6.17 6.20 5.66 4.74 3.19 1.66 0.84 .58 0.0
9.94 10.12 9.82 8.79 6.32 5.69 5.67 3.19 1.90 1.80 0.0
9.425 324.06 0.0 5.0
ESI.BLADE.90.RPM.30.MPH.LD.1-27-88

```

INPUT MODIFIED SECTION PROPERTIES FILE: QUARTERCHORD

```

3.5 12.0 30.0 58.0 111.0 182.0 304.0 487.0 688.0 1250.0 880.0
167.3 400. 520. 880. 1360. 2050. 2770. 4396. 3200. 2500. 2400.
.65 2.50 6.50 13.3 24.3 39.8 63.1 99.2 136.0 180.0 144.5
2.9 3.42 3.86 4.32 4.77 5.22 5.68 6.1 5.21 4.2 4.2
19.11 25.30 31.1 38.8 49.0 59.5 74.5 94.4 94.9 120.0 152.0
ONLY THE SHEAR CENTER HAS BEEN CHANGED BELOW TO MATCH TESTDATA

```

TERMINAL SESSION AND OUTPUT: STRESSDATA

```

ENTER INPUT FILE NAME FOR SECTION GEOMETRY:
ESIBL.REGTIP
ENTER INPUT FILE NAME FOR APPLIED LOADING:
ES190LD.30MPH
ENTER INPUT FILE NAME FOR ALTERNATE SECTION PROPERTIES:
QUARTERCHORD
*****
ONLY THE SHEAR CENTER HAS BEEN CHANGED BELOW TO MATCH TESTDATA

```

ESI.BLADE.MEAS.DATA.REGULAR.TIP.1-27-88

ESI.BLADE.90.RPM.30.MPH.LD.1-27-88 ** TCORR = 1.000 ** OMEGA = 9.425 ** PITCH = 0.00 ** CONING = 5.00

STATIONS: 11 E-SKIN: 1880000.00 E-LONG: 1879999.75 # DIVISIONS: 500 GS: 132999.97 GL: 132999.97 WS: 0.025 WL: 0.025

STATION	TIP										ROOT												
	1	2	3	4	5	6	7	8	9	10	11	1	2	3	4	5	6	7	8	9	10	11	
STA POS	0.00	32.41	64.81	97.22	129.62	162.03	194.44	226.84	259.25	291.65	324.06	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	
X-LOAD	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	-1.25	-1.63	-2.30	-2.71	-2.82	-1.93	-1.16	-0.42	-0.03	0.00	0.00	
Y-LOAD	-1.25	-1.63	-2.30	-2.71	-2.82	-1.93	-1.16	-0.42	-0.03	0.00	0.00	4.79	5.47	6.17	6.20	5.66	4.74	3.19	1.66	0.84	0.58	0.00	
Z-LOAD	4.79	5.47	6.17	6.20	5.66	4.74	3.19	1.66	0.84	0.58	0.00	9.94	10.12	9.82	8.79	6.32	5.69	5.67	3.19	1.90	1.80	0.00	
P-MOMENT	9.94	10.12	9.82	8.79	6.32	5.69	5.67	3.19	1.90	1.80	0.00	EA	0.2363E+08	0.2755E+08	0.3191E+08	0.4149E+08	0.5374E+08	0.5973E+08	0.7465E+08	0.1009E+09	0.9914E+08	0.1933E+09	0.1933E+09
EA	0.2363E+08	0.2755E+08	0.3191E+08	0.4149E+08	0.5374E+08	0.5973E+08	0.7465E+08	0.1009E+09	0.9914E+08	0.1933E+09	0.1933E+09	EIYP	0.4928E+07	0.1350E+08	0.3036E+08	0.8098E+08	0.1168E+09	0.1850E+09	0.3037E+09	0.5205E+09	0.6074E+09	0.9410E+09	0.9410E+09
EIYP	0.4928E+07	0.1350E+08	0.3036E+08	0.8098E+08	0.1168E+09	0.1850E+09	0.3037E+09	0.5205E+09	0.6074E+09	0.9410E+09	0.9410E+09	EIZP	0.2516E+09	0.4161E+09	0.6144E+09	0.1036E+10	0.1673E+10	0.2233E+10	0.3327E+10	0.5204E+10	0.3736E+10	0.4259E+10	0.4259E+10
EIZP	0.2516E+09	0.4161E+09	0.6144E+09	0.1036E+10	0.1673E+10	0.2233E+10	0.3327E+10	0.5204E+10	0.3736E+10	0.4259E+10	0.4259E+10	GJ	0.2126E+07	0.4874E+07	0.9386E+07	0.2338E+08	0.3593E+08	0.5331E+08	0.8683E+08	0.1523E+09	0.1668E+09	0.3493E+09	0.3493E+09
GJ	0.2126E+07	0.4874E+07	0.9386E+07	0.2338E+08	0.3593E+08	0.5331E+08	0.8683E+08	0.1523E+09	0.1668E+09	0.3493E+09	0.3493E+09	SCALE	11.65	13.66	15.43	17.28	19.09	20.87	22.72	24.40	20.85	16.80	16.80
SCALE	11.65	13.66	15.43	17.28	19.09	20.87	22.72	24.40	20.85	16.80	16.80	TSCALE	0.69	0.83	0.96	1.19	1.15	1.23	1.29	1.38	1.77	2.42	2.42
TSCALE	0.69	0.83	0.96	1.19	1.15	1.23	1.29	1.38	1.77	2.42	2.42	Y-BCENT	5.54	6.45	7.21	8.09	9.06	9.85	10.70	11.48	9.67	7.63	7.63
Y-BCENT	5.54	6.45	7.21	8.09	9.06	9.85	10.70	11.48	9.67	7.63	7.63	Z-BCENT	0.03	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15
Z-BCENT	0.03	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15	Y-SHRCTR	2.90	3.42	3.86	4.32	4.77	5.22	5.68	6.10	5.21	4.20	4.20
Y-SHRCTR	2.90	3.42	3.86	4.32	4.77	5.22	5.68	6.10	5.21	4.20	4.20	Z-SHRCTR	0.03	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15
Z-SHRCTR	0.03	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15	WEIGHT	0.31	0.36	0.42	0.54	0.70	0.78	0.97	1.32	1.29	2.52	2.52
WEIGHT	0.31	0.36	0.42	0.54	0.70	0.78	0.97	1.32	1.29	2.52	2.52	Y-CG	5.54	6.45	7.21	8.09	9.06	9.85	10.70	11.48	9.67	7.63	7.63
Y-CG	5.54	6.45	7.21	8.09	9.06	9.85	10.70	11.48	9.67	7.63	7.63	Z-CG	0.03	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15
Z-CG	0.03	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15	I-EA	3.34	5.60	8.40	14.56	23.32	31.52	47.31	74.60	56.60	67.76	67.76
I-EA	3.34	5.60	8.40	14.56	23.32	31.52	47.31	74.60	56.60	67.76	67.76	I-CG	3.34	5.60	8.40	14.56	23.32	31.52	47.31	74.60	56.60	67.76	67.76
I-CG	3.34	5.60	8.40	14.56	23.32	31.52	47.31	74.60	56.60	67.76	67.76	BALLAST	0.93	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
BALLAST	0.93	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	TOTAL WT	1.24	0.36	0.42	0.54	0.70	0.78	0.97	1.32	1.29	2.52	2.52
TOTAL WT	1.24	0.36	0.42	0.54	0.70	0.78	0.97	1.32	1.29	2.52	2.52	NEW Y-CG	5.68	6.45	7.21	8.09	9.06	9.85	10.70	11.48	9.67	7.63	7.63
NEW Y-CG	5.68	6.45	7.21	8.09	9.06	9.85	10.70	11.48	9.67	7.63	7.63	NEW Z-CG	0.01	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15
NEW Z-CG	0.01	0.04	0.05	0.07	0.08	0.09	0.10	0.12	0.13	0.15	0.15												

NEW ICG	3.35	5.60	8.40	14.56	23.32	31.52	47.31	74.60	56.60	67.76	67.76
BLD-WGHT	0.00	23.35	33.38	48.78	68.79	92.44	120.51	158.56	201.79	266.85	351.81

V-AERO	0.00	166.21	354.72	556.72	750.39	920.63	1050.84	1127.50	1166.10	1189.97	1200.24
BM-AERO	0.00	2633.00	11011.72	25804.02	47001.52	74188.84	106244.95	141639.00	178911.84	217118.88	255890.91
Y-AEROD	-1.84	-1.56	-1.27	-1.00	-0.75	-0.53	-0.35	-0.20	-0.09	-0.02	0.00
Z-AEROD	16.00	12.92	10.04	7.51	5.32	3.54	2.19	1.21	0.53	0.13	0.00
Z-CONING	28.24	25.42	22.60	19.77	16.95	14.12	11.30	8.47	5.65	2.82	0.00

SECTION TORSION DUE TO: (+ = NOSE DOWN)

P-MOMENT	9.94	10.12	9.82	8.79	6.32	5.69	5.67	3.19	1.90	1.80	0.00
LIFT	0.03	-0.08	-0.12	-0.17	-0.17	-0.15	-0.09	-0.03	0.01	0.01	0.00
TOT-AERO	9.97	10.04	9.70	8.62	6.15	5.54	5.58	3.16	1.91	1.81	0.00

SECTION ALONE:

TNS-RCKT	0.01	-0.01	-0.04	-0.12	-0.27	-0.46	-0.85	-1.58	-1.34	-1.81	-1.81
FLAP-DEF	-6.26	-7.51	-8.45	-10.62	-13.13	-12.86	-13.64	-14.50	-7.71	-5.66	0.00
LD-LAG	0.00	0.01	0.03	0.07	0.17	0.27	0.48	0.87	0.67	0.91	0.91
TOTAL	-6.26	-7.51	-8.47	-10.66	-13.23	-13.05	-14.01	-15.21	-8.38	-6.56	-0.90

BALLAST ALONE:

TNS-RCKT	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
CENTRIF	-20.33	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
TOTAL	-20.33	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00

TOTAL SECTION + BALLAST:

TNS-RCKT	0.01	-0.01	-0.04	-0.12	-0.27	-0.46	-0.85	-1.58	-1.34	-1.81	-1.81
FLAP-DEF	-26.57	-7.51	-8.45	-10.62	-13.13	-12.86	-13.64	-14.50	-7.71	-5.66	0.00
LD-LAG	-0.02	0.01	0.03	0.07	0.17	0.27	0.48	0.87	0.67	0.91	0.91
TOTAL	-26.59	-7.51	-8.47	-10.66	-13.23	-13.05	-14.01	-15.21	-8.38	-6.56	-0.90
SECTION	-16.62	2.52	1.24	-2.04	-7.09	-7.51	-8.43	-12.05	-6.47	-4.75	-0.90

TORSION	0.00	-173.31	-57.29	-65.55	-208.67	-443.91	-700.98	-1057.55	-1382.45	-1569.92	-1667.18
EL.TWIST	-0.182	-0.139	-0.090	-0.083	-0.076	-0.062	-0.047	-0.033	-0.019	-0.008	0.000
TENSION	0.00	1698.58	2304.19	3163.47	4128.18	5099.08	6032.47	7006.76	7823.52	8488.87	8842.51
V-INPL	0.00	-45.89	-108.77	-190.77	-281.16	-357.81	-407.55	-432.20	-438.55	-438.95	-438.87
BM-INPL	0.00	-697.63	-3157.13	-7988.30	-15610.73	-26037.66	-38512.80	-52165.26	-66324.17	-80539.59	-94764.63
V-FLAP	0.00	166.21	354.72	556.72	750.39	920.63	1050.84	1127.50	1166.10	1189.97	1200.24
BM-FLAP	0.00	-2013.38	28.00	4772.89	12788.95	23762.91	37162.20	51705.61	65767.03	79859.88	93326.44
PR A A	-0.37	0.49	1.30	2.06	2.86	3.66	4.49	5.27	5.90	6.65	6.66
M.EIYP	0.0000E+00	-1.1487E-03	0.3284E-05	0.6244E-04	0.1160E-03	0.1371E-03	0.1319E-03	0.1081E-03	0.1189E-03	0.9421E-04	0.1102E-03
M.EIZP	0.0000E+00	-1.1718E-05	-5.136E-05	-7.537E-05	-8.938E-05	-1.096E-04	-1.067E-04	-9.069E-05	-1.585E-04	-1.661E-04	-1.956E-04

Y-DEF.	-1.12	-0.95	-0.78	-0.62	-0.47	-0.34	-0.23	-0.13	-0.06	-0.02	0.00
Z-DEF.	33.36	29.85	26.22	22.56	18.96	15.48	12.15	8.94	5.85	2.88	0.00

M-STRAIN 1	0.	37.	36.	38.	57.	76.	91.	93.	61.	16.	12.
M-STRAIN 2	0.	245.	52.	-88.	-226.	-324.	-362.	-341.	-438.	-413.	-489.
M-STRAIN 3	0.	233.	68.	-54.	-183.	-273.	-314.	-302.	-370.	-347.	-412.
M-STRAIN 4	0.	78.	114.	129.	117.	122.	98.	71.	134.	110.	124.
M-STRAIN 5	0.	-108.	78.	208.	336.	443.	475.	439.	530.	441.	510.
M-STRAIN 6	0.	-131.	63.	201.	346.	464.	507.	475.	554.	456.	527.

*** STOP

10.3: COMPUTER CODE: COMPOSITE BLADE BENDING FREQUENCIES OF VIBRATION

10.3.1 BRIEF DISCUSSION

The bending frequency of vibration calculations used the methodology of Chapter 4 (Holzer table). The code has the following general characteristics:

- Uses Myklestad-Prohl lumped mass analysis.

- Does not assume mode shapes; calculates frequencies and mode shapes from arbitrary section mass and stiffness data.

- Assumes uncoupled bending modes in flapping, lead-lag, and torsion; bending mode coupling corrections can be added to the routine if the frequencies of interest coincide.

- Can use cantilevered or simply supported beam model.

- Program gives frequencies and mode shapes for the principal bending, and gives influence coefficients which can be used in other aeroelastic programs.

10.3.2 PROGRAM LISTING, VIBRATE

The code VIBRATE is listed here. Comment lines explain the logic and input and output parameters

```

a      n      d
C      PROGRAM VIBRATE
C      .....
C      -----THIS PROGRAM FINDS THE NATURAL FREQUENCIES-----
C      -----IN BENDING OF A CANTILEVER BEAM BY THE MYKLESTAD METHOD-----
C      -----THE BEAM IS FIXED AT STATION 11, THE BEAM IS DIVIDED----
C      -----INTO 11 DISCRETE LUMPED MASSES, AND THE BEAM ROTATES----
C      -----AT FREQUENCY OMEGA ABOUT THE FIXED END, THE BEAM-----
C      -----THE BEAM SEGMENTS ARE OF EQUAL LENGTH.-----
C
C      DIMENSION EL(12),RM(12),EI(12),UF(12),VF(12),UM(12),VM(12),EM(12)
C      DIMENSION ELRC(12),ELR(12),EMR(12),FCR(12),FC(12),DFC(12),DFCR(12)
C      DIMENSION A(12),B(12),C(12),D(12)
C      DIMENSION BGP(12),BHP(12),HP(12),GP(12),BG(12),BH(12),H(12),G(12),
+DEF(12)
C      CHARACTER*40 INP1
C      CHARACTER*80 HEADING
C
C      50  FORMAT(3X,I5,3X,F8.4)
C      52  FORMAT(3X,F6.2)
C      55  FORMAT(3X,6F9.6/,3X,5F9.6)
C      56  FORMAT(3X,6F8.4/,3X,5F8.4)
C
C      57  FORMAT(1A80)
C      58  FORMAT(1A40)
C      59  FORMAT(/,1A80)
C
C      -----READ INPUT FILE: INP1 IS THE FILE WHICH CONTAINS THE
C      -----BEAM MASS AND STIFFNESS PROPERTIES-----
C      -----FIRST:  NO. OF STATIONS AND BEAM LENGTH (INCHES)
C      -----SECOND:  BEAM RUNNING MASS (SLUGS PER INCH)
C      -----THIRD:   BEAM EI'S (FLAP OR LAG) (LB-IN**2) 10E-6
C      -----FOURTH:  RPM OF BLADE
C      -----LAST:   HEADING FOR THE DATASET
C
C      WRITE(1,70)
C      70  FORMAT('ENTER INPUT FILE NAME FOR BEAM PROPERTIES:')
C      READ(1,58) INP1
C
C      -----READ NUMBER OF STATIONS,LENGTH,RUNNING MASS, AND EI'S----
C      -----BEGINNING AT THE TIP-----
C
C      OPEN(UNIT=5,FILE=INP1)
C      READ(5,*) NSTA,RAD
C      READ(5,*)(RM(N),N = 1,NSTA)
C      READ(5,*)(EI(N),N = 1,NSTA)
C
C      DO 80 N = 1,NSTA
C      EI(N) = EI(N) * 10 ** 6.0
C      RM(N) = RM(N) / 12.0
C      CONTINUE
C
C      -----READ RPM AND HEADING-----
C
C      READ(5,*) RPM
C      READ(5,57) HEADING
C
C      -----REWIND INPUT FILE -----
C
C      CLOSE(5)
C
C      -----PRINT TABLE HEADER-----
C
C      90  FORMAT('FREQUENCIES AND MODES OF BENDING VIBRATION OF A ROTATING
+LUMPED MASS BEAM')
C      PRINT 90
C      PRINT 59, HEADING
C
C      100 FORMAT(/,/,/,30X,'LENGTH = ',F6.2,/)
C      105 FORMAT(30X,'RPM = ',F6.2,/,/,/)
C      PRINT 100, RAD
C      PRINT 105, RPM
C
C      u      n      i      t      s
C
C      110 FORMAT(' MODE FREQUENCY (TIP)',11X,'STATION',62X,'ROOT')
C      112 FORMAT(' RAD/SEC CYC/SEC',119)
C      PRINT 110
C      PRINT 112, (N,N = 1,11)
C
C      OMEGA2 = (RPM * 2.0 * 3.1415926 / 60.0) ** 2.0
C
C      -----CALCULATE THE INFLUENCE VALUES FROM L'S AND EI'S-----
C
C      DELTR = RAD / FLOAT(NSTA-1)
C      DO 145 N = 1,NSTA
C      EL(N) = RAD-FLOAT(N-1) * (DELTR))
C      UF(N) = (DELTR ** 3.0) / (3.0 * EI(N))
C      VF(N) = (DELTR ** 2.0) / (2.0 * EI(N))
C      UM(N) = VF(N)
C      VM(N) = DELTR / EI(N)
C      145 CONTINUE
C
C      DO 150 N = 1,NSTA
C      EM(N) = 0.0
C      A(N) = 0.0
C      B(N) = 0.0
C      C(N) = 0.0
C      D(N) = 0.0
C      150 CONTINUE
C
C      JJ = 0
C
C      -----CALCULATE THE LUMPED MASSES-----
C
C      EM(1) = (DELTR / 4.0) * (RM(1) + RM(2))
C      EM(11) = (DELTR / 4.0) * (RM(10) + RM(11))
C      DO 165 N = 2,10
C      EM(N) = (DELTR / 4.0) * (RM(N-1) + (2.0 * RM(N)) + RM(N+1))
C      165 CONTINUE
C
C      -----CALCULATE THE TENSION DUE TO CENTRIFUGAL FORCE-----
C      -----STARTING AT THE TIP-----
C
C      DO 180 N = 1,NSTA
C      DFC(N) = 0.0
C      FC(N) = 0.0
C      180 CONTINUE
C
C      DO 200 N = 1,NSTA
C      DFC(N) = EM(N) * EI(N) * OMEGA2
C      FC(N) = DFC(N) + FC(N-1)
C      200 CONTINUE
C
C      -----CALCULATE THE NUMERICAL COEFFICIENTS IN THE-----
C      -----ASSUMED LINEAR FUNCTIONS: SHEAR = ( - ) BGP * -----
C      -----PHI + BG, ETC-----
C
C      DO 300 N = 1,NSTA
C      A(N) = 1.0 + VF(N) * FC(N)
C      B(N) = 1.0 + UM(N) * FC(N)
C      C(N) = DELTR + UF(N) * FC(N)
C      D(N) = C(N) * FC(N)
C      300 CONTINUE
C
C      -----SET THE INITIAL VALUE OF THE ITERATION FREQUENCY(SQUARED)
C      -----WSQ = 1 RAD/SEC OR FREQUENCY = 6.28 SECOND PERIOD-----
C
C      WSQ = 1.0
C      314 CONTINUE
C      REM1 = 100.0
C
C      *****MAIN LOOP*****
C
C      DO 580 II = 1,100000
C
C      -----SET INITIAL VALUES OF AMPLITUDE COEFFICIENTS-----
C      -----AT THE TIP, DUE TO FREE BOUNDARY CONDITION-----
C
C      DO 450 N = 1,NSTA
C      BGP(1) = 0.0
C      BHP(1) = 0.0
C      HP(1) = 1.0
C      GP(1) = 0.0
C      BG(1) = EM(1) * WSQ

```

```

BH(1) = 0.0
H(1) = 0.0
G(1) = 1.0
C
C -----CALCULATE VALUES OF AMPLITUDE COEFFICIENTS FOR EACH-----
C -----STATION-----
C
GP(N+1) = GP(N) + C(N) * HP(N) + UM(N) * BHP(N) + UF(N) * BGP(N)
G(N+1) = G(N) + C(N) * H(N) + UM(N) * BH(N) + UF(N) * BG(N)
BGP(N+1) = BGP(N) + EM(N+1) * GP(N+1) * WSQ
BG(N+1) = BG(N) + EM(N+1) * G(N+1) * WSQ
BHP(N+1) = B(N) * RHP(N) + D(N) * HP(N) + C(N) * BGP(N)
BH(N+1) = B(N) * BH(N) + D(N) * H(N) + C(N) * BG(N)
HP(N+1) = A(N) * HP(N) + VM(N) * BHP(N) + VF(N) * BGP(N)
H(N+1) = A(N) * H(N) + VM(N) * BH(N) + VF(N) * BG(N)
450 CONTINUE
C
C -----CALCULATE PHI FROM B.C. AT ROOT SO THAT SLOPE = 0-----
C
PHI = H(11) / HP(11)
C
C -----FIND RESULTING DEFLECTION AT ROOT FROM OTHER B.C.-----
C -----(THIS GIVES THE REMAINDER FOR THE ITERATION)-----
C
REM = (- GP(11) * PHI) + G(11)
DIFF = REM1 - ABS(REM)
REM1 = ABS(REM)
C
C -----START FINE CONVERGENCE INTERVAL-----
C
IF (ABS(REM) .LE. 0.01) GO TO 570
C
WSQ = WSQ + 1.0
GO TO 580
C
570 CONTINUE
C
C -----IF REMAINDER IS A MINIMUM YOU'VE FOUND A ROOT-----
C
IF (DIFF.LE.0.0) GO TO 600
C
WSQ = WSQ + 0.01
C
580 CONTINUE
C
600 CONTINUE
C
C -----FIND MODE SHAPE FOR SOLUTION FREQUENCY-----
C
DO 625 J = 1,NSTA
DEF(J) = G(J) - (PHI * GP(J))
625 CONTINUE
C
FREQ = SQRT(WSQ)
JJ = JJ + 1
FREQ1 = FREQ / (2.0 * 3.1415926)
C
C -----PRINT OUT RESULTS-----
C
650 FORMAT(1X,13,3X,F6.2,3X,F6.2,5X,11(F8.5,1X),/)
PRINT 650, JJ,FREQ,FREQ1,(DEF(K),K = 1,NSTA)
C
C -----STOP IF ENOUGH MODES HAVE BEEN FOUND---
C
IF (JJ.EQ.25) STOP
C
C -----LOOK FOR HIGHER FREQUENCIES-----
C
WSQ = WSQ + 100.0
GO TO 314
C
C -----END OF PROGRAM-----
C
END

```

10.3.3 SAMPLE RUN, UTRC BLADE, STATIC AND 108 RPM

These sample runs use the uniform pultrusion UTRC blade for simplicity. The input file contains the blade structural properties and the rotor speed. The first run is for static frequencies, and the second for the 108 rpm condition. **Note:** the output list multiple solutions for each root due to coarse convergence criteria; the best solution is the one with minimum root deflection (e.g., deflection = 0 at STA 11). Also, the program must be manually stopped when all practical roots have been found (the program will continue to find higher and higher roots).

INPUT BLADE FILE: UTRCFLAPVIBE

```

11 141.0
.01312 .01312 .01312 .01312 .01312 .01312
.01312 .01312 .01312 .01312 .01312
5.845 5.845 5.845 5.845 5.845 5.845
5.845 5.845 5.845 5.845 5.845
00.00
UTRC.BLADE.FLAPVIBE.VALUES.FROM.STRESSM.6-12-87

```

TERMINAL SESSION AND OUTPUT: VIBE

```

ENTER INPUT FILE NAME FOR BEAM PROPERTIES:
UTRCFLAPVIBE
FREQUENCIES AND MODES OF BENDING VIBRATION OF A ROTATING LUMPED MASS BEAM
UTRC.BLADE.FLAPVIBE.VALUES.FROM.STRESSM.6-12-87

```

```

LENGTH = 141.00
RPM = 0.00

```

MODE	FREQUENCY		STATION											ROOT
	RAD/SEC	CYC/SEC	1	2	3	4	5	6	7	8	9	10	11	
1	12.92	2.06	1.00000	0.86134	0.72365	0.58856	0.45858	0.33696	0.22752	0.13450	0.06241	0.01597	-0.00004	
2	81.63	12.99	1.00000	0.50597	0.04489	-0.34125	-0.60703	-0.72184	-0.68251	-0.51890	-0.29248	-0.08922	-0.00990	
3	82.24	13.09	1.00000	0.50435	0.04212	-0.34418	-0.60865	-0.72058	-0.67716	-0.50900	-0.27853	-0.07257	0.00771	
4	236.25	37.60	1.00000	0.15136	-0.47013	-0.68117	-0.43600	0.09279	0.59171	0.78145	0.58434	0.19190	0.01011	
5	236.46	37.63	1.00000	0.15095	-0.47059	-0.68118	-0.43526	0.09401	0.59259	0.78107	0.58230	0.18862	0.00619	
6	236.68	37.67	1.00000	0.15055	-0.47105	-0.68118	-0.43451	0.09523	0.59346	0.78072	0.58033	0.18549	0.00261	
7	236.89	37.70	1.00000	0.15015	-0.47150	-0.68118	-0.43377	0.09644	0.59434	0.78033	0.57823	0.18201	-0.00140	
8	237.10	37.74	1.00000	0.14974	-0.47196	-0.68118	-0.43303	0.09766	0.59521	0.77993	0.57616	0.17882	-0.00510	
9	237.31	37.77	1.00000	0.14934	-0.47241	-0.68118	-0.43229	0.09887	0.59607	0.77954	0.57420	0.17550	-0.00889	

STOP

The following run is for the same blade, but at its operating speed of 108 rpm.

INPUT BLADE FILE: UTRCFLAPVIBE.108

```
11 141.0
.01312 .01312 .01312 .01312 .01312 .01312
.01312 .01312 .01312 .01312 .01312
5.845 5.845 5.845 5.845 5.845 5.845
5.845 5.845 5.845 5.845 5.845
108.0
UTRC.BLADE.FLAPVIBE.108RPM.SPEED
```

ENTER INPUT FILE NAME FOR BEAM PROPERTIES:

```
UTRCFLAPVIBE.108
FREQUENCIES AND MODES OF BENDING VIBRATION OF A ROTATING LUMPED MASS BEAM
UTRC.BLADE.FLAPVIBE.108RPM.SPEED
```

```
LENGTH = 141.00
RPM = 108.00
```

MODE	FREQUENCY		(TIP)	STATION											ROOT
	RAD/SEC	CYC/SEC		1	2	3	4	5	6	7	8	9	10	11	
1	17.89	2.85	1.00000	0.86456	0.72995	0.59758	0.46959	0.34880	0.23867	0.14332	0.06765	0.01751	-0.00002		
2	86.71	13.80	1.00000	0.50897	0.05146	-0.33068	-0.59306	-0.70665	-0.66925	-0.51015	-0.28868	-0.08836	-0.00993		
3	87.28	13.89	1.00000	0.50731	0.04858	-0.33381	-0.59506	-0.70604	-0.66487	-0.50158	-0.27635	-0.07352	0.00582		
4	241.56	38.45	1.00000	0.14769	-0.47402	-0.68199	-0.43363	0.09555	0.59252	0.78085	0.58413	0.19197	0.01000		
5	241.77	38.48	1.00000	0.14729	-0.47447	-0.68198	-0.43289	0.09675	0.59339	0.78050	0.58219	0.18875	0.00636		
6	241.97	38.51	1.00000	0.14690	-0.47491	-0.68198	-0.43216	0.09795	0.59427	0.78017	0.58027	0.18571	0.00309		
7	242.18	38.54	1.00000	0.14651	-0.47535	-0.68198	-0.43142	0.09915	0.59515	0.77985	0.57838	0.18257	-0.00096		
8	242.39	38.58	1.00000	0.14611	-0.47579	-0.68197	-0.43068	0.10035	0.59602	0.77951	0.57642	0.17931	-0.00470		
9	242.59	38.61	1.00000	0.14572	-0.47623	-0.68196	-0.42995	0.10155	0.59688	0.77917	0.57449	0.17628	-0.00789		

STOP

10.3.4 PROGRAM LISTING, VIBETIP, (TIP WEIGHT VERSION)

The following version of the vibration program has provision for the addition of a tip mass to the blade from the keyboard.

```

VIBETIP
C PROGRAM VIBRATE
C .....
C -----THIS PROGRAM FINDS THE NATURAL FREQUENCIES-----
C -----IN BENDING OF A CANTILEVER BEAM BY THE MYKLESTAD METHOD.
C -----THE BEAM IS FIXED AT STATION 11, THE BEAM IS DIVIDED----
C -----INTO 11 DISCRETE LUMPED MASSES, AND THE BEAM ROTATES---
C -----AT FREQUENCY OMEGA ABOUT THE FIXED END, THE BEAM-----
C -----THE BEAM SEGMENTS ARE OF EQUAL LENGTH.-----
C
C DIMENSION EL(12),RM(12),EI(12),UF(12),VF(12),UM(12),VM(12),EM(12)
C DIMENSION ELRC(12),ELR(12),EMR(12),FCR(12),FC(12),DFC(12),DFCR(12)
C DIMENSION A(12),B(12),C(12),D(12)
C DIMENSION BGP(12),BHP(12),HP(12),GP(12),BG(12),BH(12),H(12),G(12),
C +DEF(12)
C REAL MT
C CHARACTER*40 INP1
C CHARACTER*80 HEADING
C
C ..... see VIBE .....
70 FORMAT(ENTER INPUT FILE NAME FOR BEAM PROPERTIES:)

READ(1,58) INP1
C
C -----READ NUMBER OF STATIONS,LENGTH,RUNNING MASS, AND EI'S---
C -----BEGINNING AT THE TIP-----
C
C OPEN(UNIT=5,FILE=INP1)
C READ(5,*) NSTA,RAD
C READ(5,*)(RM(N),N = 1,NSTA)
C READ(5,*)(EI(N),N = 1,NSTA)
C
C WRITE(1,75)
75 FORMAT('READ TIP MASS IN LB:')
C READ*, MT
C
C ..... see VIBE .....
C -----CALCULATE THE LUMPED MASSES-----
C
C EM(1) = (DELTR / 4.0) * (RM(1) + RM(2)) + (MT/386.)
C EM(11) = (DELTR / 4.0) * (RM(10) + RM(11))
C DO 165 N = 2,10
C EM(N) = (DELTR / 4.0) * (RM(N-1) + (2.0 * RM(N)) + RM(N+1))
165 CONTINUE
C
C ..... see VIBE .....

```

10.4: COMPUTER CODE: COMPOSITE BLADE TORSIONAL FREQUENCIES OF VIBRATION

10.4.1 SUMMARY

The code that calculates torsional frequencies and mode shapes is also a Holzer table (see Chap. 4). There are two versions of this program: one for general torsional beams (TORSVIB) and the other with provision for added tip moment of inertia (TVIBTIP). This latter program was used to find the frequencies of the blades with tip weights and mechanisms (ESI). The sample runs are for this blade first bare and then with the regular tip (see Chap. 5).

10.4.2 PROGRAM LISTING, TORSVIB

```

TORSVIB
C PROGRAM VIBRATE
C .....
C -----THIS PROGRAM FINDS THE NATURAL FREQUENCIES-----
C -----IN TORSION OF A CANTILEVER BEAM BY A HOLZER TABLE----
C -----THE BEAM IS FIXED AT STATION 11, THE BEAM IS DIVIDED---
C -----INTO 11 DISCRETE LUMPED INERTIAS, THE BLADE SEGMENTS---
C -----ARE OF EQUAL LENGTH.-----
C
C REAL I(12),K(12),IS(12)
C DIMENSION GJ(12),QN(12),THETA(12),SQN(12),DTHETA(12)
C CHARACTER*40 INP1
C CHARACTER*80 HEADING
C
C 50 FORMAT(3X,I5,3X,F8.4)
C 52 FORMAT(3X,F6.2)
C 55 FORMAT(3X,6F9.6,/,3X,5F9.6)
C 56 FORMAT(3X,6F8.4,/,3X,5F8.4)
C 57 FORMAT(1A80)
C 58 FORMAT(1A40)
C 59 FORMAT(/,/,1A80)
C
C -----READ INPUT FILE: INP1 IS THE FILE WHICH CONTAINS THE
C -----BEAM INERTIAS AND TORSIONAL STIFFNESSES
C -----FIRST: NO. OF STATIONS AND BEAM LENGTH (INCHES)
C -----SECOND: BEAM SECTION RUNNING MASS MOMENTS OF INERTIA
C -----      (LB-IN**2 PER INCH) WHICH INCLUDES CONTRIBU-
C -----      TIONS FROM THE SKIN, SPAR, WEB, AND COUNTER-
C -----      WEIGHTS. THE MASS MOMENTS ARE REFERRED TO THE
C -----      AXIS OF TORSION, OR ELASTIC AXIS.
C
C -----THIRD: BEAM SECTION TORSIONAL RIGIDITY GJ (LB-IN**2)
C -----LAST: HEADING FOR THE DATASET
C
C WRITE(1,70)
70 FORMAT(ENTER INPUT FILE NAME FOR BEAM TORSIONAL PROPERTIES:)
C READ(1,58) INP1
C
C -----READ NUMBER OF STATIONS, LENGTH, INERTIAS, AND GJ'S---
C -----BEGINNING AT THE TIP-----
C
C OPEN(UNIT=5,FILE=INP1)
C READ(5,*) NSTA,RAD
C READ(5,*)(IS(N),N = 1,NSTA)
C READ(5,*)(GJ(N),N = 1,NSTA)
C
C -----CORRECT FOR DECIMAL FACTORS:-----
C
C DO 80 N = 1,NSTA
C IS(N) = IS(N) * 1.0
C GJ(N) = GJ(N) * 10**6.
80 CONTINUE
C
C -----READ HEADING-----
C
C READ(5,57) HEADING
C
C -----REWIND INPUT FILE -----
C
C CLOSE(5)
C
C -----PRINT TABLE HEADER-----
C
C 90 FORMAT('FREQUENCIES AND MODES OF TORSIONAL VIBRATION OF A
C +LUMPED INERTIA BEAM)

```

```

PRINT 90
PRINT 59, HEADING
C
100 FORMAT(/,30X,'LENGTH = ',F6.2,/)
PRINT 100, RAD
C
110 FORMAT(' MODE FREQNCY (TIP)',11X,'STATION',62X,'ROOT')
112 FORMAT(' RAD/SEC CYC/SEC',11B)
PRINT 110
PRINT 112, (N,N = 1,11)
C
C ----CALCULATE MASS MOMENTS OF INERTIA (LB-IN**2) AND TORSIONAL
C ----STIFFNESS (IN-LB/RADIAN) FOR EACH STATION----
C
DELTR = RAD / FLOAT(NSTA-1)
K(1) = GJ(1) / (DELTR/2)
I(1) = IS(1) * (DELTR/2)
C
DO 145 N = 2,NSTA
C
C ----TORSIONAL STIFFNESS FOR THE SEGMENT IS THE AVERAGE OF THE
C ----ADJACENT GJ'S----
C
K(N) = ( GJ(N) + GJ(N+1) ) / ( 2 * DELTR )
C
C ----MASS MOMENT OF INERTIA IS THE SECTION VALUE TIMES THE SECTION
C ----LENGTH----
C
I(N) = IS(N) * DELTR
C
145 CONTINUE
C
C ----INITIALIZE VALUES----
C
JJ = 0
C
C ----CHANGE UNITS OF MASS MOMENT OF INERTIA FROM (LB-IN**2) TO
C ----(LB-IN-SEC**2) BY DIVIDING BY THE ACCELERATION OF GRAVITY
C ----IN THE ENGLISH SYSTEM (= 386 INCHES/SEC**2)----
C
DO 180 N = 1,NSTA
I(N) = I(N) / 386
180 CONTINUE
C
C -----SET THE INITIAL VALUE OF THE ITERATION FREQUENCY (SQRD)
C -----WSQ = 1 RAD/SEC OR FREQUENCY = 6.28 SECOND PERIOD-----
C
WSQ = 1.0
C
314 CONTINUE
C
C .....
C .....MAIN LOOP.....
C .....
C
DO 580 II = 1,100000
C
C ----SET INITIAL VALUES FOR THE TIP, WHICH IS FREE OF RESTRAINT
C ----AND HAS (NORMALIZED) TWIST ANGLE AMPLITUDE OF 1.0
C ----TWIST ANGLE, INERTIAL TORQUE, SUMMED TORQUES, AND TWIST ANGLE FOR
C ----EACH SECTION STARTING AT THE TIP-----
C
THETA(1) = 1.0
QN(1) = I(1) * WSQ
SQN(1) = QN(1)

DTHETA(1) = SQN(1) / K(1)
C
C ----CALCULATE VALUES OF TWIST ANGLE, TORQUE, AND SUMS FOR
C ----EACH STATION----
C
DO 450 N = 2,NSTA
C
C ----FIND TWIST ANGLE FOR THIS STATION----
C
THETA(N) = THETA(N-1) - DTHETA(N-1)
C
C ----CALCULATE INERTIAL TORQUE FOR THIS STATION----
C
QN(N) = I(N) * WSQ * THETA(N)
C
C ----SUM TORQUES FOR ALL STATIONS SO FAR----
C
SQN(N) = SQN(N-1) + QN(N)
C
C ----FIND TWIST ANGLE FOR THIS STATION----
C
DTHETA(N) = SQN(N) / K(N)
C
450 CONTINUE
C
C ----FIND TWIST ANGLE AT ROOT OF BLADE----
C
THETAR = THETA(10) - DTHETA(10)
THETA(11) = THETAR
C
C ----IF ROOT TWIST ANGLE IS ZERO, YOU'VE FOUND A ROOT----
C
IF ( ABS(THETAR) .LE. 0.001 ) GO TO 600
C
C ----OTHERWISE INCREASE THE WSQ VALUE, AND TRY AGAIN----
C
WSQ = WSQ + 1.0
C
580 CONTINUE
C
600 CONTINUE
C
C ----NOW PRINT OUT MODE NUMBER, SOLUTION FREQUENCY IN RAD/SEC, AND
C ----CYCLES/SEC, AND TORSIONAL MODE SHAPE FOR EACH STATION----
C
FREQ = SQRT(WSQ)
FREQ1 = FREQ / (2.0 * 3.1415926)
JJ = JJ + 1
C
C -----PRINT OUT RESULTS-----
C
650 FORMAT(1X,13,3X,F6.2,3X,F6.2,5X,11(F8.5,1X),/)
PRINT 650, JJ,FREQ,FREQ1,(THETA(J),J = 1,NSTA)
C
C -----IF ALL 10 MODES HAVE BEEN FOUND, END PROGRAM-----
C
IF ( JJ .EQ. 10) STOP '10 MODES COMPLETED'
C
C -----LOOK FOR HIGHER FREQUENCIES-----
C
WSQ = WSQ + 100.0
GO TO 314
C
C -----END OF PROGRAM-----
C
END

```

10.4.3 SAMPLE RUN,ESI BLADE

INPUT BLADE FILE: ESITORSVIBE.FINAL

11 324.06
 3.34 5.60 8.40 14.56 23.32 31.52
 47.31 74.60 56.60 67.76 67.76
 2.10 4.90 9.40 23.40 35.90 53.30
 86.80 152.30 166.80 349.30 349.30
 TORSION.VIBE.ESI.BLADE.CALC.VALUES.1-26-88

TORSVIB
 ENTER INPUT FILE NAME FOR BEAM TORSIONAL PROPERTIES:
 ESITORSVIBE.FINAL
 FREQUENCIES AND MODES OF TORSIONAL VIBRATION OF A LUMPED INERTIA BEAM
 TORSION.VIBE.ESI.BLADE.CALC.VALUES.1-26-88

LENGTH = 324.00

MODE	FREQUENCY (TIP)		STATION								ROOT		
	RAD/SEC	CYC/SEC	1	2	3	4	5	6	7	8	9	10	11
1	252.85	40.24	1.00000	0.93084	0.76341	0.62240	0.49125	0.35938	0.24730	0.16460	0.08925	0.03926	0.00100
2	253.05	40.27	1.00000	0.93073	0.76305	0.62187	0.49057	0.35861	0.24648	0.16378	0.08848	0.03854	0.00034
3	253.24	40.30	1.00000	0.93062	0.76270	0.62133	0.48990	0.35783	0.24565	0.16296	0.08771	0.03782	-0.00031
4	253.44	40.34	1.00000	0.93052	0.76234	0.62079	0.48922	0.35706	0.24483	0.16214	0.08694	0.03711	-0.00097
5	405.38	64.52	1.00000	0.82223	0.42990	0.16041	-0.02387	-0.14080	-0.18692	-0.18087	-0.13853	-0.09877	-0.06083
6	437.01	69.55	1.00000	0.79341	0.34918	0.06258	-0.11191	-0.19751	-0.20583	-0.16839	-0.09942	-0.04545	-0.00100
7	437.13	69.57	1.00000	0.79330	0.34888	0.06223	-0.11221	-0.19767	-0.20585	-0.16829	-0.09925	-0.04524	-0.00078
8	437.24	69.59	1.00000	0.79319	0.34859	0.06188	-0.11250	-0.19784	-0.20587	-0.16820	-0.09907	-0.04503	-0.00056
9	437.36	69.61	1.00000	0.79308	0.34829	0.06154	-0.11280	-0.19800	-0.20589	-0.16811	-0.09890	-0.04481	-0.00034
10	437.47	69.63	1.00000	0.79297	0.34799	0.06119	-0.11309	-0.19817	-0.20591	-0.16801	-0.09872	-0.04460	-0.00012

**** STOP 10 MODES COMPLETED

10.4.4 PROGRAM LISTING, TVIBTIP, (ESI BLADE WITH TIP INERTIA)

```

TVIBTIP
C PROGRAM VIBRATE
C
C .....
C -----THIS PROGRAM FINDS THE NATURAL FREQUENCIES-----
C -----IN TORSION OF A CANTILEVER BEAM BY A HOLZER TABLE----
C -----THE BEAM IS FIXED AT STATION 11, THE BEAM IS DIVIDED---
C -----INTO 11 DISCRETE LUMPED INERTIAS. THE BLADE SEGMENTS---
C -----ARE OF EQUAL LENGTH.-----
C
REAL I(12),K(12),IS(12),ITIP
DIMENSION GJ(12),QN(12),THETA(12),SQN(12),DTHETA(12)
CHARACTER*40 INP1
CHARACTER*80 HEADING
C
..... see TORSVIBE .....

75 FORMAT('READ TIP INERTIA IN LB-IN**2:')
READ*, ITIP
C
C -----CORRECT FOR DECIMAL FACTORS:-----
..... see TORSVIBE .....
C -----CALCULATE MASS MOMENTS OF INERTIA (LB-IN**2) AND TORSIONAL
C -----STIFFNESS (IN-LB/RADIAN) FOR EACH STATION-----
C
DELTR = RAD / FLOAT(NSTA-1)
K(1) = GJ(1) / (DELTR/2)
I(1) = IS(1) * (DELTR/2) + ITIP
C
DO 145 N = 2,NSTA
..... see TORSVIBE .....

```

10.4.5 SAMPLE RUN, ESI BLADE WITH TIP INERTIA

TVIBTIP

ENTER INPUT FILE NAME FOR BEAM TORSIONAL PROPERTIES:

ESITORSVIBE.FINAL

READ TIP INERTIA IN LB-IN**2:

831.

FREQUENCIES AND MODES OF TORSIONAL VIBRATION OF A LUMPED INERTIA BEAM

TORSION.VIBE.ESI.BLADE.CALC.VALUES.1-26-88

LENGTH = 324.06

MODE	FREQUENCY (TIP)		STATION											ROOT										
	RAD/SEC	CYC/SEC	1	2	3	4	5	6	7	8	9	10	11	1	2	3	4	5	6	7	8	9	10	11
1	148.29	23.60	1.00000	0.61094	0.35378	0.23082	0.15603	0.10142	0.06393	0.04045	0.02172	0.00986	0.00098											
2	306.79	48.83	1.00000	-0.66524	-1.51000	-1.68026	-1.56314	-1.27600	-0.94616	-0.65701	-0.36169	-0.15879	-0.00100											
3	306.95	48.85	1.00000	-0.66701	-1.51232	-1.68229	-1.56454	-1.27657	-0.94598	-0.65631	-0.36060	-0.15749	0.00039											

QUIT.

10.5: COMPUTER CODE: STEADY ELASTIC TWIST CALCULATION

10.5.1: PROGRAM LISTING, STRESSELASTIC

The code STRESSELASTIC is the chief program used for this study. It contains all the calculations required for finding elastic twist and all the bending deflections. A listing is given here in its entirety.

```

C .....
C ***** STRESSELASTIC *****
C .....
C ***** ALTERNATIVE ENERGY INSTITUTE *****
C ***** WEST TEXAS STATE UNIVERSITY *****
C ***** CANYON TX 79016 806-656-2295 *****
C .....
C THIS PROGRAM CALCULATES THE SECTION MASS AND STIFFNESS PROPERTIES
C OF AN ARBITRARY EXTERNAL GEOMETRY (TWIST, TAPER) AND SPECIFIED
C INTERNAL SPAR-SHELL STRUCTURAL GEOMETRY, COMPOSITE WIND TURBINE
C BLADE, AND THEN FINDS THE BENDING DEFLECTIONS, ELASTIC TWIST, AND
C STRAINS IN THE LAMINATE FOR A SPECIFIED EXTERNAL LOAD AND ROTOR
C ROTATIONAL SPEED.
C
C THIS VERSION OF THE PROGRAM ("STRESSELASTIC") CONTAINS ALL THE ROUTINES
C AND PRINTS OUT THE TORSIONAL DEFLECTIONS (ELASTIC TWIST) DUE TO ALL
C INERTIAL AND APPLIED LOADS, AND DUE TO BLADE BENDING.
C
C THIS PROGRAM WAS ORIGINALLY WRITTEN BY E.VAN DUSEN (COMPOSITE
C ENGINEERING, INC., 277 BAKER AVE., CONCORD MA 01742 617-371-3132)
C IN 1977 FOR COMPOSITE RACING SHELLS AND HAS BEEN EXTENSIVELY REWRITTEN
C BY F.S. STOODARD (AEI, BOX 248 WTSU, CANYON TX 79016 806-656-2295) FOR
C WIND TURBINE BLADES.
C
C -----
C GENERAL PROGRAM DESCRIPTION:
C
C PART 1 OF THIS PROGRAM CALCULATES THE SECTION PROPERTIES OF AN
C ARBITRARY COMPOSITE BEAM, USING EITHER ACTUAL THICKNESSES AND
C MODULI OF PLIES, OR USING MODULUS-WEIGHTING (THE LATTER ROUTINE
C HAS BEEN SUPPRESSED FOR THE CALCULATION OF ELASTIC TWIST). THE
C BEAM (BLADE) IS MODELED BY A SHELL OF CONSTANT THICKNESS WITH
C ARBITRARY LONGITUDINAL MODULUS AND SHEAR MODULUS, AND A SPAR
C CONSISTING OF A NOSE SECTION EMBEDDED NEXT TO THE SHELL IN THE
C BLADE LEADING EDGE, AND A WEB (OR WEBS) LOCATED INTERNALLY. THE
C SPAR ALSO HAS CONSTANT THICKNESS AND ARBITRARY LONGITUDINAL AND
C SHEAR MODULI. THE WEB IS DESCRIBED ONLY BY ITS MOMENTS OF INERTIA
C AND CENTROID IN THE SECTION AXIS SYSTEM. MULTI-WEB BEAM DESIGNS ARE
C HANDLED BY INCLUDING ALL THE ACTUAL WEBS INTO THIS ONE MODEL WEB.
C
C THE SECTION AIRFOIL SHAPES ARE DETERMINED BY SCALING AN INPUT
C TABLE OF OFFSETS. THE CHORD SCALING FACTOR (SCALE) DETERMINES THE
C AIRFOIL CHORD AND THE THICKNESS SCALING FACTOR (TSKALE) DETERMINES
C THE SECTION THICKNESS PROPORTIONAL TO THE TABLE OF OFFSETS.
C ALL THESE INPUT DATA ARE INCLUDED IN A DATAFILE GIVING THE GEOMETRY
C AND STRUCTURAL DESIGN OF THE BLADE OF INTEREST.
C
C IN PART 1 PARABOLIC SEGMENTS ARE FITTED THROUGH EACH GROUP OF THREE
C SECTION OFFSET POINTS. THE I OF THE SKIN+SPAR+WEB IS CALCULATED BY
C DIVIDING THE GROUP OF POINTS INTO (DIV) STRAIGHT LINE SEGMENTS OF
C ROUGHLY EQUAL SIZE AND SUMMING THE APPROPRIATE VARIABLES.
C THE OUTPUT OF PART 1 IS NOT PRINTED IN THIS VERSION OF THE PROGRAM.
C SECTION DETAILS ARE GIVEN IN THE PROGRAM VERSION "STRESSM".
C
C PART 2 OF THE PROGRAM SUBJECTS THE BLADE TO APPLIED LOADING AND TO
C CENTRIFUGAL (INERTIAL) LOADING. THE BLADE IS ASSUMED TO BE
C CANTILEVERED AT ITS ROOT (STATION 11) AND FREE AT THE TIP (STATION 1).
C THE PROGRAM THEN CALCULATES THE EQUILIBRIUM DEFLECTIONS OF THE
C ROTATING BLADE AND THE RESULTING STRAINS IN THE SKIN LAMINATE AT
C 6 SPECIFIED INPUT LOCATIONS AROUND THE SECTION FOR EACH BLADE STATION.
C
C THE DISTRIBUTED LOADING IS RESOLVED INTO X,Y, AND Z COMPONENTS.
C X LIES ALONG THE (UNDEFLECTED) AXIS OF THE BLADE WITH THE ORIGIN AT
C THE BLADE ROOT. Y-AXIS IS IN THE LAGGING DIRECTION AND Z-AXIS IS
C IN THE FLAPPING DIRECTION. THE FREE STREAM WIND IS IN THE POSITIVE
C Z-DIRECTION, AND ROTATION IS THUS CLOCKWISE LOOKING UPSTREAM FROM
C AFT OF THE ROTOR.
C
C THE BLADE STATION SPACING AND APPLIED LOADING (WHICH CONSISTS OF
C PREVIOUSLY CALCULATED AERODYNAMIC LOAD AND ANY OTHER EXTERNAL
C BLADE
C LOAD) ARE LISTED IN ANOTHER INPUT DATAFILE, WHICH ALSO INCLUDES
C ROTOR SPEED, BLADE RADIUS, BLADE PITCH ANGLE, AND BLADE CONING ANGLE.
C
C THE X-Y PLANE CONSTITUTES THE PLANE OF ROTATION OF THE ROTOR.
C THE Z-DEFLECTIONS ARE THE FLAPPING VALUES AND THE Y-DEFLECTIONS
C THE LEAD-LAG VALUES. TWIST AND PITCH ARE MEASURED POSITIVE AIRFOIL
C NOSE-DOWN, OR TENDING TO DECREASE ANGLE-OF-ATTACK.
C
C NOTE: IF THE BLADE OF INTEREST HAS A FLEXIBLE ROOT ATTACHMENT (IE
C FLEXBEAM) THE GJ OF THE ROOT CAN BE SPECIFIED AS THE MINIMUM IN THE
C GJ2N SECTION OF PART 1 OF THE PROGRAM. EIS ARE HANDLED AS THE ROOT WEB.
C
C .....
C . . . . .

```

```

C .....
C C .....INITIALIZE VARIABLES.....
C C
C DIMENSION XC(5000),YC(5000),R(4),XX(33),YY(33),DFZ2(11),DFZC(11)
C DIMENSION RX(11),YSO(6),ZSO(6),DFY(11),DFZ(11),DFY1(11),DFZ1(11)
C DIMENSION ZTOT(11),DFZT(11),ANGTOT(11),TANGTOT(11),AANG(11)
C REAL MLAG(11),MFLAP(11),MLAGP(11),MFLAPP(11),MCENT(11),RCENT(11)
C REAL MCX(11),MC1(11),MFLAPA(11)
C DIMENSION Q(11),QT(11),QS(11),TLAST1(11),TLAST2(11),FCENT(11)
C +ELAST1(11),ELAST2(11),TELAST1(11),TELAST2(11),MCENT1(11),BWT(11)
C COMMON X(33),Y(33)
C DIMENSION STX(11),EIYP(11),EIZP(11),FX(11),PANG(11)
C +VY(11),VZ(11),EA1(11),SCALE(11),AANG(11),FXT(11),TSKALE(11)
C +SXTN(11),SX(11),DEFY(11),DEFZ(11),YCB(11)
C +ZCB(11),GJ1(11),YCT(11),ZCT(11),PM(11),TANG(11),TANGI(11)
C +GJ2(11),EIFLAP(11),EILAG(11)
C DIMENSION WA1(11),YGB(11),ZGB(11),YGBB(11),ZGBB(11),WB(11),
C +YBAL(11),ZBAL(11),WT(11),RS(11),RB(11),RBAL(11)
C DIMENSION QA(11),QC1(11),QC2(11),QTRM(11),QT(11),QC1B(11),
C +QC2B(11),QTRMB(11),QTIB(11),OCB(11),OB(11),QBTR(11),OL(11)
C REAL IXMO,IYMO,IXMT,IYMT,IXMG,IYMG,IBMT,IBMG,IMBB,IMBG
C REAL IZO(11),IYO(11),IZT(11),IYT(11),IZG(11),IYG(11)
C REAL IMT(11),IMG(11),IMGB(11)
C INTEGER STA,IST,NPT,I,J,DIV,ID,KC,JC,IC,J1,NSO,TOGGLE
C REAL CY,LX(11),LY(11),LZ(11),MY(11),MZ(11),MPY(11),MPZ(11)
C +NANG(11),MYE(11),MZE(11),MYP(11),MZP(11),LYC(11),LZC(11),
C +LXI(11),LYI(11),LZI(11)
C CHARACTER*40 INP1
C CHARACTER*40 INP2
C CHARACTER*40 HEADING
C CHARACTER*80 HEAD2
C DATA DFZ,DFZ1,DFZ2,ZTOT,DFZT/55*0.0/
C DATA ANGTOT,TANGTOT,AANG/133*0.0/
C
C -----READ INPUT FILES: INP1 IS THE BLADE SECTION GEOMETRY FILE
C -----AND INP2 IS THE APPLIED LOADING FILE. BOTH ARE READ FROM
C -----THE CURRENT FILE WORKSPACE UNDER THE NAMES INPUT FROM THE
C -----KEYBOARD VIA THE FOLLOWING COMMANDS:-----
C
C WRITE(1,100)
C 100 FORMAT('ENTER INPUT FILE NAME FOR SECTION GEOMETRY:')
C READ(1,101) INP1
C 101 FORMAT('A40')
C WRITE(1,102)
C 102 FORMAT('ENTER INPUT FILE NAME FOR APPLIED LOADING:')
C READ(1,101) INP2
C
C -----OPEN INPUT TAPES-----
C
C OPEN(UNIT=5,FILE=INP1)
C OPEN(UNIT=6,FILE=INP2)
C
C -----READ NUMBER OF CALCULATIONS-----
C
C READ (6,*) NCAL
C
C -----RUN ENTIRE PROGRAM NCAL TIMES-----
C
C DO 2000 I=1,NCAL
C
C -----READ NUMBER OF STATIONS, E-SKIN, E-LONGITUDINAL,-----
C -----* DIVISIONS PER OFFSET FOR NUMERICAL INTEGRATION,-----
C -----G-SKIN, G-LONGITUDINAL-----
C
C READ (5,*) STA,ES,EL,DIV,GS,GL
C ES = ES * 10**6.0
C EL = EL * 10**6.0
C GS = GS * 10**6.0
C GL = GL * 10**6.0
C
C -----READ NUMBER OF POINTS AND SKIN THICKNESS CORRECTION FOR RUN
C -----ALSO READ WEIGHT DENSITIES (WS) AND (WL) FOR SKIN & SPAR
C -----IN LB PER INCHES**3
C
C READ (5,*) NPT,TCORR,WS,WL
C
C -----READ SECTION COORDINATES -X..., Y... -----
C -----NOTE: X-AXIS IS CHORD LINE-----
C
C READ (5,*) (X(I),I=1,NPT)
C READ (5,*) (Y(I),I=1,NPT)
C DO 105 I = 1,NPT
C XX(I) = X(I)
C YY(I) = Y(I)*TCORR

```

```

105 CONTINUE
C
C -----READ NUMBER OF STRESS OFFSETS-----
C
C READ (5,*) NSO
C
C -----READ STRESS OFFSETS-----
C
C READ (5,*) (YSO(I), I=1,NSO)
C READ (5,*) (ZSO(I), I=1,NSO)
C
C -----READ STATION POSITION FROM FREE END AND LOAD PER UNIT---
C -----LENGTH. X-LOAD IS TOWARDS TIP; Y-LOAD POSITIVE IN LAG---
C -----DIRECTION; Z-LOAD POSITIVE IN FLAP DIRECTION; PITCHING--
C -----MOMENT IS POSITIVE NOSEDOWN (INCH-LB); LOADS ARE IN---
C -----LBS. PER INCH.-----
C
C READ (6,*) (STX(I), I=1,STA)
C READ (6,*) (LX(I), I=1,STA)
C READ (6,*) (LY(I), I=1,STA)
C READ (6,*) (LZ(I), I=1,STA)
C READ (6,*) (PM(I), I=1,STA)
C
C ----READ ROTATIONAL SPEED OMEGA (RAD/SEC), BLADE RADIUS (IN.),
C ----NOSEDOWN PITCH ANGLE, AND CONING ANGLE (BOTH IN DEGREES)----
C
C READ (6,*) OMEGA,RBL,PITCH1,CONING1
C
C ----CHANGE ANGLES FROM DEGREES TO RADIANS----
C
C PITCH = PITCH1/57.29577951
C CONING = CONING1/57.29577951
C
C -----READ TWIST ANGLE. TWIST ANGLE IS IN DEGREES, POSITIVE NOSEDOWN--
C -----COUNTER-CLOCKWISE, LOOKING DOWN THE BLADE FROM THE TIP--
C
C READ (5,*) (TANGI(I), I=1,STA)
C
C -----READ HEADING FOR THIS RUN-----
C
C READ (6,101) HEADING
C
C ***** P A R T 1 : CALCULATIONS FOR EACH STATION *****
C
C -----PRINT HEADER-----
C
C PRINT 998,HEADING,TCORR
C PRINT ', '
C PRINT 910,STA,ES,EL,DIV,GS,GL,WS,WL
C PRINT 973
C PRINT ', '
C
C PRINT ', ' *****
C PRINT ', '
C
C DO 570 IST = 1,STA
C
C -----READ THE SCALE FACTORS FOR SIMILAR SECTIONS, SPAR THICKNESS
C -----TSPAR), SKIN THICKNESS (TSKIN), AND X-COORDINATE OF CHANGE
C
C READ (5,*) SCALE(IST),TSCALE(IST),TSPAR,TSKIN,XTHK
C
C -----CONVERT COORDINATES TO SCALE-----
C
C DO 131 I = 1,NPT
C X(I) = SCALE(IST)*XX(I)
C Y(I) = SCALE(IST)*YY(I)*TSCALE(IST)
C
C -----CALCULATE (SPAR) MODULUS-WEIGHTED THICKNESSES FOR BOTH
C -----BENDING AND TORSION. THK WEIGHTED TO SPAR MODULUS AND
C -----GTHK WEIGHTED TO SKIN MODULUS-----
C
C THK1 = TSPAR + TSKIN * (ES/EL)
C THK2 = TSKIN * (ES/EL)
C GTHK1 = TSKIN + TSPAR * (GL/GS)
C GTHK2 = TSKIN
C
131 CONTINUE
C
C -----PRINT OUT RESULTS-----
C
C PRINT 930,IST,THK1,THK2,XTHK,TSPAR,TSKIN,TSCALE(IST)
C PRINT 925, (X(I), I=1,NPT)
C PRINT 925, (Y(I), I=1,NPT)
C
TXSM = 0
TXSMS = 0
TXSML = 0
TYSM = 0
TYSMS = 0
TYSML = 0
TXSI = 0
TXSIS = 0
TXSIL = 0
TYSI = 0
TYSIS = 0
TYSIL = 0
TXYSI = 0
TXYSIS = 0
TXYSIL = 0
TAREA = 0
TSA = 0
TLA = 0
TAA = 0
TDST = 0
C
J1 = NPT - 2
C
DO 340 J = 1,J1,2
C
C *****
C *****CALCULATION OF SKIN MOMENT OF INERTIA AND CENTROID*****
C *****FROM CHORD LINE*****
C
CALL LCOE (COE1,COE2,COE3,J)
C
C -----SELECTION OF DIVISIONS BY DIVIDING A STRAIGHT-----
C -----LINE BETWEEN POINTS INTO EQUAL SEGMENTS-----
C
DX1 = (X(J+1)-X(J))/DIV
DX2 = (X(J+2)-X(J+1))/DIV
I5 = DIV + 1
DO 210 ID = 1,I5
XC(ID) = X(J) + (ID-1) * DX1
XC(DIV + ID) = X(J+1) + (ID - 1) * DX2
210 CONTINUE
I3 = 2 * DIV + 1
DO 230 ID = 1,I3
YC(ID) = COE1 * ((XC(ID)**2)) + COE2*XC(ID) + COE3
YC(ID+DIV) = COE1*((XC(ID+DIV)**2))+COE2*XC(ID+DIV)+COE3
230 CONTINUE
TA = 0
TSKINA = 0
TSPARA = 0
XSM = 0
XSMS = 0
XSML = 0
YSM = 0
YSMS = 0
YSML = 0
XSI = 0
XSIS = 0
XSIL = 0
YSI = 0
YSIS = 0
YSIL = 0
XYSI = 0
XYSIS = 0
XYSIL = 0
SDST = 0
SDA = 0
C
C -----CORRECT FOR SHELL THICKNESS; OUTER SKIN GEOMETRY IS----
C -----GIVEN; XLEV,YLEV ARE COORDINATES OF MIDPOINT OF DAREA--
C
DO 305 ID = 2,I3
XLV = (XC(ID-1) + XC(ID))/2.0
YLV = (YC(ID-1) + YC(ID))/2.0
C
C -----CHANGE VALUES IF SPAR HAS ENDED-----
C -----NOTE: THIS VERSION OF THE PROGRAM USES THE ACTUAL THICKNESSES
C -----TO CALCULATE SECTION PROPERTIES SINCE E-MODULUS AND G-
MODULUS
C -----MUST BE INDEPENDENT. TO USE MODULUS-WEIGHTING STATEMENTS
BELOW
C -----SHOULD BE CHANGED, EG THK, GTHK.
C
IF ((XLV-XTHK) .GE. 0.0) GO TO 250
C
THK = THK1
GTHK = GTHK1

```

```

THK = TSPAR + TSKIN
GTHK = TSPAR + TSKIN
TSPARI = TSPAR
GO TO 260
250 CONTINUE
THK = THK2
GTHK = GTHK2
THK = TSKIN
GTHK = TSKIN
TSPARI = 0
260 CONTINUE
C
  C   DYC = YC(ID) - YC(ID-1)
  C   DXC = XC(ID) - XC(ID-1)
  C   HYP = SQRT (DYC ** 2 + DXC ** 2)
  C   DX = THK * DYC / (2.0*HYP)
  C   DY = THK * DXC / (2.0*HYP)
  C   XLEV = XLV + DX
  C   YLEV = YLV - DY
  C   DAREA = THK * HYP
  C   DSAREA = TSKIN * HYP
  C   DLAREA = TSPARI * HYP
  C   DST = HYP / GTHK
  C   DA = DXC * YLV
C
C   ----CALCULATE TOTAL MODULUS-WEIGHTED AREA----
C
  C   TA = TA + DAREA
C
C   ----CALCULATE TOTAL SKIN AREA----
C
  C   TSKINA = TSKINA + DSAREA
C
C   ----CALCULATE TOTAL SPAR AREA----
C
  C   TSPARA = TSPARA + DLAREA
C
C   ----FIRST MOMENT ABOUT X-AXIS----
C
  C   XSM = XSM + YLEV * DAREA
  C   XSMS = XSMS + YLEV * DSAREA
  C   XSML = XSML + YLEV * DLAREA
C
C   ----FIRST MOMENT ABOUT Y-AXIS----
C
  C   YSM = YSM + XLEV * DAREA
  C   YSMS = YSMS + XLEV * DSAREA
  C   YSML = YSML + XLEV * DLAREA
C
C   ----X MOMENT OF INERTIA----
C
  C   XSI = XSI + DAREA * (YLEV ** 2)
  C   XSI = XSI + DSAREA * (YLEV**2)
  C   XSIL = XSIL + DLAREA * (YLEV**2)
C
C   ----Y MOMENT OF INERTIA----
C
  C   YSI = YSI + DAREA * (XLEV ** 2)
  C   YSI = YSI + DSAREA * (XLEV**2)
  C   YSIL = YSIL + DLAREA * (XLEV**2)
C
C   ----PRODUCT OF INERTIA----
C
  C   XYSI = XYSI + DAREA * XLEV * YLEV
  C   XYSIS = XYSIS + DSAREA * XLEV * YLEV
  C   XYSIL = XYSIL + DLAREA * XLEV * YLEV
C
C   ----CALCULATE SHEAR FLOW INTEGRALS FOR SKIN ---
C
  C   SDST = SDST + DST
  C   SDA = SDA + DA
305 CONTINUE
C
C   ----CALCULATE RUNNING TOTALS----
C
C   -----AREAS---
C
  C   TAREA = TAREA + TA
  C   TSA = TSA + TSKINA
  C   TLA = TLA + TSPARA
C
C   -----1ST MOMENT OF AREA ABOUT X-AXIS---
C
  C   TXSM = TXSM + XSM
  C   TXSMS = TXSMS + XSMS
  C   TXSML = TXSML + XSML

```

```

C
C   -----1ST MOMENT OF AREA ABOUT Y-AXIS---
C
  C   TYSM = TYSM + YSM
  C   TYSMS = TYSMS + YSMS
  C   TYSML = TYSML + YSML
C
C   -----X-MOMENT OF INERTIA---
C
  C   TXSI = TXSI + XSI
  C   TXSIS = TXSIS + XSI
  C   TXSIL = TXSIL + XSIL
C
C   -----Y-MOMENT OF INERTIA---
C
  C   TYSI = TYSI + YSI
  C   TYSIS = TYSIS + YSI
  C   TYSIL = TYSIL + YSIL
C
C   -----PRODUCT OF INERTIA---
C
  C   TXYSI = TXYSI + XYSI
  C   TXYSIS = TXYSIS + XYSIS
  C   TXYSIL = TXYSIL + XYSIL
C
C   -----SHEAR FLOW INTEGRALS---
C
  C   TAA = TAA + SDA
  C   TDST = TDST + SDST
340 CONTINUE
C
C   *****
C   **CALCULATE SECTION VALUES BASED ON MODULUS-WEIGHTING**
C   *****
C
C   ----MODULUS-WEIGHTED BENDING CENTROID----
C
  C   SXB = TYSM / TAREA
  C   SYB = TXSM / TAREA
C
C   ----MODULUS-WEIGHTED E'S----
C
  C   EXSI = TXSI * EL
  C   EYSI = TYSI * EL
C
C   -----READ WEB INFORMATION - CROSS-SECTIONAL AREA (AL)-----
C   -----IX,IY ALL ABOUT WEB CENTROID, XL,YL-----
C
  C   READ (5,*) AL,XLI,YLI,XYLI,XL,YL
C
C   -----PRINT OUT THESE NUMBERS-----
C
  C   PRINT 940,AL,XLI,YLI,XYLI,XL,YL
C
C   ----1ST AREA MOMENTS OF WEB----
C
  C   XLM = AL * YL
  C   YLM = AL * XL
C
C   ----E'S OF WEB----
C
  C   EXLI = EL * XLI
  C   EYLI = EL * YLI
C
C   -----CALCULATION OF CENTROID AND EI FOR TOTAL COMPOSITE STRUC
C
  C   EA = EL * TAREA + EL * AL
  C   EA1(IST) = EA
C
C   ----TOTAL COMPOSITE STRUCTURE BENDING CENTROID BASED ON
C   ----MODULUS-WEIGHTING----
C
  C   XB = (EL * TYSM + EL * YLM) / EA
  C   YB = (EL * TXSM + EL * XLM) / EA
  C   YCB(IST) = XB
  C   ZCB(IST) = YB
C
C   -----FIND EIX,EIY,EIXY FOR TOTAL STRUCTURE ABOUT ITS CENTROID
C   ----- (XB,YB) BY USING THE PARALLEL AXIS THEOREM IN SEQUENCE,
C   -----WORKING BACKWARDS----
C
  C   EXI = EXSI+EXLI-EL*TAREA*(SYB**2-(SYB-YB)**2)+EL*AL*((YL-YB)**2)
  C   EYI = EYSI+EYLI-EL*TAREA*(SXB**2-(SXB-YB)**2)+EL*AL*((XL-YB)**2)
  C   EXYI = TXYSI*EL+XYLI*EL-EL*TAREA*(SXB*SYB-(SXB-YB)*(SYB-YB))
  C   + +AL*EL*(XL-YB)*(YL-YB)
C

```

```

C -----FIND PRINCIPAL AXIS OF INERTIA BY USING MOHR'S CIRCLE---
C
SUM = (EXI+EYI)/2.0
DIF = (EYI-EXI)/2.0
PP = ATAN(EXY/DIF)
PHI = PP/2.0
PANG(IST) = PHI
RAD = SQRT(EXY**2 + DIF**2)
C
C ----PRINCIPAL EIX AND Eiy----
C
PXI = SUM - RAD
PYI = SUM + RAD
EYYP(IST) = PXI
EIZP(IST) = PYI
C
C ----PRINCIPAL RADIUS OF GYRATION----
C
PRX = SQRT(PXI/EA)
PRY = SQRT(PYI/EA)
C
C -----PRINT OUT RESULTS-----
C
PRINT 945,XB,YB,TAREA,PHI,PXI,PYI,PRX,PRY,EA
C
C -----CALCULATE POLAR MOMENT OF INERTIA: J = IX + IY-----
C
C -----IF G-SKIN & E-SKIN ARE DIFFERENT THE TORSIONAL CENTROID-
C
C -----WILL DIFFER FROM THE BENDING CENTROID-----
C
GA = GS * TAREA*(ES/EL) + GL * AL
C
C ----CALCULATE SHEAR CENTER (TORSIONAL CENTROID) BASED ON
C
C ----MODULUS-WEIGHTED VALUES----
C
XBT = (GS * TYSM + GL * YLM) / GA
YBT = (GS * TXSM + GL * XLM) / GA
YCT(IST) = XBT
ZCT(IST) = YBT
C
C -----CALCULATE TORSIONAL STIFFNESS-----
C
GJ1N = GS*TXSI+GL*XLI-GS*TAREA*(ES/EL)*(SYB**2-(SYB-YBT)**2) +
+ GL*AL*(YL-YBT)**2 + GS*TYSI + GL*YLI - GS*TAREA*(ES/EL)*
+ (SXB**2-(SXB-XBT)**2) + GL*AL*((XL-XBT)**2)
GJ1(IST) = GJ1N
C
C ----CALCULATE ALTERNATE TORSIONAL STIFFNESS BASED ON SHEAR FLOW-
C
C ----NOTE: IF GJ IS BELOW THE MINIMUM FOR CONVERGENCE, MAKE IT EQUAL
C
C ----TO THE MINIMUM----
C
GJ2N = GS* 4.0 * (TAA**2)/ TDST
IF (GJ2N .LE. 240000.) GJ2N = 240000.
C
GJ2(IST) = GJ2N
C
C -----PRINT FURTHER RESULTS-----
C
PRINT 950,XBT,YBT,GJ1N,GJ2N
C
PRINT ' '
PRINT ' '
C
C **CALCULATE SECTION VALUES BASED ON ACTUAL THICKNESSES**
C
C -----SKIN AND SPAR BENDING CENTROIDS----
C
SXBS= TYSMS / TSA
SXBL = TYSSL / TLA
SYBS = TXSMS / TSA
SYBL = TXSSL / TLA
C
C ----CALCULATION OF CENTROID AND EI FOR TOTAL COMPOSITE STRUCTURE
C
EA = ES * TSA + EL * TLA + EL * AL
EA1(IST) = EA
C
C ----TOTAL COMPOSITE STRUCTURE BENDING CENTROID BASED ON
C
C ----ACTUAL THICKNESSES----
C
XB = ( ES * TYSMS + EL * TYSSL + EL * YLM ) / EA
YB = ( ES * TXSMS + EL * TXSSL + EL * XLM ) / EA
YCB(IST) = XB
ZCB(IST) = YB
C

```

```

C -----FIND EIX,EIY,EIXY FOR TOTAL STRUCTURE ABOUT ITS CENTROID
C
C -----(XB,YB) BY USING THE PARALLEL AXIS THEOREM IN SEQUENCE,
C
C -----WORKING BACKWARDS, FOR THE SKIN, SPAR, AND WEB----
C
EXI = ES * TXSIS - ES * TSA * (SYBS**2) + ES * TSA * ((SYBS
+ - YB)**2) + EL * TXSIL - EL * TLA * (SYBL**2) + EL * TLA
+ ((SYBL - YB)**2) + EL * XLI + EL * AL * ((YL - YB)**2)
EYI = ES * TYSIS - ES * TSA * (SXBS**2) + ES * TSA * ((SXBS
+ - XB)**2) + EL * TYSIL - EL * TLA * (SXBL**2) + EL * TLA
+ ((SXBL - XB)**2) + EL * YLI + EL * AL * ((XL - XB)**2)
EXYI = ES * TXYSIS - ES * TSA * SYBS * SXBS + ES * TSA
+ (SYBS - YB) * (SXBS - XB) + EL * TXYSIL - EL * TLA
+ SYBL * SXBL + EL * TLA * (SYBL - YB) * (SXBL - XB)
+ EL * XLYI + EL * AL * (XL - XB) * (YL - YB)
C
C -----FIND PRINCIPAL AXES OF INERTIA BY USING MOHR'S CIRCLE---
C
SUM = (EXI+EYI)/2.0
DIF = (EYI-EXI)/2.0
PP = ATAN(EXY/DIF)
PHI = PP/2.0
PANG(IST) = PHI
RAD = SQRT(EXY**2 + DIF**2)
C
C ----PRINCIPAL EIX AND Eiy----
C
PXI = SUM - RAD
PYI = SUM + RAD
EYYP(IST) = PXI
EIZP(IST) = PYI
C
C ----PRINCIPAL RADIUS OF GYRATION----
C
PRX = SQRT(PXI/EA)
PRY = SQRT(PYI/EA)
C
C -----PRINT OUT RESULTS-----
C
PRINT 946,XB,YB,TSA,TLA,PHI,PXI,PYI,PRX,PRY,EA
C
C -----CALCULATE POLAR MOMENT OF INERTIA: J = IX + IY-----
C
C -----IF G-SKIN & E-SKIN ARE DIFFERENT THE TORSIONAL CENTROID-
C
C -----WILL DIFFER FROM THE BENDING CENTROID-----
C
C -----IT WILL ALSO INCLUDE CONTRIBUTIONS FROM SKIN,SPAR&WEB-
C
GA = GS * TSA + GL * TLA + GL * AL
C
C ----CALCULATE SHEAR CENTER (TORSIONAL CENTROID) BASED ON
C
C ----ACTUAL THICKNESSES OF LAMINATES----
C
XBT = (GS * TYSMS + GL * TYSSL + GL * YLM) / GA
YBT = (GS * TXSMS + GL * TXSSL + GL * XLM) / GA
YCT(IST) = XBT
ZCT(IST) = YBT
C
C -----CALCULATE TORSIONAL STIFFNESS-----
C
GJ1N = GS * TXSIS - GS * TSA * (SYBS**2) + GS * TSA * ((SYBS
+ - YBT)**2) + GL * TXSIL - GL * TLA * (SYBL**2) + GL
+ TLA * ((SYBL - YBT)**2) + GL * XLI + GL * AL * ((YL
+ - YBT)**2) + GS * TYSIS - GS * TSA * (SXBS**2) + GS
+ TSA * ((SXBS - XBT)**2) + GL * TYSIL - GL * TLA
+ (SXBL**2) + GL * TLA * ((SXBL - XBT)**2) + GL * YLI
+ GL * AL * ((XL - XBT)**2)
GJ1(IST) = GJ1N
C
C -----PRINT FURTHER RESULTS-----
C
PRINT 950,XBT,YBT,GJ1N,GJ2N
C
C ***** MASS CALCULATIONS *****
C
C -----SECTION MASS CALCULATIONS CAN NOW BE DONE USING THE ACTUAL
C
C -----THICKNESSES-----
C
C ----TOTAL RUNNING WEIGHT IS THE SUM OF ALL COMPONENTS----
C
WA = WS * TSA + WL * TLA + WL * AL
WA1(IST) = WA
C
C ----1ST WEIGHT MOMENTS----
C
XBG = ( WS * TYSMS + WL * TYSSL + WL * YLM ) / WA

```



```

YBG = ( WS * TXSMS + WL * TXSML + WL * XLM ) / WA
YGB(IST) = XBG
ZGB(IST) = YBG
C
C -----SECTION MASS MOMENTS OF INERTIA ABOUT ORIGIN (NOSE)---
C
IXMO = WS * TXSIS + WL * TXSIL + WL * (XL1 + AL * (YL**2))
IYMO = WS * TYSIS + WL * TYSIL + WL * (YLI + AL * (XL**2))
IYO(IST) = IXMO
IZO(IST) = IYMO
C
C -----MASS MOMENTS NOW MOVED TO THE SECTION CG (XBG,YBG)---
C
C -----NOTE: THIS VALUE WILL BE THE MINIMUM VALUE OF J ---
C
IXMG = IXMO - ( WA * (YBG**2) )
IYMG = IYMO - ( WA * (XBG**2) )
IYG(IST) = IXMG
IZG(IST) = IYMG
C
C -----ALSO MOVED TO THE ELASTIC AXIS (XBT,YBT)-----
C
IXMT = IXMG + WA * ( YBT - YBG )**2
IYMT = IYMG + WA * ((XBT - XBG)**2)
IYT(IST) = IXMT
IZT(IST) = IYMT
C
C -----THE SECTION MASS MOMENTS OF INERTIA (PER UNIT LENGTH) ARE NOW
C
C -----THE SUMS OF IX, IY, FOR BOTH THE ELASTIC AXIS AND SECTION CG---
C
IBMT = IXMT + IYMT
IBMG = IXMG + IYMG
IMT(IST) = IBMT
IMG(IST) = IBMG
C
C -----PRINT SECTION MASS RESULTS-----
C
C PRINT 955, WL, WS, WA, XBG, YBG, IBMT, IBMG
C
C ---READ BALLAST INFORMATION: RUNNING BALLAST WEIGHT---
C
C ---IN LB/INCH AND CHORDWISE POSITION---
C
READ (5,) WB1, XBAL1, YBAL1
C
WB(IST) = WB1
YBAL(IST) = XBAL1
ZBAL(IST) = YBAL1
C
C -----TOTAL RUNNING WEIGHT IS THE SUM---
C
WT1 = WA + WB1
WT(IST) = WT1
C
C -----1ST WEIGHT MOMENTS: NEW CG POSITION WITH BALLAST---
C
XBGB = (( WA * XBG ) + (WB1 * XBAL1)) / WT1
YGBB = ((WA * YBG) + (WB1 * YBAL1)) / WT1
YGBB(IST) = XBGB
ZGBB(IST) = YGBB
C
C ---NOW CALCULATE NEW MASS MOMENT OF INERTIA ABOUT OLD CG
C
C ---USING THE PARALLEL AXIS THEOREM---
C
IMBB = IBMG + WB1 * ((XBG-XBAL1)**2 + (YBG-YBAL1)**2)
C
C -----NOW MOVE TO NEW CG THEREBY FINDING NEW MINIMUM
C
C ---SECTION MASS MOMENT OF INERTIA, INCLUDING BALLAST
C
IMBG = IMBB - WT1 * ((XBG-XBGB)**2 + (YBG-YGBB)**2)
IMGB(IST) = IMBG
C
C ---PRINT NEW SECTION MASS RESULTS---
C
C PRINT 956, WB1, XBAL1, YBAL1, WT1, XBGB, YGBB, IMBB, IMBG
C
C PRINT ' : '
C
C PRINT ' : '
C
570 CONTINUE
C
C
C
C *****
C
C *** P A R T 2 : ELASTIC TWIST AND DEFLECTION PART OF PROGRAM ***
C
C
C -----READ THE HEADING FOR THE BLADE RUN-----
C

```

```

READ(5,994) HEAD2
C
C -----PRINT 1ST LINES OF OUTPUT-----
C
C PRINT ' : '
C PRINT ' : '
PRINT 994, HEAD2
PRINT 998, HEADING, T CORR, OMEGA, PITCH1, CONING1
PRINT 910, STA, ES, EL, DIV, GS, GL, WS, WL
PRINT ' : '
PRINT 964
PRINT 965, (I, I=1, STA)
PRINT 973
PRINT 970, 'STA POS', STX
PRINT 970, 'X-LOAD', LX
PRINT 970, 'Y-LOAD', LY
PRINT 970, 'Z-LOAD', LZ
PRINT 970, 'P-MOMENT', PM
PRINT 992, 'EA', EA1
PRINT 992, 'E1YP', E1YP
PRINT 992, 'E2YP', E2YP
PRINT 992, 'GJ', GJ2
PRINT 970, 'SCALE', SCALE
PRINT 970, 'TSCALE', TSCALE
PRINT 973
C PRINT ' : '
C PRINT 996, (I, I=1, NSO)
C PRINT 972
C PRINT 975, 'Y STR. OF, YSO
C PRINT 975, 'Z STR. OF, ZSO
C PRINT 972
C PRINT ' : '
C PRINT 965, (I, I=1, STA)
C PRINT 973
PRINT 970, 'Y-BCENT', YCB
PRINT 970, 'Z-BCENT', ZCB
PRINT 970, 'Y-SHRCTR', YCT
PRINT 970, 'Z-SHRCTR', ZCT
PRINT 973
PRINT 970, 'WEIGHT', WA1
PRINT 970, 'Y-CG', YGB
PRINT 970, 'Z-CG', ZGB
PRINT 970, 'I-EA', IMT
PRINT 970, 'I-CG', IMG
PRINT 973
PRINT 970, 'BALLAST', WB
PRINT 970, 'TOTAL WT', WT
PRINT 970, 'NEW Y-CG', YGBB
PRINT 970, 'NEW Z-CG', ZGBB
PRINT 970, 'NEW I-CG', IMGB
CALL INTEG(STX, WT, BWT, 1, STA)
PRINT 970, 'BLD-WGHT', BWT
PRINT 973
C
C -----FIND TWIST ANGLE FOR EACH STATION, THEN ADD THE
C
C -----PRINCIPAL AXIS ANGLE-----
C
DO 640 I = 1, STA
TANG(I) = PITCH + (TANG(I)/57.29577951)
AANG(I) = PANG(I) + TANG(I)
TELAST1(I) = 0.0
TELAST2(I) = 0.0
LXI(I) = LX(I)
LYI(I) = LY(I)
LZI(I) = LZ(I)
DFY1(I) = 0.0
DFZ1(I) = 0.0
RCENT(I) = RBL - STX(I)
FCENT(I) = 0.0
MCENT(I) = 0.0
WA1(I) = WA1(I) / 32.2
WT(I) = WT(I) / 32.2
WB(I) = WB(I) / 32.2
640 CONTINUE
C
C -----SET TOGGLE TO NULL---
C
C TOGGLE = 0
FCENT(0) = 0.0
C
C -----IF THIS IS THE FIRST TIME THROUGH, MUST CALCULATE THE
C
C -----APPROXIMATE BLADE FLAPPING DEFLECTION FIRST---
C
C IF (TOGGLE .LE. 0) GO TO 760
C
650 CONTINUE

```

```

C TOGGLE = TOGGLE + 1
C
C .....
C ..... ELASTIC TORSION SECTION .....
C .....
C
C NOTE: WILL ENTER THIS SECTION AFTER INITIAL DEFLECTIONS
C ARE FOUND (TOGGLE = 1) AND THEN WILL ITERATE
C UNTIL CONVERGENCE IS REACHED ON ZDEF (DFZT) INCLUDING CONING
C
C DO 690 I=1,STA
C
C ----CALCULATE TORSION ABOUT THE SHEAR CENTER: THIS WILL
C ----PRODUCE THE ELASTIC TWIST OF THE SECTION----
C
C -----AERODYNAMIC TORSION DUE TO LIFT (LZ IN THE FLAPPING
C -----DIRECTION), DRAG (LY IN THE LAGGING DIRECTION), AND
C -----PITCHING MOMENT (PM) ALL POSITIVE NOSE-DOWN----
C
C   QL(I) = LZ(I)*(SIN(TANG(I))*ZCT(I)-COS(TANG(I))*(YCT(I) -
C + (0.25*SCALE(I)))) +
C + LY(I)*(SIN(TANG(I))*(YCT(I)-(0.25*SCALE(I))) +
C + COS(TANG(I))*ZCT(I))
C
C   QA(I) = QL(I) + PM(I)
C
C -----CENTRIFUGAL TORSION DUE TO FLAP DEFLECTION (Z-DEF),
C -----NO BALLAST ADDED. NOTE: THIS CANT BE CALCULATED
C -----UNTIL THE TOTAL BEAM DEFLECTIONS ARE CALCULATED FIRST
C -----WHICH MUST INCLUDE THE CONING ANGLE DEFLECTION----
C
C   QC1(I) = (DFZT(I)/12.0) * (OMEGA**2) * WA1(I) * ( COS(TANG(I)) *
C + (YCT(I)-YGB(I)) - SIN(TANG(I))*(ZCT(I)-ZGB(I)) )
C
C -----CENTRIFUGAL TORSION DUE TO HORIZONTAL (LEAD-LAG)
C -----COMPONENT OF CENTRIF FORCE AT A VERTICAL MOMENT ARM
C -----FROM SHEAR CENTER, AND NO BALLAST ADDED. NOTE: THIS
C -----TERM IS A SIMPLE MANIFESTATION OF THE PARALLEL AXIS THM--
C
C   RS(I) = SQRT((YCT(I)-YGB(I))**2 + (ZCT(I)-ZGB(I))**2)
C   QC2(I) = -RS(I) * (OMEGA**2) * WA1(I) * ( 1.0/12.0) *
C + (SIN(TANG(I))*(YCT(I)-YGB(I)) + COS(TANG(I))*(ZCT(I)-ZGB(I)) )
C
C -----TORSION DUE TO THE TENNIS RACKET MOMENT WHEN PRINCIPAL
C -----MASS AXIS (HERE ASSUMED SAME AS PRINCIPAL BENDING AXIS)
C -----OF THE SECTION, LESS BALLAST, IS ROTATED AN ANGLE (AANG)
C -----FROM THE PLANE OF ROTATION----
C
C   QTRM(I) = -IMG(I) * AANG(I) * (OMEGA**2) / 386.
C
C -----TOTAL INERTIAL TORSION IS THE SUM----
C
C   QT(I) = QC1(I) + QC2(I) + QTRM(I)
C
C -----NOW FIND TOTAL INERTIAL TORSION INCLUDING BALLAST, INCL
C -----THE TWO CENTRIFUGAL TERMS AND THE NEW TENNIS RACKET MOMENT
C -----NOTE: RECALL THAT THE BALLAST WEIGHT HAS CHANGED THE
C -----SECTION MASS, CG LOCATION, AND MASS MOMENT OF INERTIA----
C
C   RB(I) = SQRT((YCT(I)-YGB(I))**2 + (ZCT(I)-ZGB(I))**2)
C   QC1B(I) = DFZT(I)*(OMEGA**2)*WT(I)*(COS(TANG(I))*(YCT(I)-YGB(I))
C - SIN(TANG(I))*(ZCT(I)-ZGB(I))) / 12.0
C   QC2B(I) = -RB(I)*(OMEGA**2)*WT(I)*(SIN(TANG(I))*(YCT(I)-YGB(I))
C + COS(TANG(I))*(ZCT(I)-ZGB(I))) / 12.0
C   QTRMB(I) = -IMGB(I)*AANG(I)*(OMEGA**2) / 386.
C
C   QTIB(I) = QC1B(I) + QC2B(I) + QTRMB(I)
C
C -----CENTRIFUGAL TORSION DUE TO ADDED BALLAST WEIGHT, CONSISTS
C -----OF THE TWO CENTRIFUGAL TERMS DESCRIBED ABOVE----
C
C   RBAL(I) = SQRT((YCT(I)-YBAL(I))**2 + (ZCT(I)-ZBAL(I))**2)
C   QCB(I) = (OMEGA**2)*WB(I) * ((DFZT(I))*COS(TANG(I))-RBAL(I))*
C + SIN(TANG(I)) * (YCT(I)-YBAL(I)) - (RBAL(I))*COS(TANG(I)) +
C + DFZT(I)*SIN(TANG(I)) * (ZCT(I)-ZBAL(I))) / 12.0
C
C -----NOW THE TOTAL CONTRIBUTION OF THE BALLAST WEIGHT----
C
C   OB(I) = QTIB(I) - QT(I)
C
C -----TENNIS RACKET CONTRIBUTION OF BALLAST WEIGHT--
C
C   OBTR(I) = OB(I) - QCB(I)
C
C -----FINALLY THE TOTAL SECTION TORSION FOR THE ELASTIC TWIST

```

```

C -----CALCULATION---
C
C   QS(I) = QA(I) + QTIB(I)
C
C 690 CONTINUE
C
C -----ELASTIC TORSION SECTION PRINT STATEMENTS-----
C
C PRINT 973
C PRINT 999
C PRINT 970, 'P-MOMENT',PM
C PRINT 970, 'LIFT ',QL
C PRINT 970, 'TOT-AERO',QA
C PRINT 973
C PRINT 1000
C PRINT 970, 'TNS-RCKT,QTRM
C PRINT 970, 'FLAP-DEF',QC1
C PRINT 970, 'LD-LAG ',QC2
C PRINT 970, 'TOTAL ',QT1
C PRINT 973
C PRINT 1010
C PRINT 970, 'TNS-RCKT,OBTR
C PRINT 970, 'CENTRIF',QCB
C PRINT 970, 'TOTAL ',QB
C PRINT 973
C PRINT 1020
C PRINT 970, 'TNS-RCKT,QTRMB
C PRINT 970, 'FLAP-DEF,QC1B
C PRINT 970, 'LD-LAG ',QC2B
C PRINT 970, 'TOTAL ',QT1B
C PRINT 970, 'SECTION ',QS
C PRINT 973
C
C -----FIND TOTAL TORSION BY INTEGRATING FROM THE BLADE TIP--
C
C CALL INTEG(STX,OS,OT,1,STA)
C
C -----PRINT TOTAL SECTION TORSION-----
C
C PRINT 970, 'TORSION ',QT
C
C -----TWIST1 = TORSION*TORK/GJ
C -----TORK IS A VALUE FROM HANDBOOKS. 2.0 IS APPROX. FOR AIRFOIL
C -----SKIN SHAPES-----
C
C TORK = 2.0
C
C -----TWIST2 = TWIST CALCULATED USING SHEAR FLOW----
C -----TWIST2 = TORSION / GJ2-----
C
C -----REVERSE THE INDEX TO PERFORM INTEGRATION STARTING-----
C -----AT ROOT END-----
C
C DO 705 I = 1,STA
C IR = STA + 1 - I
C ELAST1(I) = QT(IR)*TORK/GJ1(IR)
C ELAST2(I) = QT(IR) / GJ2(IR)
C 705 CONTINUE
C
C CALL INTEG(STX,ELAST1,TLAST1,1,STA)
C CALL INTEG(STX,ELAST2,TLAST2,1,STA)
C
C DO 725 I=1,STA
C IR = STA + 1 - I
C
C -----RENUMBER THE INDEX-----
C
C TELAST1(I) = TLAST1(IR)
C TELAST2(I) = TLAST2(IR)
C 725 CONTINUE
C
C -----PRINT TWIST, RENUMBERED-----
C
C PRINT 993, 'TORSDEF1',TELAST1
C PRINT 993, 'TORSDEF2',TELAST2
C
C -----FIND STATION CENTRIFUGAL MOMENT FOR FINAL BENDING DEFLECTIONS
C -----OF BLADE-----
C
C DO 750 I=1,STA
C RCENT(I) = RBL - STX(I)
C FCENT(I) = WT(I) * RCENT(I) * OMEGA**2 * (1./12.)
C LX(I) = LX(I) + FCENT(I)
C LZ(I) = LZ(I)
C MCX(I) = FCENT(I) * DFZT(I)
C 750 CONTINUE

```

```

C CALL INTEG(STX,LXI,FX,1,STA)
C CALL INTEG(STX,MCX,MC1,1,STA)
C DO 755 I=1,STA
C MCENT(I) = MC1(I) -FX(I) * DFZT(I)
755 CONTINUE
C
760 CONTINUE
C
C ***** BENDING DEFLECTIONS SECTION *****
C
C CALCULATE BEAM SHEARS AND BENDING MOMENTS IN REFERENCE AXES,
C WHICH ARE THE Y,Z DIRECTIONS. BLADE SHEARS AND CENTRIF FORCE
C ARE SUMMED FROM THE FREE END OF THE BEAM.-----
C
C FLAPPING LOADS (LZ), SHEARS (VZ), AND MOMENTS (MFLAP) ARE ALL
C POSITIVE IN THE FLAPPING DIRECTION (SUCH AS TO INCREASE FLAPPING)
C LAGGING LOADS (LY), SHEARS (VY), AND MOMENTS (MLAG) ARE ALL
C POSITIVE IN THE LAGGING DIRECTION (SUCH AS TO RETARD BLADE TORQUE).
C -----
C
C CALL INTEG(STX,LXI,FX,1,STA)
C CALL INTEG(STX,LYI,VY,1,STA)
C CALL INTEG(STX,VY,MLAG,1,STA)
C CALL INTEG(STX,LZI,VZ,1,STA)
C CALL INTEG(STX,VZ,MFLAPA,1,STA)
C
C -----PRINCIPAL AXIS ANGLE (AANG) = TWIST ANGLE (TANG) +
C -----CHORD TO PRINCIPAL AXIS ANGLE OF SECTION (PANG) +
C -----ELASTIC TWIST ANGLE (TELAST 1 OR TELAST 2).-----
C
C NOTE: THE 1ST TIME THROUGH THIS LOOP TO FIND THE
C APPROXIMATE DEFLECTIONS DOES NOT INCLUDE ELASTIC TWIST
C
C ***** STRESS CALCULATIONS *****
C
C NOTE: THIS PROGRAM DOES NOT PRINT OUT LAMINATE STRESSES.
C
C -----CALCULATE THE STRESS DUE TO AXIAL TENSION. RESOLVE-----
C -----BENDING MOMENT TO PRINCIPAL AXES. FIND NEUTRAL AXIS-----
C -----RELATIVE TO DISC.-----
C
C DO 870 I=1,STA
C TANG(I) = TANG(I) + TELAST2(I)
C AANG(I) = PANG(I) + TANG(I)
C SXTN(I)=FX(I) * ES/EAI(I)
C
C -----NOTE: ROTATION OF AXES FROM SECTION COORDINATES TO-----
C -----PRINCIPAL AXES COORDINATES IS THROUGH POSITIVE ANGLE-----
C -----PANG (COUNTERCLOCKWISE).
C
C MFLAP(I) = MFLAPA(I) - MCENT(I)
C MFLAPP(I) = MFLAP(I)*COS(AANG(I))-MLAG(I)*SIN(AANG(I))
C MLAGP(I) = MLAG(I)*COS(AANG(I))+MFLAP(I)*SIN(AANG(I))
C
C 870 CONTINUE
C
C -----PRINT X-STRESS DUE TO TENSION-----
C
C PRINT 970, 'SX TENS.', SXTN
C
C -----CALCULATE STRESSES DUE TO BENDING. STRESS IS THE SUM-----
C -----OF BENDING IN PLANES OF PRINCIPAL AXES.-----
C
C DO 1005 K=1,NSO
C
C DO 885 J=1,STA
C
C -----TRANSLATE YSO AND ZSO TO COORDINATES THROUGH CENTROID--
C
C YK = YSO(K)*SCALE(J)-YCB(J)
C ZK = ZSO(K)*SCALE(J)-ZCB(J)
C
C -----C'S ARE DISTANCES TO STRESSED FIBER
C -----IN PRINCIPAL AXIS SYSTEM, FOR THE SELECTED STRESS OFFSETS
C -----CALCULATED ABOVE-----
C
C CY = YK*COS(PANG(J))+ZK*SIN(PANG(J))
C CZ = ZK*COS(PANG(J))-YK*SIN(PANG(J))
C
C -----NEGATIVE SIGN DENOTES FIBER IN COMPRESSION.-----

```

```

C -----THE FLEXURE FORMULA IS: BEND STRESS =M*C/I-----
C -----POSITIVE MLAGP MOMENT IS IN LAGGING DIRECTION,---
C -----AND GIVES TRAILING EDGE COMPRESSION. POSITIVE MFLAPP MOMENT
C -----IS IN FLAPPING DIRECTION, AND GIVES UPPER----
C -----SURFACE COMPRESSION.-----
C -----BOTH HAVE BEEN RESOLVED INTO THE PRINCIPAL AXES OF EACH
C -----SECTION, SO THEY ARE NOT EXACTLY IN THE FLAP AND LAG
C -----DIRECTIONS (Z, Y).-----
C
C SX(J) = -MFLAPP(J)*CZ*EL/EIYP(J)-MLAGP(J)*CY*EL/EIZP(J)
885 CONTINUE
C
C -----PRINT BEAM TENSILE STRESS DUE TO BENDING MOMENT--
C
C PRINT 970, 'SX-BM ', SX
C
C DO 890 J=1,STA
C SX(J) = SX(J) + SXTN(J)
C
890 CONTINUE
C
C -----PRINT BEAM TENSILE STRESS TOTAL-----
C
C PRINT 970, 'SX TOTAL', SX
C
1005 CONTINUE
C
C -----CALCULATE BENDING DEFLECTION. DEFINE CURVE M/EI IN-----
C -----REVERSE ORDER TO INTEGRATE FROM FIXED END-----
C
C DO 1110 I=1,STA
C IR = STA +1-I
C RX(IR) = STX(STA)-STX(I)
C MYP(I) = MFLAPP(I)/EIYP(I)
C MZP(I) = MLAGP(I)/EIZP(I)
C MY(IR) = (MYP(I)*COS(AANG(I))) + (MZP(I)*SIN(AANG(I)))
C MZ(IR) = (MZP(I)*COS(AANG(I))) - (MYP(I)*SIN(AANG(I)))
C
C 1110 CONTINUE
C
C -----REVERSE THE ORDER-----
C
C DO 1120 I=1,STA
C IR = STA+1-I
C MYE(I) = MY(IR)
C MZE(I) = MZ(IR)
C LYC(I) = LYI(I)
C LZC(I) = LZI(I)
C
C 1120 CONTINUE
C
C -----FIND DEFLECTION BY INTEGRATION: D2Y/DX2 = M/EI-----
C -----WHERE LYC = DY/DX AND LZC = DZ/DX ARE THE ROOT SLOPES
C -----NOTE: THIS SECTION IS SIMILAR TO THE SUBROUTINE "INTEG"
C
C LYC(1) = 0.0
C LZC(1) = 0.0
C DEFY(1) = 0.0
C DEFZ(1) = 0.0
C I6 = STA - 1
C DO 1115 JJ=1,I6,2
C J = JJ
C IF(J.EQ. I6) J = J-1
C CALL LCOE1(C1,C2,C3,J,RX,MZ)
C CALL LCOE1(D1,D2,D3,J,RX,MY)
C IF(J.EQ. I6) J = J+1
C C4 = LYC(J)-C1*RX(J)**3/3-C2*RX(J)**2/2-C3*RX(J)
C D4 = LZC(J)-D1*RX(J)**3/3-D2*RX(J)**2/2-D3*RX(J)
C C5 = DEFY(J)-C1*RX(J)**4/12-C2*RX(J)**3/6-C3*RX(J)**2/2-C4*RX(J)
C D5 = DEFZ(J)-D1*RX(J)**4/12-D2*RX(J)**3/6-D3*RX(J)**2/2-D4*RX(J)
C K = J+1
C LYC(K) = C1*RX(K)**3/3+C2*RX(K)**2/2+C3*RX(K)+C4
C LZC(K) = D1*RX(K)**3/3+D2*RX(K)**2/2+D3*RX(K)+D4
C DEFY(K)=C1*RX(K)**4/12+C2*RX(K)**3/6+C3*RX(K)**2/2+C4*RX(K)+C5
C DEFZ(K)=D1*RX(K)**4/12+D2*RX(K)**3/6+D3*RX(K)**2/2+D4*RX(K)+D5
C IF(J.EQ. I6) GO TO 1115
C K = J+2
C LYC(K) = C1*RX(K)**3/3+C2*RX(K)**2/2+C3*RX(K)+C4
C LZC(K) = D1*RX(K)**3/3+D2*RX(K)**2/2+D3*RX(K)+D4
C DEFY(K)=C1*RX(K)**4/12+C2*RX(K)**3/6+C3*RX(K)**2/2+C4*RX(K)+C5
C DEFZ(K)=D1*RX(K)**4/12+D2*RX(K)**3/6+D3*RX(K)**2/2+D4*RX(K)+D5
C
C 1115 CONTINUE
C
C -----REVERSE ORDER-----

```

```

DO 1215 I=1,STA
IR = STA + 1 - I
DFY(IR) = DEFY(I)
DFZ(IR) = DEFZ(I)
RCENT(I) = RBL - STX(I)
DFZC(I) = RCENT(I) * SIN(CONING)
C
1215 CONTINUE
C
C -----TOGGLE LOGIC-----
C
IF (TOGGLE .GE. 1) GO TO 1230
C
1220 CONTINUE
C
PRINT 970, 'V-AERO ',VZ
PRINT 970, 'BM-AERO ',MFLAPA
PRINT 970, 'Y-AEROD ', DFY
PRINT 970, 'Z-AEROD ', DFZ
PRINT 970, 'Z-CONING',DFZC
C
1230 CONTINUE
C
C -----FIND ACTUAL DEFLECTION INCLUDING CONING CONTRIBUTION----
C
DO 1250 I=1,STA
DFZ(I) = DFZ(I) + DFZC(I)
1250 CONTINUE
C
C -----IF CONVERGENCE IS REACHED THEN PRINT OUT, OTHERWISE RETURN
C -----NOTE: SINCE THIS PROGRAM HAS TWO DEGREES OF FREEDOM, FLAP AND
C -----TWIST, THERE ARE TWO CONDITIONS ON CONVERGENCE, AND THE LIMIT
C -----SERIES CONVERGENCE STRATEGY MUST BE USED ON BOTH----
C
IF (TOGGLE .LE. 1) GO TO 1270
C
IF (ABS(DFZ(1)-DFZ1(1)) .GE. 0.005) GO TO 1270
IF (ABS(AANG(1)-AANG1(1)) .LE. 0.0005) GO TO 1290
C
1270 CONTINUE
C
DO 1280 I=1,STA
DFZ1(I) = DFZ(I)
ZTOT(I) = ZTOT(I) + DFZ(I)
DFZT(I) = ZTOT(I) / FLOAT (TOGGLE + 1)
C
AANG1(I) = AANG(I)
ANGTOT(I) = ANGTOT(I) + AANG(I)
AANG(I) = ANGTOT(I) / FLOAT (TOGGLE + 1)
C
TANGTOT(I) = TANGTOT(I) + TANG(I)
TANG(I) = TANGTOT(I) / FLOAT (TOGGLE + 1)
C
1280 CONTINUE
C
GO TO 650
C
1290 CONTINUE
C
DO 1300 I=1,STA
EIFLAP(I) = E IYP(I)*COS(AANG(I))+EIZP(I)*SIN(AANG(I))
EILAG(I) = -E IYP(I)*SIN(AANG(I))+EIZP(I)*COS(AANG(I))
C
1300 CONTINUE
C
C -----ELASTIC TORSION PRINT STATEMENTS-----
C
PRINT 973
PRINT 999
PRINT 970, 'P-MOMENT',PM
PRINT 970, 'LIFT ',QL
PRINT 970, 'TOT-AERO',QA
PRINT 973
PRINT 1000
PRINT 970, 'TNS-RCKT',QTRM
PRINT 970, 'FLAP-DEF',QC1
PRINT 970, 'LD-LAG ',QC2
PRINT 970, 'TOTAL ',QT1
PRINT 973
PRINT 1010
PRINT 970, 'TNS-RCKT',QBTR
PRINT 970, 'CENTRIF',QC8
PRINT 970, 'TOTAL ',QB
PRINT 973
PRINT 1020
PRINT 970, 'TNS-RCKT',QTRMB

```

```

PRINT 970, 'FLAP-DEF',QC1B
PRINT 970, 'LD-LAG ',QC2B
PRINT 970, 'TOTAL ',QT1B
PRINT 970, 'SECTION',QS
PRINT 973
C
PRINT 970, 'TORSION',QT
C PRINT 993, 'TORSDEF1',TELAST1
C
DO 1390 I=1,STA
TELAST2(I)=TELAST2(I)*57.29578
1390 CONTINUE
C
PRINT 995,'EL.TWIST',TELAST2
C
C -----PRINT TENSION-----
C
PRINT 970, 'TENSION',FX
C
C -----PRINT INPLANE SHEAR-----
C
PRINT 970, 'V-INPL ',VY
C
C -----PRINT INPLANE MOMENT-----
C
PRINT 970, 'BM-INPL ',MLAG
C
C -----PRINT FLAPPING SHEAR-----
C
PRINT 970, 'V-FLAP ',VZ
C
C -----PRINT FLAPPING MOMENT-----
C
PRINT 970, 'BM-FLAP ',MFLAP
C
C -----PRINT PRINCIPAL AXIS ANGLE AFTER CONVERTING TO DEGREES--
C
DO 1400 I=1,STA
AANG(I) = AANG(I) * 57.29577951
1400 CONTINUE
C
PRINT 970, 'PR A A ',AANG
C
C -----PRINT X-STRESS TENSION-----
C
PRINT 970, 'SX TENS',SXTN
C
C -----PRINT THE CURVE M/EI-REVERSE, Y-DIRECTION, INTEGRATED FROM
C -----THE FIXED END-----
C
PRINT 992, 'M.EIYP ',MYP
C
C -----PRINT THE CURVE M/EI-REVERSE, Z-DIRECTION, INTEGRATED FROM
C -----THE FIXED END-----
C
PRINT 992, 'M.EIZP ',MZP
PRINT 973
C
C -----PRINT Y DEFLECTION-----
C
PRINT 970, 'Y-DEF ',DFY
C
C -----PRINT Z DEFLECTION-----
C
PRINT 970, 'Z-DEF ',DFZ
C
C -----PRINT THE FLAP AND LAG STIFFNESSES-----
C
PRINT 992, 'EI-FLAP ',EIFLAP
PRINT 992, 'EI-LAG ',EILAG
PRINT 963,TOGGLE
C
PRINT 973
C
C -----STRAIN CALCULATIONS -----
C
C
FIND STRAINS DUE TO AXIAL TENSION (CENTRIFUGAL FORCE)
AND BENDING MOMENT.
C
C STRAIN DUE TO TENSION (SXTN) IS JUST TENSION AT A
C STATION DIVIDED BY THE COMPOSITE EA OF THE STATION:
C
DO 1500 I = 1, STA
SXTN(I) = FX(I) / EA1(I)

```


STATION	TIP										ROOT
	1	2	3	4	5	6	7	8	9	10	11
STA POS	0.00	19.20	38.40	57.60	76.80	96.00	115.20	134.40	153.60	172.80	192.00
X-LOAD	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
Y-LOAD	-0.87	-0.89	-0.72	-0.44	-0.34	-0.18	-0.04	0.03	0.04	0.03	0.00
Z-LOAD	5.17	4.51	4.00	2.99	2.17	1.57	1.13	0.81	0.61	0.47	0.00
P-MOMENT	5.48	3.97	3.09	3.03	2.49	2.06	1.70	1.65	1.50	1.49	0.00
EA	0.2286E+08	0.2286E+08	0.2286E+08	0.2286E+08	0.2286E+08	0.2286E+08	0.2286E+08	0.2286E+08	0.2286E+08	0.2510E+08	0.2510E+08
EIYP	0.5867E+07	0.5867E+07	0.5867E+07	0.5867E+07	0.5867E+07	0.5867E+07	0.5867E+07	0.5867E+07	0.5867E+07	0.1372E+07	0.1372E+07
EIZP	0.3240E+09	0.3240E+09	0.3240E+09	0.3240E+09	0.3240E+09	0.3240E+09	0.3240E+09	0.3240E+09	0.3240E+09	0.1370E+09	0.1370E+09
GJ	0.5580E+07	0.5580E+07	0.5580E+07	0.5580E+07	0.5580E+07	0.5580E+07	0.5580E+07	0.5580E+07	0.5580E+07	0.2400E+06	0.2400E+06
SCALE	15.00	15.00	15.00	15.00	15.00	15.00	15.00	15.00	15.00	1.00	1.00
TSCALE	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	0.10	0.10	0.10

Y-BCENT	3.40	3.40	3.40	3.40	3.40	3.40	3.40	3.40	5.50	5.50	5.50
Z-BCENT	0.18	0.18	0.18	0.18	0.18	0.18	0.18	0.18	0.00	0.00	0.00
Y-SHRCTR	6.93	6.93	6.93	6.93	6.93	6.93	6.93	6.93	5.50	5.50	5.50
Z-SHRCTR	0.17	0.17	0.17	0.17	0.17	0.17	0.17	0.17	0.00	0.00	0.00
WEIGHT	0.43	0.43	0.43	0.43	0.43	0.43	0.43	0.43	0.32	0.32	0.32
Y-CG	4.69	4.69	4.69	4.69	4.69	4.69	4.69	4.69	5.50	5.50	5.50
Z-CG	0.18	0.18	0.18	0.18	0.18	0.18	0.18	0.18	0.00	0.00	0.00
I-EA	10.51	10.51	10.51	10.51	10.51	10.51	10.51	10.51	1.74	1.74	1.74
I-CG	8.38	8.38	8.38	8.38	8.38	8.38	8.38	8.38	1.74	1.74	1.74

BALLAST	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
TOTAL WT	0.43	0.43	0.43	0.43	0.43	0.43	0.43	0.43	0.32	0.32	0.32
NEW Y-CG	4.69	4.69	4.69	4.69	4.69	4.69	4.69	4.69	5.50	5.50	5.50
NEW Z-CG	0.18	0.18	0.18	0.18	0.18	0.18	0.18	0.18	0.00	0.00	0.00
NEW I-CG	8.38	8.38	8.38	8.38	8.38	8.38	8.38	8.38	1.74	1.74	1.74
BLD-WGHT	0.00	8.18	16.36	24.54	32.72	40.90	49.08	57.44	64.74	70.80	76.86

V-AERO	0.00	92.68	174.18	241.06	290.36	325.99	351.64	370.06	383.43	394.27	399.29
BM-AERO	0.00	907.62	3487.33	7501.78	12631.62	18564.69	25085.99	32022.73	39264.74	46740.13	54367.75
Y-AEROD	-0.20	-0.19	-0.18	-0.18	-0.16	-0.15	-0.12	-0.09	-0.04	-0.01	0.00
Z-AEROD	292.85	257.55	222.31	187.30	152.77	119.03	86.46	55.13	26.49	6.96	0.00
Z-CONING	16.73	15.06	13.39	11.71	10.04	8.37	6.69	5.02	3.35	1.67	0.00

SECTION TORSION DUE TO: {+ = NOSE DOWN}

P-MOMENT	5.48	3.97	3.09	3.03	2.49	2.06	1.70	1.65	1.50	1.49	0.00
LIFT	-16.54	-14.46	-12.82	-9.58	-6.95	-5.01	-3.60	-2.56	-3.18	-2.46	0.00
TOT-AERO	-11.06	-10.49	-9.73	-6.55	-4.46	-2.95	-1.90	-0.91	-1.68	-0.97	0.00

SECTION ALONE:

TNS-RCKT	0.05	0.05	0.05	0.05	0.05	0.05	0.05	0.05	0.00	0.00	0.00
FLAP-DEF	9.59	8.52	7.45	6.39	5.33	4.29	3.28	2.31	0.00	0.00	0.00
LD-LAG	0.01	0.01	0.01	0.01	0.01	0.01	0.01	0.01	0.00	0.00	0.00
TOTAL	9.65	8.58	7.51	6.44	5.38	4.35	3.33	2.37	0.00	0.00	0.00

BALLAST ALONE:

TNS-RCKT	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
CENTRIF	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
TOTAL	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00

TOTAL SECTION + BALLAST:

TNS-RCKT	0.05	0.05	0.05	0.05	0.05	0.05	0.05	0.05	0.00	0.00	0.00
FLAP-DEF	9.59	8.52	7.45	6.39	5.33	4.29	3.28	2.31	0.00	0.00	0.00
LD-LAG	0.01	0.01	0.01	0.01	0.01	0.01	0.01	0.01	0.00	0.00	0.00
TOTAL	9.65	8.58	7.51	6.44	5.38	4.35	3.33	2.37	0.00	0.00	0.00
SECTION	-1.42	-1.91	-2.23	-0.11	0.93	1.40	1.43	1.45	-1.67	-0.97	0.00

TORSION	0.00	-32.26	-72.29	-92.96	-83.37	-60.35	-32.48	0.26	3.18	-22.56	-32.23
EL.TWIST	-0.251	-0.248	-0.238	-0.221	-0.203	-0.189	-0.179	-0.176	-0.182	-0.132	0.000
TENSION	0.00	493.97	935.94	1325.91	1663.89	1949.88	2183.87	2368.11	2486.89	2544.70	2563.97
V-INPL	0.00	-17.18	-32.90	-43.79	-51.05	-56.04	-58.17	-58.15	-57.36	-56.64	-56.29
BM-INPL	0.00	-167.21	-650.28	-1392.38	-2308.66	-3341.29	-4442.29	-5560.12	-6670.22	-7763.91	-8847.33
V-FLAP	0.00	92.68	174.18	241.06	290.36	325.99	351.64	370.06	383.43	394.27	399.29
BM-FLAP	0.00	53.62	192.20	371.32	483.01	447.34	316.59	269.40	850.81	2398.64	4282.23
PR A A	-0.79	-0.79	-0.78	-0.76	-0.74	-0.73	-0.72	-0.72	-0.18	-0.13	0.00
M.EIYP	0.0000E+00	0.8745E-05	0.3125E-04	0.6012E-04	0.7720E-04	0.6898E-04	0.4442E-04	0.3405E-04	0.6047E-03	0.1735E-02	0.3121E-02
M.EIZP	0.0000E+00	-5.183E-06	-2.015E-05	-4.312E-05	-7.144E-05	-1.033E-04	-1.372E-04	-1.717E-04	-4.870E-04	-5.670E-04	-6.457E-04

Y-DEF.	-0.52	-0.45	-0.39	-0.32	-0.26	-0.20	-0.14	-0.09	-0.04	-0.01	0.00
Z-DEF.	29.97	26.79	23.62	20.45	17.31	14.19	11.10	8.02	4.97	2.16	0.00

49

M-STRAIN 1	0.	21.	38.	51.	59.	60.	55.	50.	-179.	-232.	-253.
M-STRAIN 2	0.	13.	12.	2.	2.	23.	58.	77.	-171.	-230.	-261.
M-STRAIN 3	0.	17.	27.	32.	46.	77.	120.	149.	-158.	-213.	-241.
M-STRAIN 4	0.	31.	75.	128.	181.	228.	269.	314.	-129.	-172.	-188.
M-STRAIN 5	0.	31.	76.	127.	168.	187.	190.	203.	-151.	-195.	-208.
M-STRAIN 6	0.	29.	67.	109.	139.	146.	137.	138.	-164.	-210.	-223.

**** STOP

10.6: CALCULATION OF DRAG POLARS FOR EIGHT WIND TURBINE BLADE SECTIONS

10.6.1 PROCEDURE AND ANALYSIS:

The airfoil templates were digitized with a Houston Instruments Hi-Pad digitizer with a 10 in. x 10 in. working area. Each template was divided into two or three parts as necessary. Reference points were marked on each template before dividing. The individual data files for each template were combined into one file per template. Using coordinate transformations, the complete airfoil was reconstituted into a single coordinate table with the leading edge at 0.0,0.0 and the trailing edge at 1.0,0.0. The finite trailing edge thickness was maintained.

The coordinate data then were subjected to smoothing to meet the sensitivity requirements of the Eppler Code. Smoothing required the data to be transformed so that the independent variable is monotonically increasing. Neither the x nor y coordinate can be used for this because neither varies in a monotonic fashion. A smoothing procedure utilizing the coordinates used in the Theodorsen Transformation was tried first. The coordinate reference is shifted to the fifty percent chord location so that the leading edge is at -0.5,0.0 and the trailing edge is at 0.5,0.0. The transformation variables are θ and ψ according to:

$$x = \cosh \psi * \cos \theta \qquad y = \sinh \psi * \sin \theta$$

The transformation is very sensitive near the leading and trailing edges as $\sin \theta$ goes to zero here. Cubic spline smoothing applied to the θ - ψ coordinates was unsuccessful. Modifications to the trailing edge geometry occurred that could not be controlled. The problem was traced to the fact that smoothing the ψ variable modified not only the y coordinate, but also the x coordinate. A new transformation was then tried by replacing the x coordinate with θ so that

$$x = 0.5 \cos \theta_i \qquad y = y$$

where θ_i is the i th knot of n knots. This weighted the leading edge, where geometric accuracy is more critical. Each airfoil was smoothed and a coordinate table of 61 points was generated for a range of knot sets increasing up to the maximum allowed of twenty-seven. Each set was tested with the Eppler potential solution. The coordinate table with the highest number of knot points before waves appeared in the velocity distribution was selected. This criterion resulted in the same knot point sets for all airfoils. The airfoil data used were based on twenty-one knot points, ten upper surface, ten lower surface, and one at the leading edge.

All airfoils were then subjected to a drag polar analysis using the Eppler Code. Reynolds number ranges were checked for each, and where there was a significant span, the low and the high ends of the range were used. Only the Carter airfoils required more than one Reynolds number.

The generated data for each airfoil consists of the following: plot of the airfoil and velocity distributions for six angles of attack; plot of the drag polar, lift and pitching moment coefficients versus angle of attack; laminar-turbulent transition and turbulent separation locations for upper and lower surfaces; tabulated values of all plotted data. The transition and separation locations are measured from the trailing edge along the surface contour. In considering the lift and drag values, the separation location should be consulted. Any separation location greater than ten percent (ninety percent chord location) suggests a warning. There is presently no good method for calculating accurate values for large regions of separated flow.

10.6.2 PLANFORM DATA FILES AND DRAG POLARS

Tabulated Data Lists:

ESI	0.75 radius
ESI	0.95 radius
UTRC	0.75 radius
UTRC	0.95 radius
Carter	0.75 radius uncut
Carter	0.95 radius uncut
Carter	0.75 radius cut
Carter	0.95 radius cut

Note: All these data files and drag polar plots can be obtained at AEI.

10.7: CHECKLIST SENT TO BLADE MANUFACTURERS

This information is necessary for the formulation of an accurate computer model of the blade structure for use in aeroelastic and structural dynamic codes. These data will be verified, insofar as possible, by bench tests at AEI using the actual blades mounted in test stands, and from coupon tests and measurements from sectioned blades. Proprietary data furnished by the blade manufacturer will be treated as such if properly marked and/or otherwise specifically indicated. Results of the test program and the computer simulations will not generally be treated as proprietary, and will likely be reported in the public domain.

BLADE SPECIFICATIONS

Geometric:

- blade dimensions
- blade twist distribution
- blade taper (chord) distribution
- airfoil shape distribution and offsets if not a standard airfoil
- location of control (pitch) axis
- procedure and benchmark method for setting
- blade pitch angle

Note: Any engineering drawings, airfoil lift and drag data, or layout sketches of the rotor/blade will be greatly appreciated.

Structural:

- blade internal geometry
- root attachment method
- balancing and ballast weights
- shell/skin/spar nominal locations and thicknesses
- locations of shear webs
- structural or internal control attachments
- spanwise mass distribution
- chordwise mass distribution, or location of mass centers
- location of shear centers (elastic axis)
- blade natural frequencies (first mode) and mode shapes in bending (flapping and lead-lag) and torsion

Note: Engineering drawings and cross-section detail drawings are particularly needed.

Laminate:

- laminate schedule
- individual laminate thicknesses
- engineering tensile, compressive, and shear moduli for each laminate layer (i.e., cloth, sandwich, etc.)
- laminate flexural modulus, if known
- cross sectional views
 - for cloth and unidirectional plies, the fiber orientation (i.e., 45 degree, etc.)
 - for wound plies, the tape or filament advance angle
- glass (or reinforcement) to resin ratios for the laminates
- mass densities for the laminates
- mass density for the catalyzed resins
- mass density and thickness of surface finish and/or gelcoat

Note: Cross-sectional construction drawings, test results on individual laminates (especially ASTM burnoff and outgassing tests), manufacturers' specifications on glass rovings and woven goods, and particularly any composite modulus (E) and section modulus (EI) data are needed. References already published with such material will be greatly appreciated.

REPORT DOCUMENTATION PAGE

Form Approved
OMB No. 0704-0188

The public reporting burden for this collection of information is estimated to average 1 hour per response, including the time for reviewing instructions, searching existing data sources, gathering and maintaining the data needed, and completing and reviewing the collection of information. Send comments regarding this burden estimate or any other aspect of this collection of information, including suggestions for reducing the burden, to Department of Defense, Executive Services and Communications Directorate (0704-0188). Respondents should be aware that notwithstanding any other provision of law, no person shall be subject to any penalty for failing to comply with a collection of information if it does not display a currently valid OMB control number.

PLEASE DO NOT RETURN YOUR FORM TO THE ABOVE ORGANIZATION.

1. REPORT DATE (DD-MM-YYYY) 29-08-2006			2. REPORT TYPE subcontract report			3. DATES COVERED (From - To) August 1989		
4. TITLE AND SUBTITLE Determination of Elastic Twist in Horizontal Axis Wind Turbines (HAWTs)					5a. CONTRACT NUMBER DE-AC36-99-GO10337			
					5b. GRANT NUMBER			
					5c. PROGRAM ELEMENT NUMBER			
6. AUTHOR(S) F. Stoddard, V. Nelson, K. Starcher, B. Andrews					5d. PROJECT NUMBER NREL/SR-500-12087			
					5e. TASK NUMBER WER6.7502			
					5f. WORK UNIT NUMBER			
7. PERFORMING ORGANIZATION NAME(S) AND ADDRESS(ES) F. Stoddard, V. Nelson, K. Starcher, B. Andrews, Alternative Energy Institute, West Texas State University, Canyon, Texas						8. PERFORMING ORGANIZATION REPORT NUMBER RL-6-06013		
9. SPONSORING/MONITORING AGENCY NAME(S) AND ADDRESS(ES) National Renewable Energy Laboratory 1617 Cole Blvd. Golden, CO 80401-3393						10. SPONSOR/MONITOR'S ACRONYM(S) NREL		
						11. SPONSORING/MONITORING AGENCY REPORT NUMBER NREL/SR-500-12087		
12. DISTRIBUTION AVAILABILITY STATEMENT National Technical Information Service U.S. Department of Commerce 5285 Port Royal Road Springfield, VA 22161								
13. SUPPLEMENTARY NOTES NREL Technical Monitor: A. Laxson								
14. ABSTRACT (Maximum 200 Words) This report presents the results of a project at the Alternative Energy Institute that measured and calculated the elastic twist of three representative composite horizontal-axis blades. It covers the experimental method used to inspect and test the blades, the theory and computer programs used to develop semi-empirical analytical blade models, calculation of the steady elastic twist of the three blades in simulated operation, and comparison with actual field performance of the wind turbines.								
15. SUBJECT TERMS wind energy; wind turbine; composite wind turbine blades; horizontal-axis blades								
16. SECURITY CLASSIFICATION OF:			17. LIMITATION OF ABSTRACT UL	18. NUMBER OF PAGES	19a. NAME OF RESPONSIBLE PERSON			
a. REPORT Unclassified	b. ABSTRACT Unclassified	c. THIS PAGE Unclassified			19b. TELEPHONE NUMBER (Include area code)			

Standard Form 298 (Rev. 8/98)
Prescribed by ANSI Std. Z39.18